

**THE CONNECTION BETWEEN AN INTEGRAL STEEL CAP GIRDER
AND A CONCRETE PIER**

by

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DISSERTATION

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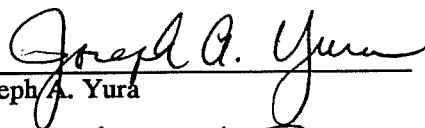
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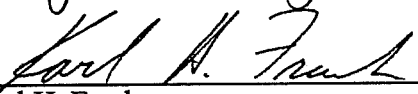
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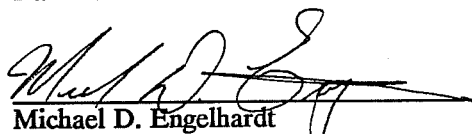
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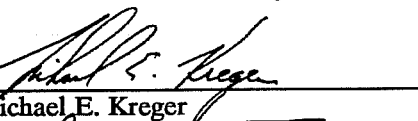
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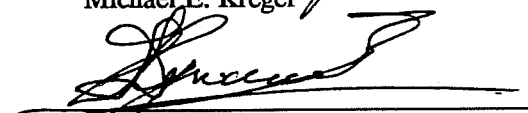
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To Mom, Dad, and Sim

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At congested highway interchanges, horizontal and vertical clearance requirements may dictate the use of narrow piers and shallow depth cap girders to accommodate the various roadways and overpasses. In situations such as this the state of Texas uses horizontally curved steel plate girders as the bridge structural system, supported on integral steel cap girders at single column piers. Two bearings are used to connect the steel cap girder to the concrete pier. Due to the narrow pier, unbalanced loading may produce a transverse overturning moment at the pier; the bearings resist this moment by developing a couple, with one bearing loaded in compression and one bearing loaded in tension. When the unbalanced loading is caused by truck traffic, which is cyclic, the bearing resisting the uplift is subject to fatigue loading.

The standard connection used by the state of Texas is an in-house design that comprises a line rocker bearing, which accommodates the horizontal rotation, and embedded anchor bolts that are used to both resist potential uplift and to provide a positive connection from the cap girder to the pier. The behavior of this connection, however, is not well understood and the detailing is complex. The objectives of the research were to examine and categorize the behavior of the existing connection and to develop a new detail that is simpler and cost-effective.

The research showed that the standard TxDOT connection performs adequately with respect to horizontal rotation but that it cannot resist uplift because of the poor fatigue characteristics of threaded anchor bolts. The fatigue problem may be mitigated by post-tensioning the uplift bearing but this option is not available for the standard connection. Two new connection details were developed, one to be used for situations in which uplift does not occur and one that is capable of resisting uplift. The new details replace the line rocker with a rolled wide-flange section and the threaded anchor bolts are replaced with high-strength threadbar, which is

specifically designed for post-tensioning. The new connections proved to be more cost-effective than the standard connection.

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CHAPTER 1 : INTRODUCTION

1.1 PROBLEM STATEMENT

At congested interchanges, horizontal and vertical clearance requirements may dictate the use of narrow piers and shallow depth cap girders to accommodate the various roadways and overpasses. In most cases in the state of Texas, horizontally curved steel plate girders are used as the structural system and a steel cap girder is used at the piers to support the longitudinal stringers, as shown in Figure 1.1. The bearing connections between the cap girder and the pier cap must be capable of transferring the deck superstructure forces into the pier. In the longitudinal direction, which is the direction parallel to traffic flow, it is assumed in the structural analysis that the connections allow free rotation of the longitudinal girders; no moments or forces are transferred to the bearings or the pier. In the transverse direction, which is the direction perpendicular to traffic flow, it is assumed in the structural analysis that no vertical deflection occurs at the pier; the connections are treated as a rigid moment resisting element that can transfer forces caused by unbalanced loading. Unbalanced loading is typically produced by a vehicle located eccentric to the pier centerline. This vehicle produces an overturning moment at the pier and may also produce uplift at one of the bearings. These conflicting requirements (connection flexibility in longitudinal direction, connection rigidity in transverse direction) present a difficult detailing problem.

The detailing problem has resulted in the use of many types of bearing designs. Six types of bearings have been used with integral steel cap girders by the Texas Department of Transportation [21].

- Separated Steel Bolster Shoes
- Single Steel Bolster Shoes
- Long Steel Convex Plate bearing on Base Plate, with Prestressed Anchor Bolts
- Preformed Fabric Pad with High Strength Anchor Bolts
- Pot and Disc Bearings
- Separated Steel Pin and Rocker Bearing with High Strength Anchor Bolts

A schematic of a steel bolster shoe is shown in Figure 1.2. The pin allows free longitudinal rotation of the cap girder while simultaneously providing a rigid positive connection between the

cap girder and the pier. The flexibility of the supporting pier is used to accommodate horizontal movement since the bearing itself does not permit this type of displacement. A typical pot bearing is shown in Figure 1.3. A pot bearing consists of a circular, non-reinforced rubber or elastomeric pad enclosed by a steel pot. The steel pot prevents the pad from bulging when loaded. This confinement results in the pad behaving as a fluid under high pressure, producing a bearing that can accommodate significant rotation yet deflect very little vertically. A pot bearing may be designed with a center pin to resist horizontal movement or a vertical key mechanism to resist uplift. Though both of the bearings just described satisfy the conflicting requirements, they tend to be expensive and are limited in the loads they can handle. The standard bearing now used by the Texas Department of Transportation with an integral steel cap girder is the last one listed above, the Separated Steel Pin and Rocker Bearing with High Strength Anchor Bolts. This is the connection detail shown in Figure 1.1.

A schematic of the standard connection is shown in Figure 1.4. The connection consists of steel bearing plates, which transfer the compressive loads to the pier; a rocker that allows free rotation in the longitudinal direction; anchor bolts, embedded in the pier cap, used to handle uplift; vertical stiffeners and a horizontal cross-plate, which provides the anchor bolt attachment to the cap girder; and bearing pads, located under the plate-washer at the top of the anchor bolt. The long length of the anchor bolts and the bearing pads are used to reduce the axial stiffness of the anchor bolts, which will reduce the force developed by the anchor bolts in response to the longitudinal rotation. The main advantage of this connection is that it is less expensive than the proprietary bearings described above. The disadvantages of this connection are that its behavior is not well understood, the detailing is complex, and, though less expensive than the proprietary bearings, it is not considered cost-effective by the Texas Department of Transportation. The expense of the connection is due to the fabrication requirements, which include the precise machining of the rocker detail and the complex stiffener detailing.

The complex detailing prompts concerns regarding the behavior of the connection and the design of the individual components of the connection. The standard connection has not been subject to controlled laboratory experimentation and so the assumptions made with respect to the behavior are not supported by any established set of data. In the transverse direction, the direction perpendicular to traffic flow, the primary concern is the behavior of the connection with respect to uplift. A truck located on the roadway eccentric with respect to the centerline of the pier will produce an overturning moment at the pier. The close spacing of the bearings, which

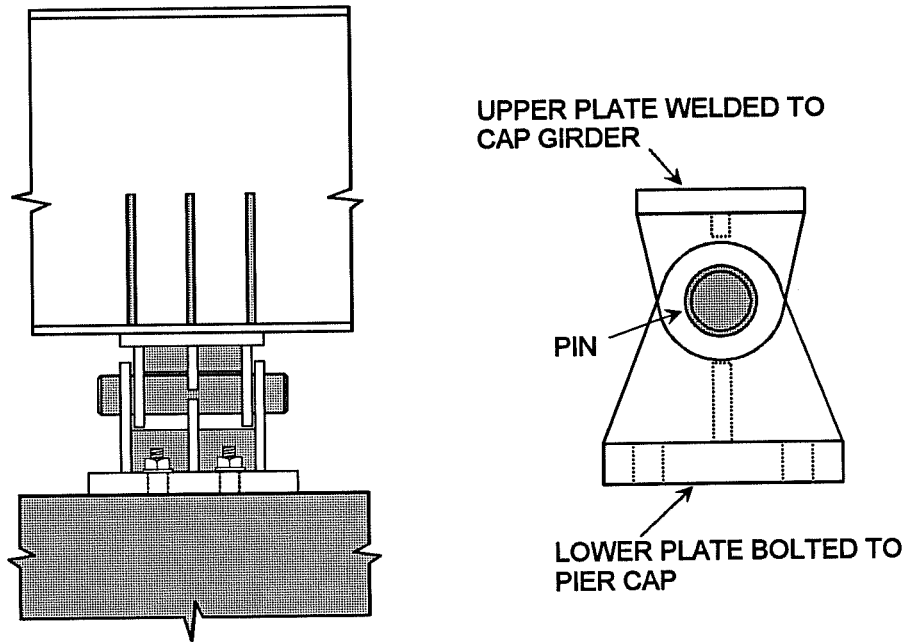


Figure 1.2 Steel Bolster Shoe

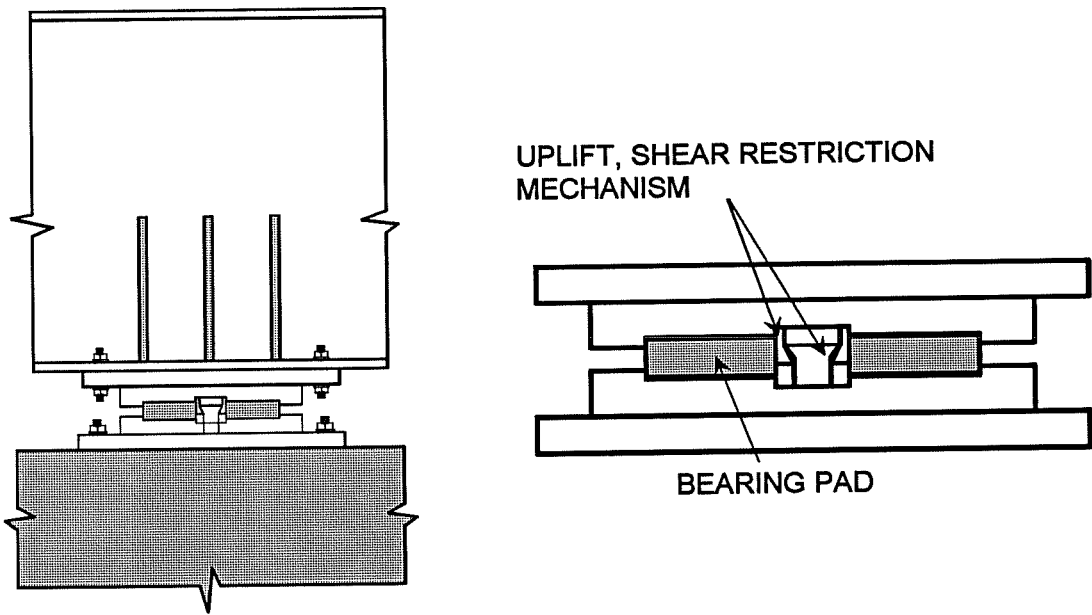


Figure 1.3 Pot Bearing

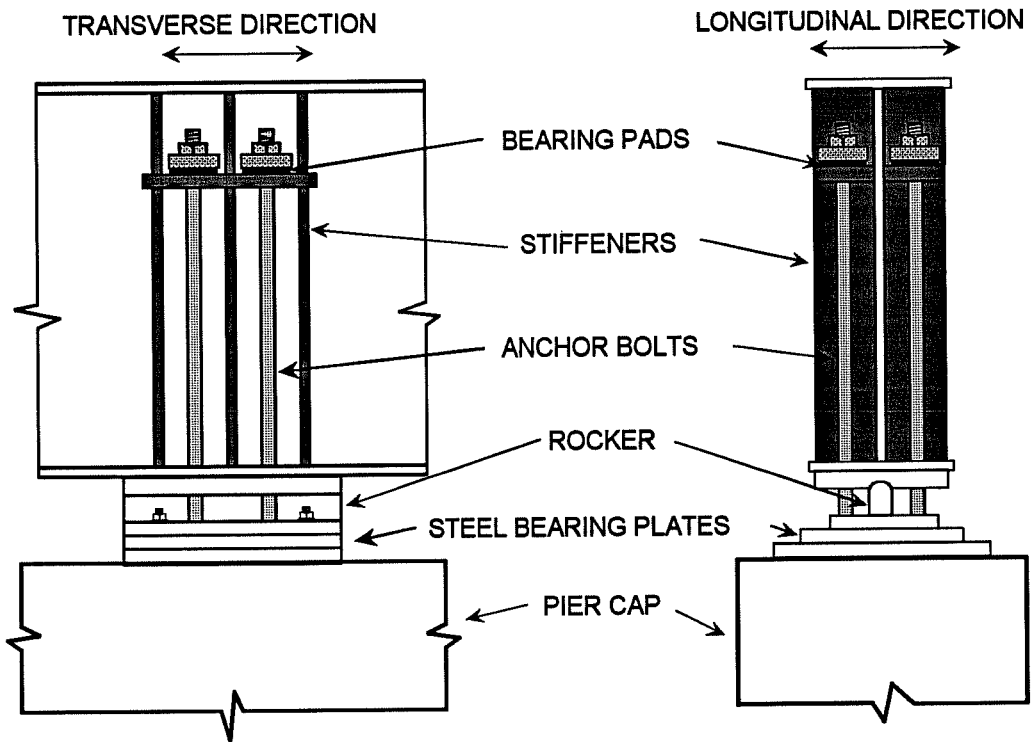


Figure 1.4 Standard TxDOT Connection

is a result of the narrow pier, increases the likelihood of uplift at one of the connections. Since the truck loading is cyclic, the uplift connection will be subject to fatigue. The uplift resisting element is the anchor bolt, which is classified as a Category E detail for fatigue design [1,9]. The failure of the anchor bolts in the connection resisting uplift may result in the collapse of the bridge. Presently the standard connection is used almost exclusively in situations in which no uplift will occur because of the questions regarding the behavior, but future plans envision the use of cap girders placed on the piers eccentrically, a condition that will almost certainly produce uplift. In the longitudinal direction, the direction parallel to traffic flow, the primary concern is the determination of the forces and moments developed in the elements of the connection. The forces and moments will be produced primarily by the rotation caused by the vehicle load and the horizontal displacement resulting from the expansion and contraction of the superstructure in response to temperature changes. The magnitudes of the forces and moments will depend on the restraining stiffnesses of the connection elements. The questions raised above and the cost-ineffectiveness of the standard connection were the impetus for the research presented herein.

The two major purposes of the research were to develop an understanding of the behavior of the standard connection and to develop a cost-effective alternative. Phase I of the research was an examination of the standard connection, in which a large-scale experimental model of the connection was the subject of study. The objectives of the testing were to determine the transverse and longitudinal stiffnesses of the connection and to determine the ultimate strength of the connection. Mathematical modelling was used as a supplement to understanding the behavior. Based on the results of the first phase of the research, a new detail was developed that addressed the deficiencies of the standard connection. Phase II of the research was an examination of the new connection detail, again through large-scale experimentation. Component tests were also conducted on one element of the new detail.

1.2 CONNECTION STATICS

1.2.1 Transverse Direction Behavior

Typically, the cap girder is placed on the pier cap symmetrically as shown in Figure 1.5. The vertical forces that are transferred to the bearings can be decomposed into a vertical force component, P , and a transverse moment, M_T . The transverse moment is a measure of the

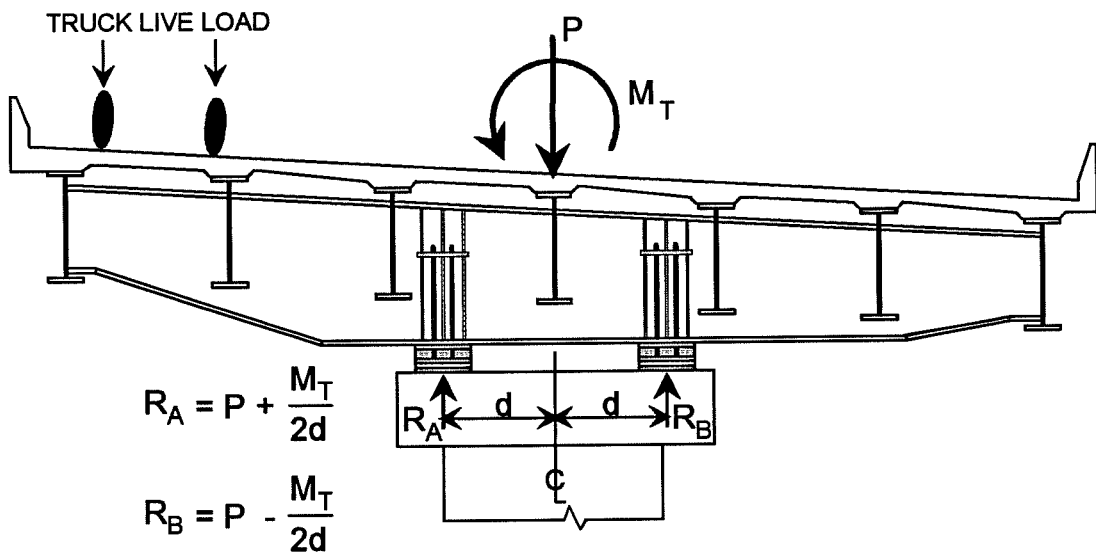


Figure 1.5 Statics of Cap Girder Centered on Pier

eccentricity of the load with respect to the pier centerline and also of the effect of the horizontal curvature of the longitudinal girders. The primary vertical loads on the bridge are dead loads and live loads. If the roadway is straight in plan, equal dead load compressive reactions will be produced in the bearings. If the bridge is horizontally curved, the curvature effect will produce a slight eccentricity and a correspondingly small transverse moment. The overwhelming percentage of the eccentric load comes from the vehicle live load, which is placed on the bridge such that it produces the maximum transverse moment at the bearings. The transverse moment produces a downward compressive force at bearing A and an upward tensile force at bearing B. Since the vehicle loading is cyclic, any component that must resist a tensile force becomes susceptible to fatigue damage. If the dead load compressive reaction at bearing B is greater than its live load tensile reaction, the bearing will remain in compression. If, however, the dead load reaction is less than the live load reaction, uplift occurs and bearing B is loaded in tension. This situation will most likely occur if the bearing connection spacing is small or if the bridge superstructure is placed on the piers eccentrically. A threaded rod or bolt that resists the uplift is classified as a category E detail, where the allowable stress range of 8 ksi. For a connection consisting of four 1.5 inch diameter anchor bolts, the live load uplift force of only 45 kips would produce the category E allowable stress range in each bolt.

To increase the transverse moment at which uplift will occur, the bearings can be pretensioned. The behavior of a pretensioned connection [12] is illustrated in Figures 1.6 through 1.8. The connection consists of a bolt and two sets of compressive elements, shown in Figure 1.6.a. There are four distinct regions of behavior:

Region 1: (Figure 1.6.b)

This is the initial condition of the connection; the pretension force has been applied and no external force is present. The bolt force $B = B_0$ is equal to the compressive reaction between the plate $R = C_0$.

Region 2: (Figure 1.6.c,d)

In this region an external force is present and the compression plates are in contact. The external force is either tensile or compressive. The bolt force increases very little with the increase in the applied load, shown in Figure 1.7. This is because the axial stiffness of the plates is much greater than the axial

stiffness of the bolt and the same change in length (or thickness) produces a much smaller increase in the bolt force than the decrease in the plate reaction. The reaction force between the plates decreases almost linearly with the increase in the external force, as shown in Figure 1.8.

Region 3: (Figure 1.6.e)

The plates have separated due to an applied tensile load; "uplift" has occurred. The bolt force now equals the applied tensile load, $B = T$ and the reaction force between the plates is zero. The external force necessary to produce plate separation is typically a few percent greater than the pretension force; it depends on the axial stiffnesses of the bolt and plates.

Region 4: (Figure 1.6.f)

This region is similar to region 3 but the external load is compression and it is the bolt that is not longer in contact with plates. The reaction force between the plates is equal to the applied compression load and the bolt force is zero. The applied load that produces this condition is typically many times greater than the pretension load. This region is not a practical concern.

Fatigue is primarily a function of the tensile stress range. The shaded areas in Figure 1.7 indicate where the bolt is in tension. In region c.) the bolt is in tension but the change in bolt tension is very small compared to the change in the applied tension; fatigue would probably not be a problem in this area. In region e.) the bolt force is equal to the applied force and fatigue becomes a concern. This of course depends on the actual magnitude of the load range. When a compressive load is applied to the connection, region d.), the bolt is not loaded in tension but the compression relieves some load in the bolt. The bolt cycles between some initial tensile force and a lower tensile force, effectively producing a tensile load range in the bolt. The change in the bolt force, however, is small compared to the change in the applied compression so fatigue is not a concern in this area.

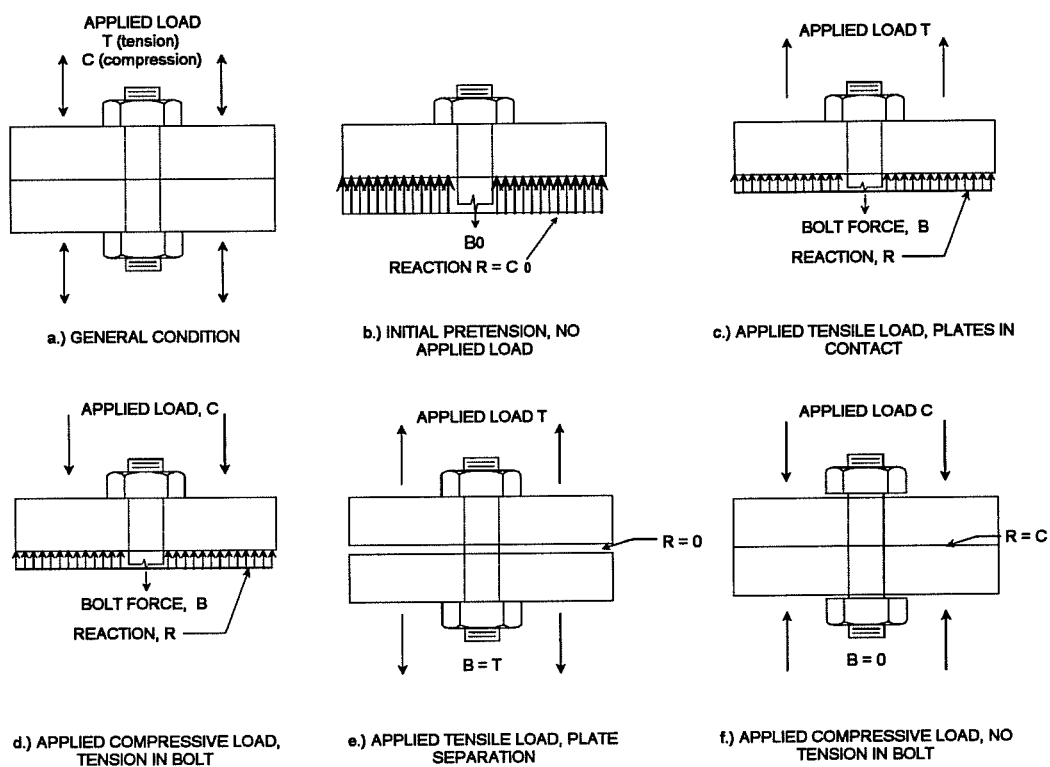


Figure 1.6 Behavior of Pretensioned Connection

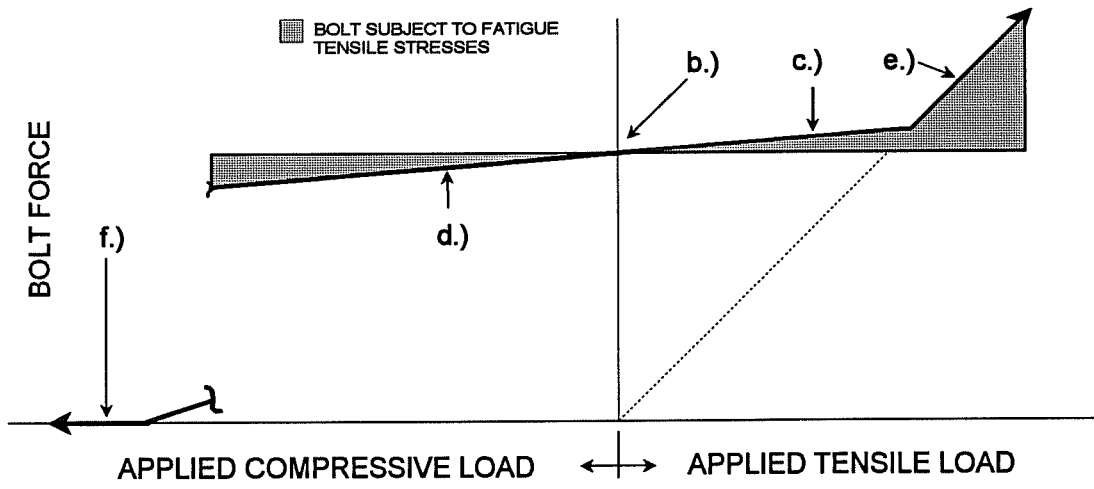


Figure 1.7 Bolt Force vs. Applied Load

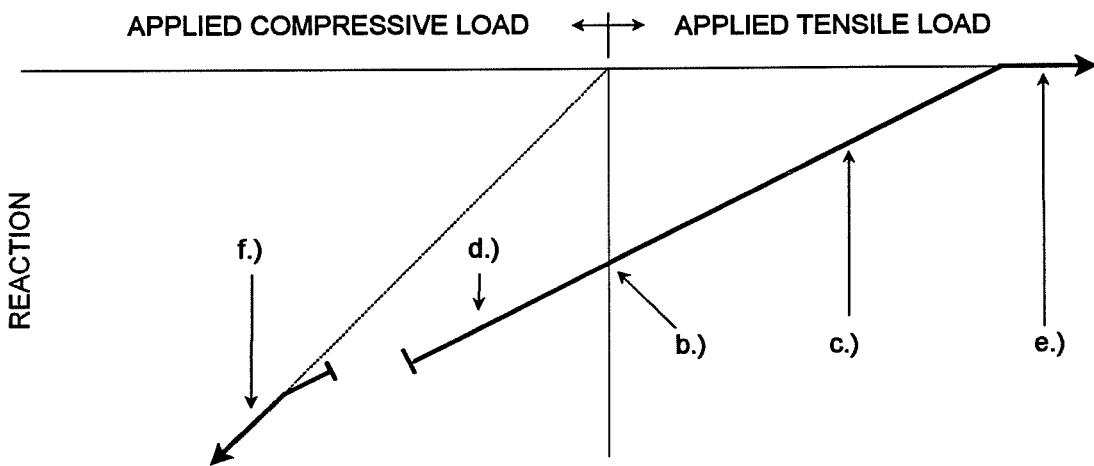


Figure 1.8 Reaction vs. Applied Load

1.2.2 Longitudinal Direction Behavior

Rotation

In the longitudinal direction, the cap girder is subject to rotation caused by the longitudinal steel girders. If the connection restrains the rotation, moments and forces will be produced in the cap girder, the connections, and in the pier. The resisting moment is proportional to the rotational stiffness of the connection. The center of rotation of a cap girder that is free to rotate is about the neutral axis of the longitudinal girders; this position may change, however, for a cap girder that is restrained. Since the bearings are located near the bottom flange of the cap girder and are not coincident with the center of rotation, a horizontal displacement is produced at the bearings. This is shown in Figure 1.9.

Temperature-Induced Displacement

Horizontal forces will develop at the bearings due to the temperature-induced expansion and contraction of the superstructure. A simplified method has been developed [23] to determine the distribution of horizontal forces in continuous bridges. Consider the continuous beam in Figure 1.10. The beam is supported on two types of bearings, designated as Type A and Type B. Type A bearings (expansion bearings, as an example) are free to displace horizontally, but develop a resisting force through friction; the resisting force is

$$H_A = \mu N \quad (1.1)$$

where μ is the coefficient of friction and N is the dead load. Type B bearings (elastomeric bearings, as an example) resist horizontal displacement and develop a resisting force proportional to the displacement; the resisting force is

$$H_B = k \delta \quad (1.2)$$

where k is the shear stiffness of the bearing and δ is the horizontal displacement of the bearing. The shear stiffness of the bearing includes the flexural stiffness of the supporting pier.

The movement of the structure is governed by a temperature coefficient

$$c = \alpha \Delta T \quad (1.3)$$

where α is the thermal coefficient of expansion of the longitudinal girders and ΔT is the change in temperature. At some point along the length of the beam the horizontal displacement will be zero; this is called the point of zero movement and is distance x from the left end of the beam. The distance x is given as

$$x = \frac{c \sum k_i L_i + \sum \mu_i N_i}{c \sum k_i} \quad (1.4)$$

where the subscript i indicates an individual bearing and L_i is the distance from the bearing to the point of zero movement. The variable x is present on both sides of the equation (it is necessary to determine L_i) and so manipulation is needed to determine its value. The quantity $\mu_i H_i$ is positive if the bearing is to the right of the point of zero movement and is negative if the bearing is to the left. This process is similar to finding the centroid of a section, using the bearing shear stiffness in place of the area. The forces in the Type B bearings can now be found by applying Equation 1.2, with $\delta = c L_i$. The bearings in this research are Type B. The effects of the Type A bearings may be conservatively ignored, reducing Equation 1.4 to

$$x = \frac{\sum k_i L_i}{\sum k_i} \quad (1.5)$$

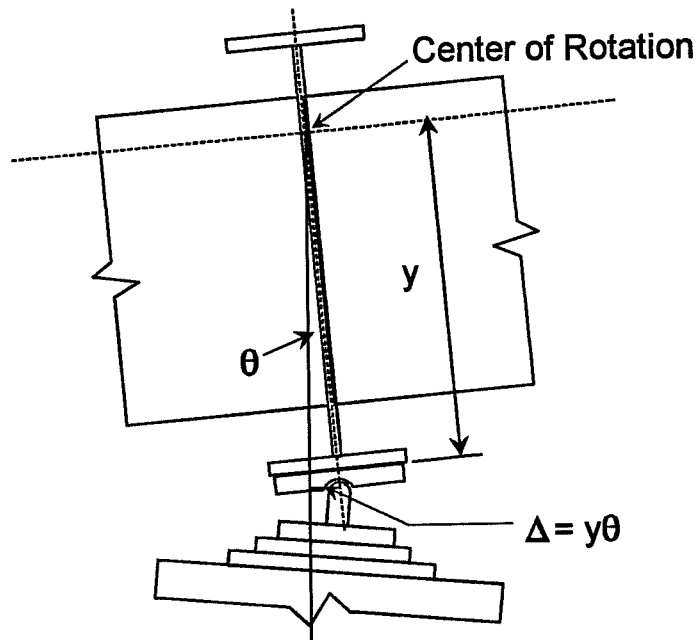


Figure 1.9 Horizontal Displacement of Bearing

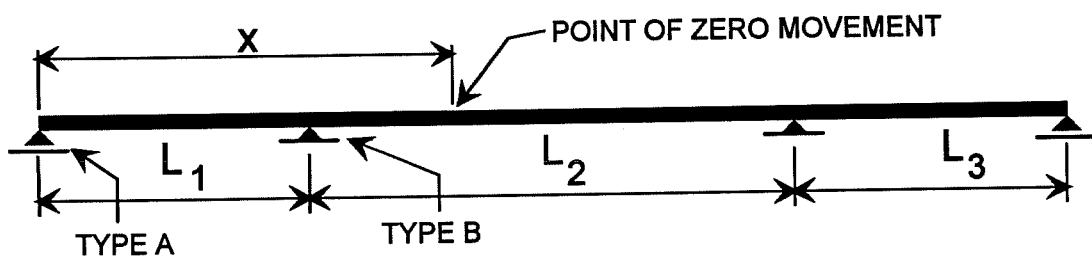


Figure 1.10 Distribution of Horizontal Forces in Continuous Beam

1.3 STANDARD PRACTICE IN THE UNITED STATES

1.3.1 Survey

To assist in the process of developing a new connection detail, a questionnaire was sent to the fifty states to determine the types of details used in situations where the pier dimensions are restricted. Thirty-three responses were received and a summary of the results is shown in Table 1.1. The responses are divided into six categories. These categories were determined by consolidating the numerous connection details and practices of the several states into a manageable number. None of the states that responded had a standard design detail for those situations in which a narrow pier and integral cap girder were necessary. Connections seemed to be designed on a case-by-case basis. Typically, each state had only a few situations in which a narrow pier and integral cap girder were used.

The most common response, cited by 55% of the respondents, was that the integral cap girder / uplift resistant detail is not used or that its use is specifically avoided. The "Does Not Use" response indicates that restricted geometries (limited horizontal and vertical clearances) are not encountered in that state and that standard hammerhead caps are used in conjunction with single column piers. This response could also indicate that the state had concerns regarding the design and behavior of the detail. The majority of the concerns regarded the fatigue behavior of integral steel cap girders and the fatigue strength of anchor bolts. The lack of redundancy of a single steel cap girder was also a major concern. Ten percent of the respondents indicated that they used details that included an integral steel cap girder, but one that was supported at the ends, thereby eliminating uplift. In the states that did encounter situations where the integral cap girder / uplift resistant detail was needed, many different designs were used. Three of the typical details are described in the following sections.

Table 1.1 : Survey Results

DESCRIPTION	%
Does Not Use or Designs to Avoid Uplift	55
Post-Tensioned Concrete Cap Girder	12
Proprietary Bearings (includes Pot Bearings)	10
Steel Box Girder Supported at Ends	10
Single, Steel Jacketed Columns with Pinned Bearing Connection	7
Steel Cap Girder Bolted Directly to Pier	6

1.3.2 Typical Details

Single, Steel-Jacketed Columns with Pinned Bearing Connection

A schematic of this detail is shown in Figure 1.11. A large diameter pin is used to connect the steel cap girder to the single column, steel-jacketed pier. The pin passes through two large plates that are embedded in the concrete column. Shear studs and large diameter holes in the pin plate provide a strong interlock with the concrete. This connection detail, as with many others that were reviewed, is specifically designed to allow longitudinal rotation. The horizontal displacements that develop must be accommodated by the pier column flexibility.

Integral Prestressed Concrete Cap Girder

This detail was used by states that were concerned about the fatigue behavior of steel cap girders. A schematic of a typical cap girder is shown in Figure 1.12. The longitudinal steel girders are initially bolted to a small steel cross-beam (not shown). Formwork is then built around the cross-beam and the connecting longitudinal girders. Metal ductwork is set in the concrete to guide the post-tensioning bars. The rods are anchored to the outside longitudinal stringers and follow a parabolic path.

Steel Cap Girder Bolted Directly to Pier

The simplest detail is probably that shown in Figure 1.13, the steel cap girder bolted directly to the pier cap. The connection hardware is essentially eliminated. The cap girder rests on some type of bearing pad, which may be used in conjunction with grout, and is secured to the pier cap with embedded anchor bolts. Since no element allows free longitudinal articulation, the moment and force that develop from the rotation and horizontal displacement must be mitigated by the pier column. A tall, slender, flexible pier column will attract a small moment and force. A short, stocky, stiff pier column can develop considerable moment and force. In the latter case, a box cap girder is used because a single web cap girder will distort considerably at the longitudinal stringer frame-in points. The single web, by flexing in response to the stringer rotation, becomes the rotational "sink", a function normally performed by a bearing. Distortion-induced fatigue is the likely outcome.

None of the connection designs submitted seemed simple or cost-effective enough to consider as an alternative to the standard TxDOT detail. Many, including the connection described above, involved the use of a pin, which requires labor-intensive machining. The post-tensioned concrete cap girder requires complex detailing of the steel connections within the cap girder. The cap girder placed directly on the pier cap must transfer force along the entire length of the pier, rather than at one or two locations. The distribution of the force along the length would be difficult to determine as compared with the determination of the forces at one or two isolated locations.

1.4 BASICS OF BRIDGE DESIGN

At present (1994), the Texas Department of Transportation designs its bridges using the Standard Specifications for Highway Bridges of the American Association of State Highway and Transportation Officials (AASHTO) [2]. For curved steel plate girder bridges that incorporate integral steel cap girders, the working stress design provisions of the specification are used.

1.4.1 Analysis of the Bridge Superstructure

Two primary methods exist for analyzing bridges that are curved in plan. The first method is to use a three-dimensional finite element analysis; this is the most accurate method of

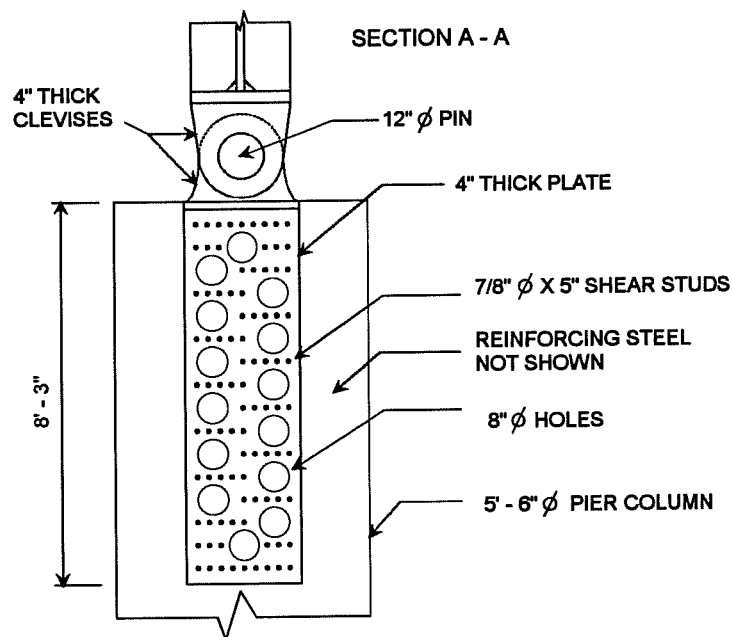
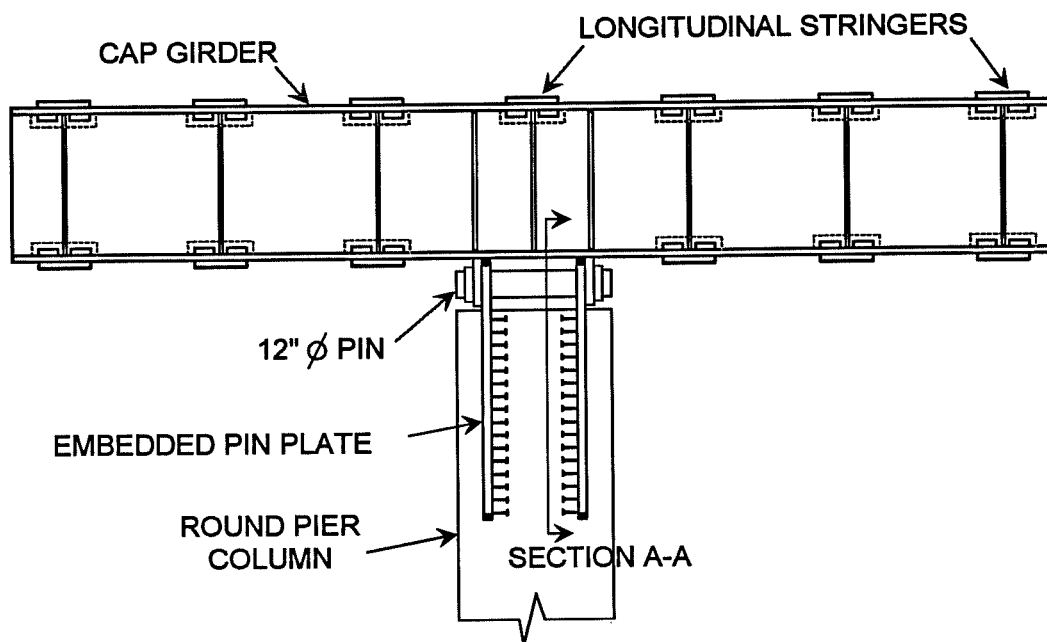


Figure 1.11 Single Column with Pinned Bearing Connection

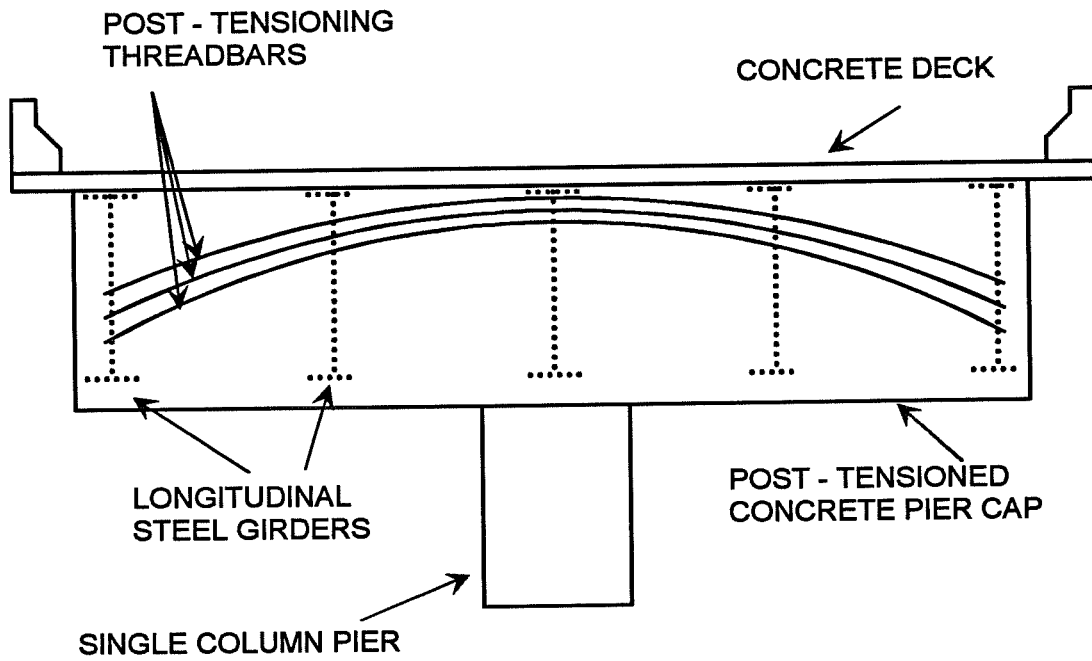


Figure 1.12 Integral Prestressed Concrete Cap Girder

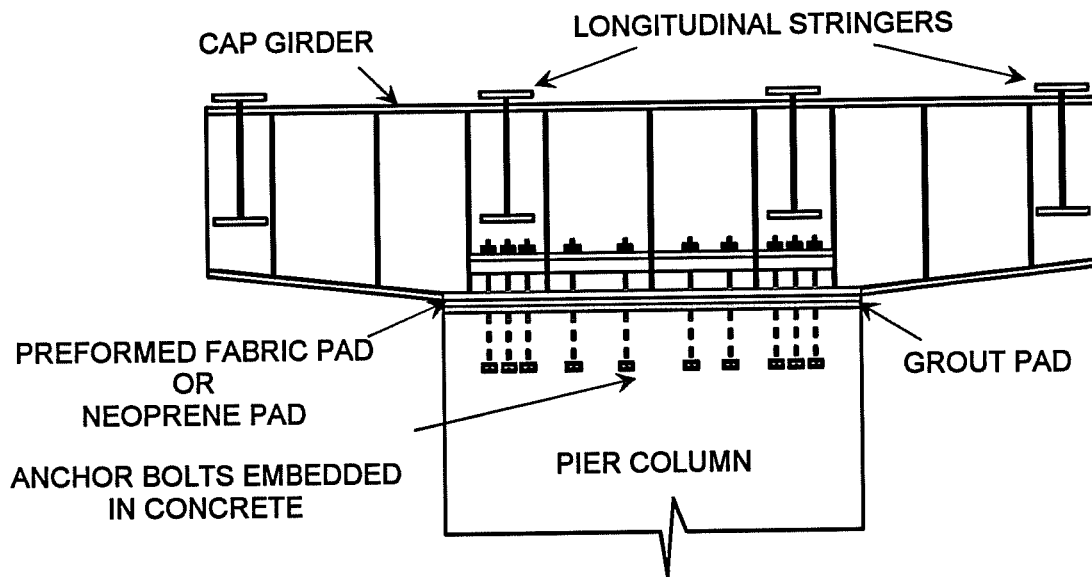


Figure 1.13 Steel Cap Girder Bolted Directly to Pier

analysis but it is also the most time consuming and requires access to a program capable of carrying out the analysis. The second approach is to use an approximate method; the most common simplified analysis technique used is the V-load method [22].

Description of V-Load Method

When a horizontally curved girder is loaded vertically, torsion is produced. In multiple girder bridges, diaphragms are used between the girders to distribute lateral loads and to resist the torsion. A typical diaphragm detail, known as an X-brace, is shown in Figure 1.14. Figure 1.15 shows a segment of a curved girder centered on a diaphragm chord. The arc length, S , is the distance between diaphragms and R is the radius of curvature. The girder flange force can be approximated as $F_f = M_i / h_f$ where M_i is the in-plane bending moment and h_f is the distance between the flange centroids. Since the flange forces are not collinear a lateral force component develops in the diaphragm chord, $H_d = (M_i S) / (h_f R)$. Equilibrium of the diaphragm requires that vertical shear reactions develop in the girders, as shown in Figure 1.16. For two girders, the magnitude of the shear force is $V = (H_{d1} + H_{d2}) h_f / D$ where the subscripts 1 and 2 indicate the girder number and D is the girder spacing. These vertical reactions are termed V-loads. They are assumed to be self-equilibrating along each line of diaphragms and therefore produce no net increase or decrease in force. The V-loads are added to the actual loads on the girder to approximate the curvature loading effects.

The V-load method is based on the assumption that at each support location all of the longitudinal girders are restrained from vertical translation. This assumption cannot be made when a steel cap girder is used at a support. The finite stiffness of the cap girder allows vertical deflections at the longitudinal girder frame-in points. Olsen [13] conducted a study to determine whether the violation of this assumption makes the V-load method inapplicable to bridge systems with integral steel cap girders. Two additional objectives of Olsen's study were to determine the range of reactions and deflections at the bearings for some typical TxDOT bridge systems and to determine the loading patterns necessary to produce the maximum reactions and deflections. The bridges were modelled using ANSYS [16], a general purpose finite element analysis program.

Influence Surfaces for Maximum Reactions and Displacements at Bearings

The critical bearing reactions are vertical compression and uplift and the critical displacement is rotation about the cap girder axis; horizontal forces and displacements are

functions of the rotation. The influence surfaces, in two dimensions, are shown in Figure 1.17. Loading in the grayed areas produces the maximum indicated reaction at the enlarged bearing. These results were determined from analyses of three span continuous bridges that met the following requirements; a constant radius of curvature, diaphragms evenly spaced along the length, the longitudinal girders composite with a concrete deck, integral steel cap girders at the interior two supports, and complete support of all longitudinal girders at the abutments.

Applicability of the V-Load Method

The conclusions of the study with regard to the applicability of the V-Load method to bridge systems with integral steel cap girders are summarized below. The results from the V-Load analyses were compared to the results from a three-dimensional finite element analysis. Unless otherwise stated, the conclusions refer to the determination of the reactions of the cap girder-pier cap connections.

- The V-Load method predicts well the longitudinal girder moments, giving results that tend to be conservative.
- The results from a V-Load analysis are not affected by a change in the radius of the curvature of the bridge.
- The V-Load method is more accurate for end spans that are long relative to the center span; a recommended minimum ratio of the end span length to the center span length is 1 to 1.2. For shorter end spans the V-Load method becomes increasingly inaccurate.
- For Live Load placed in the interior lanes, the V-Load method is conservative. For Live Load placed in the exterior lanes, the V-Load method is unconservative in predicting the compressive bearing reaction by about 5%. If the V-Load method predicts that uplift will not occur, that is, the uplift bearing will be under compression, the estimate of this compression force will be unconservative by about 10%. If the V-Load method predicts uplift, the estimate of the uplift force will be conservative.
- For Dead Load and concentric, uniform Live Load, the transverse overturning moment will produce reactions in the bearings that are small compared to the reactions produced by the vertical loads. Therefore, any inaccuracy in V-Load method will have negligible effect on the final reaction forces.
- If the pier support is off-center, the V-Load method is conservative and fairly accurate.

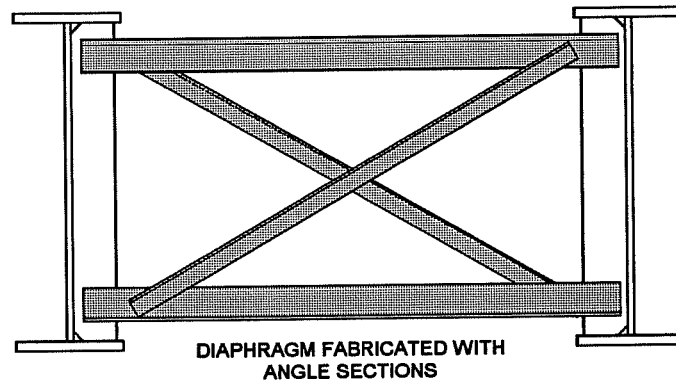


Figure 1.14 Typical Diaphragm Detail

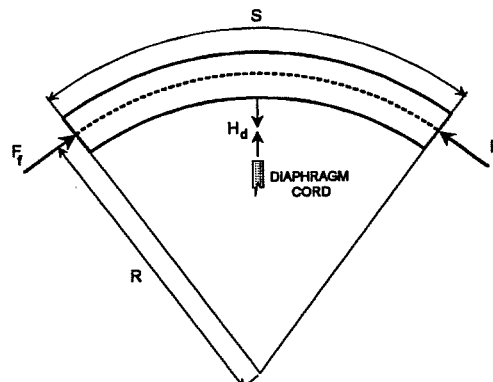


Figure 1.15 Diaphragm Equilibrium

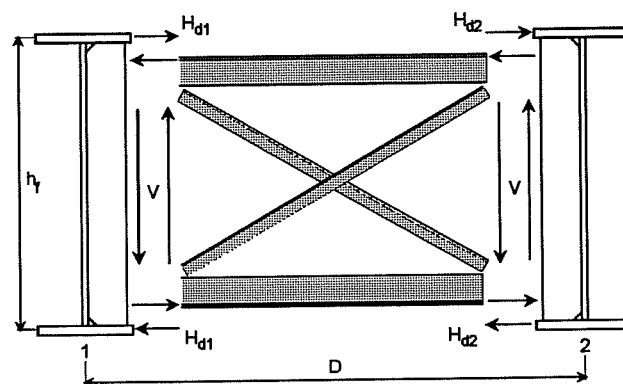
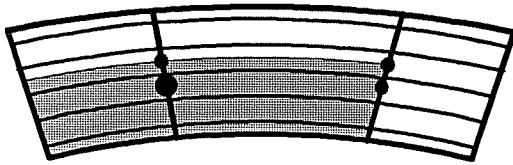
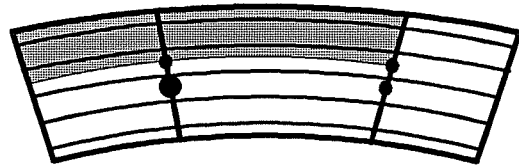


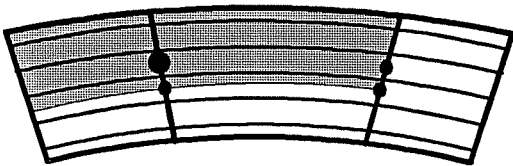
Figure 1.16 Shear Loads on Diaphragm



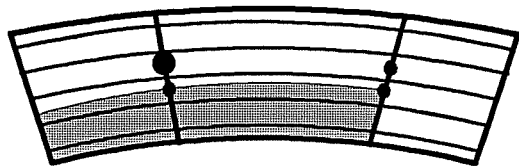
MAXIMUM COMPRESSIVE REACTION
AT INTERIOR BEARING



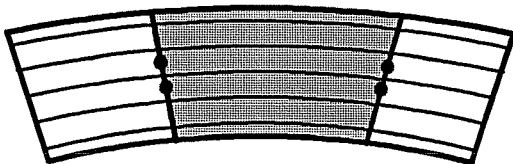
MAXIMUM UPLIFT REACTION
AT INTERIOR BEARING



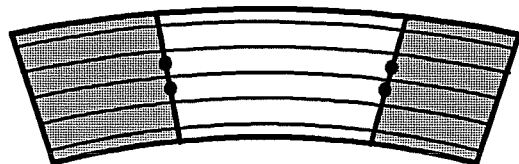
MAXIMUM COMPRESSIVE REACTION
AT EXTERIOR BEARING



MAXIMUM UPLIFT REACTION
AT EXTERIOR BEARING



MAXIMUM ROTATION
DECK ROTATES DOWNWARD
TOWARD CENTER SPAN



MAXIMUM ROTATION
DECK ROTATES DOWNWARD
TOWARD END SPANS

Figure 1.17 Influences Surfaces for Maximum Bearing Reactions

1.4.2 Design Loads

The loads that affect the design of the cap girder and bearing connection are listed below:

● Dead Load	D
● Live Load plus Impact	$(L+I)_n$
● Centrifugal Force	CF
● Wind Pressure on Structure	W
● Wind Pressure on Moving Load	WL
● Braking and Acceleration Force	LF
● Thermal Displacement Force	T

The dead load (D) includes the weight of the concrete deck slab and railings, the longitudinal steel girders, diaphragms, and cap girders, and any permanent utilities.

The live load (L) is the weight of the vehicular traffic. The load is represented by either a standard truck with trailer or an equivalent uniform load in combination with a concentrated load, as shown in Figure 1.18. The load is placed in a design lane, which is 12 feet wide. The standard truck and the uniform load occupy a 10 foot width of the design lane, and the loads are positioned, in their respective lanes, such that they produce the maximum stress in the member being analyzed. When the standard truck loading is used, only one truck per lane is used on each span, and when the equivalent uniform lane load is used it is placed on the entire length of the span; partial span loading is also allowed. The number of design lanes on a bridge is a function of the roadway width and a load reduction factor is used when more than two design lanes are loaded. The factor is 0.9 for three loaded lanes and 0.75 for four loaded lanes. The live load is increased by an impact factor (I) to account for the dynamic nature of the loading.

The centrifugal force (CF) is the normal component of the inertia vector of the moving truck and represents the tendency of the truck to leave its curved path. It is applied at six feet above the deck in the transverse direction.

Wind forces (W) are treated as moving loads that act horizontally in any direction. They can act on the exposed areas of the superstructure and substructure and on any traffic on the bridge. When applied to bridge traffic (WL), the centroid of the force is assumed to act at 6 feet

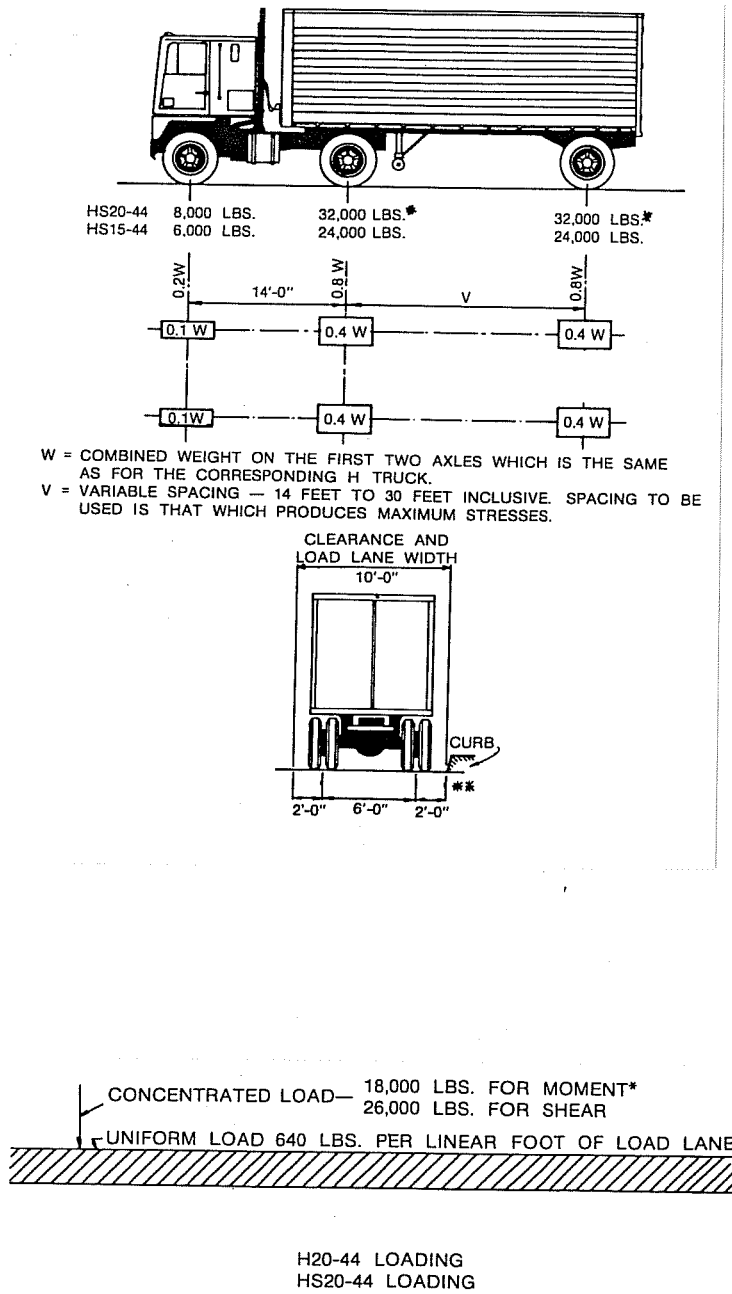


Figure 1.18 Standard Truck and Equivalent Uniform Lane Load
(adapted from AASHTO [2])

above the bridge deck. The acceleration and braking forces (LF) are the tangential component of the inertia vector of the moving truck and are given a magnitude of 5% of the live load headed in one direction.

The thermal displacement forces (T) occur due to the temperature induced expansion and contraction of bridge components. Forces are present only in those components that are restrained from moving.

1.5 DESIGN PROCEDURE FOR STANDARD TxDOT DETAIL

There are five major steps in designing the standard TxDOT connection detail. They are listed below.

- 1.) Determine the loads on the cap girder and bearings
- 2.) Design the bearing plates and rocker elements
- 3.) Size the cap girder web
- 4.) Design the bearing stiffeners
- 5.) Design the anchor bolts and related plates

The following sections will describe each step in detail. The design example given in Appendix A clarifies the general descriptions given below. The design of the cap girder is outside the scope of this investigation and is not addressed.

Determining the Loads on the Cap Girder and Bearing

The load combination that usually controls the design of the connection is shown below:

$$D + (L + I)_n + CF + 0.3W + WL + LF + T$$

The live load, which may consist of one or more lanes, is positioned such that it produces the largest reaction in the member under investigation. The largest vertical forces are produced by the dead load and live load and the largest horizontal forces are typically produced by temperature.

Design of the Bearing Plates and Rocker Pin

The bearing plates and rocker pin are designed to resist the maximum compressive reaction. The bearing plates typically number three or more and are stepped, as shown in Figure

1.19. The bottom plate is sized such that the stress it places on the concrete pier, which is assumed uniform and equal to the load divided by the area of the plate, does not exceed the allowable concrete bearing stress of $0.3f_c' (A_2 / A_1)^{0.5}$. The total thickness of the plates is determined by applying the uniform bearing stress to the bottom of the plate, calculating the moment at the center of the plate, and determining the minimum thickness necessary to limit the bending stress to the allowable of $0.55F_y$. The thicknesses of the individual plates are determined by applying the uniform bearing stress under the plate to the cantilevered portion and carrying out the procedure used for calculating the total thickness. The plates are welded together to form a monolithic unit.

The rocker pin is treated as a pin subject to rotation. The length of the pin is established from the design of the bearing plates; the thickness is determined such that the bearing stress on the pin does not exceed the allowable of $0.4F_y$. The radius at the top of the pin is machined to an ANSI 125 finish. The rocker pin is welded to the top bearing plate with a partial penetration groove weld. The minimum weld size usually governs the design. The dimensions of the sole plate are typical, with the radius of the groove 1/16 inch larger than the radius of the pin.

Sizing the Cap Girder Web

The cap girder web is designed to preclude the need for intermediate stiffeners. It must resist the shear caused by the vertical loads and is tapered outside of the connection area.

Design of the Bearing Stiffeners

The bearing stiffeners, three to a connection and located on both sides of the web, extend along the entire depth of the web and are interrupted by a horizontal cross-plate. The stiffeners, along with a portion of the web, are designed as an axially loaded column. A mill-to-bear specification is made for the bottom flange-stiffener junction and a tight fit specification is made for the remaining junctions. The stiffeners are welded to the web and the cross-plate but are not welded to the cap girder flanges.

Design of the Anchor Bolts and Related Plates

The minimum diameter of anchor bolts for bridges with spans over 100 feet is 1.5 inches. Four anchor bolts per connection are typical and the embedment depth is 20 bolt diameters. Four 1 inch diameter bolts are also used to secure the bearing plates to the concrete pier. The

anchorage for the bolts is shown in Figure 1.20. It consists of a metal template to position the bolts and double nuts to hold the bolts in place. It is not necessary to check the anchorage and pull-out capacity of the anchor bolts if uplift does not occur.

The cross-plate is designed as a fixed-fixed beam with a point load at the center. The magnitude of the load is equal to the allowable tensile load of the anchor bolt. The bearing pad, which is located between the cross-plate and the nut-washer plate, is also designed to resist the allowable tensile load of the anchor bolt. A 1 inch thick preformed fabric pad is typically used, with an allowable compressive stress of 2000 psi. The nut-washer plate is designed in a manner similar to the bearing plates.

The anchor bolts are tightened using the turn-of-the-nut-method, typically to a condition of snug tight plus one-quarter turn. The snug tight condition is defined as the tightness that exists when all plies of the joint are in firm contact; this can be achieved by the full effort of a man using an ordinary spud wrench [3]. After the bolt is snugged the nut is turned some fraction of a turn further, in this case one-quarter turn. The final tightening is done after all of the dead load is on the structure.

1.6 SCOPE OF RESEARCH

The two major purposes of the research were to develop an understanding of the behavior of the standard connection used by TxDOT between an integral steel cap girder and a concrete pier and to develop an alternative connection that is simpler and more cost-effective. A brief examination of the connection behavior in the two directions was presented in Section 1.2. Possible alternative designs were examined in Section 1.3. The analysis and design of the standard connection and of any new connection will be based on the provisions of the AASHTO bridge code, which was introduced in Section 1.4. The end result of the research will be the development of a new connection detail and a design procedure for that connection. The design procedure for the standard connection was presented in Section 1.5.

Phase I of the research, which dealt with an examination of the standard connection, is covered in Chapters 2 through 5. Chapter 2 describes the test program and the development of the test specimen. A three-quarter scale experimental model of a prototype connection was the subject of the study. Tests were conducted in the transverse and longitudinal directions. In addition, tests were conducted to determine the pretension force that could be induced in the

anchor bolts. The primary objectives of the Phase I research were to determine if the anchor bolts developed fatigue critical stresses due to longitudinal rotation and to determine if the connection could be used to resist uplift. The test results are presented in Chapter 3 and a discussion of the results are presented in Chapter 4. A summary of the results and the conclusions of the Phase I research are given in Chapter 5. A new detail is developed based on the results of the Phase I research and is introduced in Chapter 5.

Phase II of the research focused on the testing of the new detail. The testing consisted of large-scale tests and component tests on the bearing detail. Chapters 6 and 7 cover the component tests on the bearing detail, which consisted of compression tests and fatigue tests. Chapter 8 covers the large-scale tests on the new connection system. These tests were conducted in a manner similar to the large-scale tests conducted in the first phase of the research. A summary of the results and the conclusions of the Phase II research are given in Chapter 9. The design procedures for the new connection detail are presented in Chapter 10 and a synopsis of the research is given in Chapter 11.

The research was limited to an examination of the connection itself, which included the bearing hardware, the anchor bolts, and the stiffener details. The pull-out capacity of the bolt anchorage was not part of the experimental program, but some design recommendations are included. The design of the cap girder was not addressed, though its interaction with the connections was considered. Additional research has been conducted by Denio [6], who studied the compressive capacity of the pier cap, and by Olsen [13], who examined the distribution of the vertical forces to the bearing connections.

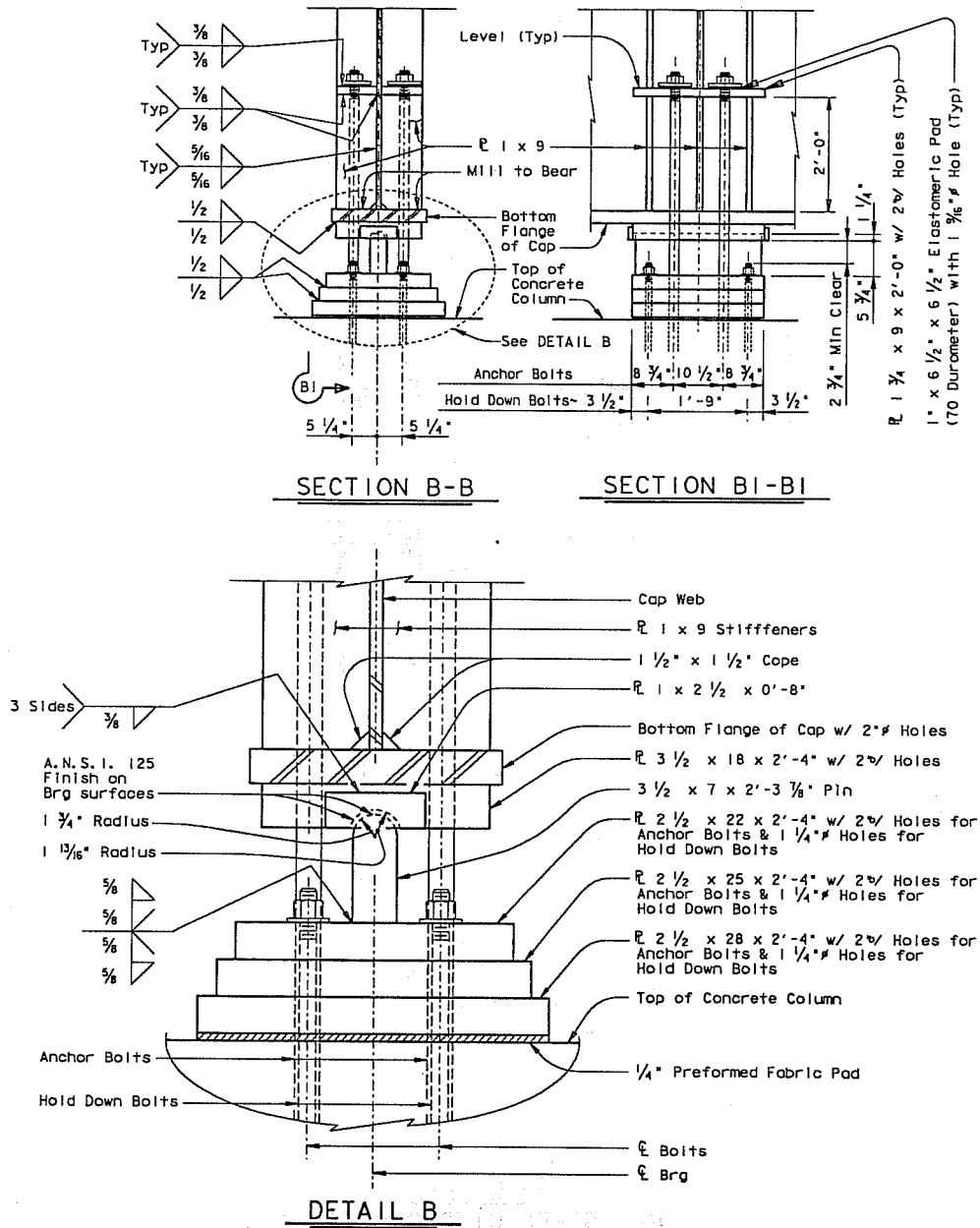


Figure 1.19 Typical Details of Standard TxDOT Connection
 (from TxDOT Bridge Design Guide [21], p.7-70)

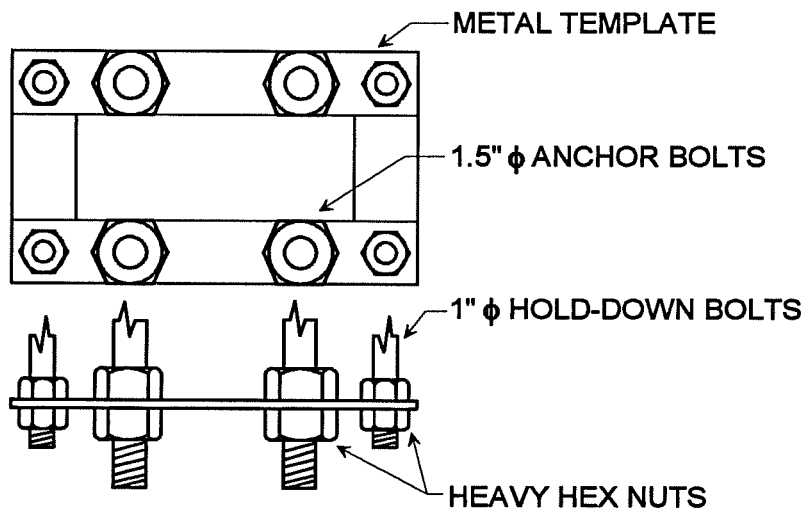


Figure 1.20 Bolt Anchorage

CHAPTER 2 : PHASE I TEST PROGRAM

2.1 PROTOTYPE BRIDGE DESIGN

The first step in the testing program was to design and build the test specimen. The test specimen included the cap girder, the pier cap, and the connection. The design parameters included the span lengths and deck width of the roadway, the longitudinal girder spacing, the bearing connection spacing, and the slab thickness; TxDOT provided a range of values for each of the parameters. Once the parameters were established, a particular set of parameters was selected to be used in the design of prototype components. The test specimen was an approximate 3/4 scale model of the prototype.

Design Parameters

TxDOT provided the following information with regard to the design parameters; the overpasses are typically three span, with a constant radius of curvature, and a center span 20% longer than the end spans. The curvature was ignored in the parameter study. The minimum length of the end spans is 100 feet, corresponding to a 120 foot center span, and the maximum length of the end spans is 180 feet, corresponding to a 220 foot center span. The deck width is 44 feet for 2 lane bridges and 56 feet for three lane bridges; the girder spacing is not less than 7.5 feet and does not exceed 9 feet. There are two types of girder configurations, odd and even; the odd configuration indicates that there is an odd number of girders. In this type of configuration, one girder is aligned between the bearing connections. The even configuration signifies the use of an even number of girders, with the bearing connections between two girders. The slab thickness is usually 8, 9, or 10 inches and the connection spacing varies from 3 feet to 12 feet. This information is shown schematically in Figure 2.1.

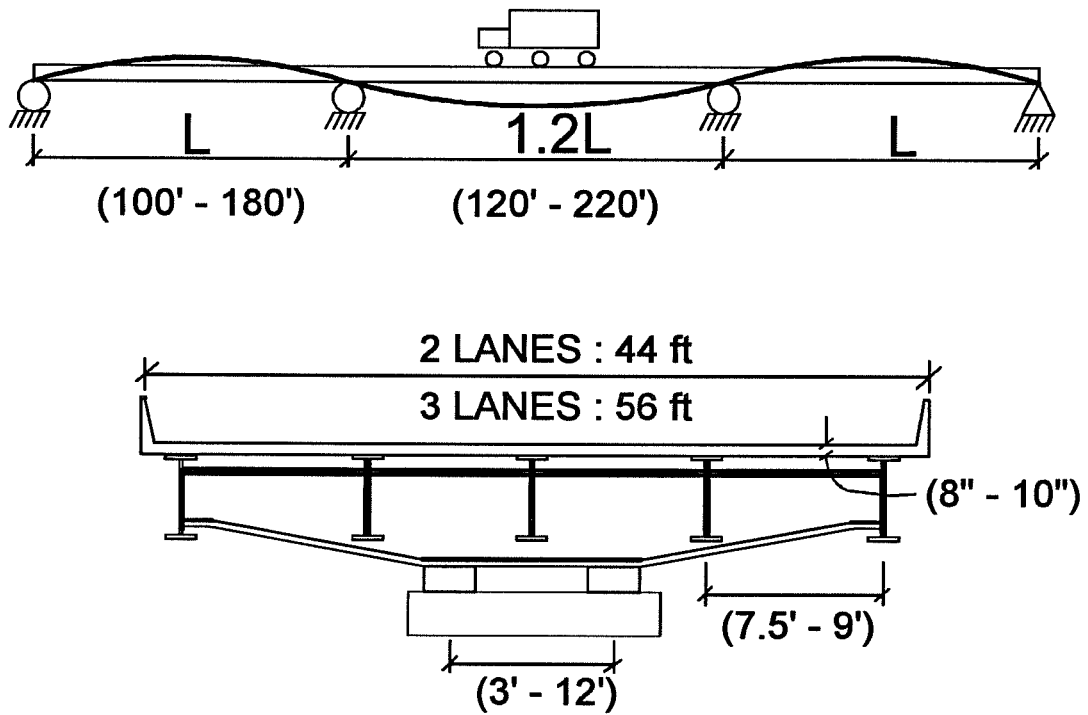


Figure 2.1 Design Parameters

Bridge Systems Analyzed

The prototype bridge was selected based on the results from an analysis of four basic bridge configurations. The parameters of these bridge systems are listed in Table 2.1.

Table 2.1 : Bridge System Parameters

# of Lanes	Span Lengths	# of Girders @ Spacing	Slab Thickness
2	100' - 120' - 100'	6 @ 8' - 0"	8"
			9"
			10"
2	180' - 220' - 180'	6 @ 8' - 0"	8"
			9"
			10"
3	100' - 120' - 100'	7 @ 8' - 8"	8"
			9"
			10"
3	180' - 220' - 180'	7 @ 8' - 8"	8"
			9"
			10"

A two lane bridge and a three lane bridge, each with the minimum span lengths and maximum span lengths, were examined. For each system, the concrete slab thickness was varied from 8 inches to 10 inches. The purpose of the analyses was to determine the uplift force as a function of the bearing connection spacing. By using the minimum and maximum span lengths, bounds for the uplift force could be established. A plan view of the bridge systems and the loading pattern is shown in Figure 2.2. A cross-section view of the two lane bridge and its loading pattern is shown in Figure 2.3.

Two load cases were examined, dead load and vehicle live load. The dead load comprised the weight of the concrete deck slab, given as S_{DL} , and the self-weight of the longitudinal girders, given as G . The span-to-depth ratio of the longitudinal girders was assumed

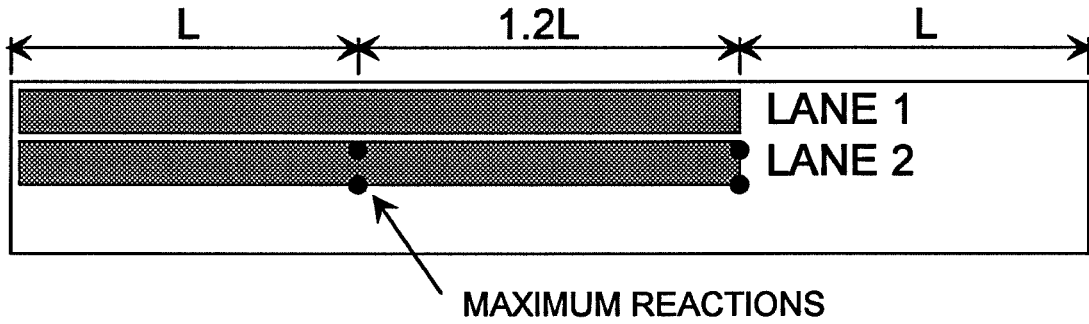


Figure 2.2 Plan View of Loading for Prototype Bridges

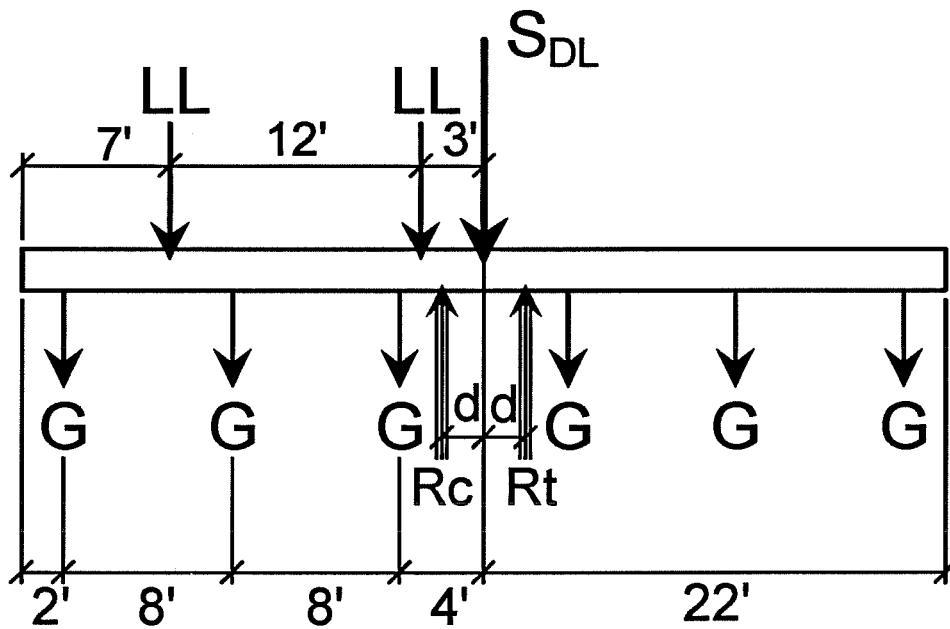


Figure 2.3 Cross-section View of Loading for Prototype Bridges

to be 30, using the center span length, and the self-weight was estimated. The vehicle live load consisted of truck lane loads and an accompanying point load, given as LL. Each lane is 12 feet wide and the lane load is assumed to be distributed over a length of 10 feet. The lane load can be located anywhere within the 12 foot wide lane. The AASHTO lane loading is used to simulate the load of a line of trucks and is equal to 640 plf. The maximum reaction at the bearing locations occurs when the two adjacent spans are loaded. An 18 kip point load, one for each lane, was placed at the bearing location, centered within the lane load. To produce uplift at one of the bearings, the live load must be placed on the roadway eccentrically with respect to the centerline of the roadway. In both the two lane and the three lane bridges, two loaded lanes were necessary to produce the worst-case uplift condition.

Results of Analyses

The results of the analyses are shown graphically in Figure 2.4, as a plot of bearing uplift force R_t versus the connection spacing, which is defined as $2d$ (shown in Figure 2.3). Only the results from the analyses using an 8 inch slab thickness are shown. The change in slab thickness did not have much affect on the connection spacing necessary to produce uplift. The connection spacing at which uplift occurs is primarily a function of the deck width and the span length. A wider deck width increases the eccentricity of the live load; this produces a larger overturning moment at the bearings and consequently a larger tensile force in the bearing subject to uplift. The bearings must be farther apart in order to prevent uplift from occurring. An increase in span length increases both the magnitude of the live load and the magnitude of the dead load. An increase in dead load causes the curves to move to the left, indicating that the bearings can be spaced at a smaller distance without uplift occurring. The most critical bridge configuration is that in which uplift commences at the widest bearing spacing; this configuration is the three lane short-span bridge and uplift begins at a connection spacing of 6.26 feet. Uplift begins in the two lane short-span bridge at a connection spacing of 4.46 feet, at 4.22 feet in the three lane long-span bridge, and at 3.00 feet in the two lane long-span bridge. The three lane short-span bridge was chosen for the prototype design. The result show that, in general, uplift will not occur for symmetrically placed pier caps until the connection spacing becomes less than 10% of the roadway width. This translates into connection spacings from about 3 feet to 6 feet, spacings which are for the most part impractical because they would place limitations on the widths of the bearings.

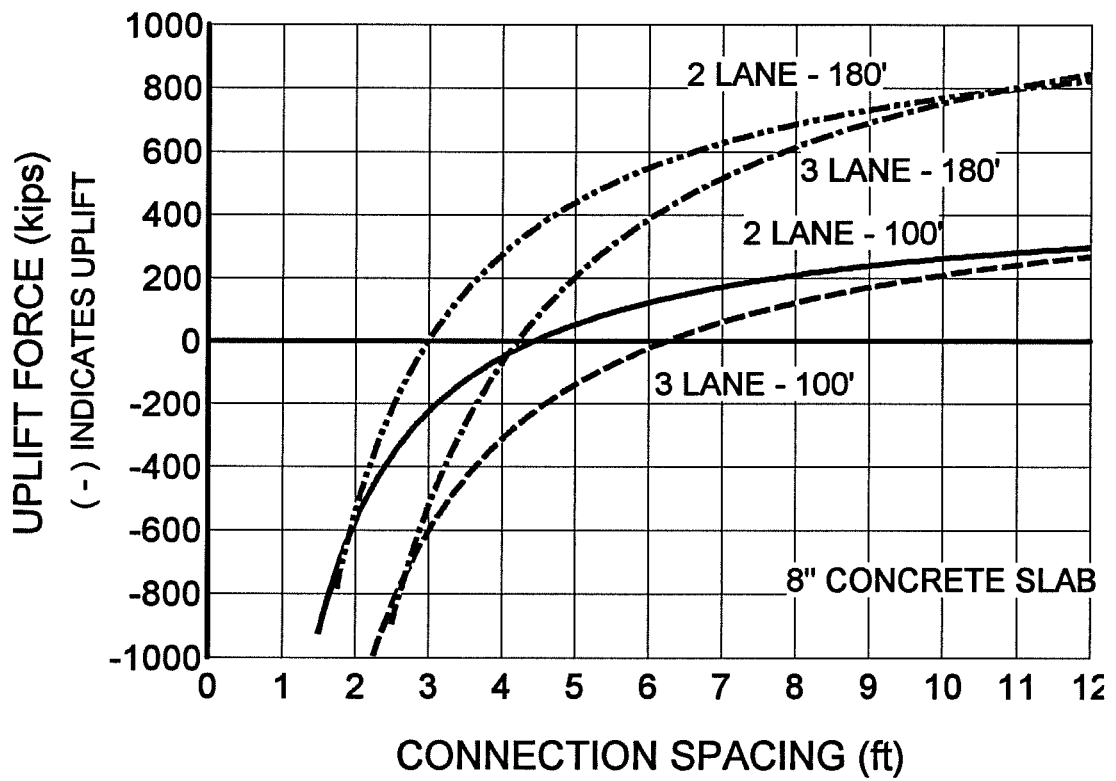


Figure 2.4 Bearing Uplift Force vs. Connection Spacing

Uplift, therefore, will not occur for the vast majority of situations in which the cap girder is placed on the pier cap symmetrically.

Prototype Bridge Design

The purpose of designing the prototype bridge system was to produce a cap girder design and connection detail that could be scaled down and tested experimentally. The connection spacing of the three lane short span bridge was decreased to 4 feet and the deck width was decreased to 47 feet. The deck width was decreased to lessen the load on the cap girder, which would result in a smaller size, and the connection spacing was reduced to match the test floor bolt pattern spacing. By reconciling the spacing difference, a more efficient test setup could be designed. The analysis was simplified by using the same size for all of the longitudinal girders; the section was prismatic. A summary of the sizes of the primary components of the prototype cap girder and the test specimen is shown in Table 2.2. The test specimen is an approximate 3/4 scale model of the prototype. The cap girder acted as a lever arm that applied an overturning moment to the bearings. Only the vehicle live load was simulated during testing. A schematic of the test connection is shown in Figure 2.5.

Table 2.2 : Cap Girder Dimensions

	PROTOTYPE	3/4 SCALE MODEL	TEST SPECIMEN
FLANGE	20" X 2"	15" X 1.5"	16" X 1.5"
WEB	60" X .75"	45" X 9/16"	48" X 3/4"
STIFFENER	9" X 1"	6.75" X .75"	7" X .75"
ANCHOR BOLT	2"φ / 2.5 in ²	1.75"φ / 1.88 in ²	1.5"φ / 1.41 in ²
UPLIFT FORCE	64 kips	48 kips	45 kips
COMPRESSIVE REACTION	1050 kips	788 kips	750 kips

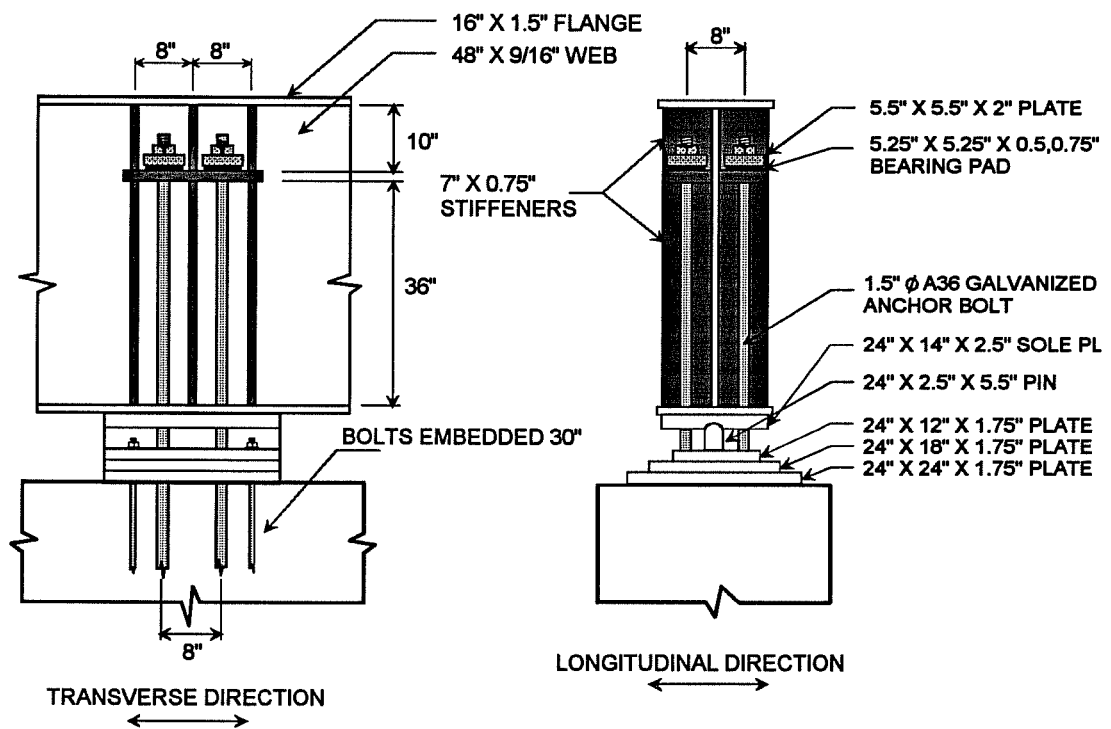


Figure 2.5 Schematic of Test Connection

The design for the concrete pier cap and pier column was a scaled down version of a typical design used in bridges where the connection spacing is 12 feet. The actual design and detailing of the concrete and its reinforcement is addressed by Denio [6].

2.2 TEST PROGRAM

2.2.1 Test Variables

The experimental test program comprised pretensioning tests, transverse stiffness tests, longitudinal stiffness tests, cyclic loading tests in the transverse direction, and an ultimate strength test. The primary variable in the tests was the bearing pad. The bearing pad is used to reduce the axial stiffness of anchor bolt and thereby reduce the force in the anchor bolt produced by the longitudinal rotation.

The stiffness of the connection is a direct function of the flexibility of the anchor bolts and the compressibility of the bearing pads. To examine a connection with different stiffnesses, a number of bearing pads, with a wide range of stiffnesses, were used as variables in the tests. Three types of materials were used, each of two thicknesses; an unreinforced elastomeric bearing pad, made from neoprene rubber (NEO); a non-engineered, pre-formed structural bearing pad that is constructed from a homogeneous mixture of new, unvulcanized rubber and unused, randomly-oriented synthetic fiber reinforcement (ROF); and preformed fabric pad, which is composed of multiple layers of cotton duck impregnated and bound with a high quality natural rubber (PFP). Approximate static stiffnesses of the pads, in kips per in, are shown in Table 2.3. The stiffnesses vary from 52 k/in for the 3/4" neoprene to 890 k/in for the 1/2" preformed fabric pad. Also shown is the stiffness of the anchor bolt, which is a calculated value (see Section 4.1.1).

Table 2.3 : Bearing Pad Stiffnesses

BEARING PAD	STIFFNESS
	kips per inch
1/2" PFP	890
3/4" PFP	799
1/2" ROF	241
3/4" ROF	171
5/8" NEO	75
3/4" NEO	52
7/16" BOLT, 1.5" dia	723

2.2.2 Stiffness Tests

Test Schedule

The test schedule for the transverse stiffness tests is shown in Table 2.4. Six of the tests were conducted with the bearing pads; the variables considered were the pad material and pad thickness. One test was conducted with no bearing pads. Also shown is the amount of pretension used in each test. The bolt tightening procedure used was the turn-of-the-nut method; in the field, the bolts are typically tightened to snug plus 1/4 turn.

Table 2.4 : Test Schedule for Stiffness Tests

TEST	BEARING PAD	PRETENSION
1	3/4" NEO	SNUG + 1/2 TURN
2	5/8" NEO	SNUG + 1/2 TURN
3	3/4" ROF	SNUG + 1/4 TURN
4	1/2" ROF	SNUG + 1/4 TURN
5	3/4" PFP	SNUG + 1/4 TURN
6	1/2" PFP	SNUG + 1/4 TURN
7	NO PAD	SNUG

Test Setup

A schematic of the test setup for the transverse loading tests is shown in Figure 2.6. The setup consisted of the cap girder, the pier cap, a 1' section of the pier column, and a foundation to anchor the setup to the test floor. The concrete portion was cast in two stages, as is done in the actual construction. The pier column is cast first and is allowed to set; the pier cap is then cast at a later date. The cap girder was 15 feet long and cantilevered out past the left bearing 8 feet, acting as a loading arm to apply an overturning moment to the bearings. The girder and bearing steel were A572 GR50 material (yield stress is 50 ksi) and the anchor bolts were 1.5 inch diameter galvanized A36 steel (yield stress is 36 ksi); the concrete strength was 3500 psi. The foundation was secured to the test floor at each end with high strength bolts. Figure 2.7 shows the test setup as constructed in the lab. A schematic of the test setup for the longitudinal rotation tests is shown in Figure 2.8. A truss structure composed of angles and T-sections was designed to simulate a longitudinal girder and to act as a loading arm. The rotation was applied by loading the end of the truss structure. The connection to the cap girder web is typical of what is done in the field; clip angles are used to transfer shear, the bottom flange of the longitudinal girder was welded to a plate that was bolted to the cap girder web, and the top flange was connected over the top of the cap girder. The test setup as constructed in the lab is shown in Figure 2.9.

Instrumentation

The instrumentation consisted of strain gages, used on the anchor bolts to measure bolt force, and linear potentiometers, located along the length of the girder to measure the vertical bottom flange deflection and the bottom flange longitudinal rotation. Electronic rotation gages were located on the top flange of the cap girder. The instrumentation is shown in Figure 2.10.

Test Procedure

The test procedure for the transverse stiffness tests was the same for each test. After the bolts were pretensioned, small static loads were applied to the test specimen and sensor readings were taken after each. Load was then applied in increments such that 10 to 15 readings could be taken before the test end. Each load was paused for 5 minutes or 10 minutes, depending on the bearing pad material, to allow creep of the pads. Readings were taken before and after the pause and the load was held constant during this time.

2.2.3 Pretensioning Tests

The primary objective of the pretensioning tests was to determine the magnitude of the pretension force that could be put in the anchor bolts using a typical installation procedure. A secondary objective was to determine how the tightening sequence affects the distribution of the bolt forces. The schedule is shown in Table 2.5. Each test is identified by an alphanumeric code. The first three letters indicate bearing pad material; the next two numbers are the nominal durometer hardness of the bearing pad; the following two numbers are the thickness of the pad in sixteenths of an inch; the last number is the bolt tightening sequence number.

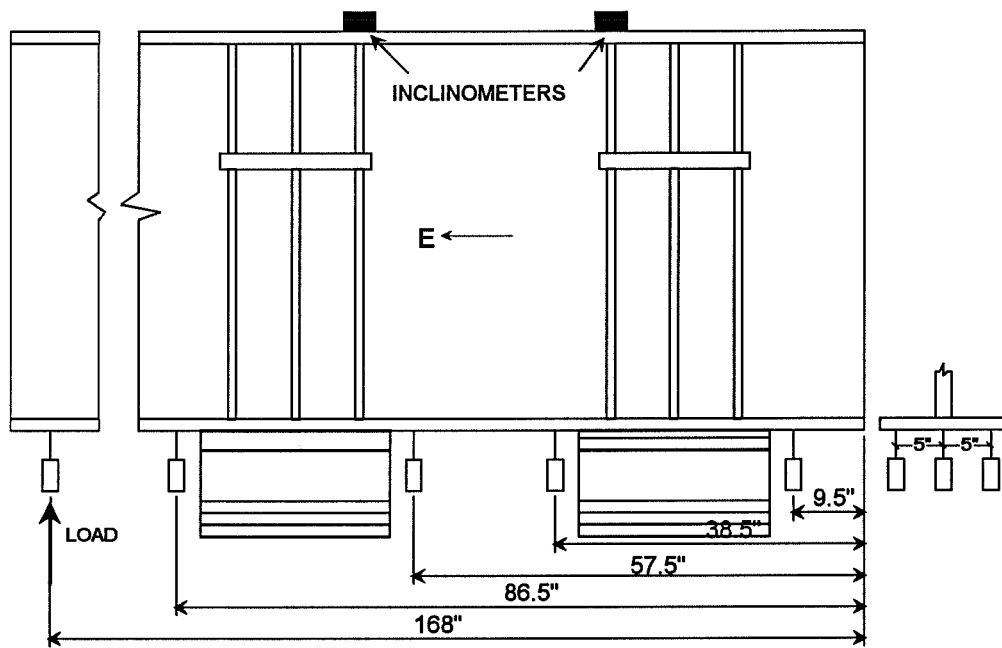


Figure 2.10 Schematic of Instrumentation

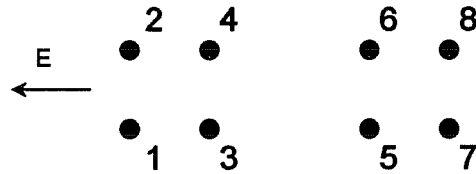


Table 2.5 : Test Schedule for Pretensioning Tests

TEST	MATERIAL	HARDNESS	THICK	TENSION	TIGHTENING SEQUENCE
		(durometer)	(inches)		
NEO70101	Neoprene	70	5/8	Snug+1/2T	1423
NEO70102	Neoprene	70	5/8	Snug+1/2T	1342
NEO70121	Neoprene	70	3/4	Snug+1/2T	1423
NEO70122	Neoprene	70	3/4	Snug+1/2T	1342
PFP90081	Fabric Pad	90	1/2	Snug+1/4T	1423
PFP90082	Fabric Pad	90	1/2	Snug+1/4T	1342
PFP90121	Fabric Pad	90	3/4	Snug+1/4T	1423
PFP90122	Fabric Pad	90	3/4	Snug+1/4T	1342
ROF85081	ROF	85	1/2	Snug+1/4T	1423
ROF85082	ROF	85	1/2	Snug+1/4T	1342
ROF85121	ROF	85	3/4	Snug+1/4T	1432
ROF85122	ROF	85	3/4	Snug+1/4T	1342
STE00321	Steel Plate	-	2	Snug+1/4T	1432
STE00322	Steel Plate	-	2	Snug+1/4T	1342

The pretensioning tests were conducted on the east, or uplift, connection. A pipe wrench with a 12 inch long handle was used for the tightening. A pipe extension was used as a lever arm to tighten the bolts past snug. The number of turns past snug was determined by marking the nut and then marking the cross plate at the location where the nut had to be turned to.

Two bolt tightening sequences were used. The procedure for each test is shown below.

- 1.) Hand-tighten all four bolts.
- 2.) Snug tighten first bolt in sequence using 12" long pipe wrench.
- 3.) Tighten same bolt fraction of a full turn further, either 1/4 turn or 1/2 turn. Bolt is fully tightened. Record strain readings.
- 4.) Repeat steps 2.) and 3.) for next bolt in tightening sequence. (Each step took approximately two minutes to complete).
- 5.) After all four bolts are tightened, pause ten minutes to allow creep and record strain readings.

CHAPTER 3 : PHASE I TEST RESULTS

3.1 MATERIAL PROPERTIES

Anchor Bolts

The anchor bolts used in the full scale tests were made from 1.5 inch diameter A36 round stock and had cut threads for six inches at each end. Two rod lengths were ordered, 91" and 66". The anchor bolts were hot - dip galvanized and the nuts were tapped oversized to accommodate the galvanizing. The strength properties of the anchor bolt-nut assemblage was determined by a tension test on one rod of each length. The bolts were tested in a 600 kip capacity hydraulic test machine. The strain was measured with an 8" extensometer and the total bolt deflection was measured by a linear potentiometer. The bolt shank was held by grips at the top and the bottom was secured by the nut and washer plate used in the full scale tests. The results of the tests are shown in Table 3.1. In both cases, the bolt fractured at the first thread that engaged the nut. A "G" indicates that calculations were made using the gross area and a "T" indicates that calculations were made using the tensile stress area [3, p.4-147], which is the area of the threaded portion of the bolt and is defined as

$$A_t = 0.7854 \left(d_b - \frac{0.9743}{p} \right)^2 \quad (3.1)$$

where d_b is the nominal diameter of the bolt and p is the number of threads per inch.

Table 3.1 : Anchor Bolt Material Properties

DESC	AREA in ²	STATIC YIELD		DYNAMIC YIELD		ULT. STRENGTH	
		STRESS ksi	LOAD kips	STRESS ksi	LOAD kips	STRESS ksi	LOAD kips
		91" G	1.77	42.1	74.5	45.2	79.8
91" T	1.41	52.8	74.5	56.6	79.8	69.1	97.4
66" G	1.77	40.6	71.9	43.2	76.4	58.2	103
66" T	1.41	51.0	71.9	54.2	76.4	73.0	103
MILL	-			45.5		69.0	
ASTM	-			36		58	

Concrete Pier Cap and Foundation

The compressive strengths of the concrete elements used in the large scale tests are shown in Table 3.2. The foundation and column were specified to have a strength of 5000 psi and the pier cap was specified to have a strength of 3600 psi. The target pier cap concrete strength was the same as that specified in the field. Shown in the table are the strengths at 7, 14, 21, and 28 days and the strength at the time of the ultimate strength test. The values are the average of strengths obtained from three standard cylinder tests.

Table 3.2 : Concrete Material Properties

AGE (days)	COMPRESSIVE STRENGTH (psi)	
	PIER CAP	COLUMN & FOUND.
7	2561	4247
14	3047	4813
21	3353	5120
28	3474	5543
201	3880	6243

3.2 PRETENSIONING TESTS

The functions of the anchor bolts in the detail are to provide a positive connection from the cap girder to the pier cap and to resist uplift. Most of the existing bridge systems using this detail are not subject to uplift, so the anchor bolts serve no purpose once the bridge is opened to traffic. During the erection stage, the anchor bolt nuts are tightened to hold the cap girder in place as the deck slab is placed. Once placement is complete, the nuts are tightened using the turn-of-the-nut method, one-quarter turn past snug. The engagement of the anchor bolts introduces a rotational restraint into the connection system. To reduce the magnitude of the restraint, which will reduce the fatigue stress in the anchor bolt, bearing pads are placed between the cross plate and the plate-washer to increase the anchor bolt axial flexibility. Future plans envision the use of uplift resistant details. As shown earlier, pretensioning of the anchor bolts is necessary to improve the fatigue resistance of the connection. The purposes of the pretensioning tests were to determine the magnitude of the force that could be put in the bolts using a typical installation method and the effect that the tightening sequence had on the force.

Magnitude of the Pretension Force

The total pretension force that could be developed at the connection with each type of bearing pad is shown in Figure 3.1. Also shown is the average stress per bolt, based on the tensile stress area. The minimum pretension force, 19.2 kips, occurred when the 3/4" neoprene pad was used, which is the softest pad. The magnitude of the pretension force increased steadily as the stiffness of the bearing pad type increased. The largest force, 86.1 kips, was developed in the connection with no pad; it was extremely difficult to turn the nut the required amount, 1/4 turn past snug. The bearing pads creep and as a result the pretension force decreases with time. The percent loss of the pretension force after 10 minutes for each configuration is shown in Table 3.3. The loss varied from 19% for the connection with the 5/8 inch neoprene pads to 5.4% for the connection with the 1/2 inch preformed fabric pad. Long term creep of 70 durometer bearing pads is approximately 45% [8], so a good portion of the pretension force in the anchor bolt will be eliminated over time.

Table 3.3 : Percent Loss of Pretension Force Due to Pad Creep

BEARING PAD	% LOSS	FORCE AFTER LOSS
	after 10 minutes	kip
3/4" NEO	14.1	19.2
5/8" NEO	19.0	27.1
3/4" ROF	13.2	29.2
1/2" ROF	8.3	37.4
3/4" PFP	7.0	45.1
1/2" PFP	5.4	54.2
NO PAD	1.3	86.1

Effect of Tightening Sequence on Pretension Force

The tightening sequence did not have much effect on the total pretension force that could be developed in the connection. Table 3.4 shows the total pretension force as a function of the tightening sequence. The same data are shown in Figure 3.2. The scatter was minimal for the

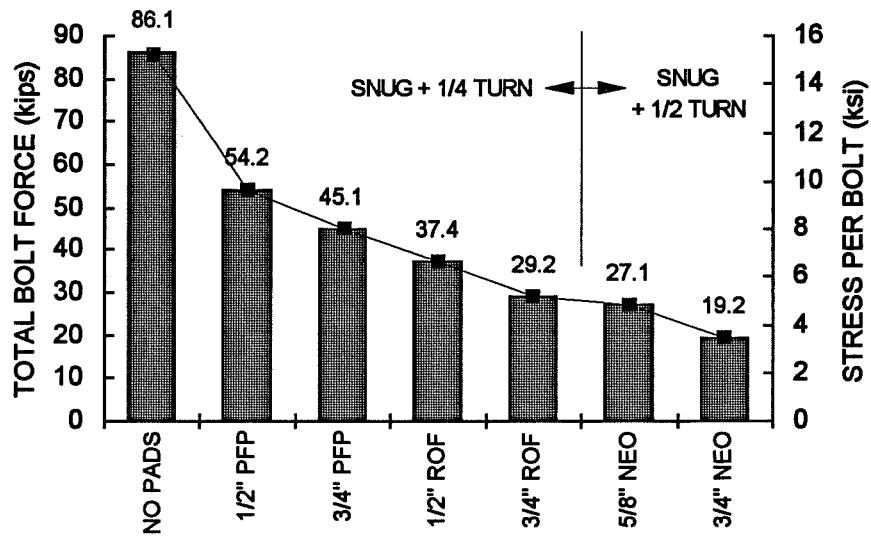


Figure 3.1 Total Pretension Force as Function of Bearing Pad

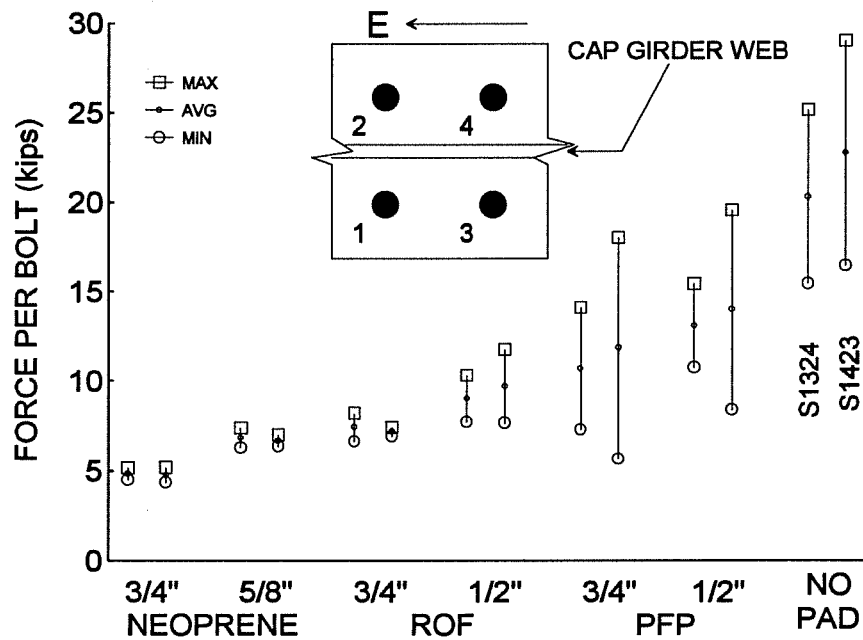


Figure 3.2 Effect of Tightening Sequence on Pretension Force

softer pads, but increased considerably for the stiffer pads. The tightening sequence S1324, in which both of the bolts on one side of web were tightened first and second, produced less scatter than the tightening sequence S1423. In that sequence, the first and second bolts tightened were on opposite sides of the web. The web is in plane with the rocker, which is the center of rotation in the longitudinal direction. As a bolt on one side of the web was tightened, the cap girder rotated towards the tightened bolt. This produced an increase in the force of the bolts on the opposite side of the web and a reduced force in the bolt on the tightened side of the web. The force in each bolt of the group changed as each individual bolt was tightened. The scatter of the bolt forces will be significant only if the cap girder is allowed to rotate. In practice the longitudinal girders and the concrete deck are in place before the final tightening is done, so longitudinal rotation of the cap girder is effectively precluded.

Table 3.4 : Effect of Tightening Sequence on Pretension Force

BEARING PAD	PRETENSION FORCE	
	(kips)	
	S1423	S1324
NO PADS	82.5	91.1
1/2" PFP	52.5	54.9
3/4" PFP	44.6	45.0
1/2" ROF	36.3	37.8
3/4" ROF	30.1	29.1
5/8" NEO	27.3	26.9
3/4" NEO	19.2	19.4

3.3 TRANSVERSE STIFFNESS TESTS

A typical plot of the load vs. vertical deflection, as recorded by a pen plotter during a test, is shown in Figure 3.3. The responses of the system have been classified as dynamic and static.

The dynamic response was constructed by connecting the points on the graph located at the peaks of the non-zero portions of the curve. The static response was constructed by connecting the points on the graph located at the ends of the zero slope portions of the curve. All subsequent data refer to the static response of the system.

3.3.1 Load-Deflection and Moment-Rotation Behavior

The deflection and the rotation were measured as shown in Figure 3.4. The load vs. vertical deflection curves of the seven stiffness tests are shown in Figure 3.5. The stiffness is primarily a function of the axial stiffness of the bearing pad-anchor bolt combination. This is confirmed by the curves, where the steepest slopes are for the preformed fabric pads and for no pads and the flattest slope is for the 3/4" neoprene pads. Almost no difference exists between the stiffnesses of the systems with preformed fabric pads and the system with no pads, suggesting that the use of preformed fabric pads adds no additional flexibility to the system. The responses of these stiff systems are also relatively linear, in contrast to the responses of the systems using the softer pads. In the tests where the randomly-oriented-fiber pads and the neoprene pads were used, the stiffness of the system increased with increasing load. This is due to the hyperelastic behavior of the rubber-like materials; the thickness of the pad reduces considerably as lateral bulging increases, thereby increasing the compressive stiffness of the pad.

The transverse moment vs. rotation curves of the seven stiffness tests are shown in Figure 3.6. The rotation of the cap girder is measured between the two connections and the moment is calculated as the end load multiplied by the distance to the centerline of the two connections, 120 inches. Initially the moment is increasing with little change in rotation because the cap girder is still in contact with the bearing and the anchor bolts have not picked up much additional load. At this range of load the connection is essentially behaving as a rigid moment resisting element, as is assumed in the analysis. The stiffness of the connection reduces when the cap girder is no longer in contact with the bearing and the anchor bolts and bearing pads are resisting the entire uplift force. A photo of the separation at the uplift bearing is shown in Figure 3.7. The rotational stiffnesses of the systems, both before and after uplift, are shown in Table 3.5. The stiffnesses were determined using linear regression for all of the data within each group and the regression lines for the test with the 3/4 inch bearing pads are shown in Figure 3.6..

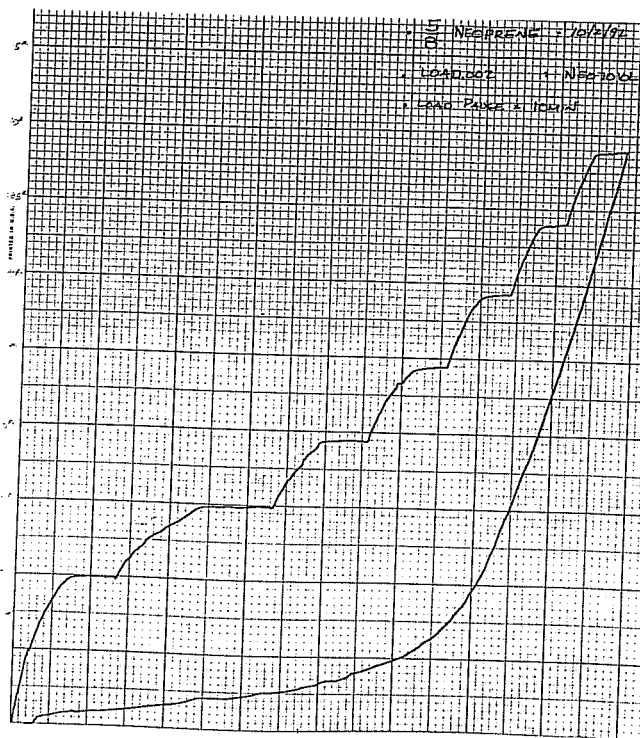


Figure 3.3 Typical Plot of End Load vs. Deflection for Transverse Stiffness Tests

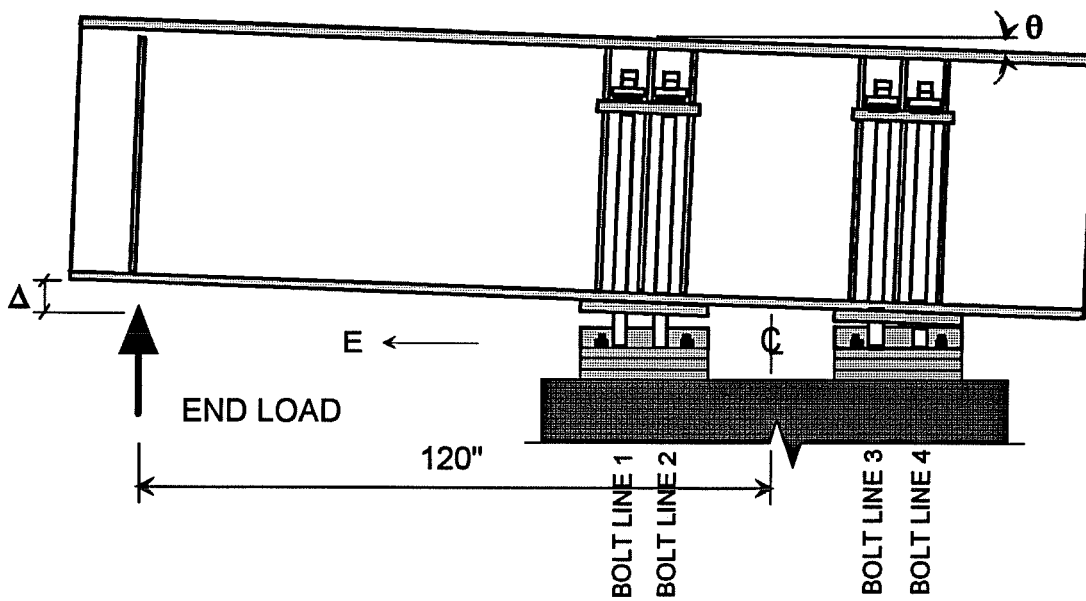


Figure 3.4 Location of Deflection and Rotation Points, Transverse Stiffness Tests

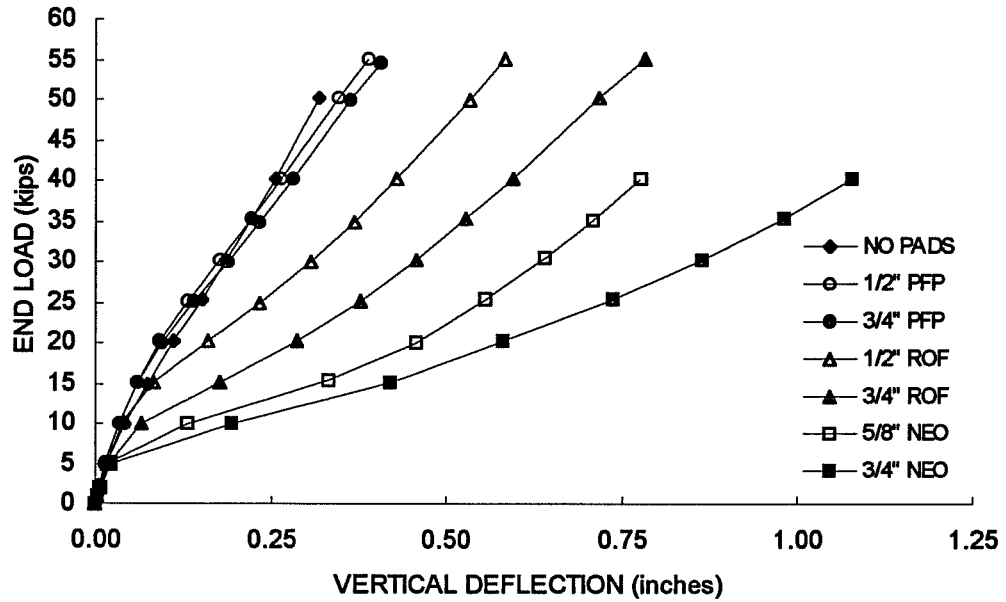


Figure 3.5 End Load vs. Vertical End Deflection for Transverse Stiffness Tests

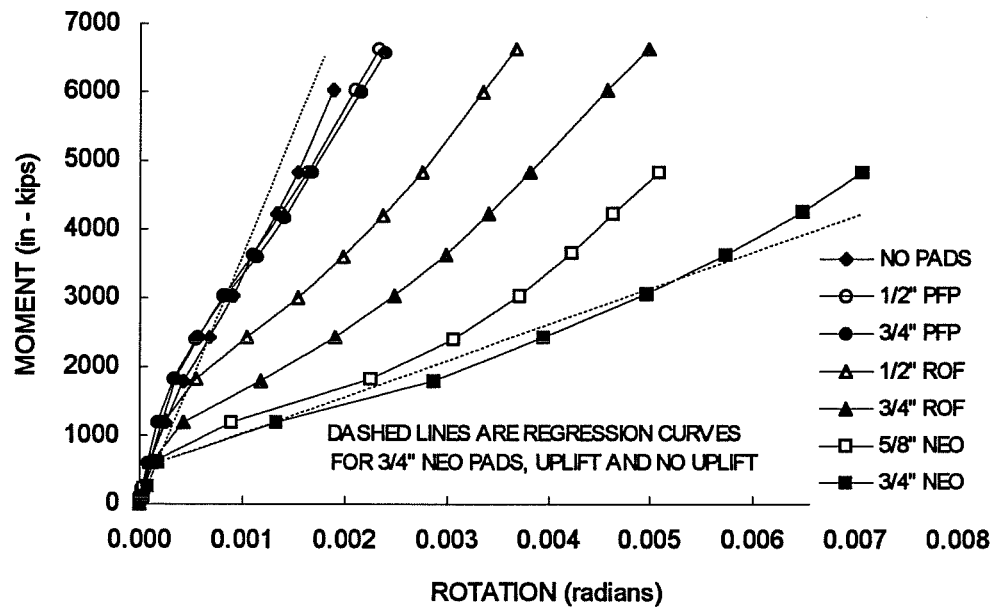


Figure 3.6 Moment vs. Rotation for Transverse Stiffness Tests

Table 3.5 : Transverse Rotational Stiffnesses

BEARING PAD	NO UPLIFT (in-kips / rad)	UPLIFT (in-kips / rad)
NO PADS	4,930,000	2,850,000
1/2" PFP	3,900,000	2,440,000
3/4" PFP	4,470,000	2,310,000
1/2" ROF	3,360,000	1,610,000
3/4" ROF	2,710,000	1,300,000
5/8" NEO	3,960,000	872,000
3/4" NEO	3,570,000	635,000

3.3.2 Distribution of the Bolt Forces and the Pier Reactions

The behavior of a pretensioned connection was described in Section 1.2.1. In comparing the behavior of the tested connections with the idealized connection, it is assumed that the center of rotation of the cap girder is at the center of the two connections and that the resultant bolt force is at the center of each bolt group. If these assumptions are invalid, the behavior of a tested connection will deviate somewhat from the behavior predicted by the idealized model. The reaction force at each connection was calculated using simple statics, as shown in Figure 3.8. The resultant uplift force, in kips, at the east bearing is

$$A = 3P - 6.25 \quad (3.2)$$

where P is the end load, defined as positive, and the value 6.25 accounts for the dead load of the cap girder, G, which was 5 kips and was applied at one foot to the left of the uplift connection. The compression force, in kips, at the west bearing is

$$A = - (2P - 1.25) \quad (3.3)$$

The bolt group force B is the sum of the forces in the four bolts at each connection, is defined as positive in Figure 3.8, and is assumed to act at the centroid of each connection. The pier reaction is the difference between the applied load and the bolt force and is given by

$$R = A - B \quad (3.4)$$

The bolt group force and pier reaction data are presented in four quadrant graphs in Figures 3.9 to 3.12. Each graph combines the two charts shown in Figures 1.7 and 1.8. Both of the connections are represented on each graph. The portion of the graph to the right of zero shows the response of the uplift connection (solid data points). The portion of the graph to the left of zero shows the response of the compression connection (open data points). For each connection the bolt group force (B) and pier reaction (R) are plotted against the connection reaction force (A). One graph is shown for each type of bearing pad, with the results from both of the tests using the two different thicknesses. The responses were not significantly affected by the change in thickness.

The response of the stiff systems, the connections in which no pads and the preformed fabric pads were used, correlated qualitatively very well with the response predicted by the idealized model. For the uplift connection, the initial slope of the bolt group force curve was relatively flat, indicative of a pretensioned connection. When uplift occurred, the bolt force matched the reaction force. The system responded equally well with respect to the pier reaction, which went to zero after uplift occurred. For the compression connection, the load at which the pretension force became zero coincided with the load at which the pier reaction began to match the compression force. A deviation in the expected and actual behavior is seen in the responses of the soft systems, the connections in which the neoprene pads and the ROF pads were used. This deviation began after uplift occurred and was due to the flexibility of the pads. It is assumed that the bolts of the compression connection can only lose load, i.e., that once the tension force in the bolts reduces to zero they are no longer effective because the applied load is compressive and the bolts do not resist compression. Similarly, it is assumed that the uplift connection resists all of the tension in the system. With a flexible vertical support, however, the point of rotation of the cap girder moves toward the compression connection and as a result, the bolts in the compression connection actually begin picking up tensile force. The bolts of the uplift connection

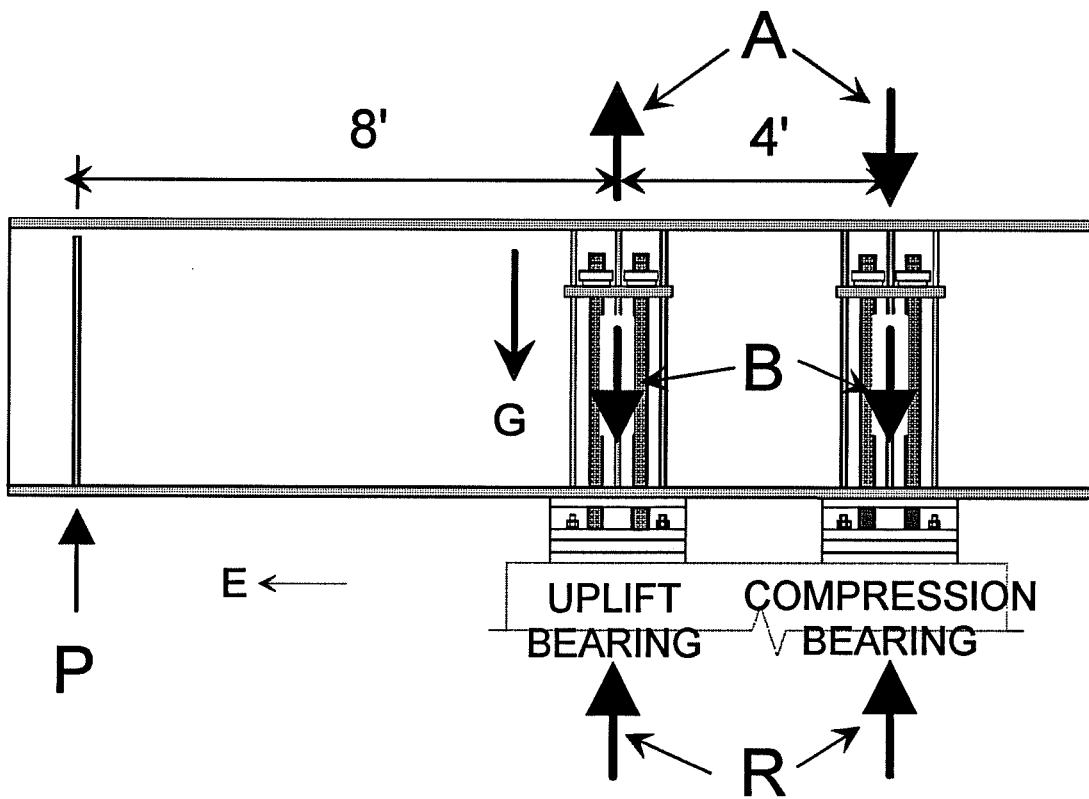


Figure 3.8 Statics of System

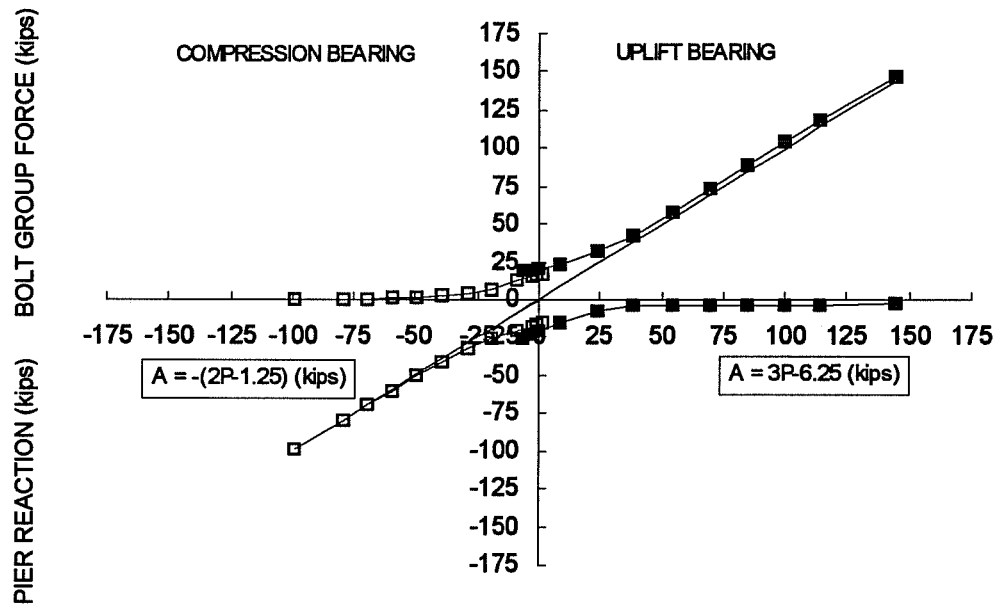


Figure 3.9 Bolt Group Force, Pier Reaction vs. Reaction Force, No Pads

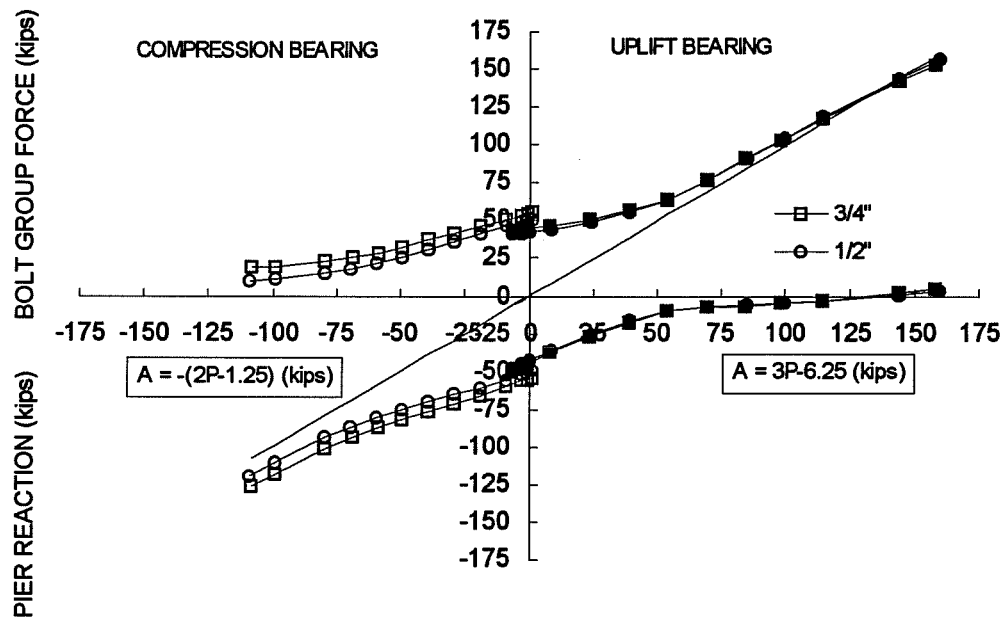


Figure 3.10 Bolt Group Force, Pier Reaction vs. Reaction Force, Preformed Fabric Pads

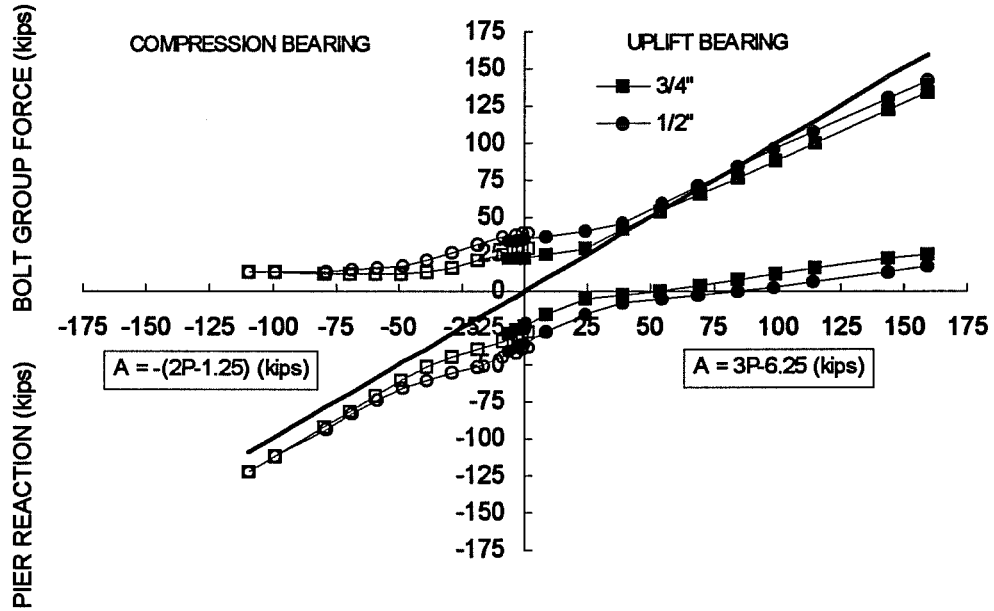


Figure 3.11 Bolt Group Force, Pier Reaction vs. Reaction Force, ROF Pads

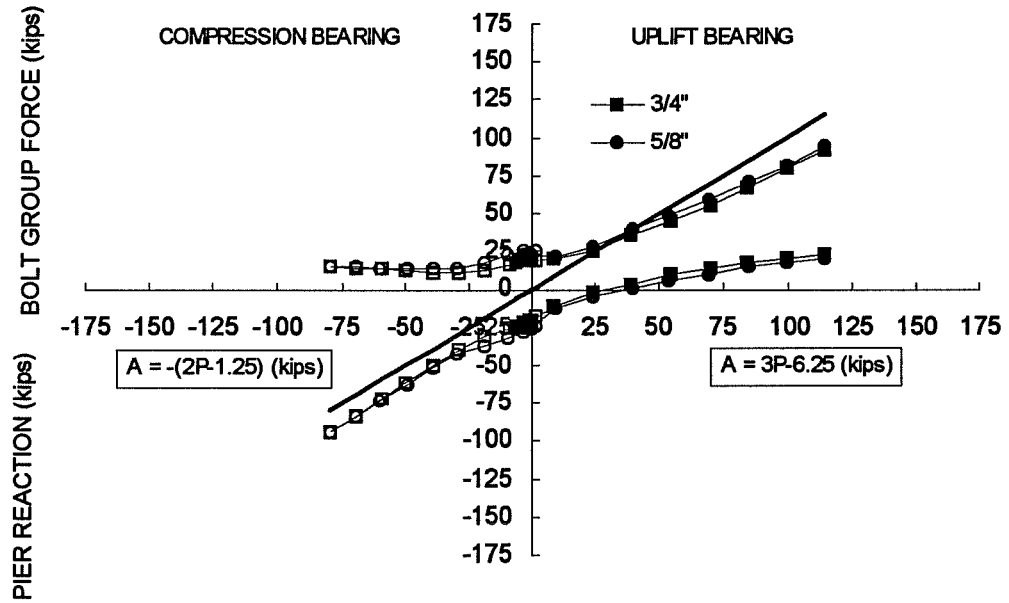


Figure 3.12 Bolt Group Force, Pier Reaction vs. Reaction Force, Neoprene Pads

no longer resist all of the tensile force and the bolt force curve drops below the applied load curve. The difference between an ordinate of the applied load curve and the corresponding ordinate of the bolt force curve is equal to the tensile force that is resisted by the compression connection. This is also the difference that appears in the other quadrants of the graph.

The bolt force values used in the above graphs were the sum of the forces of the four bolts in the connection. Since the bolts in each connection were spaced at a finite distance, the forces in the bolts were not the same. Figure 3.13 shows a typical plot of the change in the bolt line force vs. the load applied at the end of the cap girder for the four lines of bolts. Also shown is the force curve that the bolts would follow if all were concentrated at the center of the connection, as has been assumed previously. The first line of bolts always develops more force than the second line of bolts, and the ratio of these two bolt line forces is shown in Table 3.6. In general, the disparity in the force between the lines of bolts increases with an increase in the flexibility of the bearing pads.

Table 3.6 : Ratio of Force of First Line of Bolts to Second Line of Bolts

BEARING PAD	RATIO
NO PADS	1.27
1/2" PFP	1.27
3/4" PFP	1.24
1/2" ROF	1.37
3/4" ROF	1.34
5/8" NEO	1.75
3/4" NEO	1.59

The transverse direction stiffness tests showed that a flexible system does not respond as predicted once uplift has occurred. Once the dead load compressive force is overcome, only the maintenance of the pretension force precludes uplift. The magnitude of the pretension force that can be induced is small, probably less than the range within which the uplift force is known. If uplift occurs, only the tensile resistance of anchor bolts prevents failure. Since the loading is

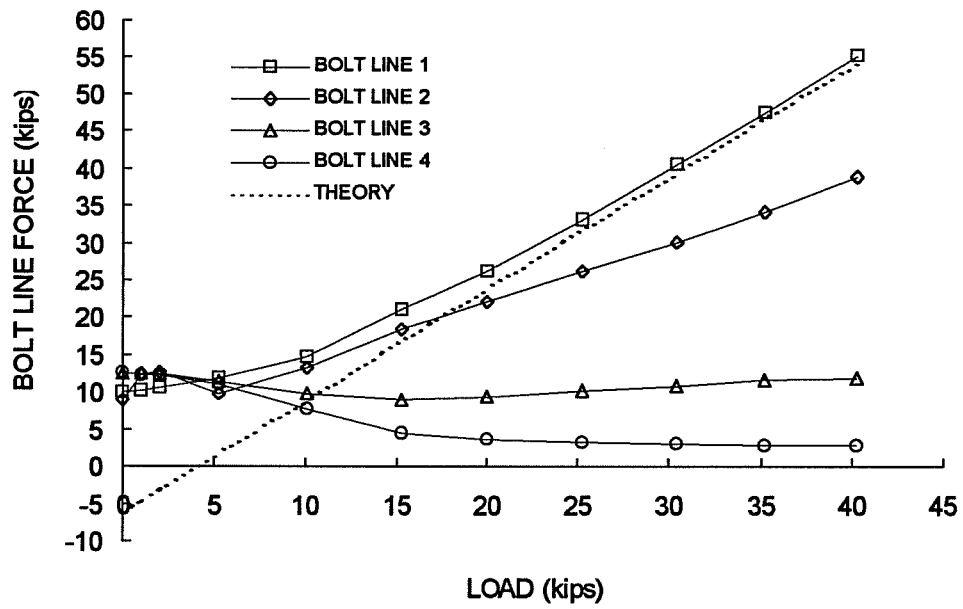


Figure 3.13 Typical Plot of Bolt Line Force vs. Load

cyclic, the anchor bolts are then subject to fatigue. Threaded rods, however, are rated as a category E detail, with an allowable stress of 8 ksi; using four 1.5 inch diameter anchor bolts, the allowable live load uplift force is only $T_L = 4 (1.41) (8) = 45$ kips. Given the uncertainty in knowing the actual uplift load and the results of the pretensioning tests and the transverse stiffness tests, it is recommended that the current detail not be used in situations where uplift can occur.

3.4 LONGITUDINAL STIFFNESS TESTS

Moment vs. Rotation Behavior

The moment and rotation are defined as shown in Figure 3.14. The moment vs. rotation curve of each connection is shown in Figure 3.15. As with the transverse direction tests, the rotational stiffness is proportional to the stiffness of the bearing pads used in the connection. Similarly, a change in stiffness occurs during each test. Initially, the bolts are pretensioned; as the cap girder rotates counterclockwise, the bolts on the south side of the girder immediately begin picking up load and the bolts on the north side begin losing load. As long as the bolts on both sides of the girder remain in tension they both contribute to the stiffness of the connection. When the bolts on the north side lose all pretension force, they no longer contribute to the stiffness of the connection and the stiffness is cut in half. This is the case for the connection with no pads and the connection with the 3/4" ROF pads, which are shown in Figure 3.16. All of the bolts are in tension at the beginning of the loading and the connection has a certain stiffness, indicated by the dashed line. At a certain point, the pretension in the bolts on the north side is lost, which means a reduction in the connection stiffness.

Moment Resisted by the Anchor Bolts

The moment resisted by the anchor bolts is

$$M_b = (F_{ss} - F_{ns}) d \quad (3.5)$$

where F_{ss} is the sum of the bolt forces on the south side of the web, F_{ns} is the sum of the bolt forces on the north side of the web, and d is the distance from the anchor bolts to the centerline of the connection. It is assumed with this equation that the cap girder rotates about the centerline of the connection. The rocker, however, is not a knife-edge; it is 2.5 inches thick and the cap girder may rotate about any point within this 2.5 inches. Since the rocker thickness is a significant

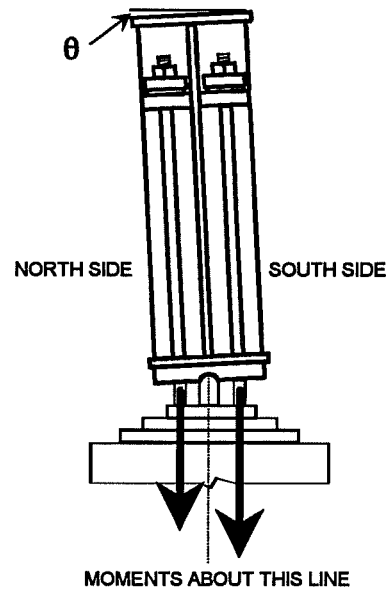


Figure 3.14 Longitudinal Rotation of Cap Girder

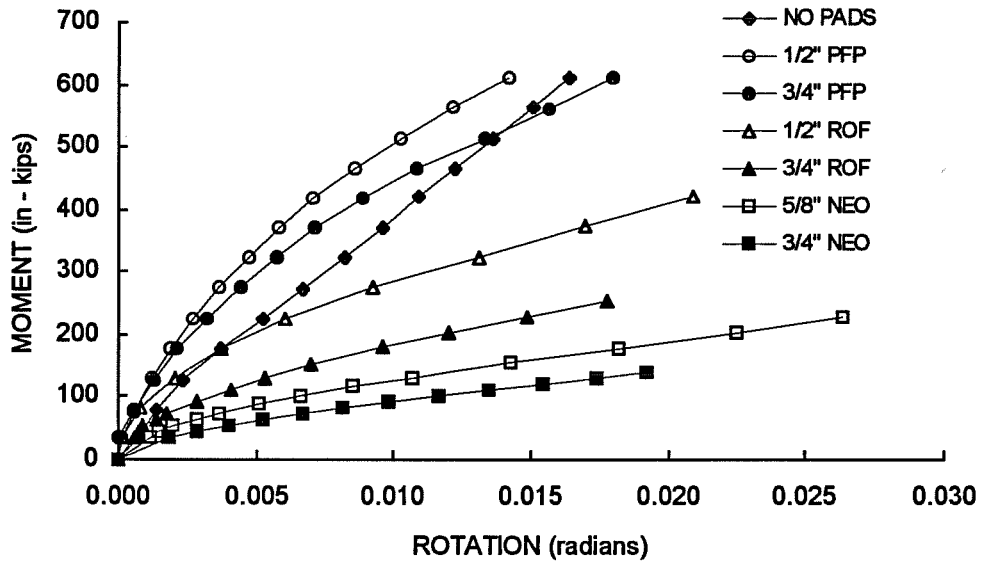


Figure 3.15 Moment vs. Rotation for Longitudinal Rotation Tests

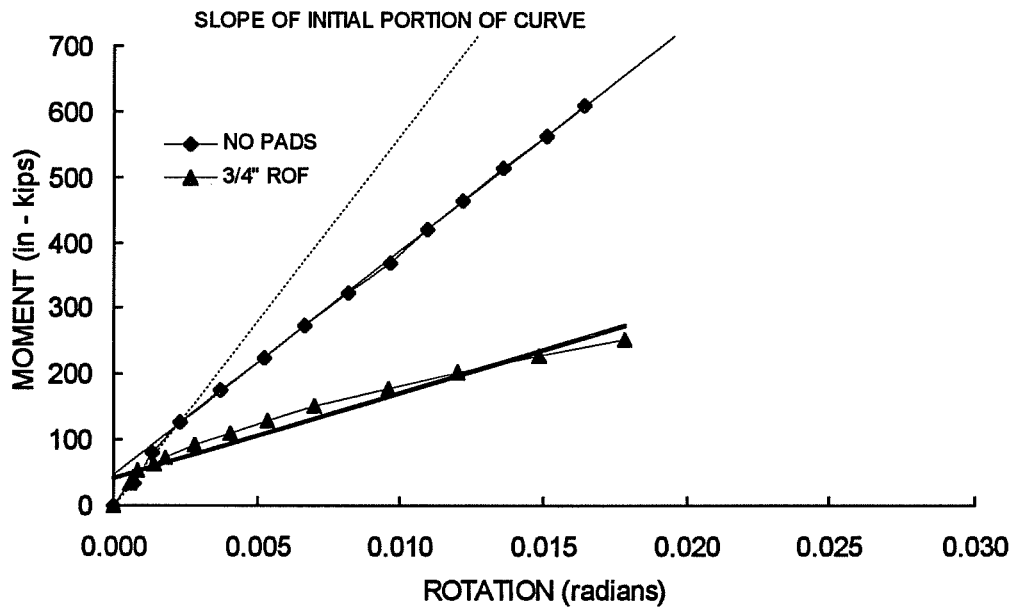


Figure 3.16 Illustration of Change in Rotational Stiffness

percentage of the moment arm distance, an appreciable difference can exist between the applied moment and the anchor bolt moment calculated from Equation 3.5. If an eccentricity of rotation, x , is defined as shown in Figure 3.17, the anchor bolt moment can be recalculated as

$$M_b = F_{ss} (d + x) - F_{rs} (d - x) \quad (3.6)$$

If M_b is replaced with M_{app} , the applied moment, the eccentricity x can be determined from Equation 3.5 and is shown in Table 3.7. For the stiffer pads an increase in eccentricity indicates an increase in the error between the applied moment and the anchor bolt moment. The connections with the softer pads are much less sensitive to a change in eccentricity but a larger discrepancy exists between the applied moment and the anchor bolts moment. The rotational stiffnesses were calculated using both the applied moment, β_b , and the anchor bolt moment, β_{app} . For most of the connections the anchor bolts on both sides of the web contributed to the stiffness. It is conservative to use the stiffness based on the applied moment rather than the stiffness based on the anchor bolt moment.

Table 3.7 : Longitudinal Rotational Stiffness

BEARING PADS	x	M_b / M_{app}	β_b	β_{app}
NO PADS	0.35	0.90	51,000	56,600
		<i>one side</i>	29,900	34,400
1/2" PFP	0.49	0.71	36,600	42,000
3/4" PFP	0.32	0.81	36,200	41,400
		<i>one side</i>	18,200	20,600
1/2" ROF	0.47	0.69	15,100	18,600
3/4" ROF	0.76	0.72	10,500	13,100
5/8" NEO	0.82	0.74	5,909	7,779
3/4" NEO	0.90	0.75	5,015	6,524

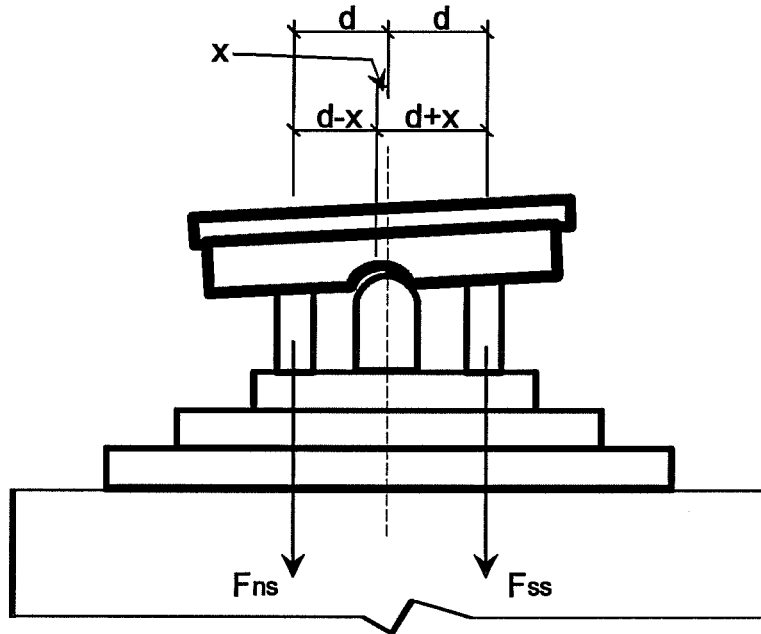


Figure 3.17 Eccentricity of Rotation

Distribution of the Bolt Forces

The applied moment was resisted entirely by the anchor bolts, though some discrepancy was noted in the previous section. The stress per bolt on the south side of the girder, calculated using the tensile stress area, is plotted against the rotation in Figure 3.18. A similar plot showing the stress in the bolts on the north side is shown in Figure 3.19. These graphs confirm that the south line of bolts pick up additional load as the rotation increases and that the north line of bolts lose load. Shown in Figure 3.20 is the stress per bolt on the south side exclusive of the pretension stress. This graph illustrates that the stiffer the connection, the greater the stress that the bolts develop. In an actual bridge, the longitudinal rotation will be less than .01 radians. If the design limits the bolt tensile stress range to 8 ksi in the threaded area, only the connections with no pads and the preformed pads approached or exceeded this level.

3.5 TRANSVERSE DIRECTION ULTIMATE STRENGTH TEST

Moment-Rotation Behavior

The moment vs. rotation curve for the ultimate strength test is shown in Figure 3.21. The load vs. deflection curve is shown in Figure 3.22. There are five principal regions of behavior and the characteristics of each of these regions is described in Table 3.8. The regions are delineated on the graphs.

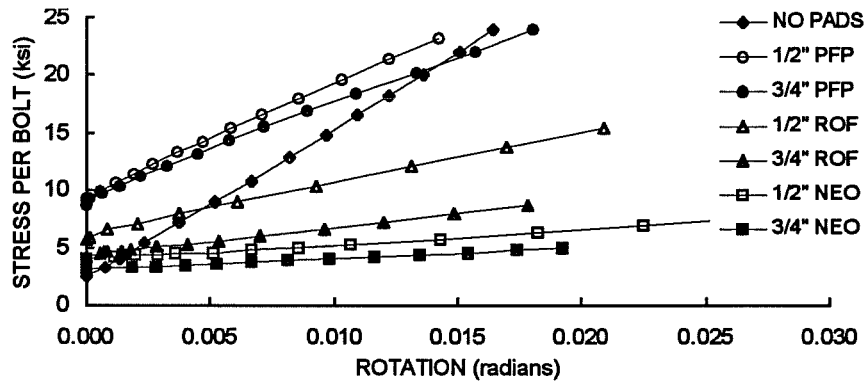


Figure 3.18 Stress per Bolt on South Side

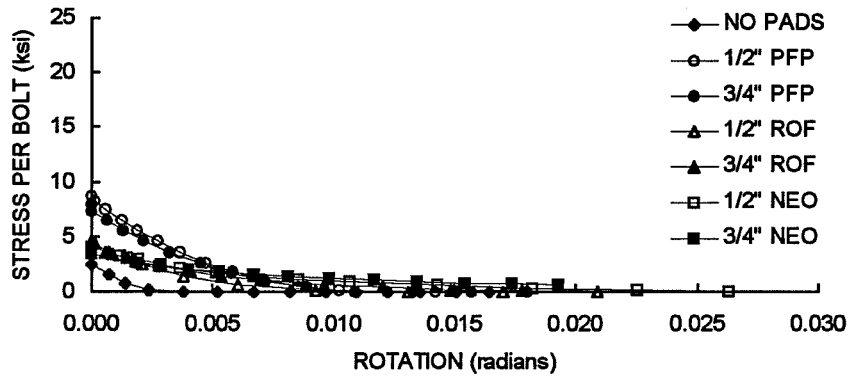


Figure 3.19 Stress per Bolt on North Side

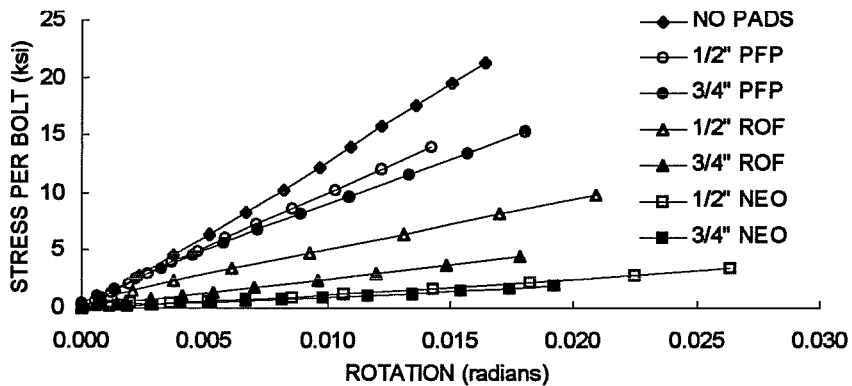


Figure 3.20 Stress per Bolt on South Side, No Pretension

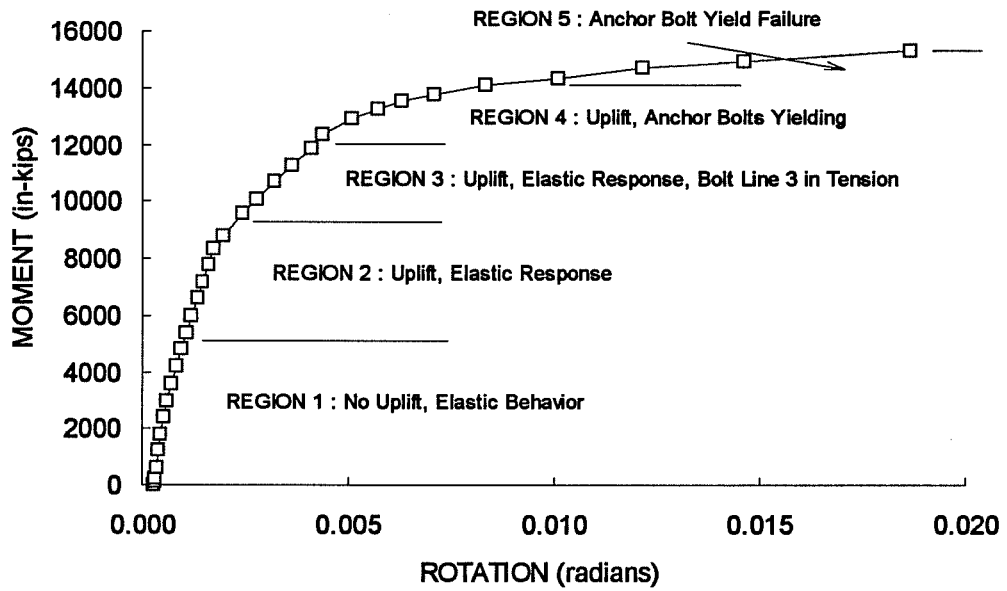


Figure 3.21 Moment vs. Rotation for Ultimate Strength Test

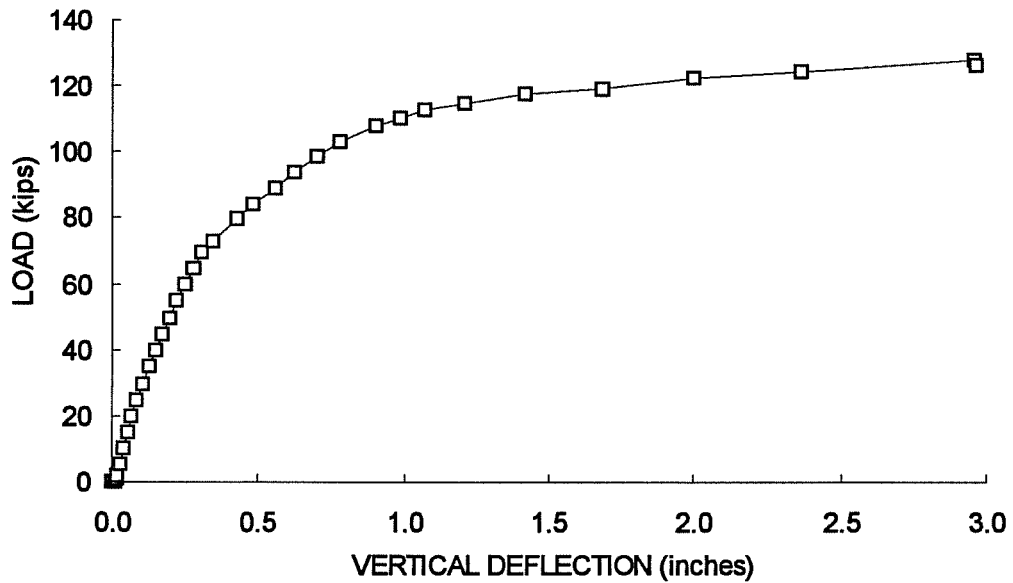


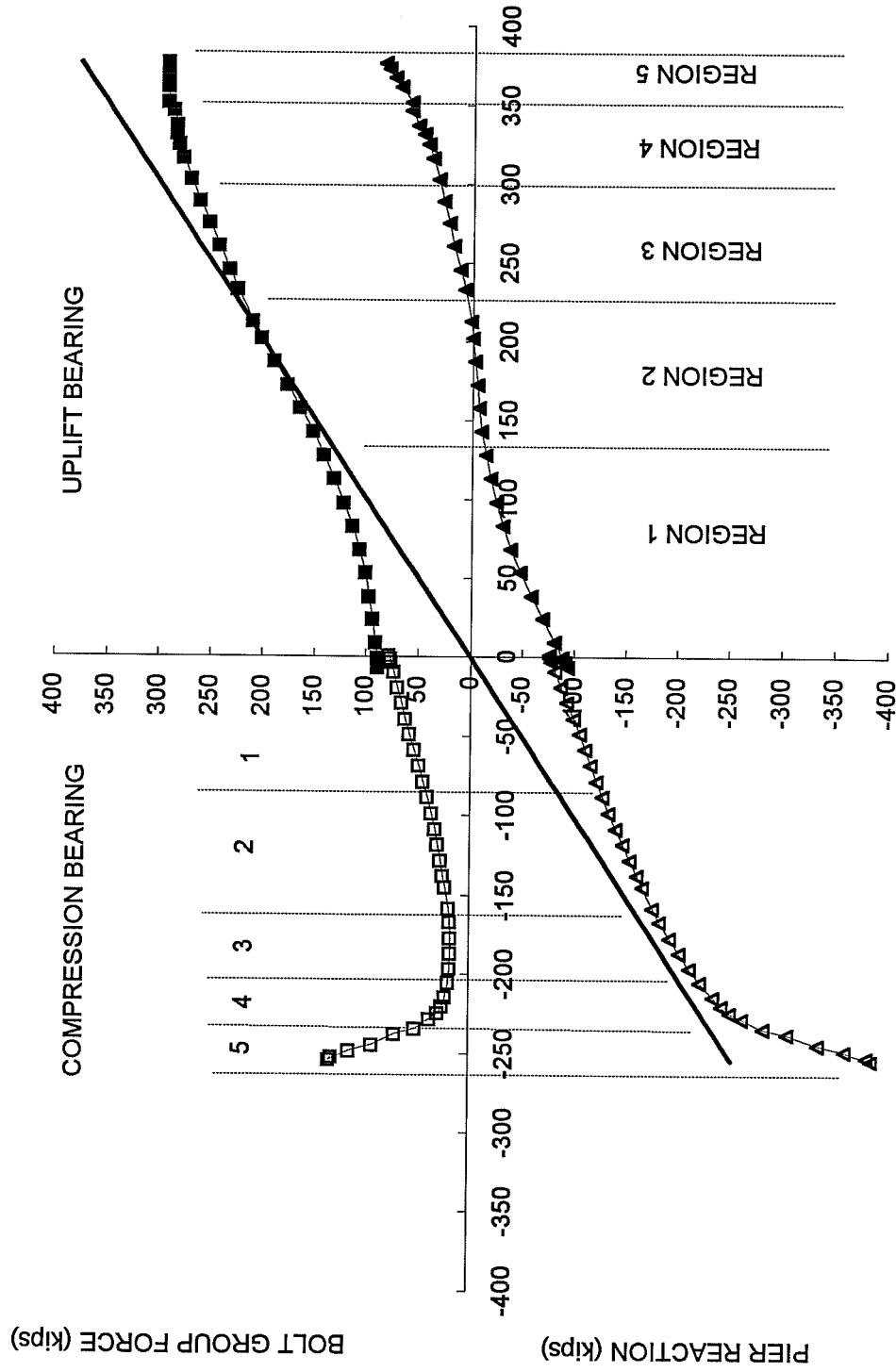
Figure 3.22 Load vs. Deflection for Ultimate Strength Test

Table 3.8 : Description of Ultimate Strength Test

REGION	P	REACTION FORCE		DESCRIPTION
	kip	UPLIFT	COMPR	
1	0- 45	0-129	0- 89	No uplift, Elastic Response
2	45- 80	129-234	89-159	Uplift, Elastic Response
3	80-100	234-294	159-199	Uplift, Elastic Response, Third line of bolts picking up tension
4	100-120	294-354	199-239	Uplift, Anchor bolts of uplift connection yielding, Third line of bolts picking up tension
5	120-128	354-378	239-255	Uplift, Anchor bolts of uplift connection completely yielded, Third line of bolts picking up tension, Increase in deformation with very small increase in load

Load Distribution in Anchor Bolts

The distribution of the bolt forces and the pier reaction is shown in Figure 3.23. The regions of the behavior listed above are shown on the graph. The pretension forces were 88 kips for the compression connection and 78 kips for the uplift connection. The shapes of the curves and the deviation from the expected behavior are consistent with the stiffness tests (Section 3.3.2) detailed earlier. The major deviation is due to the transfer of tensile load from the anchor bolts of the uplift connection, bolt lines one and two, to the first line of anchor bolts of the compression connection, bolt line 3. An additional deviation in the behavior occurred when the bolts of the uplift connection reached the yield level; there was no increase in bolt force with an increase in applied load near the end of the test. The deviation is also the cause of the upward spike in the bolt force and pier reaction at the compression bearing. The increase in bolt force is due to the transfer of tensile load to the third line of the bolts.



$A = 3P - 6.25$ (kips)

$A = -(2P - 1.25)$ (kips)

Figure 3.23 Bolt Group Force, Pier Reaction vs. Applied Load for Ultimate Strength Test

Stress Distribution in Compression Bearing

The rocker element of the compression bearing was instrumented with seven strain gages along its length to determine the stress distribution. The normalized stress distribution at the seven locations is plotted for eight different load levels in Figure 3.24. The stress values were normalized to the average stress in the bearing, calculated as the applied compression force divided by the area of the rocker. At the load levels corresponding to regions 1 and 2 of the behavior the pattern of the stress distribution is similar. The reaction load is concentrated in the first third and last third of the rocker, with little load in the center third. The stresses decrease with increasing load in the first third and increase with increasing load in the last third. At load levels corresponding to regions three through five, the stress redistributes to the right third of the bearing.

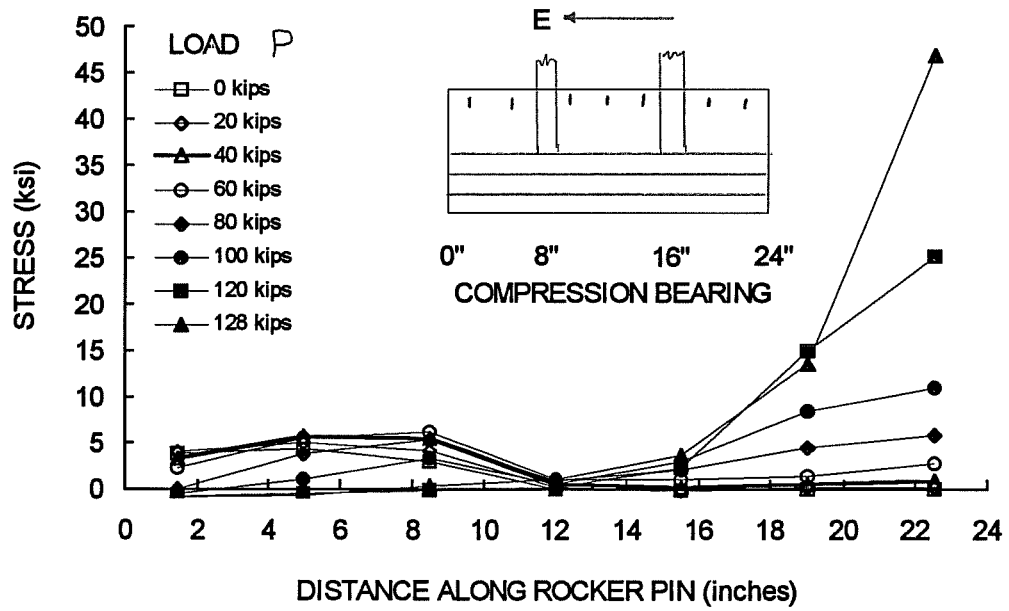


Figure 3.24 Stress Distribution Along Rocker Pin

CHAPTER 4 : PHASE I ANALYSIS AND DISCUSSION

4.1 ESTIMATION OF PRETENSION FORCE

The pretension force in a bolt can be determined as

$$B_0 = \frac{\Delta_b}{\sum \frac{1}{k_i}} \quad (4.1)$$

where Δ_b is the bolt extension and k_i is the axial stiffness of an element located between the nut and anchorage of the bolt. When the turn-of-the-nut method is used to tighten a bolt, the bolt extension is equal to the number of turns of the nut divided by the pitch of the threads. The pitch is the number of threads per inch. The connection studied in this research can be divided into five elements as shown in Figure 4.1; these element are

- Anchor bolt k_b
- Bearing Pad k_p
- Cap Girder Web, Cross Plate, and Stiffeners k_{gird}
- Bearing Hardware k_h
- Concrete Pier Cap k_{conc}

where k_{gird} and k_{conc} are shaded. Substituting the element stiffnesses into Equation 4.1 gives

$$B_0 = \frac{\text{number of turns} / \text{pitch}}{\frac{1}{n k_b} + \frac{1}{n k_p} + \frac{1}{k_{gird}} + \frac{1}{k_h} + \frac{1}{k_{conc}}} \quad (4.2)$$

where n is the number of bolts in the connection. For elements in parallel, such as the anchor bolts and the bearing pads, the total stiffness is equal to the sum of the stiffnesses. For elements in series, such as those described above, the total stiffness is equal to the inverse of the sum of the inverse of the individual stiffnesses. In the latter case, the stiffness is always less than the smallest stiffness of any element. Only those elements that have a stiffness similar to the element

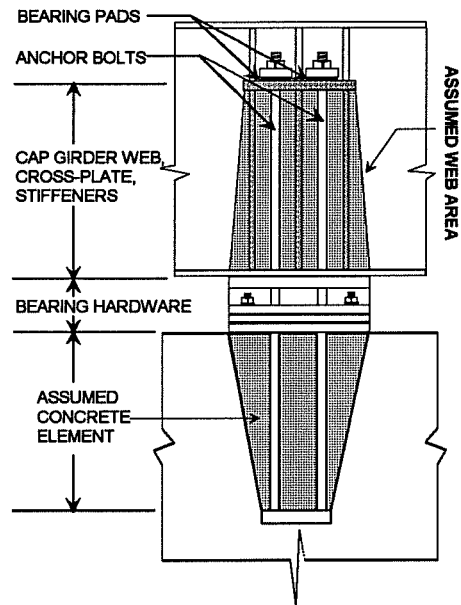


Figure 4.1 Elements of Connection

with the smallest stiffness will have a significant effect on the total stiffness. The following sections will give methods for estimating the stiffnesses of the individual elements.

4.1.1 Anchor Bolt Stiffness

The anchor bolt stiffness is calculated as

$$k_b = \frac{A_b E}{L_b} \quad (4.3)$$

where A_b is the area of the shank of the bolt, E is the modulus of elasticity for steel, 29,000 ksi, and L_b is the length of the anchor bolt. The threaded portions of the bolt, which have a smaller area than the shank, typically comprise only a few percent of the total length of the bolt and will be ignored in the calculation. The major difficulty in using this equation is selecting a proper length for the anchor bolt. The length of the anchor bolt from the top of the pier cap to the nut is free to deform, but from the top of the pier cap to the anchorage point, it is embedded in concrete. Along this length, the anchor bolt is bonded to the concrete. If no slip occurs at the anchorage the anchor bolt length is a minimum if it is assumed to be fixed at the top of the pier cap and is a maximum if it is assumed to be unbonded in the embedment length. If the bond stress is uniformly distributed along the length of the bolt, the stiffness of the embedded length of the anchor bolt is the same as the stiffness of a free anchor bolt with a length one-half of the embedment depth [20]. The length of the anchor bolt for stiffness calculations can be approximated as the free length plus one-half of the embedment depth. Using the properties of the connection being studied, the axial stiffness of the anchor bolt is bounded by the following values:

- $L_{b,\min} = 56 \text{ in} : A_b = 1.77 \text{ in}^2 : E = 29,000 \text{ ksi} : k_{b,\min} = 1.77(29000) / 56 = \underline{917 \text{ k / in}}$
- $L_{b,\max} = 85 \text{ in} : (56'' + \text{embedment depth } (29'')) : k_{b,\max} = 1.77(29000) / 85 = \underline{604 \text{ k / in}}$
- $L_{b,\text{eff}} = 71 \text{ in} : (56'' + 0.5 (29)) : k_{b,\text{avg}} = 1.77(29000) / 71 = \underline{723 \text{ k / in}}$

4.1.2 Bearing Pad Stiffness

The compressive load vs. deflection curves of the bearing pads used in this research are shown in Figure 4.2. The response of the preformed fabric pads and the ROF pads were relatively linear while the response of the neoprene pads were nonlinear, increasing in stiffness as the load increased. Also shown is the linear regression estimate of the stiffness, the values of which are shown in Table 4.1. The table also gives an estimate of the compressive modulus of elasticity for each pad and the ratio of the modulus of elasticity of steel to the modulus of elasticity of the pad. The stiffness of a bearing pad will be a function of the material and the shape factor. The shape factor is defined as the ratio of the pad area compressed to the pad area free to bulge laterally. The stress-strain relationship of the various types of pads can be provided by the manufacturer. The stiffness can be found by using Equation 4.3, substituting the pad thickness for length.

Table 4.1 : Compressive Stiffness of Bearing Pads

BEARING PAD	k_p (k / in)	SF	E_p (ksi)	E_s / E_p
1/2" PFP	932	2.44	18.2	1596
3/4" PFP	786	1.63	23.0	1262
1/2" ROF	230	2.44	4.5	6468
3/4" ROF	161	1.63	4.7	6160
5/8" NEO	80	1.95	1.9	14877
3/4" NEO	57	1.63	1.7	17400

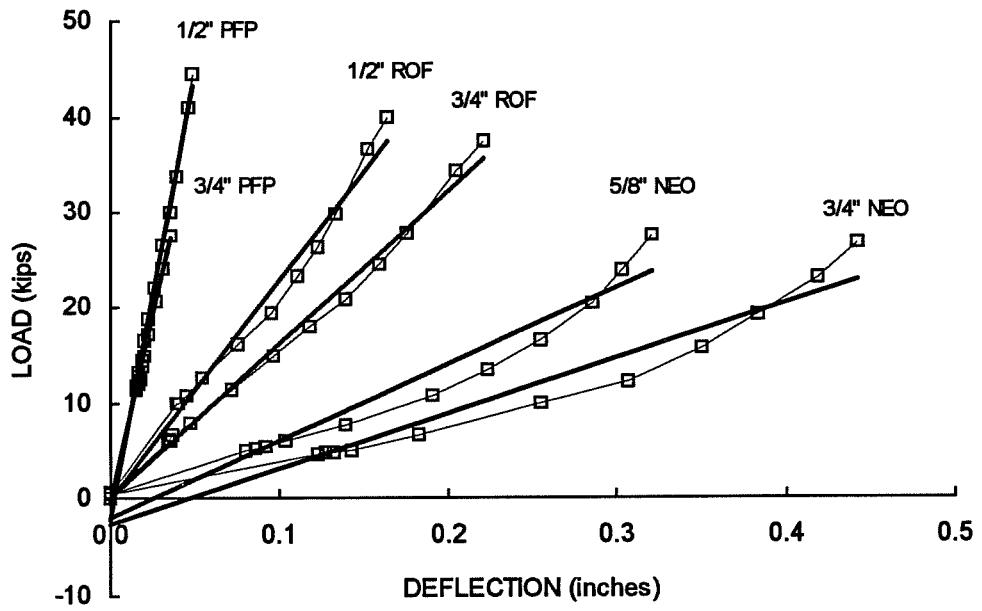


Figure 4.2 Load vs. Deflection Curves for Bearing Pads

The modulus of elasticity of neoprene bearing pads can be estimated as [2, p.295,630]

$$E_p = 3G (1 + 2 (C) SF^2) \quad (4.4)$$

where G is the shear modulus in psi, C is a constant that is a function of the durometer of the pad, and SF is the shape factor of the bearing pad. The values of G and C for three durometer hardnesses are shown in Table 4.2. Interpolation may be used for values not listed.

Table 4.2 : Values for G and C , Neoprene Pads

HARDNESS	50	60	70
G , psi	90 - 100	120	150-160
C	0.75	0.60	0.55

Using Equation 4.4, the calculated modulus of elasticity for the neoprene pads cited in Table 4.1 are 1.92 ksi for the 5/8" pad and 1.88 ksi for the 3/4" pad. These values compare very well with the measured values of 1.9 ksi (5/8") and 1.7 ksi (3/4").

4.1.3 Cap Girder Web, Cross Plate and Stiffeners

The axial stiffness of this area can be estimated as

$$k_{gird} = \left[\frac{h_{st}}{E (A_{st} + A_{web})} + \frac{S_{st}^3}{n 192 EI_{cp}} \right]^{-1} \quad (4.5)$$

where h_{st} is the height of stiffeners, A_{st} is the area of stiffeners, A_{web} is the area of web, S_{st} is the spacing of stiffeners, and I_{cp} is the moment of inertia of the cross plate. The first term is the axial stiffness of the stiffeners and the cap girder web and the second term is the flexural stiffness of the cross plate with respect to the load and deflection at midspan of the plate, which is assumed to be fixed at the ends. The effective area of the web may be estimated by multiplying its thickness by a width that is the average of the bearing width and the out-to-out width of the edge

stiffeners. Using the properties of the connection being studied, the stiffness of the cap girder section can be estimated as follows:

- $h_{st} = 36 \text{ in} : A_{st} = 6(7)(0.75) = 31.5 \text{ in}^2 : A_{web} = 0.5625(24+16)/2 = 11.25 \text{ in}^2$
- $n = 4 : S_{st} = 8 \text{ in} : I_{cp} = (1/12)(7)(2)^3 = 4.67 \text{ in}^4$

$$k_{gird} = \left[\frac{1}{34438} + \frac{1}{203145} \right]^{-1} = 29446 \text{ k / inch}$$

4.1.4 Bearing Hardware Stiffness

The stiffness of the bearing hardware may be estimate as

$$k_h = \frac{(t_p b) E}{h_h} \quad (4.6)$$

where t_p is the thickness of the rocker pin, b is the transverse width of the bearing, and h_h is the height of bearing hardware. Using the properties of the connection being studied, the bearing hardware stiffness can be estimated as follows:

- $t_p = 2.5 \text{ in} : b = 24 \text{ in} : h_h = 12.25 \text{ in} : k_h = 2.5(24)(29000) / 12.25 = \underline{142,000 \text{ k / in}}$

The measured stiffness of the bearing hardware was determined by dividing the pretension force in the bolts by the average of the deflections on the bottom of the cap girder, which measured the bottom flange deflection with respect to the top of the concrete pier cap. The average value for the stiffness was 4000 kips / inch, which is only 3% of the stiffness calculated above. The small value is most likely the result of settlement between the rocker pin and the sole plate. No dead load was present for the tests, so the rocker and sole plate were not in complete contact at all locations. Any initial settlement in the field will be eliminated by the superstructure dead load; a connection with a settlement stiffness of 4000 kips / inch requires only 40 kips of load to eliminate a gap of 0.01 inches.

4.1.5 Concrete Pier Cap Stiffness

This represents the mass of concrete between the top of the pier and the anchorage of the bolts. The axial stiffness of this element may be estimated as

$$k_{conc} = \frac{(A_{bear} + A_{anch}) E_c}{2 L_d} \quad (4.7)$$

where A_{bear} is the area of the bottom bearing plate, A_{anch} is the area encompassed by the anchor bolts, E_c is the modulus of elasticity of concrete, and L_d is the embedment depth of anchor bolts. Using properties of the connection being studied, the axial concrete stiffness can be estimated as follows:

- $A_{bear} = 576 \text{ in}^2$: $A_{anch} = 64 \text{ in}^2$: $E_c = 3420 \text{ ksi}$: $h_e = 29 \text{ in}$
- $k_{conc} = (576+64) 3420 / (2(29)) = 37738 \text{ k / in}$

4.1.6 Comparison of Calculated and Experimental Pretension Forces

A comparison of the calculated and experimentally measured pretension forces is shown in Table 4.3. The stiffness of the anchor bolt was based on the free length plus one-half of the embedded length. In all cases, the calculated pretension force is smaller than the actual pretension force. The percent difference varied from 2% to 59%. The difference is due mainly to the pretension force developed during the snugging process. The calculated values are based on the force produced by the number of turns after the snugging procedure and so do not include this initial pretensioning. As a result, the calculated force will be less than the actual force, assuming the stiffnesses were correctly derived. The elements with the smallest stiffnesses will have the largest impact on the magnitude of the pretension force. In this case these elements are the anchor bolts and the bearing pads. If the stiffness contributions of the other elements are ignored, the stiffness of the connection will be slightly overestimated, which will result in a larger calculated pretension force. For example, the pretension force of the connection with the 1/2" ROF pads was 23.5 kips when the stiffness of all of the elements was considered. If only the stiffness of the anchor bolts and the bearing pads are considered, the pretension force is 29 kips,

a 23% increase. For calculating the pretension force, it is conservative to use only the anchor bolt and bearing pad stiffnesses and the anchor bolt length should be the free length plus one-half of the embedded length. The pretension force will be used to determine the area of the bearing pad. The procedure is given in Chapter 10.

Table 4.3 : Comparison of Calculated and Experimental Pretension Force

BEARING PAD	$B_0, \text{ calc}$ (kips)	$B_0, \text{ exp}$ (kips)	$B_{0,\text{exp}}/B_{0,\text{calc}}$
NO PADS	58.8	86.1	1.47
1/2" PFP	45.6	54.2	1.19
3/4" PFP	44.3	45.1	1.02
1/2" ROF	23.5	37.4	1.59
3/4" ROF	19.0	29.2	1.53
5/8" NEO	21.9	27.1	1.24
3/4" NEO	16.2	19.2	1.18

4.2 LONGITUDINAL ROTATION BEHAVIOR

The longitudinal rotation of the stringers at the pier location is restrained by the anchor bolts and the pier column. The restraining mechanism of each element will be demonstrated in the following sections. The forces and moments developed in the restraining elements are a function of the rotational stiffnesses of the elements and will be proportional to the applied rotation.

4.2.1 Determination of the Longitudinal Rotation

The longitudinal rotation at a support is not a quantity that is normally determined in the design of a steel bridge. The rotation, however, produces forces in the anchor bolts and a

horizontal force on the rocker. The only displacements that are typically determined are the dead load deflections and the maximum live load deflection. The maximum rotation at a support occurs under pattern live loading. By linking the rotation to the maximum live load deflection, a simple method is developed for calculating the maximum longitudinal rotation.

Consider the three span continuous beam shown in Figure 4.3. The ratio of the span lengths are 1.0 - 1.2 - 1.0, the span ratios recommended for the design of the bridge systems studied in this research. The stiffness of the girder, EI , is constant. The center span is loaded with a uniform live load, w , and a point live load, P . This will produce the maximum live load deflection in the center span, point c , and will also produce the maximum rotations at supports b and d . The deflection of point c is

$$\Delta_c = 0.01312 \frac{w L^4}{EI} + 0.01864 \frac{P L^3}{EI} \quad (4.8)$$

and the rotation at point b (or d) is

$$\theta_b = 0.02572 \frac{w L^3}{EI} + 0.03145 \frac{P L^2}{EI} \quad (4.9)$$

Dividing Equation 4.9 by Equation 4.8 gives

$$\theta_b = \frac{\Delta_c}{L} \left[\frac{0.02572 wL + 0.03145 P}{0.01312 wL + 0.01864 P} \right] \quad (4.10)$$

If $P = 0$, Equation 4.10 reduces to $\theta_b = 1.96 \frac{\Delta_c}{L}$. If $wL = 0$, $\theta_b = 1.69 \frac{\Delta_c}{L}$. For long spans, lane loading governs the design; the uniform load component of a lane load, $0.64L$, is much larger than the point load component, 18 kips, so the equation in which $P=0$ can be conservatively used. For short spans, truck loading governs; there is no uniform load component of a truck load, so the equation in which $wL=0$ can be used. The maximum live load deflection is limited to $L_s / 800$ where L_s is the length of the span. If $L_s = 1.2L$ the maximum live load rotations at the interior supports are

$$\begin{aligned} \theta_{\max} &= 0.003 \text{ radians for lane loading} \\ \theta_{\max} &= 0.0025 \text{ radians for truck loading} \end{aligned} \quad (4.11)$$

Although the equations derived above apply only to the bridge system shown in Figure 4.3, the values should be reasonable estimates for bridge systems with different, but similar, span ratios;

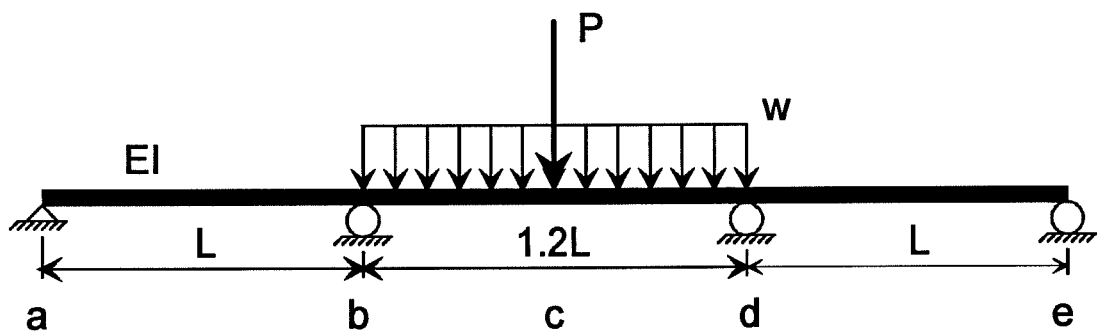


Figure 4.3 Loading to Produce Maximum Rotation at Interior Supports

for bridges with non-constant section properties; and for bridge systems where the maximum interior support rotations are produced by loading the end spans. The largest rotation would occur for cases in which the end spans are very long in relation to the center span, the limiting case being the center span simply supported at the ends. In this case, the maximum end rotations are 0.004 radians for uniform loading and 0.00375 radians for point loading.

4.2.2 Rotational Stiffness of Elements

Anchor Bolt System

The anchor bolt system can be modelled as shown in Figure 4.4. The cap girder and stiffeners are treated as a rigid body. The anchor bolts and the bearing pads, the flexible elements in the system, are replaced with springs of axial stiffness k_s . Applying a moment M to the rigid body, a rotation θ occurs and the springs deflect a distance Δ . The spring on the left contracts and the spring on the right extends. If both of the bolts are pretensioned to a force F_0 , a change in force will occur

$$\begin{aligned} F_L &= F_0 - k_s \Delta \\ F_R &= F_0 + k_s \Delta \end{aligned} \quad (4.12)$$

where the moment is

$$\begin{aligned} M &= d (F_R - F_L) \\ &= d (F_0 + k_s \Delta - F_0 + k_s \Delta) \\ &= d (2 k_s \Delta) \end{aligned} \quad (4.13)$$

and $\Delta = \theta d$. Substitution into Equation 4.13 gives the rotational stiffness as

$$\beta_b = \frac{M}{\theta} = 2 k_s d^2 \quad (4.14)$$

For multiple bolts on each side of the web of the cap girder, the equation is multiplied by the number of bolts on each side. If the bolts on one side lose all pretension force, only one side of bolts is effective and the equation is divided by two.

The term k_s is the stiffness of the anchor bolt and the bearing pad and is

$$k_s = \left[\frac{1}{k_b} + \frac{1}{k_p} \right]^{-1} \quad (4.15)$$

Substituting Equation 4.15 into Equation 4.14 gives the rotational stiffness of the anchor bolt system as

$$\begin{aligned} \beta_{1b} &= n_s d^2 \left[\frac{1}{k_b} + \frac{1}{k_p} \right]^{-1} \text{ for one side of bolts effective} \\ \beta_{2b} &= 2 n_s d^2 \left[\frac{1}{k_b} + \frac{1}{k_p} \right]^{-1} \text{ for both sides of bolts effective} \end{aligned} \quad (4.16)$$

where n_s is the number of bolts on one side of the web. The ratios of the calculated rotational stiffnesses to the experimental rotational stiffnesses are shown in Table 4.5. This table applies to the situation in which the anchor bolts on both sides of the web are effective. The experimental rotational stiffnesses are taken from Table 3.5 and the calculated stiffnesses are derived using the bearing pad stiffnesses given in Table 4.1.

Table 4.5 : Ratios of Calculated to Experimental Anchor Bolt Rotational Stiffnesses

BEARING PAD	$\beta_{\text{calc}} / \beta_{\text{exp}}$	
	$L_b = 56"$	$L_b = 85"$
NO PADS	2.07	1.36
1/2" PFP	1.41	1.12
3/4" PFP	1.31	1.06
1/2" ROF	1.27	1.15
3/4" ROF	1.34	1.25
5/8" NEO	1.21	1.16
3/4" NEO	1.05	1.02

The large difference of the stiffer systems is due to the uncertainty in the anchor bolt stiffness. In these systems, the bearing pad stiffness is comparable to the anchor bolt stiffness, so the rotational stiffness is sensitive to the anchor bolt stiffness. The difference is smaller with the softer systems because the bearing pad stiffness is much less than the anchor bolt stiffness and therefore the total stiffness is not as sensitive to the anchor bolt stiffness.

Pier Column

The pier column restrains the longitudinal rotation of the cap girder by reacting against the horizontal displacement at the bearing. The rotational stiffness of the pier column with respect to the cap girder can be derived with a method similar to that used in section 4.2.1. The cap girder is treated as a rigid body, as shown in Figure 4.5. The flexural stiffness of the pier column is represented by a spring with stiffness k_{pf} . A moment is applied to the center of rotation of the system, which acts at a distance y above the spring. The moment produces a rotation θ and a deflection Δ in the spring. The force in the spring is

$$H_{pf} = k_{pf} \Delta \quad (4.17)$$

and moment is

$$\begin{aligned} M &= H_{pf} y \\ &= k_{pf} \Delta y \end{aligned} \quad (4.18)$$

where $\Delta = \theta y$. The rotational stiffness is

$$\beta_{pf} = \frac{M}{\theta} = k_{pf} y^2 \quad (4.19)$$

The flexural stiffness of the pier column, first given in Appendix A is restated as

$$k_{pf} = \frac{3 (EI)_c}{(L_c + L_{cb})^3} \quad (4.20)$$

where $(EI)_c$ is a stiffness property of the pier column, L_c is the length of the pier column, and L_{cb} is an additional length added to the actual length to account for the flexibility of the pier column foundation.

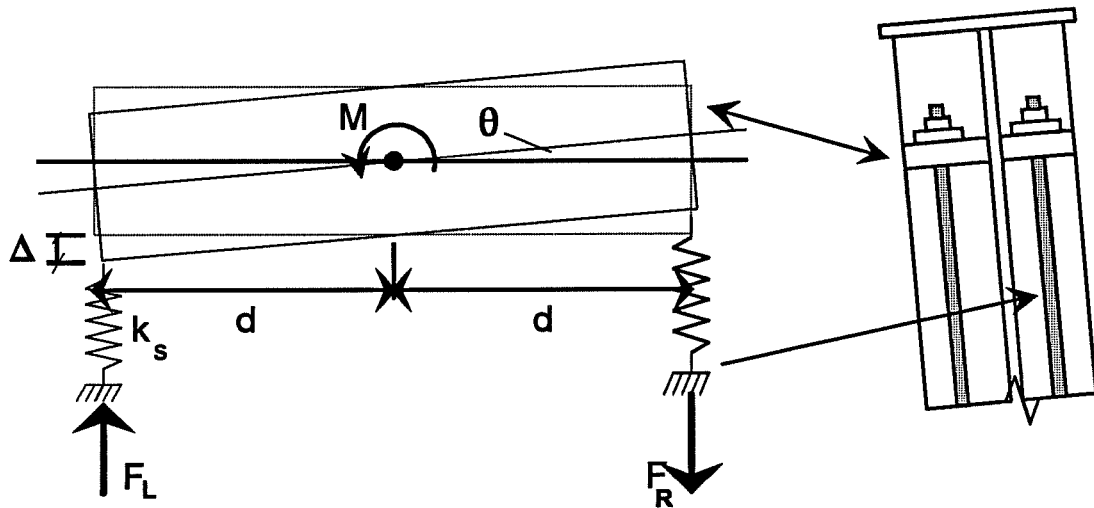


Figure 4.4 Model of Anchor Bolt System

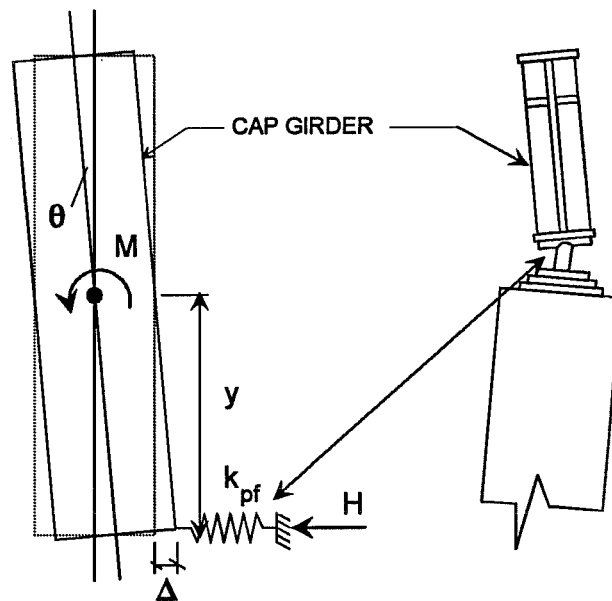


Figure 4.5 Model of Pier Used to Determine Rotational Stiffness

4.2.3 Forces Developed in the Elements

Anchor Bolts

The moment resisted by the anchor bolts is proportional to the rotational stiffness of the anchor bolt system. The force in the anchor bolts on each side of the web is a function of moment and is given as

$$\begin{aligned} F_L &= F_0 - \frac{M}{2 d n_s} \\ F_R &= F_0 + \frac{M}{2 d n_s} \end{aligned} \quad (4.21)$$

where the moment is equal to $\beta_b \theta$, giving

$$F_b = F_0 \pm \frac{\beta_b \theta}{2 n_s d} \quad (4.22)$$

The change in force in the bolt is represented by the second term in Equation 4.22. To get the tensile stress range of the bolt it is necessary to divide by the area of the bolt. The tensile stress range of a bolt is

$$S_b = \frac{\beta_b \theta}{2 n_s d A_t} \quad (4.23)$$

where A_t is the tensile stress area of the bolt. In determining the stress range for the anchor bolts, the rotation should be doubled. As a load moves along the length of the bridge the cap girder rotates; if the load is in the span to the left of support b in Figure 4.3 the cap girder at support b rotates counterclockwise. This will reduce the tension in the bolts on the left side of the cap girder web and will increase the tension in the bolts on the right side of the web; both sets of bolts experience a change in tensile stress. When the load moves into the second span the cap girder will rotate clockwise and the tension in the bolts on the left side of the web will increase while the tension in the bolts on the right side of the web will decrease. Assuming that the maximum rotation produced by load in the end span is the same as the maximum rotation

produced by the load in the center span, the tensile stress range in the bolts will be twice that produced by the load moving through the center span alone.

Determination of the tensile stress range in a bolt is illustrated by a short example. The properties of the connection designed in Appendix A will be used, with the exception that a neoprene bearing pad will be used instead of a preformed fabric pad. The parameters of the connection are listed below:

- Maximum Rotation Range θ : use $2(.003) = 0.006$ radians
- Number of bolts on each side of web $n_s = 4$
- Diameter of anchor bolt = 1.5 inches ($A_t = 1.41 \text{ in}^2$)
- Distance from anchor bolt to web centerline $d = 7$ inches
- Neoprene Pad Properties: 70 Durometer : 6.5" X 6.5" X 1" : $A = 40 \text{ in}^2$: SF = 1.53
using Equation 4.4, $E_p = 1.61 \text{ ksi}$: $k_p = 40(1.61) / 1 = 64.4 \text{ kips / in}$

The rotational stiffness of the anchor bolt system is determined using Equation 4.16 and it will be assumed that the bolts on both sides of the web are effective. The stiffness of the anchor bolt will not be considered, which will produce a conservative estimate of the rotational stiffness. The rotational stiffness of the anchor bolt system is determined as $\beta_b = 2 n_s d^2 k_p = 2(4)(7^2)64.4 = 25245 \text{ in-kips / radian}$. The stress range in each bolt is

$$S_b = \frac{25245 (0.006)}{2 (4) (7) 1.41} = 1.92 \text{ ksi}$$

which is much less than the allowable tensile stress range for anchor bolts of 8 ksi. As long as a neoprene bearing pad is used in combination with the anchor bolt, fatigue of the anchor bolts due to longitudinal rotation is not a concern.

Bearing and Pier Column

The horizontal force applied to the bearing and pier column due to the longitudinal rotation is

$$H = \frac{\beta_{df} \theta}{y} \quad (4.24)$$

which, when substituting in Equation 4.19, gives

$$H_r = \frac{3 (EI)_c}{(L_c + L_{cb})} y \theta \quad (4.25)$$

In the negative moment region over the pier supports, no shear studs are used so the concrete deck slab is not composite with the longitudinal stringers. The center of rotation can be assumed coincident with the neutral axis of the longitudinal stringers, so y will be the distance from the center of the stringer to the bottom of the cap girder.

The force produced by the change in temperature of the superstructure for a three span bridge with expansion bearings at the abutments and identical fixed bearings and pier columns at the two interior supports is

$$H_t = \frac{3 (EI)_c}{2 (L_c + L_{cb})} \Delta_t \quad (4.26)$$

where $\Delta_t = \alpha L \Delta T$; α is the coefficient of thermal expansion, which for steel is $6.5E-6 / ^\circ\text{F}$; L is the length of the center span; and ΔT is the change in temperature, which for steel structures TxDOT uses 50°F .

If Equation 4.26 is divided by Equation 4.25, the ratio of the horizontal force due to temperature change to the horizontal force due to rotation can be determined. This ratio can establish whether the horizontal force due to rotation is a quantity that needs to be considered in design.

Carrying out the division gives

$$\frac{H_t}{H_r} = \frac{\Delta_t}{y \theta} \quad (4.27)$$

where $\Delta_t = \alpha L \Delta T = 6.5e-5(50)L = 0.000325L$, $\theta = 0.003$ radians, and y can be estimated as $d / 2$ where d is the depth of the longitudinal stringer. This depth can be related to L through the span-to-depth ratio limitations recommended by AASHTO (10.5.2), which are $L / d = 25$ for the depth of the stringer including the thickness of the slab and $L / d = 30$ for the depth of the stringer alone. Using the more conservative limitation of $L / d = 25$, $y = L / 50$. Substituting these values into Equation 4.27 gives $H_t / H_r = 5.41$, which states that the horizontal force due to the temperature change is typically over five times greater than the horizontal force produced by the longitudinal rotation. The horizontal force is necessary to check the shear and bending stresses at the base of the pin where it is welded to the top bearing plate. For the primary stress check the temperature-induced force controls; for a pin with a 50 ksi yield stress the maximum bending stress is 27.5 ksi. For the fatigue check the horizontal force produced by the longitudinal rotation must be checked; given its relationship to the temperature-induced force the maximum tensile stress range in the pin would be $27.5 / 5.41 = 5$ ksi. The axial stress in the pin due to the dead load will almost certainly be greater than 5 ksi. This compressive stress will eliminate the tensile stress range, so fatigue will not be a concern. Therefore, for typical situations the horizontal force produced by the longitudinal rotation does not have to be considered in the design.

4.2.4 Effect of Removal of Upper Stiffener

The upper stiffener, shown in Figure 4.6, presents fabrication problems because of the tight fit-up required. The author witnessed the fabrication of the cap girder used in the tests and significant delays resulted from the inability of the fabricator to fit the stiffener, which was cut too long and required repeated grinding and refitting. The upper stiffener is not necessary as a bearing support since no concentrated loads are applied to the top flange of the cap girder. It restricts access to the anchor bolt nuts and so its removal would ease the installation process. Its

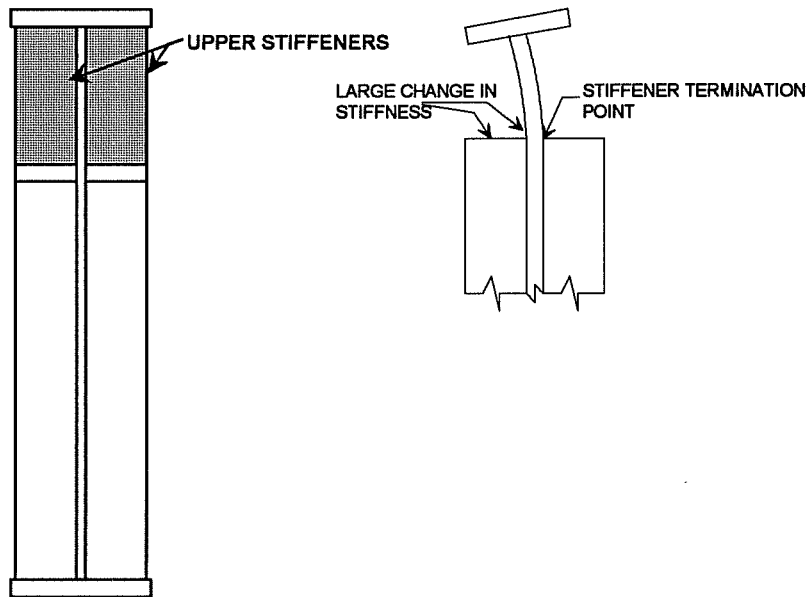


Figure 4.6 Stiffener Termination Point

removal, however, introduces a potential problem resulting from the out-of-plane rotation of the cap girder web. The longitudinal rotation of the cap girder is resisted by the anchor bolt system; with the upper stiffener in place, the web near the stiffeners is stiffened along its entire length. If the upper stiffener is removed, a sharp change in stiffness occurs at the termination of the stiffener, as shown in Figure 4.6. If the cap girder is not restrained from rotating by any anchor bolts, the web and the stiffeners rotate as a rigid body and no stress will be produced at the stiffener termination point. If anchor bolts are present and resist the cap girder rotation, however, a stress riser will develop at this location due to the sharp change in the stiffness and geometry. The area around the stiffener termination point become susceptible to distortion-induced fatigue cracking. To determine if the stresses produced in this area are high enough to produce a fatigue concern, finite element analyses using the ANSYS program were conducted on the section of the cap girder web near the connection.

Finite Element Model

The area modelled is shown in Figure 4.7. Symmetry was used to reduce the size of the problem. The objective of the study was to determine the maximum web tensile stress that would be produced as a function of four variables. The variables were the stiffener height, the anchor bolt stiffness, the spacing of the longitudinal stringer, and the web thickness. The load was an applied rotation at the stringer of 0.003 radians, the maximum probable rotation as determined in the previous section. The parameters of the analyses are listed below:

- Web Thickness, t_w : 0.5 inch, 1 inch, 1.5 inch
- Web Depth, h_w : 6 feet
- Stringer Spacing, S : 2 feet, 4 feet, 6 feet
- Stiffener Height, h_{st} : 2 feet, 4 feet
- Stiffener Thickness, t_{st} : 1 inch
- Anchor Bolt Stiff, k_b : 100, 500, 1000, 2000, 4000, 8000, ∞ kips / in
- Bearing Width, b_w : 36 inches

All of the values are typical of those used in actual designs and the maximum and minimum values for the web thickness and stringer spacing are bounds between which all designs will fall. The finite element mesh is shown in Figure 4.8. A three inch by three inch grid was used. The web and stiffeners are composed of 4-node shell elements with six degrees of freedom at each node. Beam elements were used to model the top and bottom flanges of the cap girder. The anchor

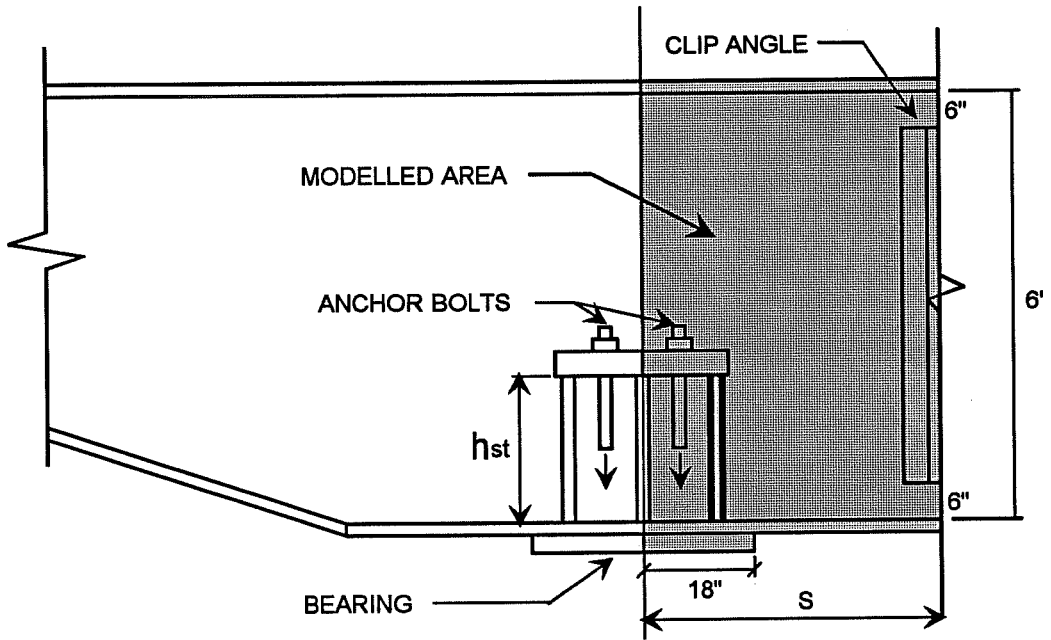


Figure 4.7 Web Area Modelled

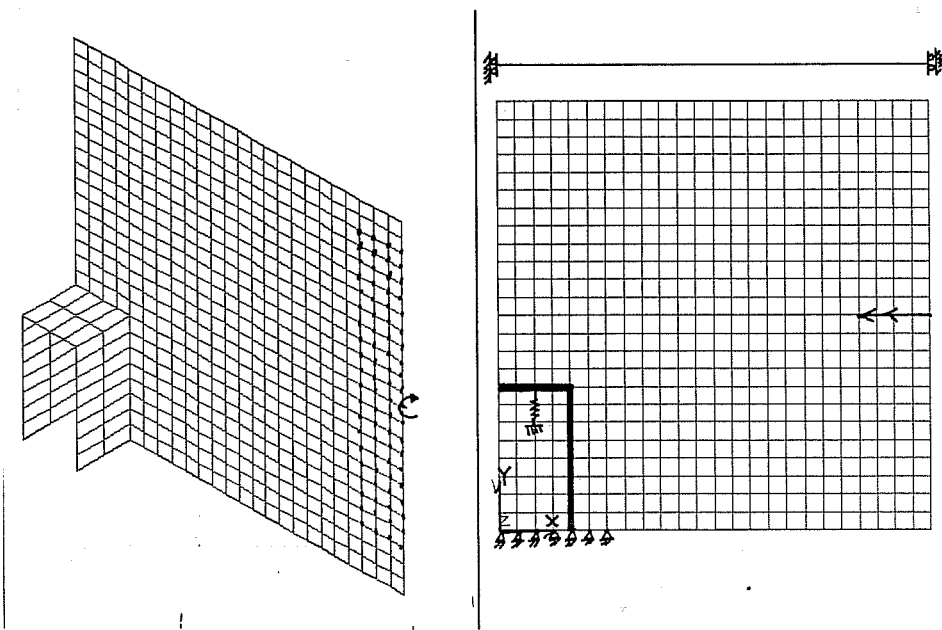


Figure 4.8 Finite Element Mesh

bolts were modelled as one-dimensional springs in the Y-direction. The cap girder was restrained in the Y and Z direction along 18 inches on the bottom flange; this simulated the bearing connection. Though the bearing actually displaces horizontally, the horizontal fixity would represent the worst case. The rotation was applied at mid-depth of the web along the stringer boundary line. An area of nodes representing the leg of the clip angle connection was rigidly coupled to the rotated node.

Results

The stringer spacing and the web thickness were varied for the first set of analyses. The stiffener height was held constant at 24 inches and the cross-plate was fixed in the vertical direction at the anchor bolt location. This fixity simulated an infinitely stiff anchor bolt, which is an upper bound on the rotational stiffness of the connection. A typical contour plot of the principal tensile stress is shown in Figure 4.9. As expected, the area of maximum stress surrounds the stiffener termination point. The plotted stresses are interpolated from the node stresses. Since the mesh is very coarse in the area around the stiffener termination point, the maximum stress occurs at the node three inches above, rather than the point itself. The ANSYS program calculates an estimated maximum bound for the stress based on the error due to the mesh discretization. This stress is shown as SMXB in the contour bar. In most cases, this bound was two to three times the maximum plotted stress. A plot of the maximum plotted principal tensile stress vs. the stringer spacing is shown in Figure 4.10. The curves, one for each web thickness, show that the stress increases as the spacing is reduced and that the stress increases with an increase in the web thickness. The purpose of this first set of analyses was to determine the combination of the stringer spacing and web thickness that produced the largest stresses. This occurred with the stringer spacing of 2 feet and the web thickness of 1.5 inches.

The anchor bolt stiffness was varied for the second set of analyses. The stiffener height was held constant at 24 inches, the stringer spacing was set at 2 feet, 4 feet, and 6 feet, the web thickness was held constant at 1.5 inches, and the anchor bolt stiffness was varied from 100 kips per inch to 8000 kips per inch. Figure 4.11 shows the maximum plotted principle stress vs. the anchor bolt stiffness. Up to a stiffness of 2000 kips per inch the stress was not affected too much by the change in stringer spacing. For anchor bolt stiffnesses greater than 2000 kips per inch the maximum stress increased as the stringer spacing decreased. The anchor bolt stiffness, given in kips per inch, does not have much meaning. The minimum anchor bolt diameter allowed by the

AASHTO code (10.29.6.2) is 1.5 inches for span lengths greater than 150 feet. If the length of the bolt is assumed to be four feet (the stiffener height plus two feet to the top of the pier cap) the stiffness of the anchor bolt is $k_b = AE / L = (1.77)29,000 / 48 = 1069$ kips per inch. The anchor bolt will be used in conjunction with a neoprene bearing pad. Using the 3/4 inch neoprene pad stiffness given in Table 4.1, 57 kips per inch, the stiffness of the anchor bolt-bearing pad system is 54 kips per inch. At this value of stiffness, the stress is insignificant. If the anchor bolt was used without the bearing pad, the stress would be approximately 6 ksi for a two foot stringer spacing. This stress, however, may be two to three times less than the actual maximum stress because of the coarse mesh.

The third set of analyses was conducted to determine the effect of the stiffener height on the maximum stress and to calculate the actual stresses using a refined mesh. For this set of analyses, the stringer spacing was held constant at two feet, the spacing producing the largest stresses; the web thickness was held constant at 1 inch, a typical value; the anchor bolt stiffness was set at 100 kips per inch and 1000 kips per inch, realistic values; and the stiffener height was set at 2 feet and 4 feet. The maximum stress bound vs. the stiffener height is shown in Figure 4.12. The maximum stress decreased with an increase in the stiffener height for both anchor bolt stiffnesses, though the change was more pronounced for the larger bolt stiffness. The maximum stress bound is an estimate by the program taking into account the error inherent in a coarse mesh. To check the accuracy of the program estimate, the mesh around the stiffener termination point was refined using a process called submodelling and the section was reanalyzed.

The area of mesh refinement is shown in Figure 4.13. In the submodelling process, the larger structure is analyzed using a coarse mesh and the results are saved. The smaller section, in which a more accurate analysis is needed, is remodelled using a finer mesh. The smaller section is oriented using the coordinates of the same area in the larger structure. To the edges of the submodel, called the cut boundaries, are applied the boundary conditions from the coarse mesh results. These boundary conditions are simply the displacements and rotations of the nodes in the coarse mesh that lie on the cut boundaries of the submodel. If the nodes of the coarse mesh do not coincide with the nodes on the cut boundaries, interpolation is used. The refined mesh around the area of the stiffener termination point is shown in Figure 4.14. The submodel analysis was run for the model with the four foot stiffener height. A typical contour plot of the principle maximum stress is shown in Figure 4.15. The peak stress occurs at the stiffener termination point; theoretically, this stress should be infinity because of the sharp corner, and

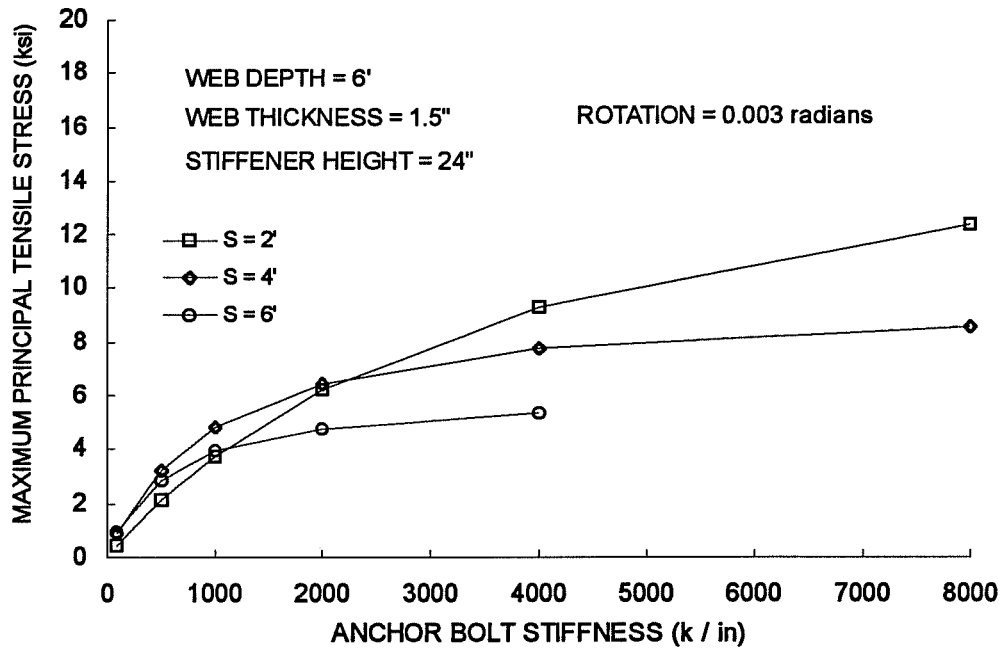


Figure 4.11 Principal Tensile Stress vs. Anchor Bolt Stiffness

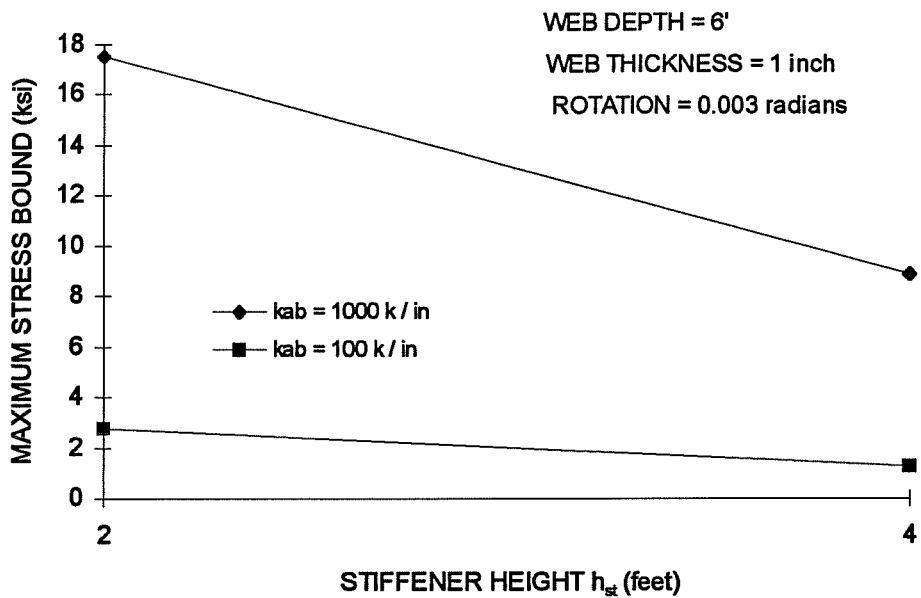


Figure 4.12 Maximum Stress Bound vs. Stiffener Height

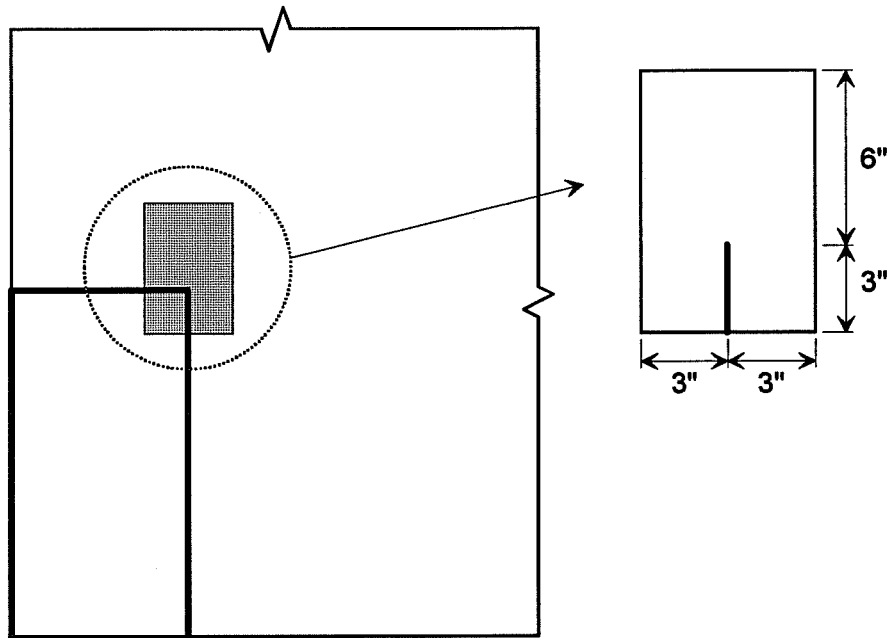


Figure 4.13 Area of Mesh Refinement

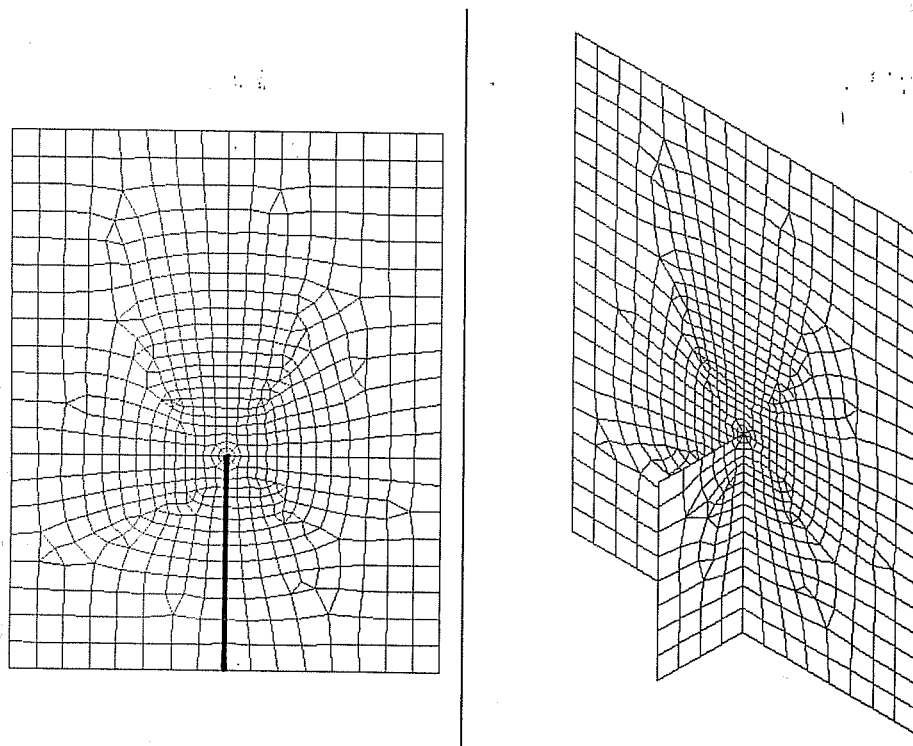


Figure 4.14 Refined Finite Element Mesh

indeed the stress is extremely high at this point. The peak stress for the refined mesh was taken as the stress of the contour band that surrounded the stiffener termination point. A comparison of the maximum stress from the coarse mesh model and the peak stress from the refined mesh is shown in Table 4.6 for the model with the two foot stringer spacing. Also shown is the maximum stress bound that was calculated for the coarse mesh. This bound appears to be a reasonable approximation of the maximum stress in the refined model.

Table 4.6 : Comparison of Maximum Stress for Coarse and Refined Mesh

$S = 2 \text{ feet} : t_w = 1 \text{ inch} : h_{st} = 48 \text{ inches} : \theta = 0.003 \text{ radians}$		
	$k_{ab} = 100 \text{ k / in}$	$k_{ab} = 1000 \text{ k / in}$
COARSE MESH	.4 ksi	3.5 ksi
REFINED MESH	1.3 ksi	9.0 ksi
Max Bounded Stress	1.25 ksi	8.9 ksi

The effect of the stiffener height on the maximum stress is shown in Table 4.7. The maximum stress bound is used since it is a reasonable estimate of the actual maximum stress. The maximum stress decreased with an increase in the stiffener height for the anchor bolt stiffnesses of 100 kips per inch and 1000 kips per inch.

Table 4.7 : Effect of Stiffener Height on Maximum Stress

$S = 2 \text{ feet} : t_w = 1 \text{ inch} : h_{st} = 48 \text{ inches} : \theta = 0.003 \text{ radians}$		
STIFFENER HEIGHT	$k_{ab} = 100 \text{ k / in}$	$k_{ab} = 1000 \text{ k / in}$
2 feet	2.74 ksi	17.5 ksi
4 feet	1.25 ksi	8.9 ksi

Conclusions

The conclusions of the finite element analyses are listed below:

- The maximum principal tensile stress at the stiffener termination point increases with a decrease in the stringer spacing and an increase in the web thickness.
- The maximum principal tensile stress at the stiffener termination point decreases with a decrease in the anchor bolt stiffness. For the combination of anchor bolt and bearing pad likely to be used in a design, the maximum stress is small and not likely to pose any fatigue problems.
- The maximum principle stress at the stiffener termination point decreases with an increase in the stiffener height.

If the stiffener detail is classified as a category C, which encompasses welded transverse stiffener details, the allowable tensile stress range would be 10 ksi for over 2 million cycles, which is the typical design requirement. The classification of the detail as such is supported by past research [], illustrated in Figure 4.16. The graph shows the fatigue performance of various types of details similar to the detail examined here. All of the data fall above the category C curve. The stress range of 10 ksi is approached and exceeded for a rotation of 0.003 radians when the anchor bolt is used without a bearing pad ($k_{ab} = 1000 \text{ k / in}$, (see Table 4.7) but is approximately seven times greater than the maximum stress for a typical anchor bolt-bearing pad combination ($k_{ab} = 100 \text{ k / in}$). Based on the results given above, it is concluded that the upper stiffener may be removed without a concern for fatigue at the stiffener termination point as long as a neoprene bearing pad is used in conjunction with the anchor bolt.

4.3 TRANSVERSE DIRECTION BEHAVIOR

It was concluded from the results presented in Chapter 3 that the standard TxDOT connection should not be used in situations where uplift occurs. A general discussion of the transverse direction behavior is provided in Chapter 8 in conjunction with a discussion on the characteristics of a new connection detail that is designed for uplift.

Specimen Type and Loading	Weld	SYMBOL
	Partial Penetration Fillet	□
	Full Penetration Fillet	■
	Fillet	●
	Non-Load Carrying Weld	△
	Type A: Full Penetration Fillet Weld	▲
	Type B: Fillet Weld	▲

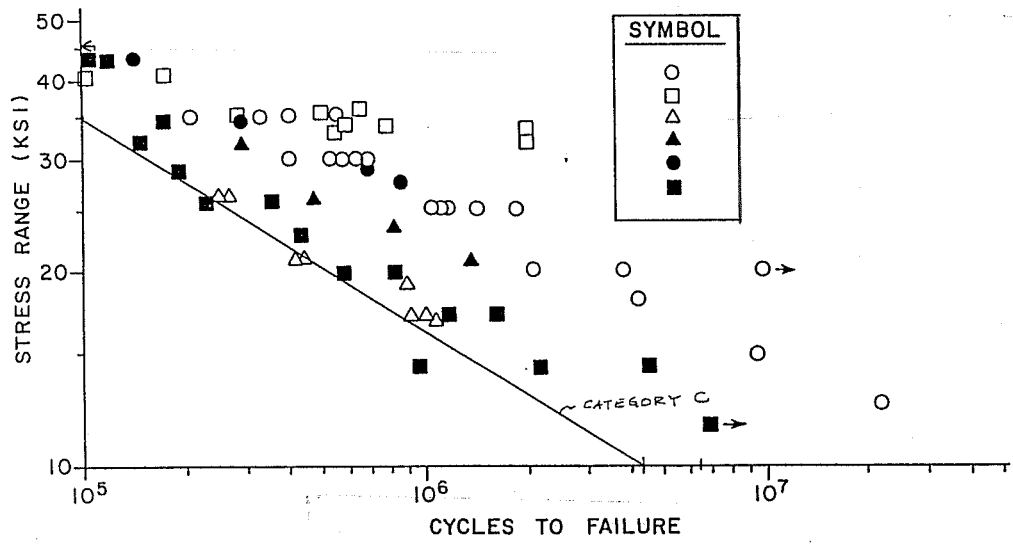
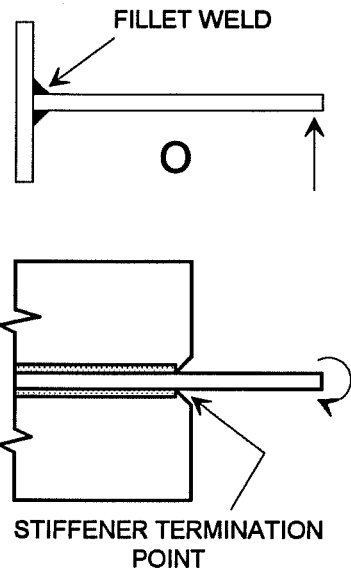


Figure 4.16 S-N Plot of Details Similar to Stiffener Termination Point
(adapted from Joehnk [11], pp.58-59)

CHAPTER 5 : PHASE I CONCLUSIONS AND RECOMMENDATIONS

5.1 CONCLUSIONS

Longitudinal Direction Behavior

There are two elements of the standard TxDOT connection that restrain the longitudinal rotation of the cap girder, the anchor bolts and the pier column. The objective of the examination of the longitudinal direction behavior was to determine whether this restraint produced reactions in the restraining elements that must be considered in design. With regard to the anchor bolts, the results and analysis showed that fatigue is not a concern if a neoprene bearing pad is used under the plate-washer. The longitudinal rotation also produces a horizontal force on the rocker pin, but this force is much smaller than the horizontal force produced by the change in length of the superstructure caused by a temperature change. It was shown that it is not necessary to consider this force in the fatigue design. In summary, the connection behaved as expected in the longitudinal direction.

Transverse Direction Behavior

The standard TxDOT connection cannot efficiently resist uplift. This is based on the transverse direction tests and the pretensioning tests. The long length of the anchor bolts and the bearing pads used under the plate-washer, the functions of which are to decrease the longitudinal rotational stiffness of the anchor bolt system, also decrease the rotational stiffness of the connection in the transverse direction. As a result, significant vertical deflection will accompany any uplift force and the anchor bolts, which resist the uplift, will be loaded in tension. The cyclic nature of the vehicle loads subject the anchor bolts to fatigue loading, but the allowable fatigue stress is much lower than the allowable static load tension stress. If four 1.5 inch diameter anchor bolts with a yield stress of 105 ksi were used in a connection, the tensile force that could be applied to the connection would be $T = 4 (0.55F_y)A_t = 4 (0.55 \cdot 105)(1.41) = 326$ kips. The allowable live load tensile stress, which will produce fatigue loading in the bolts, would only be $T_L = 4 (8)A_t = 4(8)(1.41) = 45$ kips, where the anchor bolts are treated as a category E detail with an allowable stress range of 8 ksi. The tensile stress range can be eliminated by pretensioning the anchor bolts, but this option is impractical for the current detail because the soft bearing pads, which provide for the flexibility in the longitudinal direction, preclude the

development of any significant pretensioning force in the anchor bolts. In addition, the upper stiffener detail severely restricts access to the anchor bolt nut for tightening. An uplift resistant connection is developed in the second phase of the research.

5.2 RECOMMENDATIONS

5.2.1 Design Changes

Use Neoprene Bearing Pads Under the Plate-washer

Longitudinal rotation of the cap girder produces forces in the anchor bolts. The bearing pads used under the plate-washer increase the flexibility of the connection and thereby reduce the magnitude of the forces developed. The 70 durometer neoprene bearing pads decreased significantly the axial stiffness of the anchor bolts while the 90 durometer preformed fabric pads produced a much smaller decrease.

Eliminate the Upper Stiffener or Cut Short of Top Flange

This recommendation is based on the fabrication difficulties observed by the author and the limited access for anchor bolt tightening. A full depth bearing stiffener detail must be cut to exacting tolerances, particularly when a tight fit or a mill-to-bear specification is made. An overcut stiffener must either be ground down to the correct size or forced into place, each of which are time-consuming and expensive. The upper stiffeners also restrict access to the anchor bolt nut. Elimination would significantly ease the installation process.

Decrease the Number of Bearing Plates

The sizing of the bearing plates is based on an analysis procedure that produces a very conservative design. The size of the bottom bearing plate is governed by the allowable concrete bearing stress, which is also very conservative. This limitation is the primary determinant of the bearing size. The design of the bearing plates is addressed in the second phase of the research.

5.2.1 Fabrication Changes

Implementation of the above modifications only slightly alter the basic configuration of the connection. The primary objective of the research is to develop a new detail that performs as well as or better than the current detail and is less expensive to fabricate. Consultation with a steel fabricator identified those aspects of the connection that significantly increase the fabrication cost. They are listed below:

Groove Weld Between Rocker Element and Top Bearing Plate

This weld is very expensive and time-consuming to make; considerable savings would be realized if it is eliminated. A possible solution is to make the rocker element and the top bearing plate a monolithic unit.

Machined radius of rocker and sole plate

The semi-circular groove in the sole plate and the rounded end of the rocker element require precision machining that is not justified on structural grounds. The surfaces will rarely match up in the field with the same precision with which they were machined. Fabrication cost would be reduced if this detail was eliminated or less precise machining was used.

Mill-to-Bear Specification Between Stiffeners and Bottom Flange of Cap Girder

The machining required for the mill-to-bear spec increases the fabrication costs and is not justified on structural grounds.

Based on the data from the experimental testing and the recommendations by the fabricator, a new detail is proposed. The new detail has the capability of resisting uplift.

5.3 NEW CONNECTION DETAIL

The new connection detail is shown in Figures 5.1 and 5.2. It incorporates two major changes from the standard TxDOT connection; the first change is to replace the rocker element and sole plate with a rolled wide-flange section. This change eliminates two of the problems listed above, the groove weld between the rocker element and the top bearing plate and the machined

radii of the sole plate and rocker element. The wide-flange should be capable of resisting substantial compressive loads and the web should allow relatively free out-of-plane movement. The primary movement that must be accommodated by the bearing is horizontal translation and the wide-flange web should prove to be much more flexible than the short, stiff rocker element now used. These hypotheses, of course, will be tested in the Phase II research. A major advantage of the wide-flange bearing detail is that it provides a positive connection between the cap girder and the pier cap; the current detail requires anchor bolts for a positive connection. For situations in which uplift does not occur, the wide-flange bearing connection can be used without anchor bolts, as shown in Figure 5.3.

The standard TxDOT connection uses threaded steel rod for the anchor bolts. The new detail would replace the threaded steel rod (and bearing pad) with high strength threadbar, which is specifically designed for pretensioning and for which installation procedures are well established.

- The new detail can resist uplift; the current detail cannot. When uplift does not occur, the new detail can be used without anchor bolts.
- The new detail is redundant with respect to uplift failure; a positive connection is maintained by the wide-flange bearing if the anchor bolts fail.
- The design and fabrication of the new detail is simpler and probably less expensive than the current detail.

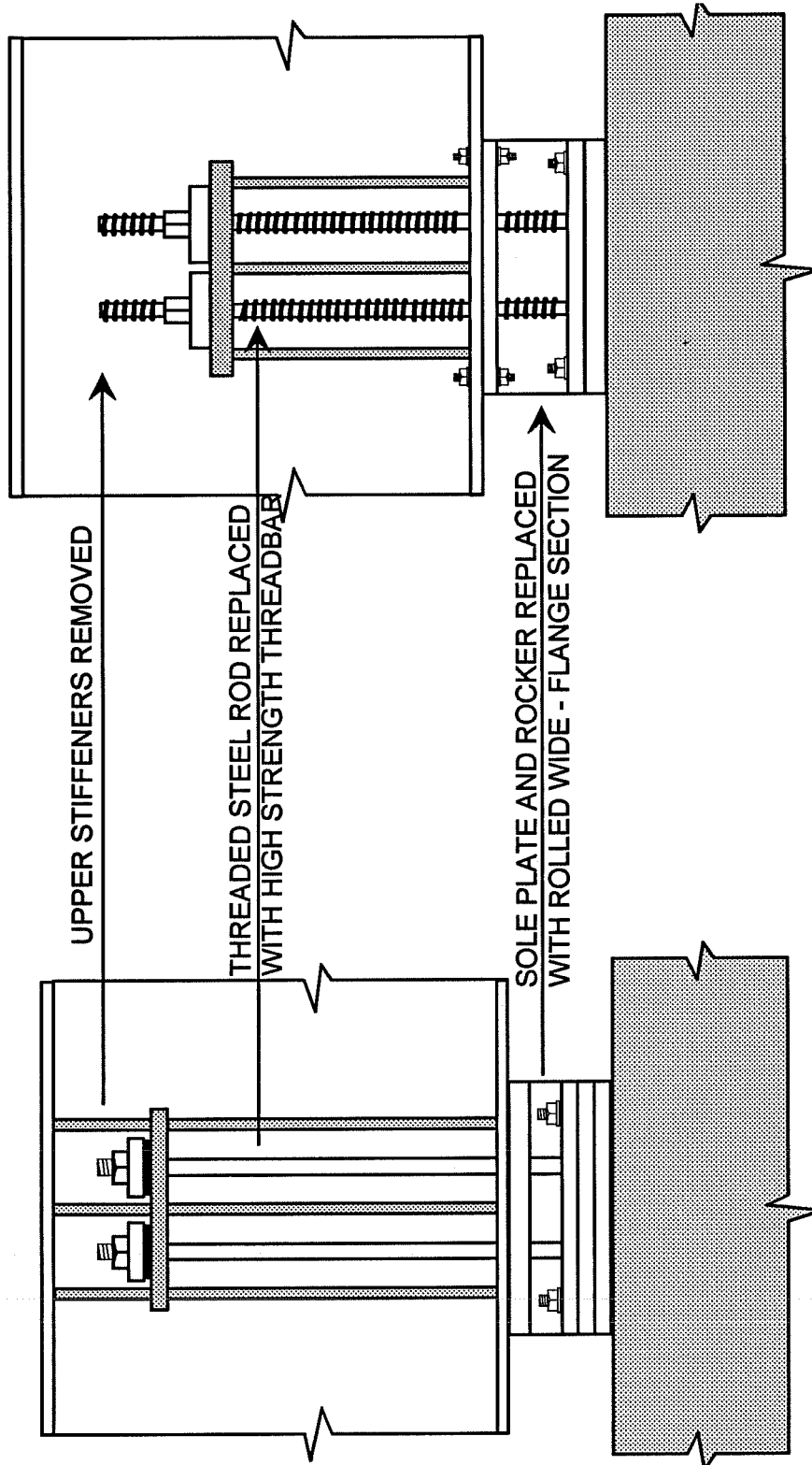


Figure 5.1 Comparison of Standard Connection and Proposed Connection, Transverse Direction

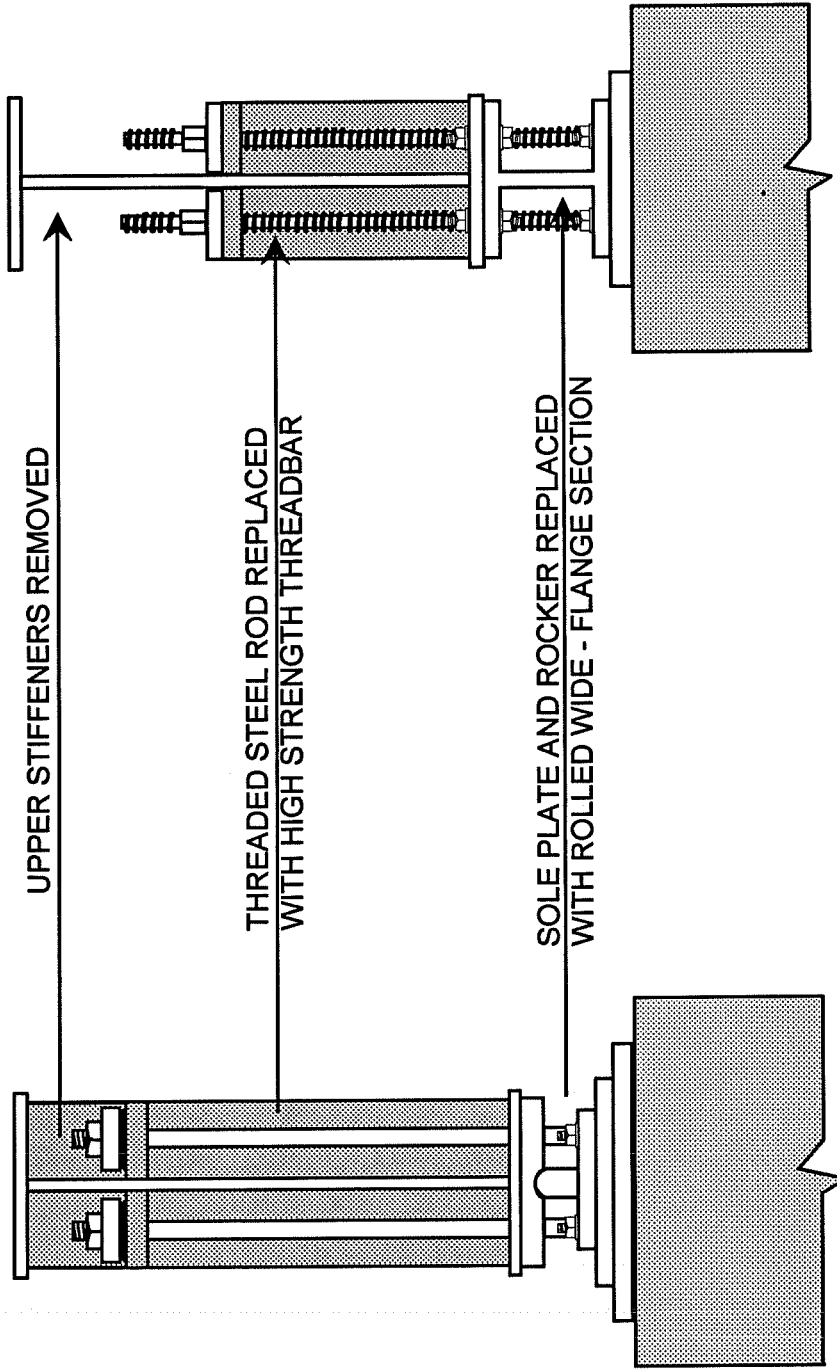


Figure 5.2 : Comparison of Standard Connection and Proposed Connection, Longitudinal Direction

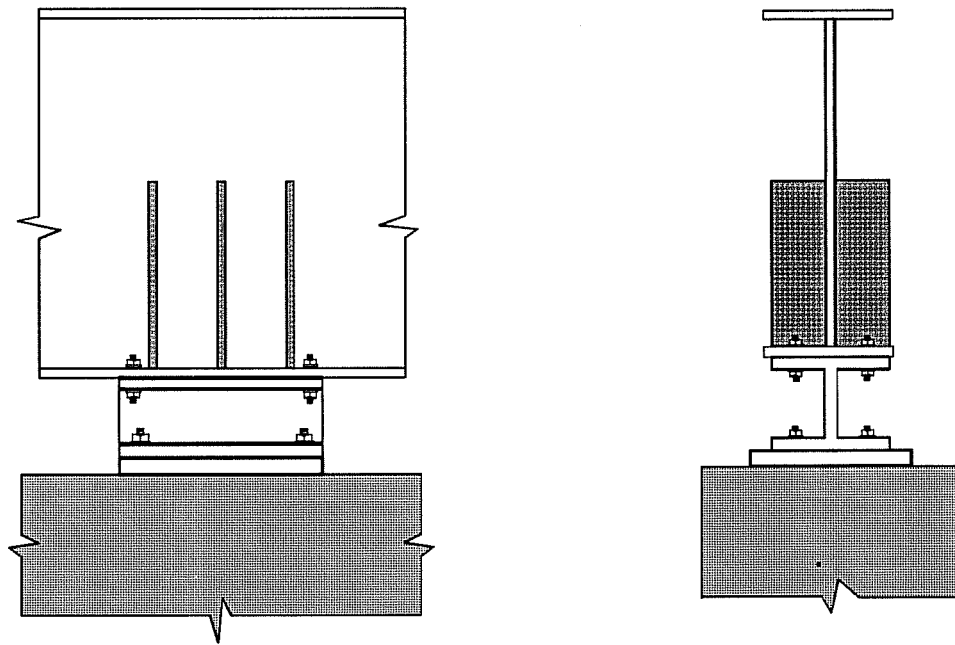


Figure 5.3 Proposed Connection, No Uplift

CHAPTER 6 : PHASE II COMPRESSION TESTS

6.1 TEST PROGRAM

The new detail is evaluated using both experimental testing and analytical modelling. The experimental testing comprised component tests and large scale tests. The component tests were conducted on the wide-flange section bearings and comprised compression tests and out-of-plane fatigue tests. Large scale tests were conducted on the new detail in a manner similar to the full scale tests conducted in the first phase testing. In this section the component compressive tests of the bearing are presented and discussed. The objective of the tests was to determine the vertical load capacity of the wide-flange bearing web.

Three sizes of wide-flange sections were selected as bearing specimens for the components tests. The sizes were a representative sample of the standard sections classified as the W12 and W14 column sections in the AISC Manual of Steel Construction [3]. The sizes tested were W12X87, W12X152, and W12X230. The material used was A572 Grade 50.

6.1.1 Variables

The primary variables for the compression tests were the slenderness ratio of the web and the thickness of the web. The slenderness ratio of the web was defined as the height of the web between the flanges, h , divided by the radius of gyration of the web, r . The range of the slenderness ratio for the W12 and W14 sections varies from 14 to 80 for web thicknesses between 0.5 inches and 2 inches. The nominal values of the web thicknesses and the slenderness ratios of the test specimens varied from 0.515 inches to 1.18 inches and from 31 to 71. An additional variable was considered, the width of the bearing. Two widths of bearings were tested, 24 inches and 36 inches, to determine whether the width of the bearing had any effect on the buckling stress. For plates with the unloaded edges supported in some way, the buckling stress is a function of the width-to-thickness ratio.

6.1.2 Test Schedule

The schedule for the compression tests is shown in Table 6.1. Each test is identified by an alphanumeric code: the first letter, C, indicates that it was a compression test; the next three digits show the weight per foot in pounds of the W12 section; the next two digits give the width of the specimen in inches; the last digit indicates the number of the specimen tested.

Replicate specimens were tested for each category. The width effect was examined using the W12X87 bearings; the widths tested were 24 inches and 36 inches. The slenderness effect and the web thickness effect were examined using all three specimen sizes. A more complete list of the specimen properties is given in Appendix C.

Table 6.1 : Bearing Specimen Compression Test Schedule and Properties

TEST	SIZE	WIDTH (inches)	WEB THICK (inches)	DEPTH (inches)	h/r
C087361	W12X87	36	0.511	10.91	71.2
C087362	W12X87	36	0.511	10.91	71.2
C087241	W12X87	24	0.516	10.89	70.3
C087242	W12X87	24	0.506	10.89	71.7
C087243	W12X87	24	0.512	10.89	70.9
C152361	W12X152	36	0.893	11.01	41.1
C152362	W12X152	36	0.879	11.01	41.8
C230241	W12X230	24	1.30	10.85	27.8
C230242	W12X230	24	1.30	10.85	27.8

6.1.3 Test Setup

The compressive loads that were expected exceeded the capacity of the largest test frame in the Ferguson laboratory. Therefore, a two million pound self-reacting frame was designed and constructed to test the bearings. A schematic of the test frame is shown in Figure 6.1 and a photograph of the frame as constructed in the lab is shown in Figure 6.2. The bottom platform supported a half section of a concrete pier cap and the top platform supported a 2000 kip capacity hydraulic ram. A 36" x 24" x 1 3/4" steel bearing plate transferred the compression load from the test specimen into the pier cap section. Shown in Figure 6.3 is the steel transfer beam that was located between the ram plunger and the test specimen to spread the load out along the width of the specimen. The bottom flange of the test specimen was secured to the concrete cap with eight 3/4 inch diameter coil rods, four rods on each side of the web. The top flange of the test specimen was bolted to the transfer beam with eight 7/8 inch diameter high strength bolts. These attachments were designed to simulate field practice.

It was assumed in the original design of the loading frame that the test specimens would fail in the fixed-fixed buckling model as illustrated in Figure 6.4. In this failure mode, the top and bottom flanges are restrained from both rotating and translating. The frame, however, lacked sufficient stiffness to prevent lateral sway of the top flange of the test specimen. A simplified model of this behavior is shown in Figure 6.5. As the test specimen is loaded, geometric imperfections cause an eccentricity in the load application, which causes a moment equal to the load, P , multiplied by the eccentricity, e . This moment must be balanced by a couple produced by a horizontal shear force, V , separated by the specimen depth, d . The shear force that develops causes the frame to sway laterally. To increase the lateral stiffness of the frame, a diagonal brace was added to the east and west sides of the frame. Though the brace did not eliminate the frame sway totally, it lessened the problem.

6.1.4 Instrumentation

The instrumentation used on the specimens consisted of strain gages, linear potentiometers, and servo inclinometers. In addition, whitewash was applied to the specimen and displacements were verified using rulers and a theodolite. A pressure transducer was used to measure applied load.

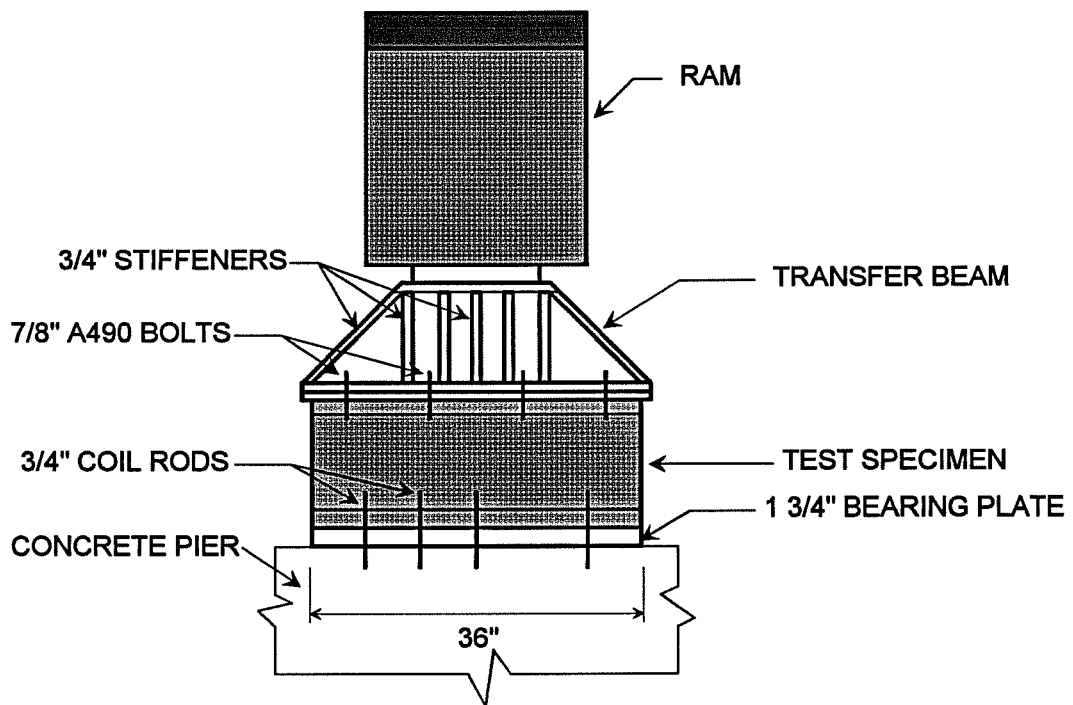


Figure 6.3 Compression Test Specimen

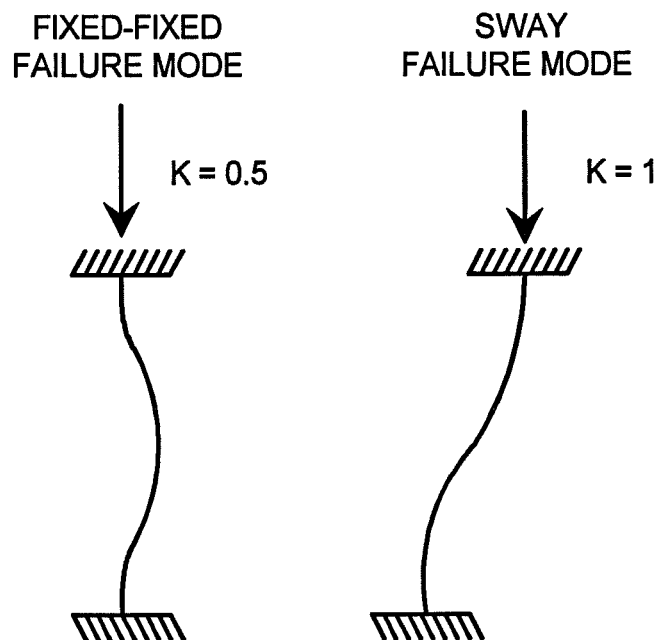


Figure 6.4 Failure Modes

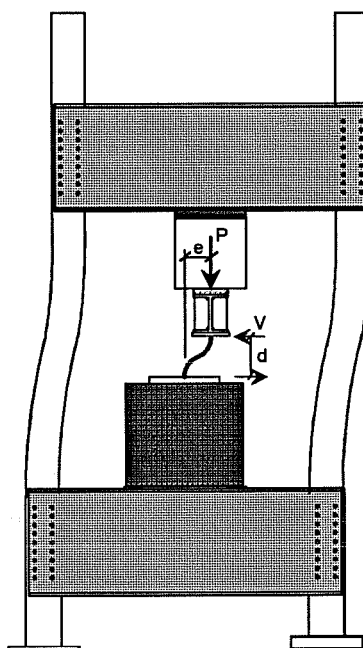


Figure 6.5 Test Frame Sway

The strain gages were used to determine axial stresses and bending stresses in the web. They were typically located at the web-fillet point of tangency at the top and bottom of the web, on both sides of the web, as shown in Figure 6.6. Strain gages were also placed on the base plate to determine the distribution of stresses. Ten gages were placed on the bottom of the plate: five gages were equally spaced along the 36" width, at the plate centerline; five gages were located on the north side of the plate, seven inches from the centerline. This location was approximately one inch past the edge of the bearing specimen flange. Five gages were located directly above this second set of gages, on the top of the base plate, as shown in Figure 6.7.

The linear potentiometers measured the lateral displacement of the test specimen at the top flange and the web centerline. In most cases, three linear pots were located along the width of the web, at the centerline, and two linear pots were placed at the top flange. The rotations at the top and bottom flanges of the test specimens were measured using servo inclinometers. The progress and extent of the web yielding was monitored by painting both sides of the web with whitewash, a mixture of lime and water. When yield lines develop, the large accompanying strains cause the mill scale to crack and flake off and the whitewash accentuates the yield line.

6.1.5 Test Procedure

Before each test, the cross-section profile of the test specimen was traced out on a piece of graph paper. The traced profile was used to measure the initial imperfections in the specimen and the depth value, h . The web thickness was measured at five different locations, evenly spaced, along the height of the web and at the strain gage locations. The measurements were taken at both ends of the specimen and the average of the ten readings is shown in Table 6.1. The specimen was then placed on the base plate and the bottom flange coil rods were tightened to secure the specimen to the concrete cap. For the W12X87 and W12X230 specimens, a layer of hydrostone was placed between the top flange and the transfer beam to mate the surfaces. The top flange bolts were then tightened. The W12X152 specimens had flanges that were not parallel. The relative angle between the two flanges was 0.01 radians, which compares with the mill tolerance value of 0.025 radians [3, p.5-251]. Rather than mate the surfaces of the top flange and transfer beam with hydrostone, contact between the surfaces was achieved by allowing the applied load to straighten out the test specimen. This is shown in Figure 6.8. This procedure was done to determine if large imperfections have any effect on load carrying capacity.

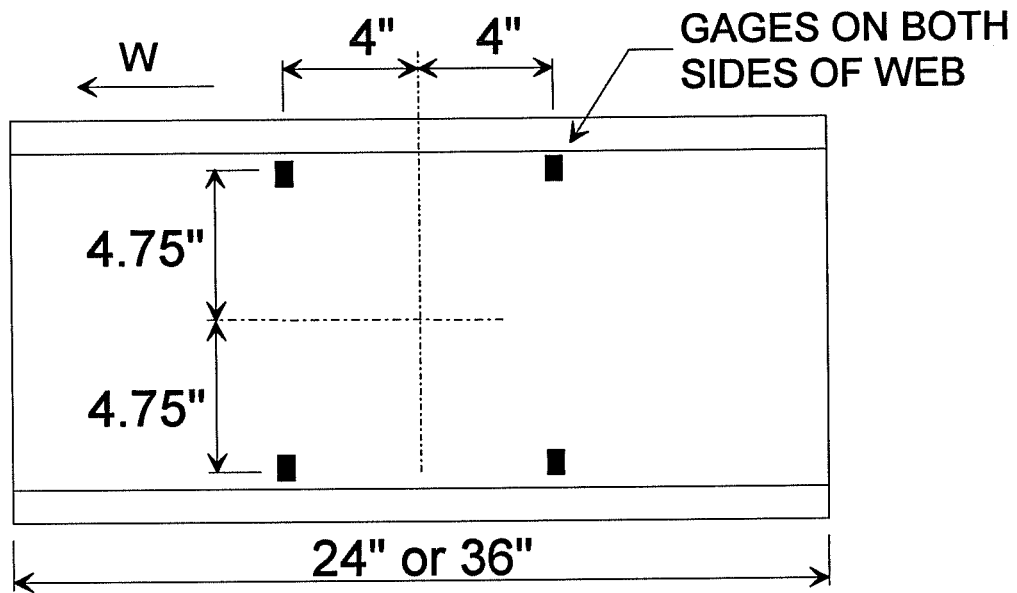


Figure 6.6 Location of Strain Gages on Test Specimen

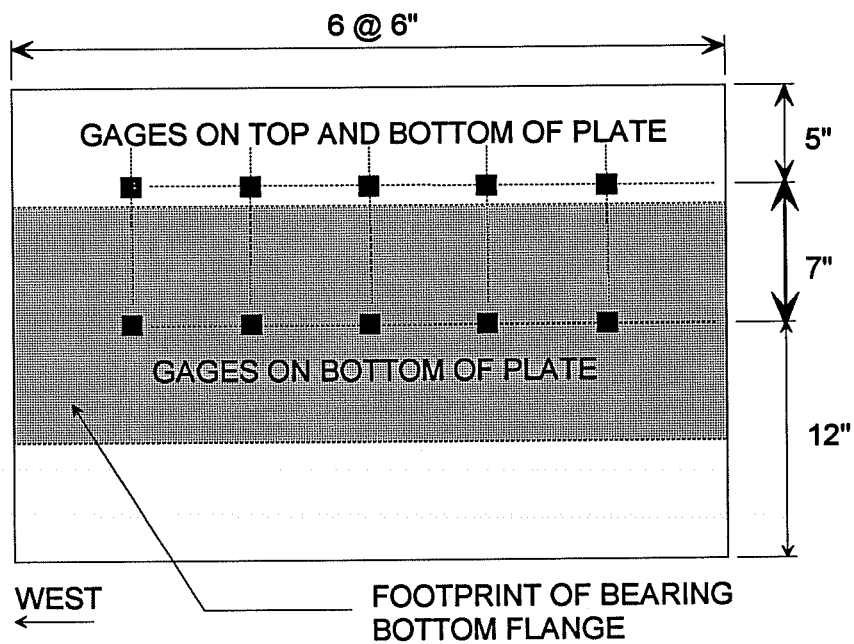


Figure 6.7 Location of Strain Gages on Bearing Plate

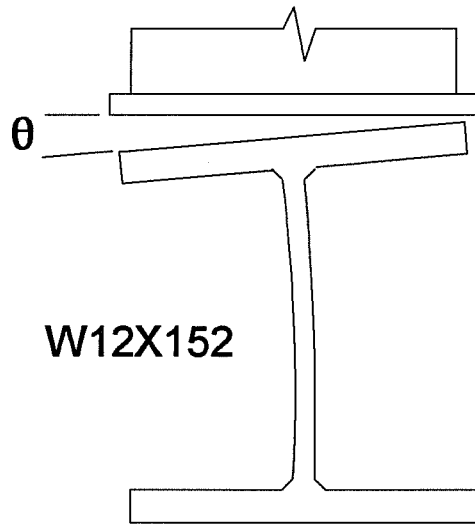


Figure 6.8 Flange Out-of-Straightness

Each test was monitored using a plotter that recorded the real time load vs. lateral deflection behavior of the specimen. The load was statically applied in increments such that approximately 10-15 readings could be taken before the expected failure load was reached. A reading was taken approximately five minutes after a load step was completed; since yielding of the web was expected before failure, the pause was used to accommodate the yielding process. The loading was stopped once a peak load was achieved or the lateral deflection became excessive.

6.2 TEST RESULTS

6.2.1 Material Properties

The material properties for the wide-flange bearing sections are given in Table 6.2. The properties were determined from tensile coupons taken from both the flange and the web. Two types of web coupons were cut from the sections, one set parallel to the direction of rolling and one set perpendicular to the direction of rolling. Standard rectangular coupons with an 8 inch gage length and a 1.5 inch width were used for the parallel sets and standard round coupons with a 2 inch gage length and a 0.5 inch diameter were used for the perpendicular sets. The only variations from this program occurred with the perpendicular web coupons for the W12X87 sections, where rectangular coupons were used instead of the round coupons, and with the parallel web coupons for the W12X230 sections, where round coupons were used instead of the rectangular coupons.

The coupons were tested in both a hydraulic test machine and a screw-type test machine. The properties measured during the tests were the yield point, if there was any, the static and dynamic yield strengths, the ultimate strength, and the percent elongation. Most of the coupons exhibited a rounded yield curve, rather than the sharp-kneed bilinear curve. As a result, the yield stresses were obtained using the 0.2% offset method. Three static yield load points were taken during each test. The loading process was interrupted for approximately 5 minutes at each point. Two of these points were used to determine the static yield stress.

The ASTM requirements for the A572 grade 50 steel are shown at the bottom of Table 7.1. The minimum specified yield strength is 50 ksi, the minimum specified ultimate strength is 65 ksi, and the minimum percent elongation is 22%. The specification requirements apply to longitudinal tensile coupons taken from the web; the yield strength applies to the yield point.

Table 6.2 : Bearing Specimen Material Properties

SPECIMEN	ORIENTATION	YIELD POINT	STATIC YIELD	DYNAMIC YIELD	TENSILE STRENGTH	% ELONG	GAGE LENGTH
	(from web)	ksi	ksi	ksi	ksi		in
W12X87	Transverse	-	58.8	61.6	81.8	31.7	2
	Longitudinal	-	53.9	56.3	77.7	24.5	8
	Mill Report	69.5	-	-	87.5	21.5	8
W12X152	Transverse	-	41.8	44.4	71.5	36.0	2
	Longitudinal	-	39.3	42.8	65.8	28.5	8
	Long, Flange	43.3	39.6	42.9	64.9	31.8	8
	Mill Report	58.5	-	-	75.1	27.0	8
W12X230	Transverse	-	47.3	49.0	83.1	22.8	2
	Longitudinal	-	47.9	51.0	84.9	28.2	2
	Long, Flange	-	48.4	51.6	80.5	28.5	8
	Mill Report	58.5	-	-	85.5	23.0	8
A572 GR50		50.0	-	-	65.0	22.0	

Only the W12X87 specimens had a static yield strength above 50 ksi. The W12X230 had a static yield strength 4.2% below the minimum specified and the W12X152 material was 21.4% below the minimum specified. The yield strengths of all three sizes were well below the yield strengths reported on the mill certifications. The extremely low yield strength of the W12X152 material prompted a chemical analysis to determine if the material was indeed Grade 50. The results of the analysis are shown in Table 6.3. The very small percentage of vanadium indicated that the W12X152 section was probably an A36 material, or a material with a yield strength of 36 ksi. The lab chemical analysis does not match the mill report, which suggests that an incorrect mill report was supplied with the steel.

Table 6.3 : Chemical Analysis of Bearing Steels, %

		C	Si	Mn	P	S	V
		Carbon	Silicon	Manganese	Phosphorus	Sulfur	Vanadium
W12X87	MILL	0.210	0.040	1.230	0.012	0.028	0.049
	LAB	0.174	0.050	1.200	0.011	0.020	0.042
W12X152	MILL	0.110	0.344	1.420	0.019	0.017	0.004
	LAB	0.173	0.270	1.060	0.016	0.027	0.001
W12X230	MILL	0.200	0.230	1.280	0.012	0.024	0.063
	LAB	0.209	0.220	1.280	0.011	0.021	0.060
A572 GR 50 maximum		0.23	0.40	1.35	0.04	0.05	0.01-0.15

6.2.2 Web Compression Results

The results of the bearing compression tests are shown in Table 6.4. A typical plot showing the load vs. the lateral centerline web displacement is shown in Figure 6.9. The dynamic failure load is the peak load reached in the test and the static failure load is the maximum static load reached in the test. The critical buckling stress, F_{cr} , is defined as the maximum static load divided by the area. All of the specimens failed in the sway mode and in most cases lateral deflection began immediately upon loading. Graphs showing the non-dimensional axial stress vs.

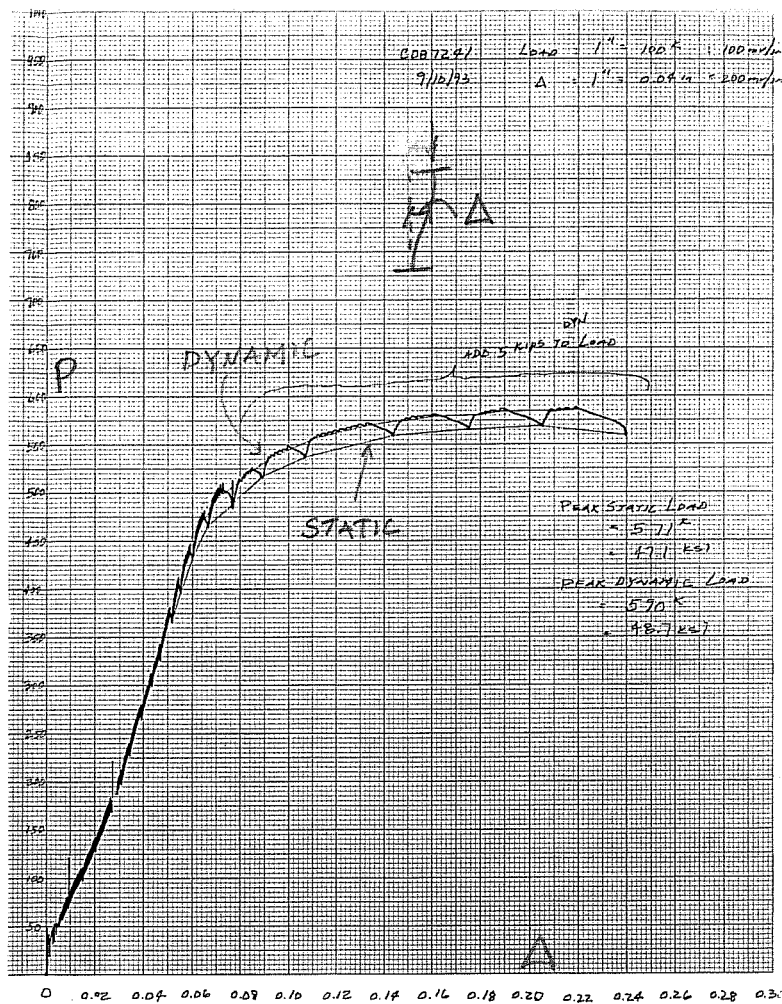


Figure 6.9 Typical Load vs. Lateral Deflection Curve

the top flange lateral deflection are shown in Figures 6.10 to 6.12. The transverse direction yield stress is used. The failure loads of the specimens in each group are repeatable and the width of the specimen, which was varied in the W12X87 group, had no effect on the results. All of the specimens buckled in the inelastic range and strain hardening occurred in the W12X152 and the W12X230 groups.

Table 6.4 : Compression Test Results

TEST ID	STATIC		
	LOAD (kips)	STRESS (ksi)	F_{cr} / F_y
C087361	830	45.1	0.78
C087362	860	46.7	0.79
W12X87 @36", AVG	845	45.9	0.79
C087241	571	46.1	0.78
C087242	578	47.6	0.81
C087243	610	49.6	0.84
W12X87 @24", AVG	586	47.8	0.81
C152361	1575	49.0	1.17
C152362	1450	45.8	1.09
W12X152 @36", AVG	1513	47.4	1.13
C230241	1750	56.1	1.19
C230242	1860	59.6	1.26
W12X230 @24", AVG	1805	57.9	1.23

The process of failure was similar for all of the specimens and the sequence of the yielding was captured by the whitewash. Figure 6.13 shows the general process and extent of the yielding at different points on the load path. The initial yielding began in the lower corner of the south web, at a stress of about $0.4 F_y$ for the W12X87 specimens and at a stress of about $0.8 F_y$ for the W12X230 specimens. Mill scale was not present on the W12X152 sections, so the

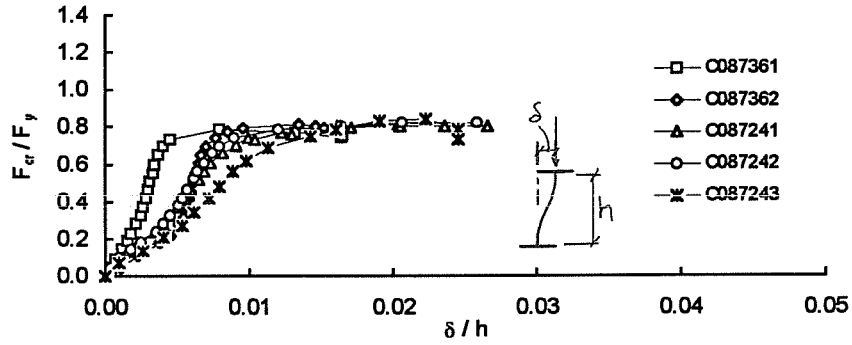


Figure 6.10 Non-Dimensional Axial Stress vs. Lateral Deflection, W12X87

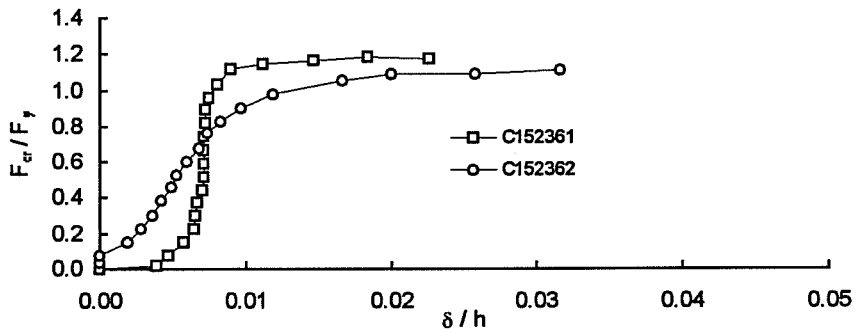


Figure 6.11 Non-Dimensional Axial Stress vs. Lateral Deflection, W12X152

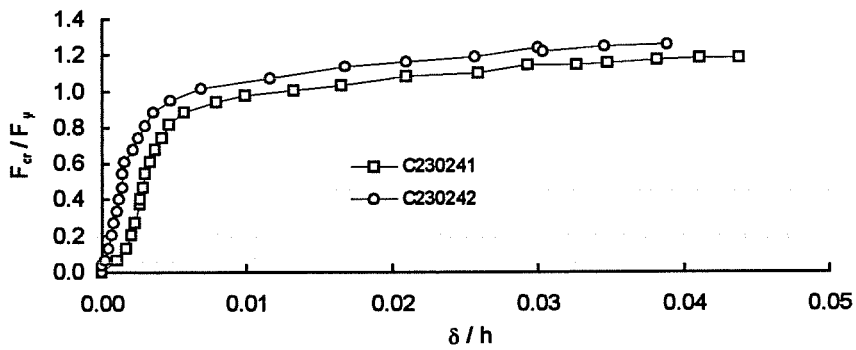


Figure 6.12 Non-Dimensional Axial Stress vs. Lateral Deflection, W12X230

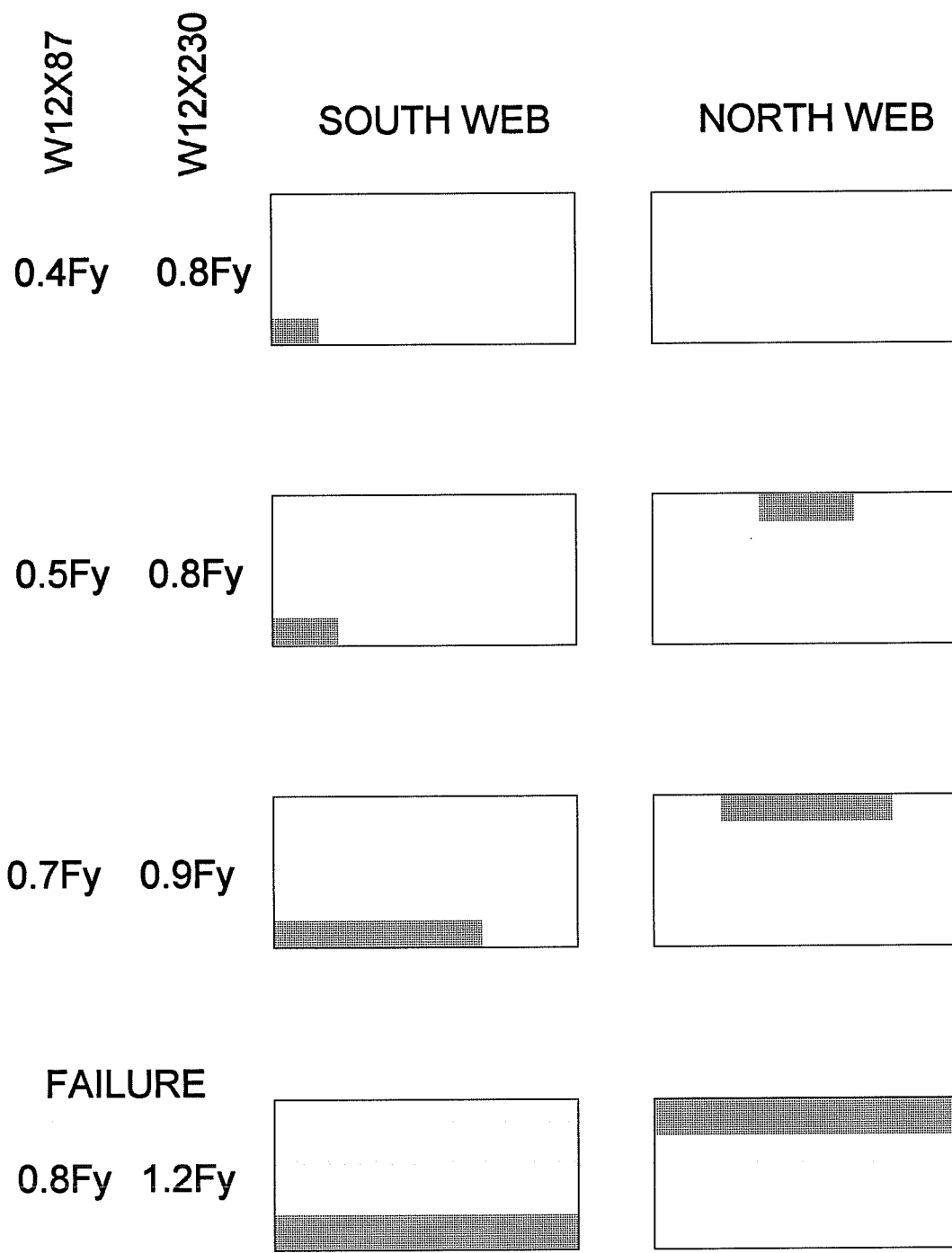


Figure 6.13 Progression of Yield

whitewash did not begin flaking off until the failure load was approached. The next part of the web to yield was the upper north, at about $0.5 F_y$ for the W12X87 specimens and at about $0.8 F_y$ for the W12X230 specimens. The lower south web began to yield along its width at $0.7 F_y$ for the W12X87 specimens and at $0.9 F_y$ for the W12X230 specimens. As the failure load was approached, the yield lines extended along the width of the web on the top north and the bottom south. Continued loading caused the yield lines to penetrate deeper into the web. The final yield patterns for each size are shown in Figure 6.14.

The load distribution along the width of the web was not uniform because the transfer beam was not perfectly rigid. The stress distribution along the width of the web at the center was measured in the C087361 test. Five strain gages spaced at 7.5 inches and 3 inches from the edge measured the stress. The non-dimensionalized stress at different load levels is shown in Figure 6.15. Over the center 15 inches of the bearing, the stress distribution is relatively even and approaches the nominal stress as the load increases. This is the stiffest section of the transfer beam; vertical bearing stiffeners are located at the centerline and at 7.5 inches from the centerline on each side. Outside this width, at 3 inches from each end, the stress drops off and reaches 80% of the nominal stress at the failure stress of $0.78 F_y$. As the load increases, the load redistributes from the center portion of the specimen to the outsider portions. This distribution is for a three foot wide specimen. The distribution is improved for the two foot section since a larger percentage of the width is under the stiffened section of the transfer beam.

6.2.3 Bearing Plate Results

The thickness of the bearing plates used to support a bearing is based on the simplified method of analysis that was described in Section 1.5. To determine the accuracy of the method, the bearing plate used in the compression tests was instrumented with strain gages along the centerline and at one inch from the edge of the bottom flange of the bearing. The centerline stress vs. load for the 36 inch wide bearings is shown in Figure 6.16. The edge stress vs. load for the 36 inch wide bearings is shown in Figure 6.17. The points on the graphs are the averages of the five values along each line.

Superimposed on each graph is the stress calculated using the simplified uniform stress method. The derivation of the stresses is shown in Figure 6.18. The plate is assumed to be loaded by a uniform force per inch of $w = P / 24$ where P is the applied load. At the centerline,

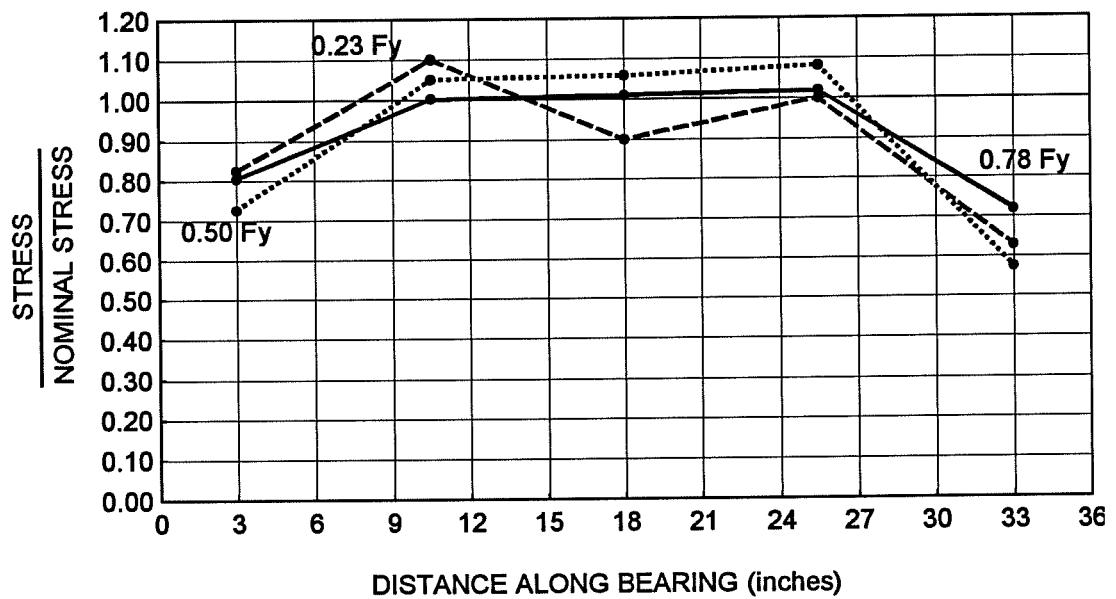


Figure 6.15 Stress Distribution Along Length of Bearing

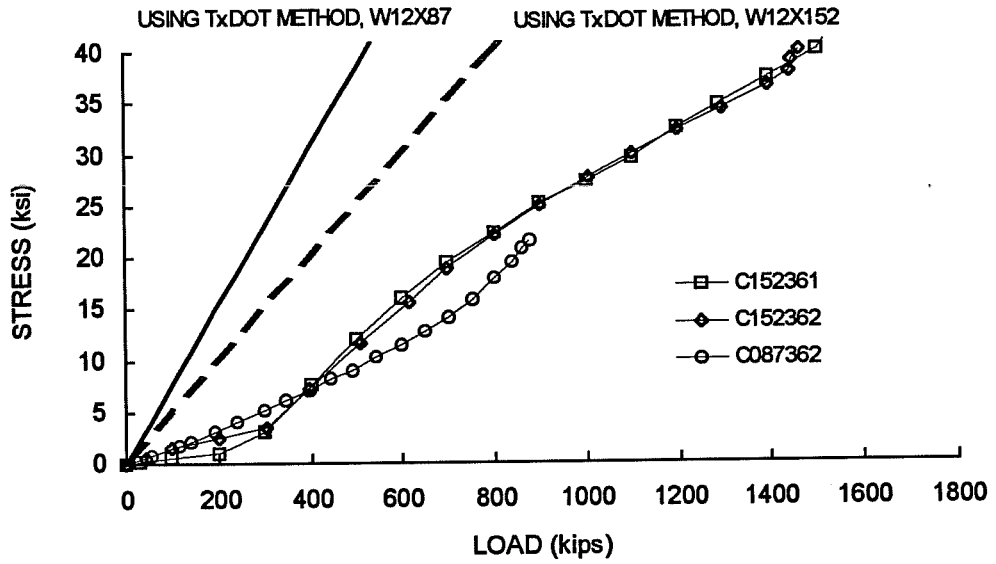


Figure 6.16 Centerline Bearing Plate Stress vs. Load

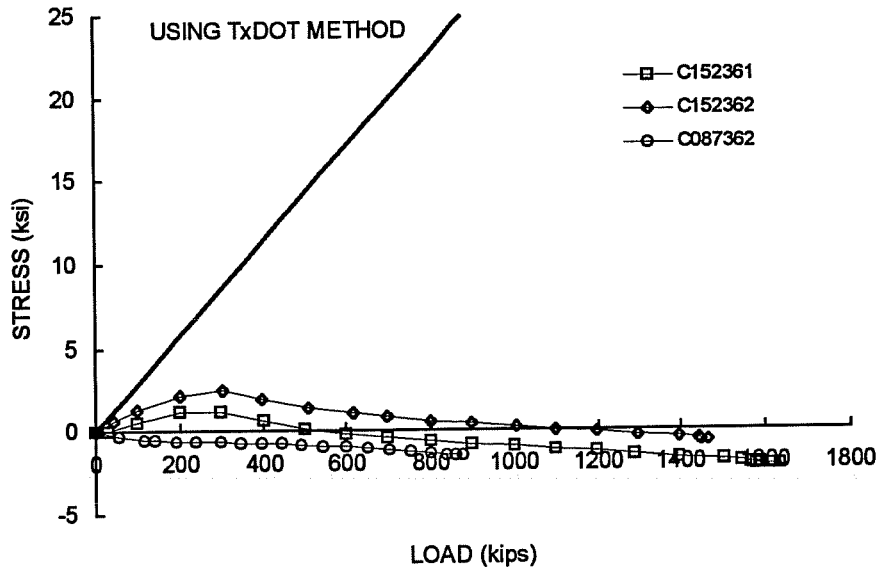
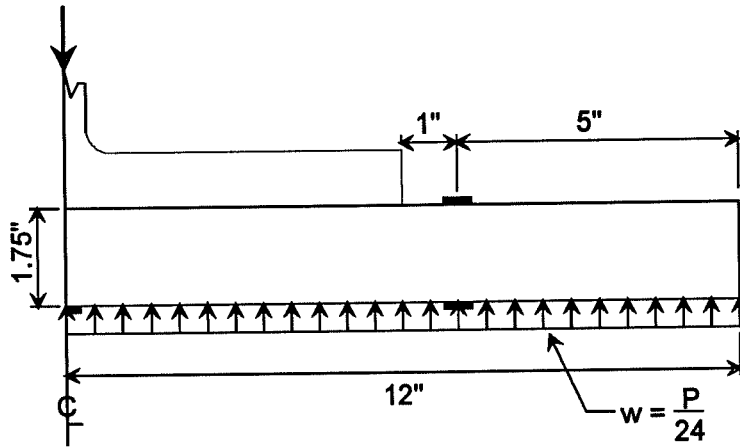
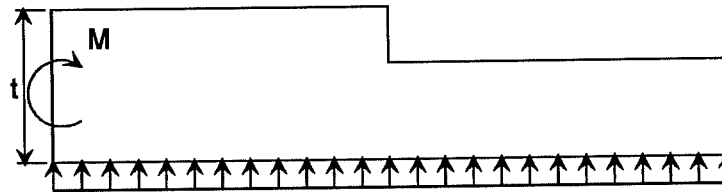


Figure 6.17 Edge Bearing Plate Stress vs. Load



DETERMINATION OF CENTERLINE STRESS



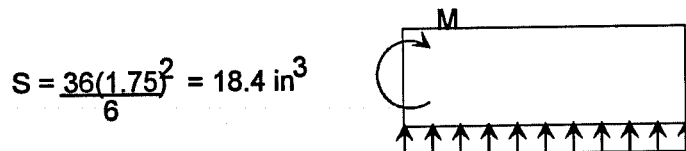
$$t_{W12X87} = 1.75 + 0.81 = 2.56 \text{ in} : S = 39.32 \text{ in}^3$$

$$t_{W12X152} = 1.75 + 1.40 = 3.15 \text{ in} : S = 59.54 \text{ in}^3$$

$$M = \frac{w(12^2)}{2} = 3P : f = \frac{3P}{S} \quad f_{W12X87} = 0.0763P$$

$$f_{W12X152} = 0.0504P$$

DETERMINATION OF EDGE STRESS



$$S = \frac{36(1.75)^2}{6} = 18.4 \text{ in}^3$$

$$M = \frac{w(5)^2}{2} = 0.521P : f = 0.0284P$$

Figure 6.18 Derivation of Bearing Plate Stresses

the thickness of the resisting element is taken as the sum of the thicknesses of the bearing plate and the bottom flange of the bearing. This procedure is not strictly correct in this case since the bearing flange and the bearing plate are not welded together; the calculated stress would be larger. The graphs show that the centerline stress, as determined using the simplified method, is larger than the experimental stress; four times greater for the W12X87 bearing and 1.7 times greater for the W12X152 bearings. The calculated edge stress is absolutely dissociated from the experimental stress values. Not only are the calculated values almost 15 times greater than the experimental values, the signs are reversed. The lack of correlation between the calculated and experimental values results from the incorrect assumption that the bearing stress is uniformly distributed. This condition can be approached only if the bearing plate is very stiff flexurally or the material it is bearing on is very flexible.

6.3 DISCUSSION OF RESULTS

6.3.1 Compressive Strength of Bearing Web

Regions of Behavior

The webs of the bearings can be considered as plates under uniform edge compression, as shown in Figure 6.19. The loaded edges are fixed and the unloaded edges are free. Structural members that fail due to compressive buckling show three regions of behavior; the strain hardening region, the inelastic buckling region, and the elastic buckling region. The primary variable that determines the mode of failure for a compression member is the slenderness ratio, $\frac{KL}{r}$, where KL is the effective length of the member and r is the radius of gyration of the member.

The non-dimensional slenderness ratio can also be expressed as

$$\lambda_c = \frac{KL}{\pi r} \sqrt{\frac{F_y}{E}} \quad (6.1)$$

where λ_c is the slenderness parameter for columns. The slenderness parameter for a plate with the unloaded edges free is

$$\lambda_{pl} = \frac{KL}{\pi r} \sqrt{\frac{F_y (1-\nu^2)}{E}} \quad (6.2)$$

The term $(1-\nu^2)$ accounts for the two-way action of the plate. If the plate consisted of a series of isolated strips, or columns, as shown in Figure 6.20 the buckling load of the plate would be equal to the sum of the buckling loads of the individual strips. A plate however, is a monolithic member and so the behavior of one strip affects the behavior of the adjacent strip. As the strips are loaded axially they expand laterally due to the poisson effect. Each strip is free to expand through the thickness but is restrained from expanding through the width by the adjacent strip. This restraint increases the buckling load of the plate over that of a series of isolated strips.

A graph of non-dimensional axial stress vs. the slenderness parameter for columns and plates is shown in Figure 6.21. The regions are delineated by λ_{st} , the slenderness parameter at which strain hardening begins, and λ_p , the slenderness parameter at which yielding begins. For both columns and plates the non-dimensional elastic buckling stress is

$$\frac{F_{cr}}{F_y} = \frac{1}{\lambda^2} \quad : \lambda \geq \lambda_p \quad (6.3)$$

where F_{cr} is the elastic buckling stress. For the mathematically perfect column or plate, the elastic buckling load would govern behavior until the yield stress was reached. Due to the residual stresses and geometric imperfections that are always present in actual columns and plates, however, an inelastic transition region develops. This region extends from the point at which yielding begins to the point at which strain hardening begins.

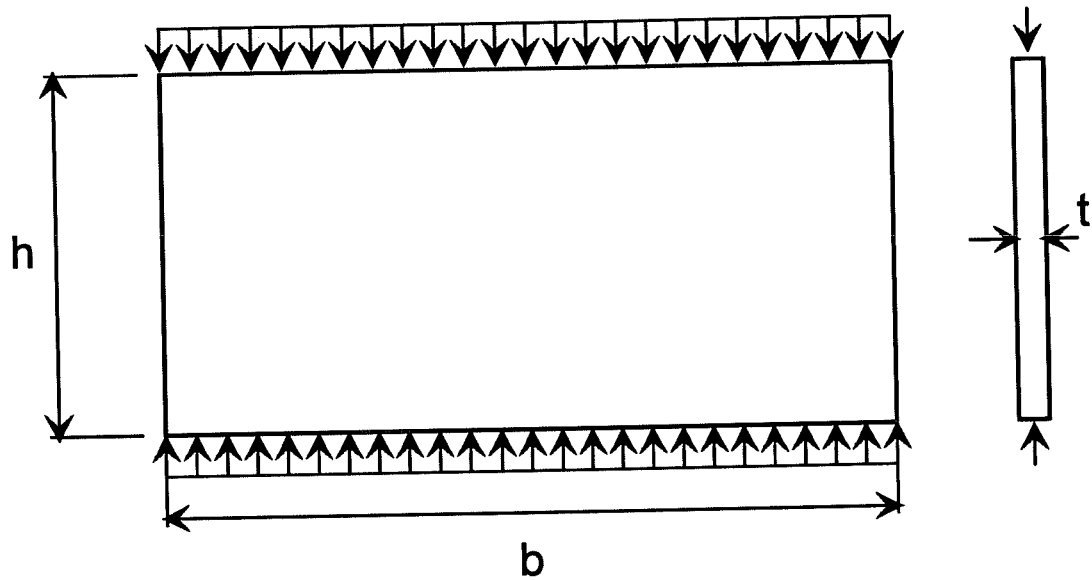


Figure 6.19 Plate Under Uniform Edge Compression

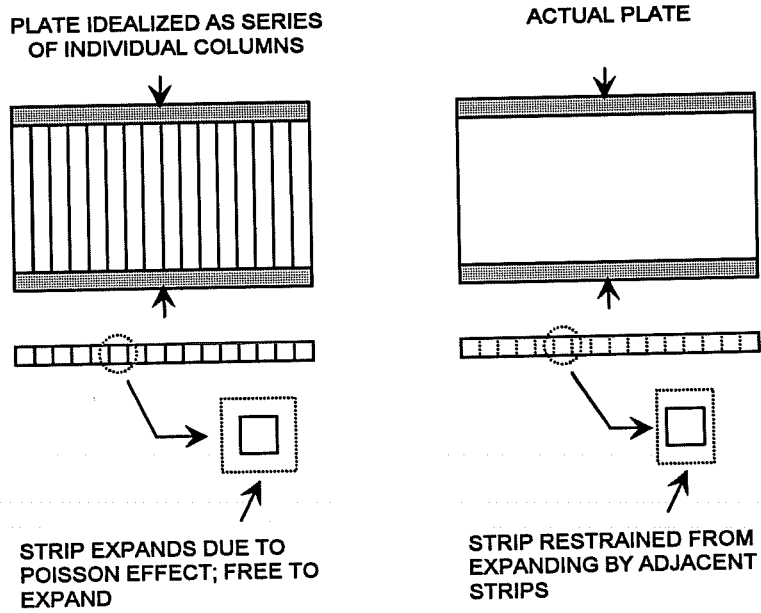


Figure 6.20 Poisson Effect In Plates

An equation commonly used to describe this region [10] is

$$\frac{F_{cr}}{F_y} = 1 - \left(1 - \frac{F_p}{F_y} \right) \left(\frac{\lambda - \lambda_{st}}{\lambda_p - \lambda_{st}} \right)^n \quad : \lambda_p \geq \lambda \geq \lambda_{st} \quad (6.4)$$

where F_p is the proportional limit, or the nominal stress at which yielding begins. The exponent n is

$$n = \frac{2 (\lambda_p - \lambda_{st})}{\lambda_p^2 - 1} \quad (6.5)$$

The behavior of a column or plate in the strain hardening range is not easily quantifiable; it is sufficient to state that $\frac{F_{cr}}{F_y} > 1$ for $\lambda \leq \lambda_{st}$.

Description of Inelastic Region

To use Equation 6.4 it is necessary to define the variables F_p , λ_{st} , and λ_p . For the design of wide - flange columns, the strain hardening region is ignored, so $\lambda_{st} = 0$. The proportional limit, F_p , can be conservatively defined as $F_y / 2$. If F_{cr} in Equation 6.3 is replaced by $F_p = 0.5F_y$, the slenderness parameter, λ_p , can be derived. Performing this substitution gives $\lambda_p = \sqrt{2}$. If $n = 2$ and all of the values are substituted into Equation 6.4, the equation defining the inelastic behavior of a column is

$$\frac{F_{cr}}{F_y} = 1 - 0.25 \lambda^2 \quad : \lambda \leq \sqrt{2} \quad (6.7)$$

which is known as the CRC (Column Research Council) column curve [4] for defining the inelastic behavior of wide - flange column sections. This equation, along with Equation 6.3 defining the elastic buckling behavior, is used by AASHTO for allowable stress design. Each equation is divided by a factor of safety of 2.12.

The inelastic buckling equation for a plate can be similarly derived. The strain hardening region for columns occurs at low slenderness ratios that are unlikely to be realized in practice.

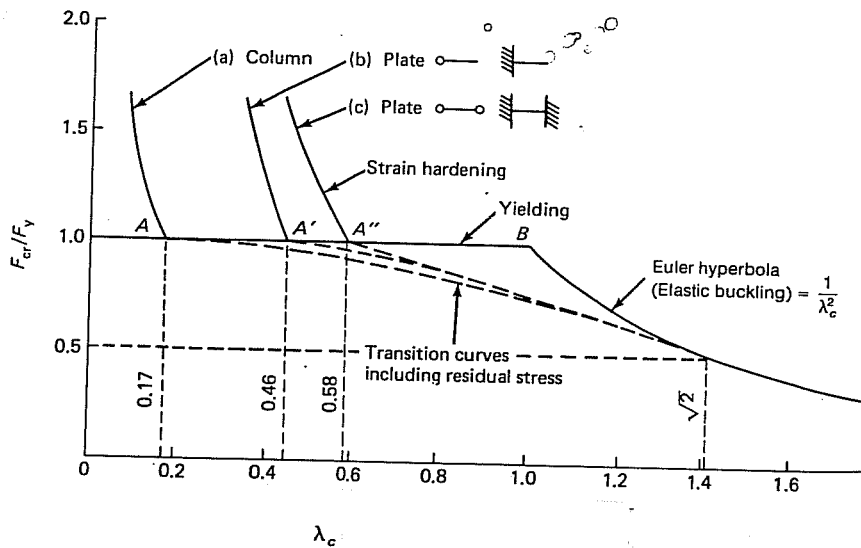


Figure 6.21 Non-Dimensional Axial Stress vs. Slenderness Parameter
 (adapted from Salmon and Johnson [15], p. 364)

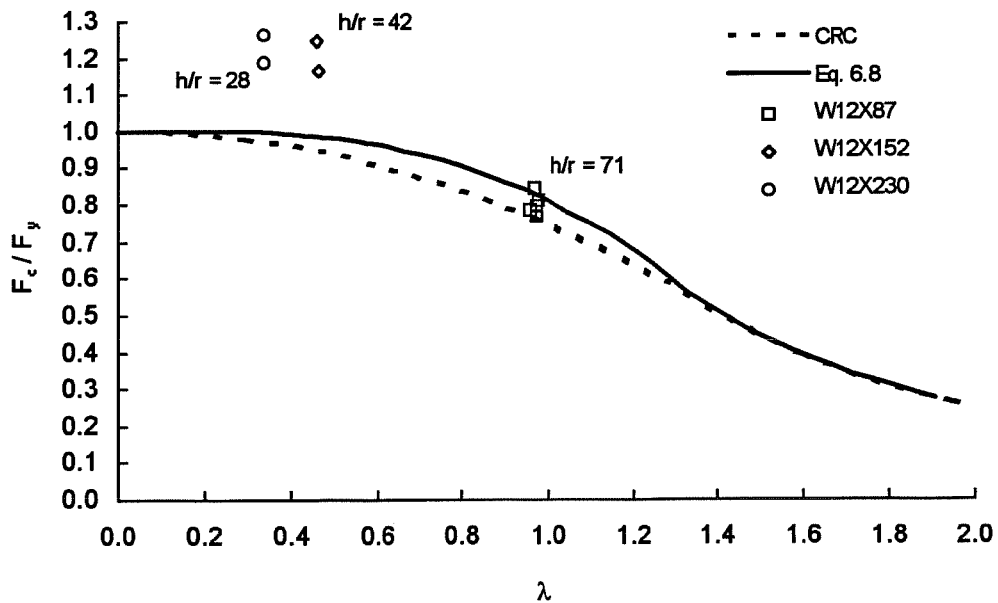


Figure 6.22 Non-Dimensional Axial Stress vs. Slenderness Parameter for Bearing Specimens

The strain hardening region for plates, however, is much larger. The size of the region depends on the boundary conditions of the unloaded edges of the plate. For a plate with one free edge and one supported edge the strain hardening behavior commences at $\lambda_{st} = 0.46$. For a plate with both unloaded edges supported the strain hardening behavior commences at $\lambda_{st} = 0.58$. These values are unconservative for a plate with two free edges, which will behave more like a column. The strain hardening slenderness parameter for columns can be derived by replacing the modulus of elasticity of steel, 29,000 ksi, with the strain hardening modulus, which may be estimated as 900 ksi []. If this substitution is made in the elastic buckling equation (6.3) and F_{cr} / F_y is set equal to one, λ_{st} is 0.17. The strain hardening region for a plate may be conservatively estimated as beginning at $\lambda_{st} = 0.17$. Also to be determined is the proportional limit and its slenderness parameter. The proportional limit was estimated based on the values obtained in the bearing compression tests; they varied from $0.78F_y$ for the W12X230 sections to $0.62F_y$ for the W12X87 sections. For the purpose of deriving the inelastic buckling equation, the proportional limit F_p was taken as $0.6F_y$. The slenderness parameter λ_p was determined by replacing F_{cr} of the elastic buckling equation with F_p and solving for λ . This substitution produced $\lambda_p = 1.29$. The exponent n is calculated by substituting the above values into Equation 6.5. For $\lambda_{st} = 0.17$, $n = 3.37$. The inelastic plate buckling curve can now be derived as

$$\frac{F_{cr}}{F_y} = 1 - 0.27 (\lambda - 0.17)^{3.37} \quad ; \quad 1.29 \geq \lambda \geq 0.17 \quad (6.8)$$

which is plotted along with the CRC column curve and the elastic buckling curve in Figure 6.22. Superimposed on the graph are the results from the bearing compression tests, with the effective length factor K set to 1. The actual effective length of the bearing specimens was difficult to determine due to the sway of the test frame. It must fall between the bounds of $K=0.5$ and $K=1$. Given that all of the bearing specimens experienced a sway failure, the effective length factor is closer to 1 than it is to 0.5. Since there was some restraint to lateral movement, however, the assumption that $K=1$ is unconservative. The inelastic plate buckling curve is unconservative for the specimen with the slenderness ratio of 71 (W12X87) but is conservative for the specimens with the lower slenderness ratios of 28 (W12X230) and 42 (W12X152). To produce a design in which yielding controls rather than buckling it is necessary to limit the slenderness parameter to 0.17,

which corresponds to a slenderness ratio of 13 for Grade 50 steel. Using the factor of safety for columns, 2.12, the allowable stress for the bearing web is

$$f_a = 0.472 F_y : \frac{h}{r} \geq 13 \quad (6.9)$$

Combined Bending and Axial Stresses

The bearing web will not be subject to axial loading only; the horizontal forces due to temperature change and live load rotation will produce bending in the web. The design of members for combined bending and axial loading is governed by an interaction equation. The AASHTO specification requires the check of two interaction equations

$$\frac{f_a}{F_a} + \frac{C_m}{\left(1 - \frac{f_a}{F_e}\right)} \frac{f_b}{F_b} \leq 1 \quad (6.10)$$

and

$$\frac{f_a}{0.472F_y} + \frac{f_b}{F_b} \leq 1 \quad (6.11)$$

where

$$F_e = \frac{\pi^2 E}{2.12 \left(\frac{KL}{r}\right)^2} \quad (6.12)$$

where f_a is the computed axial stress, f_b is the computed compressive bending stress, F_a is the allowable axial stress, F_b is the allowable bending stress, and $C_m / (1 - f_a/F_e)$ is a moment amplification factor. The first equation is appropriate for members that will fail due to instability while the second equation is appropriate for members in which buckling is precluded. It was established in the previous section that the slenderness ratio of the bearing web will be limited to ensure a yielding failure, rather than a buckling failure. Therefore, Equation 6.11 will be used to design the web. To ensure that this equation will always control the design, the amplification factor of the second term in Equation 6.10 must be less than or equal to one. If this factor is one

the second term will reduce to f_b / F_b , which is the second term of Equation 6.11. The coefficient C_m is 0.85 for a member which is loaded axially, has its maximum moments at the ends, and ends which are not prevented from translating. This is the condition of the bearing web. If $f_a = 0.472F_y$ and the effective length factor of the web is set to one the amplification factor becomes a function of the slenderness ratio, which can be solved for as

$$\frac{h}{r} \leq \frac{207}{\sqrt{F_y}} \quad (6.13)$$

and if $F_y = 50$ ksi the maximum slenderness ratio is 29. The maximum slenderness ratio, however, was previously set at 13 so that the failure would occur in the strain hardening range ($F_{cr} \geq 1.0F_y$). At $h/r \leq 29$, the buckling stress as determined by Equation 6.8 is $F_{cr} \geq 0.995F_y$, a difference of less than 0.5%. Since $r = t / \sqrt{12}$ for a rectangular section, the limiting slenderness ratio can be converted to a minimum thickness limit. If the allowable bending stress is set to $0.55F_y$ the interaction equation to be used for the design of the bearing web is

$$\frac{f_a}{0.472F_y} + \frac{f_b}{0.55F_y} \leq 1 \quad : \quad t_w \geq \frac{h \sqrt{F_y}}{60} \quad (\text{inches}) \quad (6.14)$$

6.3.2 Design of Bearing Plates

Sizing the Bearing Plate

The results of the compression tests showed that the uniform stress method for designing a bearing plate is conservative. A bearing plate on a concrete pier is best modelled as a beam on an elastic foundation, as shown in Figure 6.23. The primary variables in the model are the length of the beam, l ; the flexural stiffness of the beam, EI ; and the modulus of reaction, k , which is a function of the modulus of elasticity of the concrete, E_c . The modulus of reaction is defined as the load per unit length required to produce a unit deflection in the foundation.

The bearing stress in the concrete per unit length is

$$q = k y \quad (6.15)$$

where y is the local downward deflection of the concrete. To produce the uniform stress distribution that is assumed in the simplified analysis method, the deflection y must be uniform at all points along the length of the beam. A simple way to check the uniformity of the deflection is to compare the deflection at the center of the beam with the deflection at the end of the beam. If the ratio of the two measurements is one, the deflection and the bearing stress are uniform. For a finite length beam with a point load at the center, the ratio of the end deflection to the center deflection is

$$\frac{y_{end}}{y_{cen}} = \frac{4 \cos (\beta l / 2) \cosh (\beta l / 2)}{2 + \cos \beta l + \cosh \beta l} \quad (6.16)$$

where $\beta l = l \sqrt[4]{\frac{k}{4EI}}$. A plot of y_{end} / y_{cen} vs. βl is shown in Figure 6.24. For $\beta l < 1$, $y_{end} / y_{cen} \approx 1$; the deflection, and therefore the stress, is uniform. For $\beta l > 3$, the deflection at the end of the beam becomes negative; the beam applies a tensile stress to the foundation. Since the beam is needed to transfer compressive stress the portions of the beam transferring tensile stress are ineffective. Between the two extremes the relationship is approximately linear. If βl is set to 1, a maximum length can be calculated such that the stress distribution is uniform. If βl is set to 3, a maximum length can be calculated such that the entire length of the beam is effective. The modulus of reaction of the concrete can be estimated as

$$k = \frac{E_c}{\sqrt{l}} \quad (6.18)$$

The derivation of this equation and all of the remaining equations in the section are shown in Appendix B. For a beam of unit width, the moment of inertia $I = t^3 / 12$ where t is the thickness of the beam.

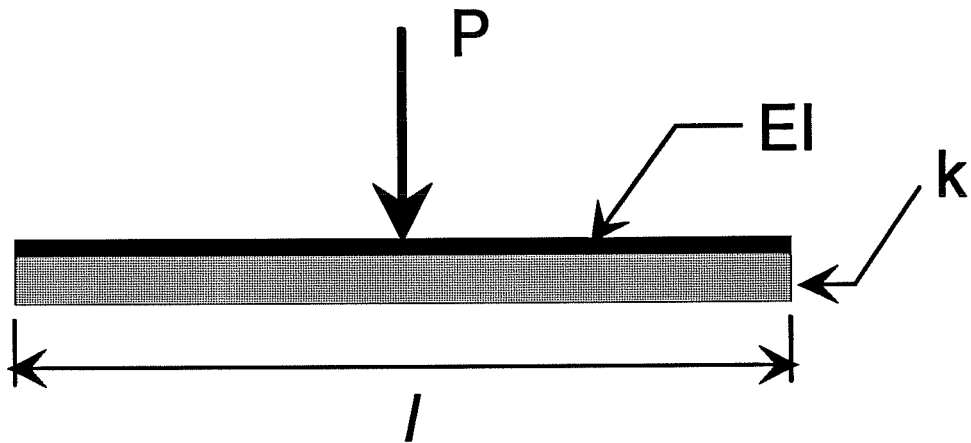


Figure 6.23 Model of Bearing Plate as Beam on Elastic Foundation

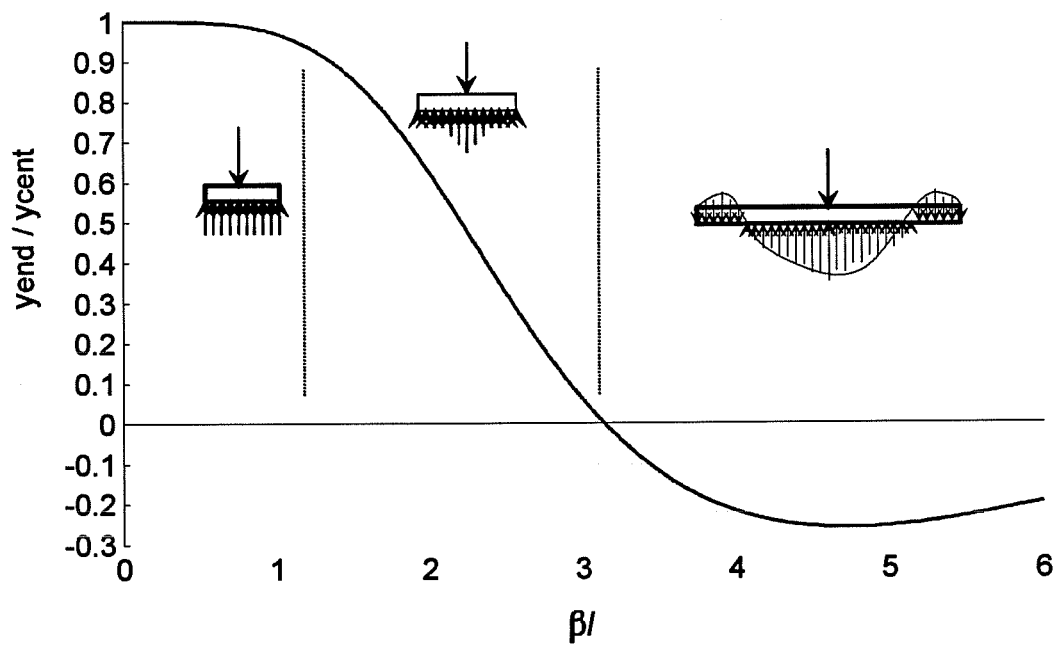


Figure 6.24 Ratio of End Deflection to Center Deflection vs. βl

The maximum length for the two conditions are

$$\begin{aligned}
 l_{stiff} &= 0.73 \sqrt[3.5]{\left(\frac{E_s}{E_c}\right)} t^3 \quad \text{for uniform stress distribution} \\
 l_{eff} &= 2.56 \sqrt[3.5]{\left(\frac{E_s}{E_c}\right)} t^3 \quad \text{for fully effective length}
 \end{aligned}
 \tag{6.19}$$

For example, if the beam thickness is 1.75 inches and the modulus of elasticity of the concrete is 4031 ksi ($F_c = 5000$ psi), which are the values from the Phase II compression tests, the maximum length allowed for the plate to justify the assumption of uniform stress is 2.07 inches. The length of the plate that is effective, l_{eff} , is 7.27 inches. The actual length of the plate was 24 inches. The calculations show that only 30% of the bearing plate was effective. The maximum length allowable for the assumption of uniform stress distribution is too small to be of practical value. It is more reasonable to determine the maximum length for which the plate is effective.

The minimum required thickness for the bearing plate can be determined by relating the moment at the center of the plate to the allowable bending stress. The moment at the centerline is

$$M_{cen} = \frac{P}{4\beta} \frac{\cosh \beta l - \cos \beta l}{\sinh \beta l + \sin \beta l}
 \tag{6.20}$$

At $\beta l = 3$, $M_{cen} = 0.09Pl$; the moment at the centerline when a uniform stress distribution is assumed is $.125Pl$, which is approximately 40% larger. Dividing the moment by the section modulus gives the bending stress. If the bending stress is set to the allowable the minimum thickness can be determined as a function of l .

Substituting the minimum thickness into Equation 6.19, the maximum length of the bearing plate that will be effective can be determined as

$$l_{eff} = 3.25 \sqrt{\frac{E_s}{E_c} \left(\frac{P}{F_b b} \right)^{0.75}} \quad (6.21)$$

and the minimum thickness is

$$t_{min} = \sqrt{\frac{0.54 P l_{eff}}{F_b b}} \quad (6.22)$$

where F_b is the allowable bending stress and b is the width of the plate.

Concrete Bearing Stress

The current TxDOT design procedure sizes the bearing plate such that the maximum stress on the concrete does not exceed $0.3f_c \sqrt{A_2 / A_1} \leq 0.6f_c$ where A_2 is the area of concrete supporting the bearing and A_1 is the area of the bearing plate. Two tests were conducted to determine the bearing strength of a concrete pier when loaded by a wide-flange bearing section. The test specimens are shown in Figure 6.25. In the first test the maximum nominal bearing stress in the concrete reached $1.36f_c$ with only a small hairline crack as the only sign of distress. The loading had to be stopped because the wide-flange section was yielding. For the second test the length of the bearing flange was cut in half, which results in more stress on the concrete for the same load. Since some of the length of the bearing flange in the first test was probably ineffective, the actual stress on the concrete would not change that much from the first test. In the second test the concrete bearing stress reached $2.85f_c$ with minor cracks around the perimeter of the bearing flange just beginning to form. It is recommended that rather than using the allowable stress given above to size the bearing plate, an allowable stress of $1.0f_c$ be used to check the concrete bearing stress under the area $l_{eff} * b$.

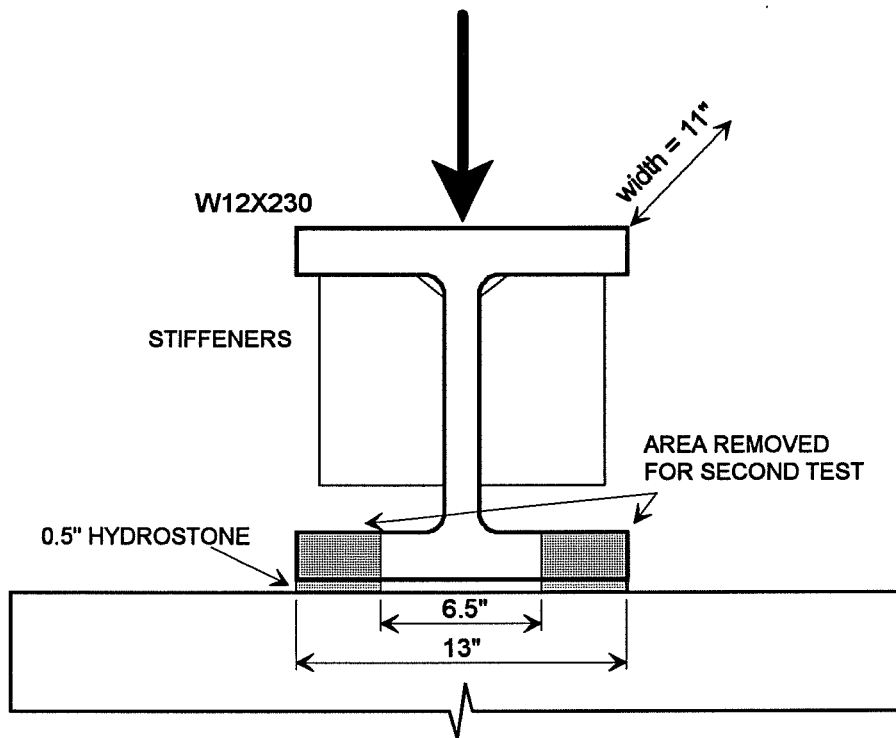


Figure 6.25 Bearing Strength Test Specimen

CHAPTER 7 : PHASE II FATIGUE TESTS

7.1 TEST PROGRAM

7.1.1 Test Variables

The bridge bearings will be subject to out-of-plane distortion due to horizontal movements and rotations. The primary horizontal movement is due to contraction and expansion caused by temperature changes. This movement, however, is not considered likely to cause fatigue damage. Fatigue damage is caused by the movements and forces produced by the cyclic live loading of truck traffic. At the pier locations, truck loading produces rotation about the transverse axis of the cap girder; the center of rotation is assumed to act at midheight of the longitudinal stringer (Section 4.2.3). Since the bearing location is not coincident with this point, a horizontal displacement, as well as a rotation, will be produced at the bearing location. This behavior is shown in Figure 1.9. The displacement that occurs at the bearing is equal to the product of the rotation and the distance from the measurement point to the center of rotation.

The maximum stress in the bearing occurs at the intersection of the web and the flange, as shown in Figure 7.1. Rolled structural shapes have an approximate parabolic fillet transition from the web to the flange. Stress risers will be present in this transitional area, the magnitude of which will depend on the geometry of the transition. A sharper transition will produce a larger stress riser. Fatigue cracks may develop at these stress riser locations. The edges of the fillet are defined by two dimensions, $k-t_f$ and $k_1-t_w/2$. There is not much variation in these dimensions for the W12 and W14 sections, as given in the AISC Manual. For the W12 sections, the mean value of $k-t_f$ is .71 inches with a standard deviation of 0.02 inches and the mean value of $k_1-t_w/2$ is 0.63 inches with a standard deviation of 0.02 inches. For the W14 sections, the mean value of $k-t_f$ is 0.67 inches with a standard deviation of 0.02 inches and the mean value of $k_1-t_w/2$ is 0.63 with a standard deviation of 0.02 inches. The fatigue specimens comprised the same three sizes that were tested in the compression tests, W12X87, W12X152, and W12X230. The properties of the specimens are listed in Appendix C.

The objective of the fatigue tests was to determine if a wide-flange section, when subject to cyclic out-of-plane shear distortion, would exhibit a fatigue life consistent with a category A detail. The category A detail encompasses details described as "Plain material, Base metal with

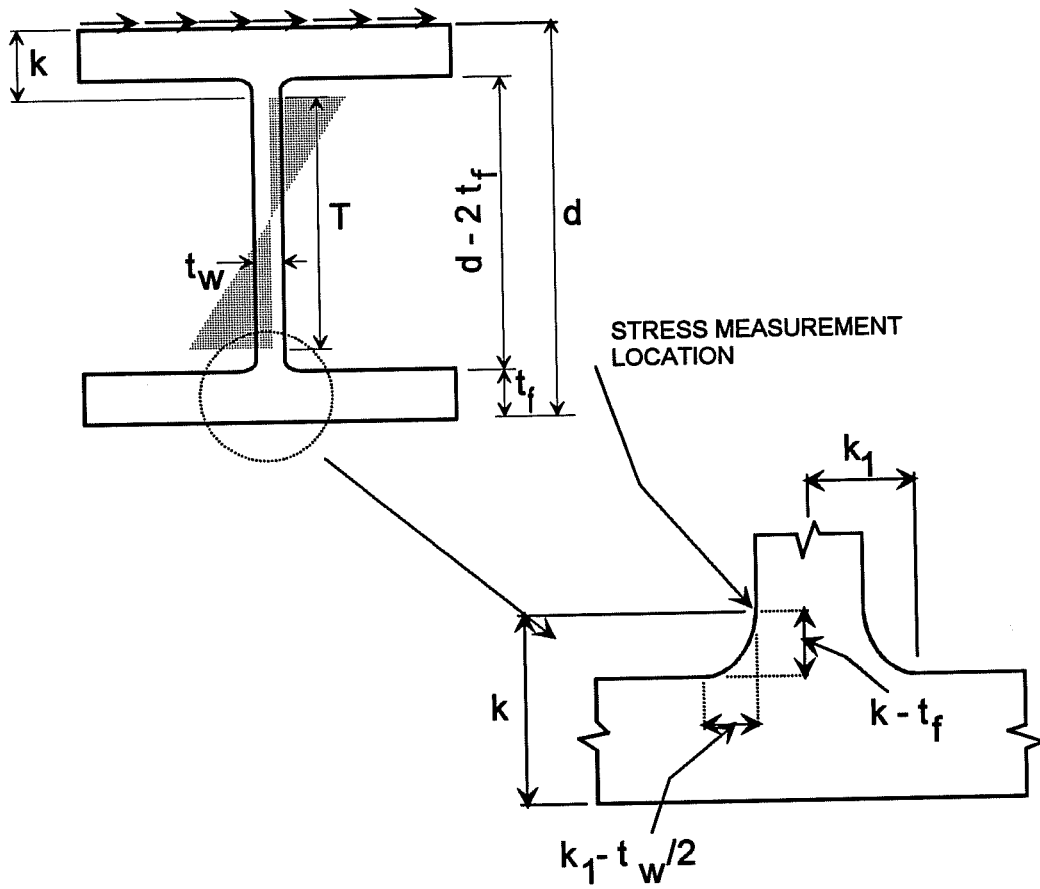


Figure 7.1 Location of Maximum Stress

rolled or cleaned surfaces" [2]. The allowable stress range is 24 ksi for an infinite life rating. Since a category A fatigue detail rarely controls the design of a bridge detail, a classification of the wide-flange section as such would simplify the design procedure by eliminating the requirement of checking for fatigue. Given that the allowable stress range for a category A detail for infinite life is 24 ksi, the fatigue specimens were tested at stress ranges above this value, 30 ksi, 40 ksi, and 50 ksi.

7.1.2 Test Setup

The rotation of the longitudinal stringers at the pier supports causes a horizontal displacement of the top flange of the bearing. This is in addition to the rotation. If the bottom flange of the bearing is assumed to be fixed, a conservative assumption, the displacement and rotation at the top of the bearing will produce a moment $M = \frac{EI}{h} \left(4\theta + \frac{6\Delta}{h} \right)$ where $\Delta = y\theta$. The second term in the parentheses, the horizontal displacement term, becomes $6y\theta/h$, which will always be greater than the rotation term, 4θ . The larger the value of y , the more dominant the displacement term becomes. For example, if $y = 60$ inches and $h = 12$ inches, the ratio of the displacement term to the rotation term becomes 7.5, meaning that the horizontal displacement component causes a stress in the bearing 7.5 times greater than the stress caused by the rotation. Taking this stress differential into account, a test setup was designed that applied a horizontal displacement to the specimens. A schematic of the test frame is shown in Figure 7.2 and a photograph of the frame as constructed in the lab is shown in Figure 7.3.

The individual specimen was twelve inches wide. In most cases, two specimens were tested simultaneously, located symmetrically above and below the load application point. The top flange of the upper specimen and the bottom flange of the lower specimen were bolted to the test frame and were restrained from rotation and displacement. The bottom flange of the upper specimen and the top flange of the lower specimen were bolted together, but separated by a one inch thick loading plate. The loading plate was attached to a hydraulic ram that applied a constant amplitude sinusoidal load. The two specimens were free to translate, but not to rotate, at the load application point. Some tests were also conducted with only one specimen in the lower position. These tests were necessary to produce a 40 ksi stress range in the W12X230 specimens and in the single specimen tests a rotation as well as a horizontal displacement was produced.

7.1.3 Test Schedule

The schedule for the fatigue tests is shown in Table 7.1. Each test is separated by a blank row. Ten tests were conducted; two specimens (four fillets) were loaded in six of the tests and one specimen (one fillet) was loaded in four of the tests. Each fillet is identified by an alphanumeric code, similar to the code used for the compression tests: the first letter, an F, indicates that it is a fatigue test; the next three digits indicate the weight per foot in pounds of the W12 section; the next two digits give the location of the fillet in the test specimen; the next digit is a sequence identifier for the section size; the final letter indicates whether two specimens were tested in the symmetric placement setup, D, or if the specimen was tested by itself, S. The code is described below:

- X _ _ _ _ _ : F, fatigue test
- _ X X X _ _ _ : weight per foot of bearing
- _ _ _ _ X X _ : location of fillet subject to tensile stress range
 - : U, upper specimen; T, top fillet
 - : L, lower specimen; B, bottom fillet
- _ _ _ _ _ X _ : section size test sequence identifier
- _ _ _ _ _ X : D, symmetric test setup
 - : S, single specimen tested

For example, test F087UT1D indicates a fatigue test with a W12X87 section, the fillet is in the upper specimen at the top, this is the first test with a W12X87 bearing, and the fillet is part of the symmetric test setup. The table shows three nominal stress ranges, 30 ksi, 40 ksi, and 50 ksi. An X under one of the headings indicates that the fillet was tested at that stress range.

Table 7.1 : Bearing Specimen Fatigue Test Schedule and Properties

FILLET	NOMINAL STRESS RANGE		
	30 ksi	40 ksi	50 ksi
F087UT1D	X	X	X
F087UB1D	X	X	X
F087LT1D	X	X	X
F087LB1D	X	X	X
F087UT2D		X	X
F087UB2D		X	X
F087LT2D		X	X
F087LB2D		X	X
F152UT1D	X		
F152UB1D	X		
F152LT1D	X		
F152LB1D	X		
F152UB1S	X		
F152UT2D		X	
F152UB2D		X	
F152LT2D		X	
F152LB2D		X	
F152UT3D		X	
F152UB3D		X	
F152LT3D		X	
F152LB3D		X	
F230UT1D	X		
F230UB1D	X		
F230LT1D	X		
F230LB1D	X		
F230UB1S		X	
F230LB1S		X	
F230LB2S		X	

7.1.4 Test Procedure

Strain gages were used to measure the stress ranges at the anticipated peak tensile stress locations, as shown in Figure 7.4. This location was the point of tangency of the web and the fillet, or the dimension k from the bottom flange. The dimension k was the nominal value given in the AISC Manual. A comparison of the nominal value of k and the measured value of k for each specimen is given in Appendix C. An LVDT was used to measure the displacement of the loading plate.

Prior to each test, the cross-section profile was traced out and the web thickness was measured. The specimens were then placed in position. Before any of the bolts that secured the specimens to the test frame were tightened, the strain gages were zeroed. This was done so that any stresses induced in the specimens due to the tightening procedure could be recorded. Once the specimens were secured, static load was applied to record the load-stress and load-displacement relationships. On completion of one static cycle, the cyclic load test began.

The constant amplitude, sinusoidal cyclic load was applied by means of a closed-loop hydraulic system. A load controller was used to define the mean load and the load range. The load was correlated to the stress range in the static load test. The values were initially set to the values predicted by the static load tests, and then adjusted until the stress range, as recorded by the strain gages, was obtained. The load range was then checked to see that it produced the theoretical stress range. The minimum load was set to produce a minimum stress of approximately +4 ksi for the fillets in tension. The load frequency varied from 3.0 Hz to 4.5 Hz and the number of cycles was recorded by a counter. Limits were placed on the load so that the system would turn off if a limit was exceeded. A fatigue crack typically activated the limit mechanism. If no fatigue crack occurred at 2 million cycles, the specimens were loaded to a 4 million cycle limit. If no fatigue crack occurred at the 4 million cycle limit, the stress range was increased and loading continued at this higher stress range until a failure occurred.

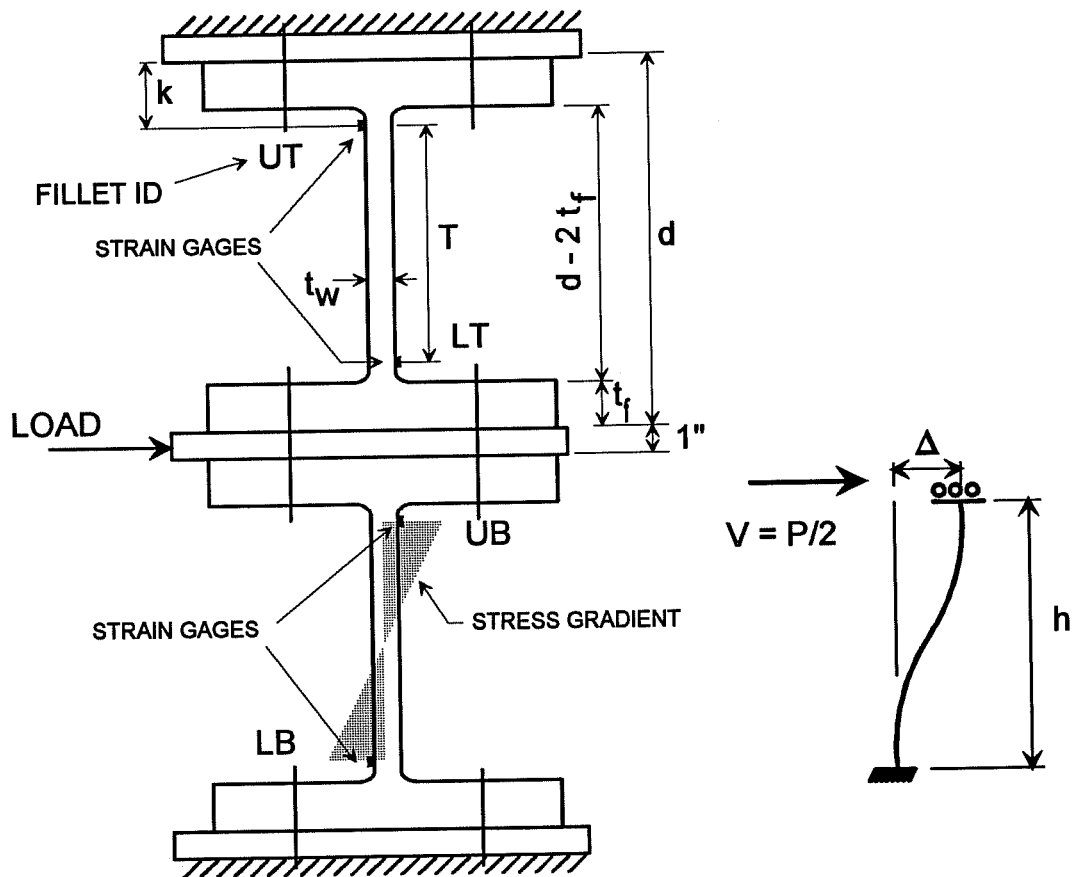


Figure 7.4 Schematic of Fatigue Test Specimen

7.2 TEST RESULTS

7.2.1 Flexural Stiffness of Specimens

Prior to each fatigue test, the flexural stiffness of the specimen was determined. The flexural stiffness was defined as the lateral deflection of the loading plate divided by the shear load. For the symmetric test setup, the shear load was defined as one-half of the applied load. A schematic of the system is shown in Figure 7.4. Each specimen was modelled as a beam that was fixed at one end and fixed but free to translate at the opposite end. For these end conditions, the flexural stiffness is

$$k_{op} = \frac{V}{\Delta} = \frac{E b t_w^3}{h^3} \quad (7.1)$$

where b is the width of the specimen, t_w is the average thickness of the web, and h is the depth of the web. The average shear vs. deflection curves for the three specimen sizes are shown in Figure 7.5. The height of the web can be considered as bounded between the height $T = d - 2k$ and the height $d - 2t_f$ where d is the depth of the specimen. If the experimental stiffnesses are substituted into Equation 7.1, the effective depth of the web, h_{eff} , can be solved for. A comparison of the effective height and the bounds given above is shown in Table 7.2.

Table 7.2 : Effective Depth of the Web

SIZE	$h = T$ (measured)	$h = d - 2t_f$ (measured)	$h = h_{eff}$ Eq. 7.1	% DIFF
	(inches)	(inches)	(inches)	$h_{eff}, d - 2t_f$
W12X87	9.61	10.89	10.12	7.06
W12X152	9.42	11.00	11.06	-5.6
W12X230	9.63	10.82	11.99	-10.81

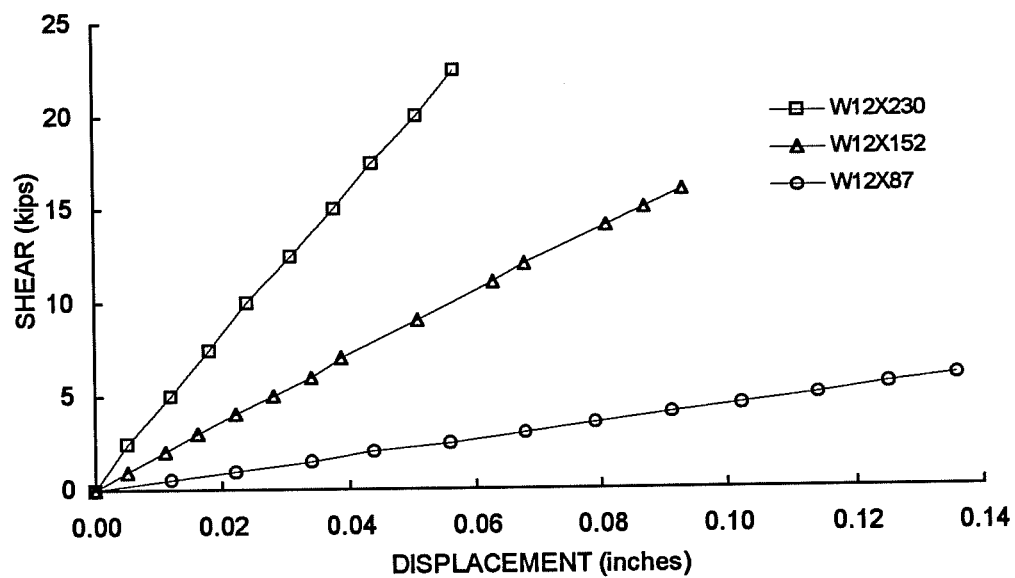


Figure 7.5 Shear vs. Displacement for Symmetric Test Setup

The measured values of T compare well with the nominal value of T given in the AISC Manual, which is 9.5 inches. The effective depth of the web is closely approximated by the depth $d-2t_f$. The percent difference between these two depths is shown in the last column of the table. The W12X87 specimens were stiffer than calculated, the W12X152 specimens matched almost exactly, and the W12X230 specimens were more flexible than calculated. The difference between the actual stiffnesses and the calculated stiffnesses can be attributed to two primary causes:

- 1.) **Increased web thickness at fillets:** This would tend to increase the flexural stiffness of the web because the fillet is ignored in the calculation.
- 2.) **Rotational flexibility of supports:** It is assumed in the calculation that the supports are restrained from rotation. The web is actually supported by an elastic support (the flange) and the test frame cannot provide perfect rigidity.

The difference in stiffness for the W12X87 specimens is probably due to cause 1.) and the difference in stiffness for the W12X230 specimen is due primarily to cause 2.).

7.2.2 Comparison of Calculated and Experimental Stresses

Strain gages were placed on the specimens at the intersection of the web and the fillet to determine if the experimental dynamic stress readings were comparable to the calculated stresses. The calculated stresses were determined as

$$f_{calc} = \frac{M}{S} = \frac{6 V (T / 2)}{b t_w^2} \quad (7.2)$$

where T is the nominal value given in the AISC Manual. The comparison of the calculated and experimental stress readings for all of the fillets that were gaged are shown in Figures 7.6 to 7.8. The mean ratio and the bounds of the ratio for each size are shown in Table 7.3. The maximum difference between the calculated and experimental stress readings is 27% and the mean difference varies between 1% and 11%. The primary error is due to the strain averaging property

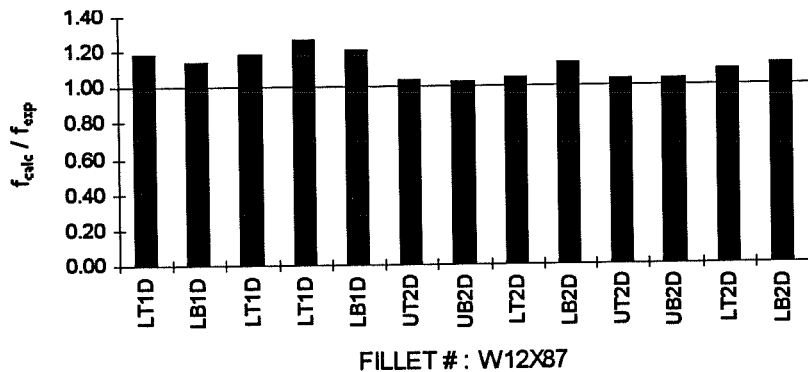


Figure 7.6 Ratio of Calculated to Experimental Stress at Fillet, W12X87

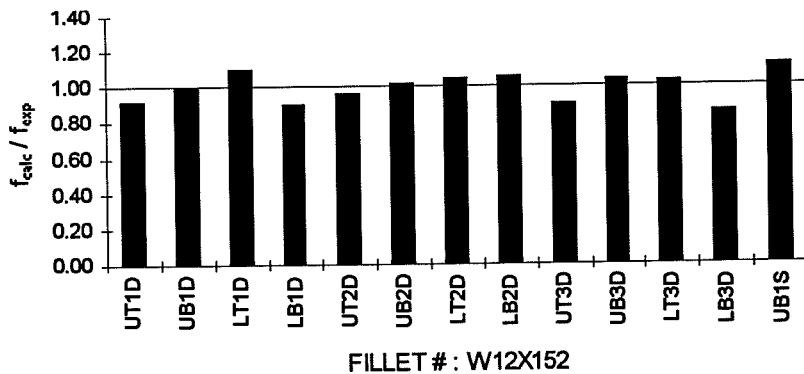


Figure 7.7 Ratio of Calculated to Experimental Stress at Fillet, W12X152

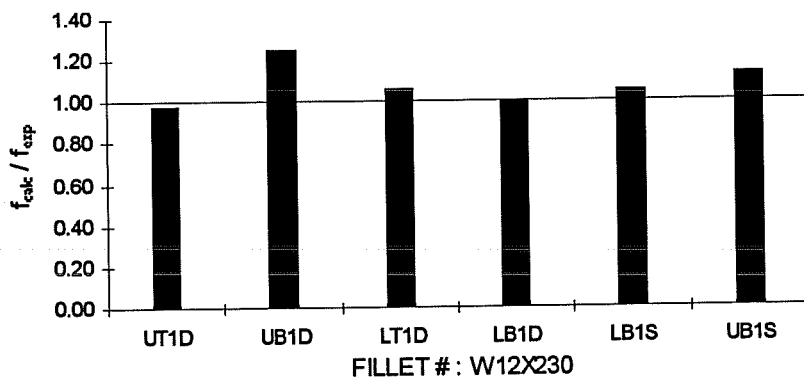


Figure 7.8 Ratio of Calculated to Experimental Stress at Fillet, W12X230

of a strain gage. A strain gage measures strain over a finite length and the output of the gage is the strain measured over this length. It is not possible to measure the strain at one point, nor is it possible to pick up a peak stress. The smaller the length of the strain gage, the more accurate the measurement will be. The length of the strain gages was 0.125 inches.

Table 7.3 : Ratio of Calculated to Experimental Stress at Fillet

	W12X87	W12X152	W12X230
MEAN RATIO	1.11	0.99	1.07
MAX RATIO	1.27	1.11	1.25
MIN RATIO	1.03	0.86	0.98

7.2.3 Fatigue Life

The results of the fatigue tests are shown in Table 7.4. The top row shows the nominal stress range in kips per square inch. The calculated stress range, S_r , and the number of cycles to failure, N , are given for each fillet. Each group of fillets that was tested simultaneously is separated by a dotted line. For each group of fillets, either one or more fillets failed or no fillets failed. Those that failed are shaded. A Y superscript indicates that the maximum stress in the stress range exceeded the yield strength of the material. The results are shown graphically in Figure 7.9 as a log-log plot of the stress range vs. the number of cycles to failure. One data point represents the results from each group of fillets. The data points for the fillets that did not fail are accompanied by an arrow. Superimposed on the graph is the equation for the category A detail. In all cases, the experimental data points lie above the category A boundary. The data points for the tests in which only one specimen was tested are indicated by an ^s. The results do not seem to be influenced by the test configuration.

In all cases, the crack initiated at some point within the center two-thirds of the web. There was usually more than one crack initiation site and typically more than one crack plane. Schematics of the crack planes and crack surfaces of some of the failed specimens are shown in Figure 7.10. The cracks usually occurred just above or just below the anticipated cracking plane (at distance k from the outer surface of the flange) and the crack propagated as an ellipse, as shown by the shaded areas. In some cases the crack was visually observed before the loading

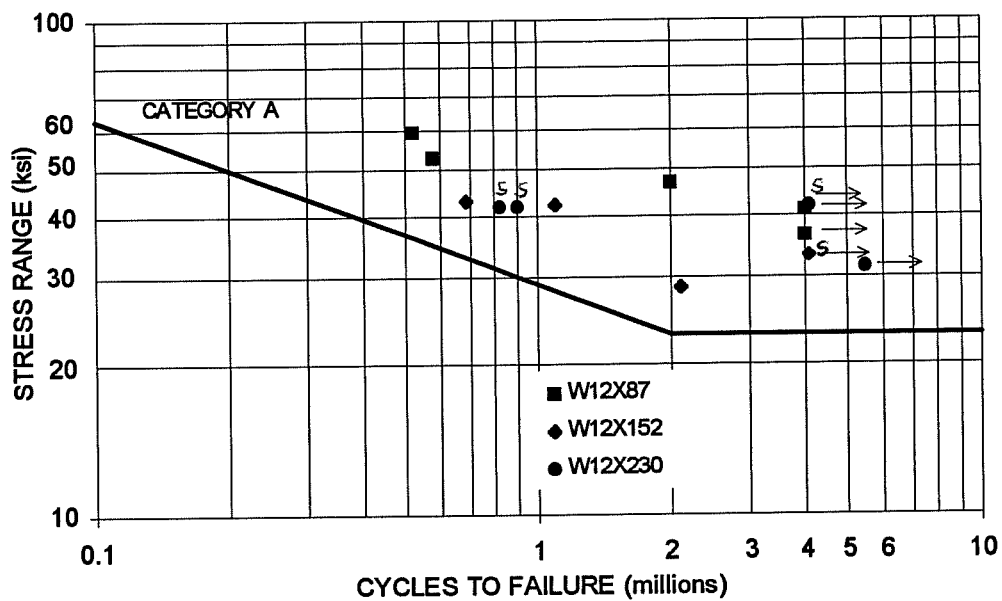


Figure 7.9 Stress Range vs. Number of Cycles to Failure

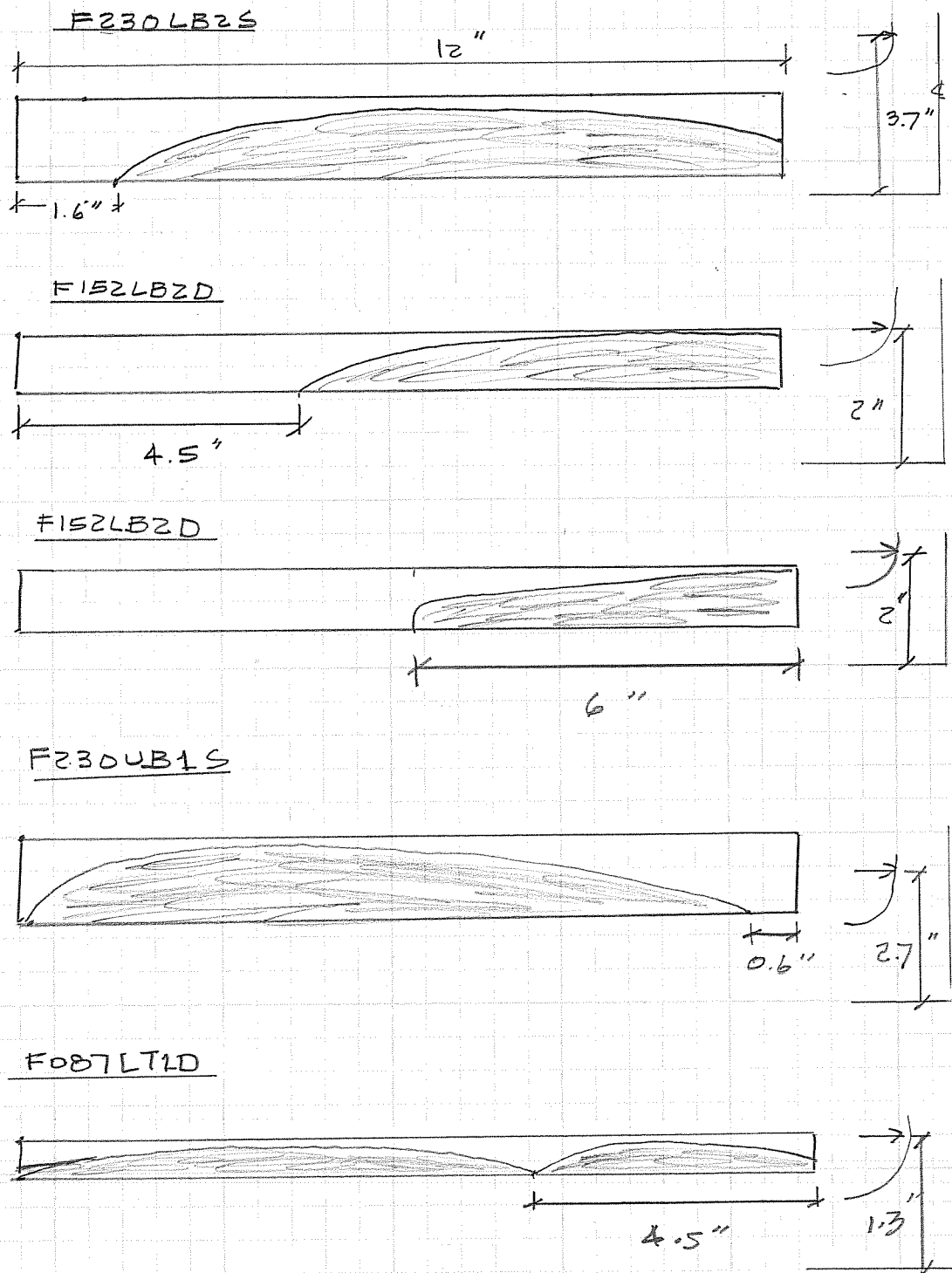


Figure 7.10 Schematics of Crack Surfaces

system shut down. The specimens typically went through a few hundred thousand cycles from the time the crack was observed to the time it propagated through most of the thickness of the web or to the edge of the specimen. The fatigue life of a cracked specimen was taken either as the cycle count at the time the crack was visually observed or the last observed cycle count before the system shut down (this was usually the case if the specimen failed during the night). There is not much apparent difference in the performances of the different sections, though the W12X87 sections had longer fatigue lives for higher stress ranges than the other two sections. This may be because the size of the fillet, which has dimensions $(k_1-t_w/2, k-t_f)$ that are larger than the web thickness, provides a smoother transition from the web to the flange and therefore has a smaller stress concentration. There was, however, no apparent stress concentration produced by the fillet transition, as shown by a thermoelastic stress analysis of the fillet in Figure F.6 in Appendix F. It should also be noted that all of the W12X87 and W12X152 specimens that failed had a peak stress above the yield stress. Photos of some of the cracked specimens are shown in Figure 7.11.

7.2.4 Notched Specimens

The fillet provides a rather smooth transition from the web to the flange. To determine the effect on the fatigue life of a sharp transition, the fillets of two W12X87 specimens were machined out, as shown in Figure 7.12. This produced a sharp right angle transition from the web to the flange. The specimens were placed in the symmetric test setup and three stress ranges were tested. The specimens were first loaded to produce a calculated stress range of 38 ksi at the intersection of the web and flange. A crack formed at the intersection of the web and flange at the upper transition in the lower specimen (tension side) at 113,000 cycles. The crack extended almost the full width (12 inches) of the specimen. This cracked web and flange intersection was then reinforced with a fillet weld (no cracking ever occurred at the weld locations) on both sides of the web and the specimen loaded to produce a stress range of 10 ksi. No crack formed at any web-flange intersection at 4 million cycles. The stress range was increased to 20 ksi and a crack subsequently formed at the intersection of the web and flange at the bottom of the lower specimen, at 2.1 million cycles. The results of the test are plotted in Figure 7.13; also shown are the results from the previous tests and superimposed on the graph is the curve defining the category C detail, which specifies an allowable stress range of 10 ksi for infinite life. The data from the notched specimen tests fall close to and above the curve for the category C detail.

Table 7.4 : Fatigue Test Results

NOMINAL S_r	30		40		50	
ID	S_r ksi	N *1M	S_r ksi	N *1M	S_r ksi	N *1M
F087UT1D	37	4	47	2	59	0.52
F087UB1D	37	4	47	2	59	0.52
F087LT1D	37	4	47	2	59 ^Y	0.52
F087LB1D	37	4	47	2	59	0.52
F087UT2D			41	4	53	0.58
F087UB2D			41	4	52	0.58
F087LT2D			41	4	52	0.58
F087LB2D			41	4	52	0.58
F152UT1D	29	2.8				
F152UB1D	29	2.8				
F152LT1D	29	2.8				
F152LB1D	29 ^Y	2.1				
F152UB1S	33	4.1				
F152UT2D			42 ^Y	1.3		
F152UB2D			43	1.3		
F152LT2D			42	1.3		
F152LB2D			42 ^Y	1.1		
F152UT3D			44	0.69		
F152UB3D			43	0.69		
F152LT3D			43	0.69		
F152LB3D			43 ^Y	0.69		
F230UT1D	31	5.49				
F230UB1D	32	5.49				
F230LT1D	32	5.49				
F230LB1D	32	5.49				
F230UB1S			42	0.9		
F230LB1S			42	4.11		
F230LB2S			42	0.82		

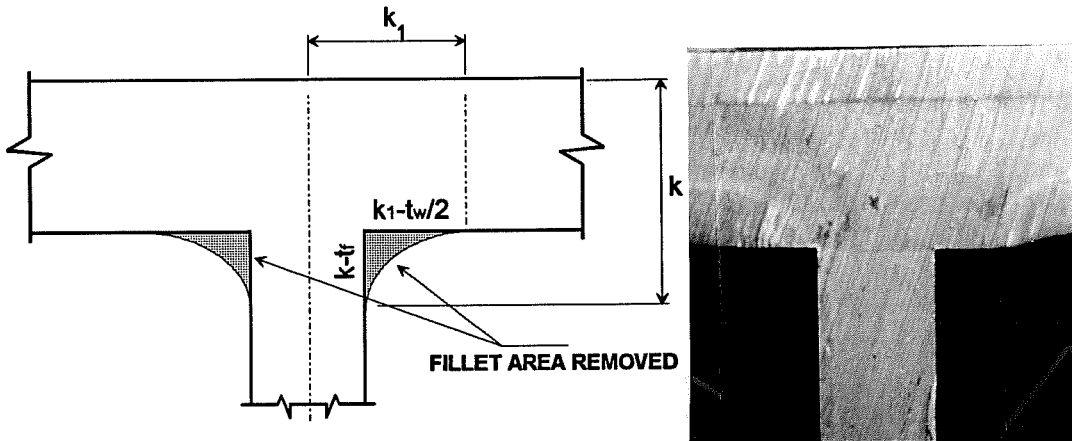


Figure 7.12 Notched Specimen Detail

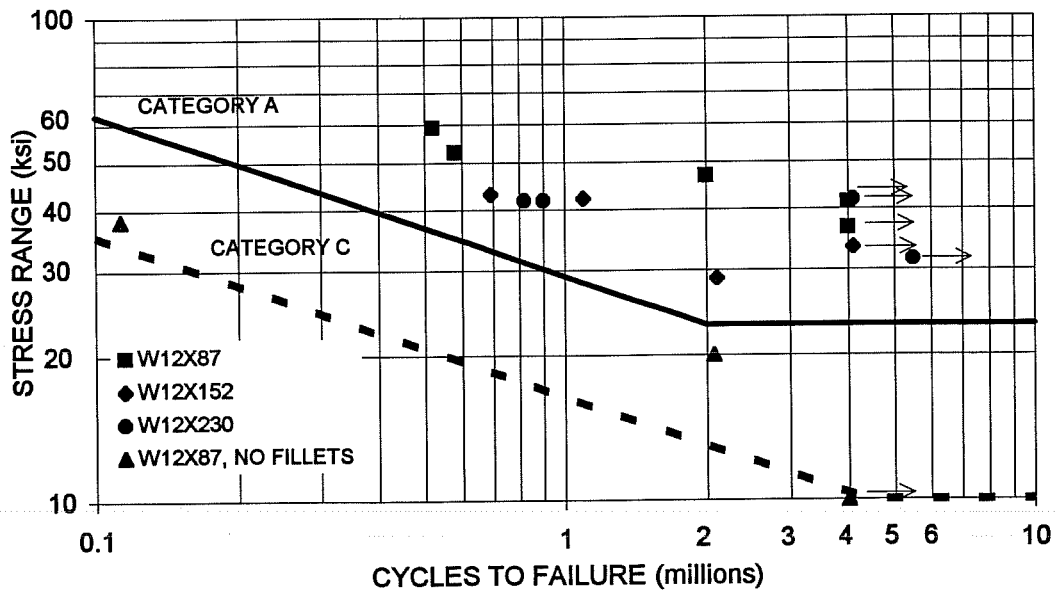


Figure 7.13 S - N Plot Including Notched Specimens

Table 8.1 : Full Scale Tests Schedule

TEST	TYPE	BEARING	PT/bolt (kips)	BRACE DISTANCE (inches)
R0870040	Rotation	W12X87	40	-
T08700	Transverse	W12X87	-	-
T08780	Transverse	W12X87	80	-
R1520000	Rotation	W12X152	-	-
R1520001	Rotation	W12X152	1	-
R1523400	Rotation	W12X152	-	34
R1523401	Rotation	W12X152	1	34
R1520080	Rotation	W12X152	80	-
R1523480	Rotation	W12X152	80	34
T15200	Transverse	W12X152	-	-
T15201	Transverse	W12X152	1	-
T15280	Transverse	W12X152	80	-

8.1.2 Test Setup

A schematic of the test setup for the transverse direction tests is shown in Figure 8.1. and a photo is shown in Figure 8.2. A schematic of the longitudinal rotation test setup is shown in Figure 8.3 and a photo is shown in Figure 8.4. The details of the connection are shown in Figure 8.5. There are three major changes in the configuration of the Phase II setup as compared with the Phase I setup. The first change is that the end load for the transverse direction tests is applied downward rather than upward as in Phase I. This models more correctly the load situation that occurs in practice. This first change necessitated the second change, the design of the pier column and foundation. The load levels required in Phase II were much larger than

those in Phase I, so the foundation was designed to engage two sets of test floor anchorages instead of one. The uplift force that could be applied to the system was 500 kips, which required a lengthening of the column section of the pier from one foot to two feet so the vertical column steel could be developed. The third major change was the orientation of the truss system that simulated the longitudinal girder. The truss was turned upside down so that the load could be applied upward. This was done to simplify the loading fixture and to use the reaction wall as a bracing element.

Two types of longitudinal direction tests were conducted. In the first set of tests, all of the lateral bracing was removed; the center of rotation was at the bottom flange of the bearing. In the second set of tests, a lateral brace was placed 34 inches above the bottom flange of the cap girder; this simulated the center of rotation for the longitudinal girder rotating about its neutral axis.

8.1.3 Instrumentation

The instrumentation used in the tests comprised strain gages, linear potentiometers, and servo and digital inclinometers. The loads were measured using both load cells and a pressure transducer. Whitewash was used on the W12X87 bearing specimens.

The strain gages were located on the cap girder, the bearings, the anchor bolts, and the hold-down bolts. The gages on the cap girder are shown in Figure 8.6. The purpose of these gages, which were T-gages, was to determine if the cap girder web was significantly stressed from the web flexure caused by longitudinal rotation. The gages on the bearings, shown in Figure 8.7, were used to determine the axial and bending stresses in the bearings. The gages on the anchor bolts were used to determine the forces in the anchor bolts. Two gages were placed opposite one another on the flat portion of the threadbar. Prior to use, one threadbar was calibrated to correlate strain and load. Subsequent testing proved the load obtained from the strain readings to be within a few tenths of a percent of the actual load.

The linear potentiometers were used to measure the bottom flange vertical and horizontal deflection of the cap girder. For the transverse loading tests, the pots were used to measure vertical deflection at the same position as the pots used in Phase I. Only one pot per location, however, was used in Phase II. Linear potentiometers were used to measure horizontal deflection of the bottom flange of the cap girder for the longitudinal rotation tests, as shown in Figure 8.8.

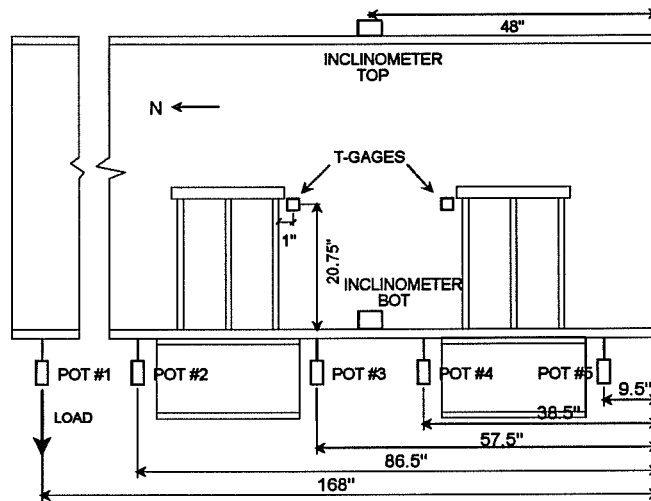


Figure 8.6 Location of Instrumentation

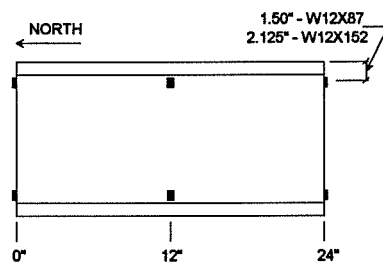


Figure 8.7 Location of Strain Gages on Bearings

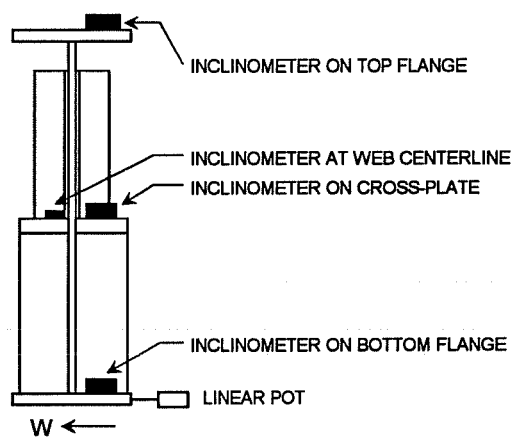


Figure 8.8 Location of Inclinometers

The inclinometers were used to measure the transverse and longitudinal rotations of the cap girder. The servo inclinometers were placed on the top and bottom flanges of the cap girder, centered between the bearing connections. A digital inclinometer was placed at the center of the clip angle that connected the truss to the cap girder web and also on the horizontal cross plates of the two connections.

8.1.4 Test Procedure

Plumbing the Cap Girder

For the x087xxx tests, the bearings and the anchor bolts were erected prior to the cap girder. For the x152xxx tests, the bearings were bolted to the cap girder before erection and the anchor bolts were placed after erection. An overhead crane was used to position the cap girder on the pier. Screw jacks were placed between the connections and near the load application point to support the girder for the plumbing operation. The cap girder was leveled horizontally along the axis by adjusting the height of the screw jacks and was plumbed vertically by selective tightening of the anchor bolts (x087xxx tests) or the hold-down bolts (x152xxx tests). A gap of approximately 3/4" was left for the grouting material. Dry pack grout was used to level the x087xxx cap girder and hydrostone was used to level the x152xxx cap girder.

Bracing the Cap Girder

The cap girder was braced to a reaction wall by four steel braces. The braces were pinned at both ends and located at the top and bottom flanges of the cap girder both at the load application point and at the north connection. The bracing scheme is shown in Figure 8.9. The braces were used for the transverse direction tests but were disconnected for the longitudinal rotation tests. A brace was added for the x152xxx tests; its purpose is explained below.

Tightening the Hold -Down Bolts

The hold-down bolts are the bolts that secure the bearing plates and the bottom flange of the bearing to the pier cap. These bolts were tightened using the turn-of-the-nut-method, snug plus 1/4 turn. All of the bolts were tightened to the snug condition before the final tightening took place.

Tensioning the Anchor Bolts

The anchor bolts were tensioned using a hydraulic ram. A schematic of the tensioning assembly is shown in Figure 8.10. The anchor bolt extended past the nut approximately four inches. A tensioning coupler was threaded onto this extension and onto a pull rod. The pull rod extended through a hole in the top flange of the cap girder and through a center-hole ram. A 100 kip load cell was placed on top of the ram and a nut and anchor plate were the reacting mechanism of the pull rod. Before the tensioning began, all of the anchor bolts were tightened, by hand, to the snug condition. Each of the anchor bolts was tightened to the suggested working load of $0.66 f_{pu} A_{ps}$ [7], which in this case was 80 kips per bolt. The bolts in each bolt group were first tightened to 40 kips and then to the 80 kip working load. The tensioning process for each bolt comprised three stages: 1) tension the bolt to its target value plus fifteen percent; 2) tighten the nut; 3) release the load. Data were recorded at each stage. Overtensioning in stage one countered the seating loss from the stage two process. The cap girder was braced during the tensioning.

Transverse Direction Tests

The load was applied in increments such that 10 to 15 data readings could be taken before nonlinear behavior began. Each load step lasted approximately 30 seconds. Once the desired load level was reached, a needle valve in the hydraulic line was closed to maintain the pressure in the ram. After five minutes, the data scan was made. For the tests in which the anchor bolts were pretensioned, the specimen was taken to failure; the test was stopped when deflection at the load point increased with no increase in load. For the tests in which the anchor bolts were not pretensioned, the test was stopped before yielding of the bearing specimens began.

Longitudinal Direction Tests

The load was applied in increments such that 5 to 10 data readings could be taken before the strain gages on the cap girder web indicated yielding. In most cases, two cycles of loading were applied.

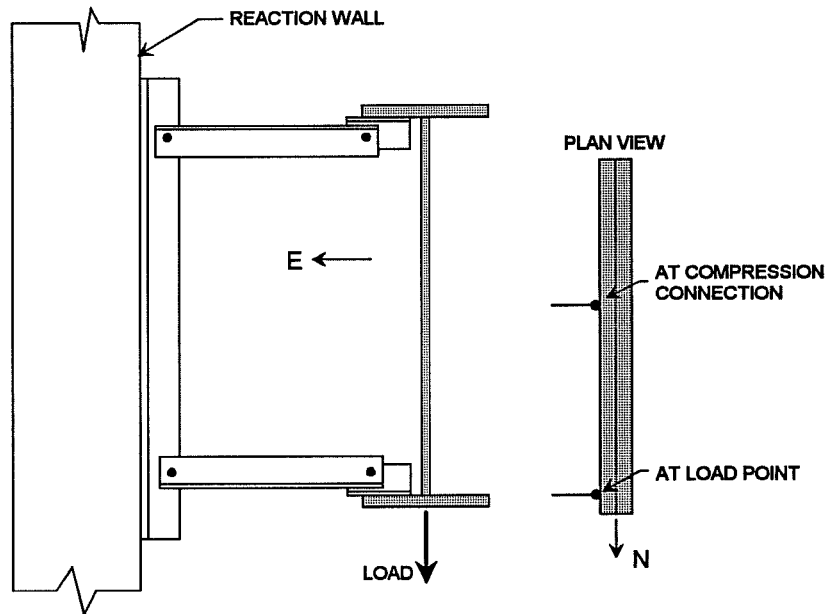


Figure 8.9 Bracing Scheme

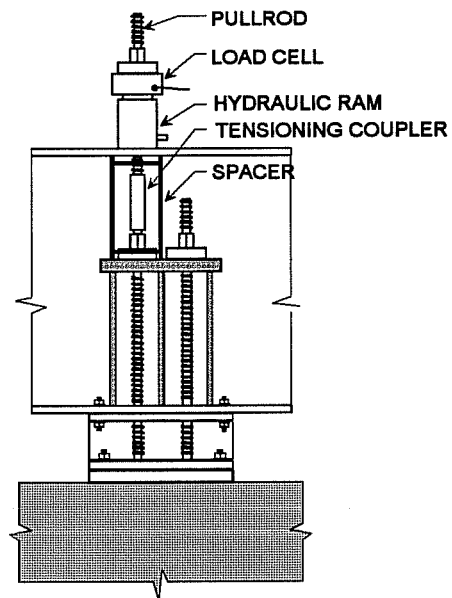


Figure 8.10 Tensioning Assembly

8.2 TEST RESULTS

8.2.1 Pretensioning Forces

The anchor bolts were tensioned using two different tightening sequences. The sequences are shown in Table 8.2.

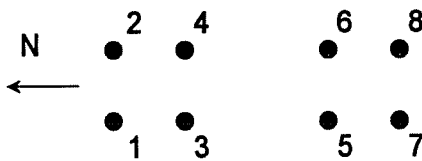


Table 8.2 : Anchor Bolt Tightening Sequences

ORDER	W12X87 BEARINGS		W12X152 BEARINGS	
	BOLT #	FORCE	BOLT #	FORCE
1	1	40	3	40
2	3	40	4	40
3	4	40	1	40
4	2	40	2	40
5	5	40	8	80
6	7	40	7	80
7	8	40	5	80
8	6	40	6	80
9	1	40	2	40
10	1	80	1	40
11	3	80	3	40
12	4	80	4	40
13	2	80	6	80
14	5	80	5	80
15	7	80	7	80
16	8	80	8	80
17	6	80	-	-
18	1	80	-	-

The tightening sequence used for the W12X87 bearings required a retensioning of the first anchor bolt after all eight bolts were tightened. The first bolt tensioned lost approximately 10% of its initial pretension force. The final pretension force in each bolt is shown in Figure 8.11 for the W12X87 bearings and in Figure 8.12 for the W12X152 bearings. The average pretension force per bolt was 80 kips for both sets of bearings, the target value. Each bolt was initially tensioned to a force greater than the target value to counter the loss that would occur due to the lock-off procedure. The percentage loss due to lock-off for each bolt is shown in Table 8.3. The average loss per bolt was approximately 12%.

Table 8.3 : Percent Loss in Pretension Force due to Lock-off

BOLT #	% LOSS	
	W12X87	W12X152
1	12.7	14.5
2	12.5	13.6
3	13.3	10.2
4	11.2	10.4
5	10.2	12.7
6	13.2	10.7
7	12.3	13.2
8	11.1	10.2
AVG	12.1	11.9

8.2.2 Longitudinal Direction Tests

Moment vs. Rotation Behavior

The primary purpose of the longitudinal rotation tests was to determine the stiffness of the connection as it relates to the rotation caused by vehicle live load. The moment vs. rotation curves of the different configurations of the connection with the W12X152 bearings are shown in

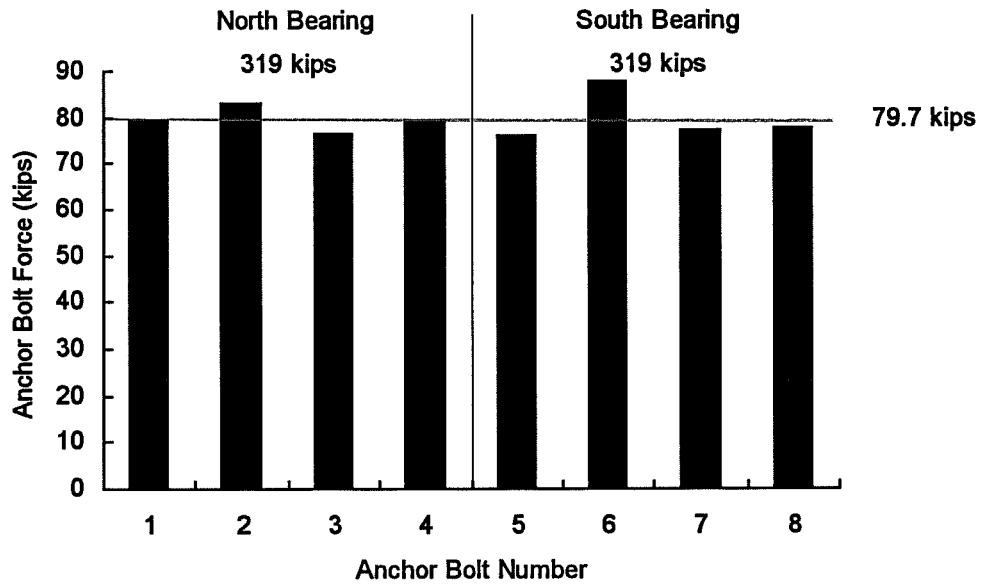


Figure 8.11 Pretension Force per Bolt, W12X87 Bearings

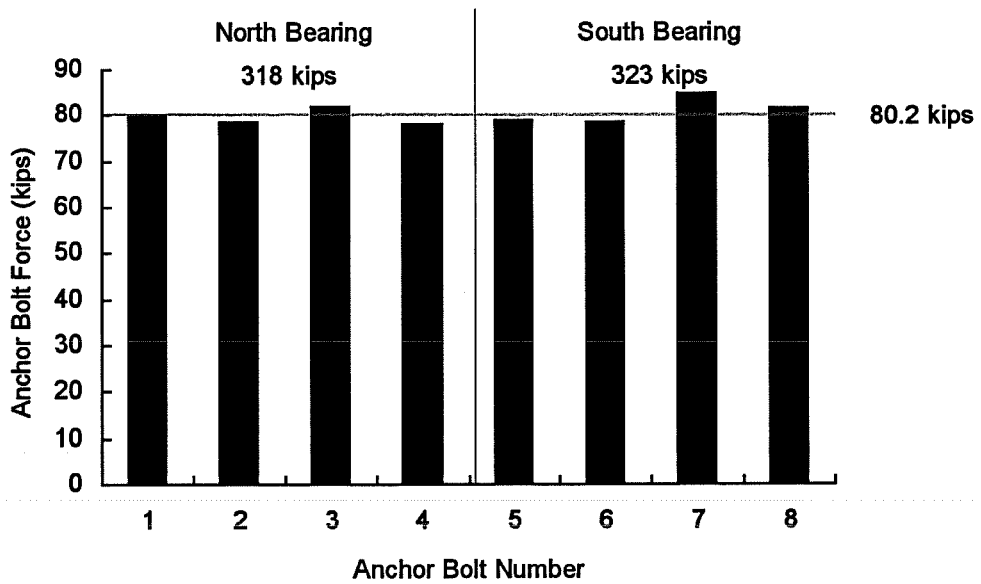


Figure 8.12 Pretension Force per Bolt, W12X152 Bearings

Figures 8.13 to 8.18. The results are not shown from the one rotation test conducted on the connection with the W12X87 bearings; its behavior was qualitatively similar to that of the connections with W12X152 bearings. Figures 8.13 to 8.15 show the results for the tests in which the lateral brace was not present, referred to as Center of Rotation (CoR) = 0". The results for the tests in which the lateral brace was present, referred to as CoR = 34", are shown in Figures 8.16 to 8.18. For each group, three sets of results are shown; the first graph in each group shows the moment vs. rotation curves for the connection in which the anchor bolts were not engaged, identified as (0) *anchor bolts* (the term *engaged* will refer to anchor bolts that resist the rotation of the cap girder); the second graph in each group shows the moment vs. rotation curves for the connection in which the anchor bolts have been tightened to a snug condition, identified as (1) *anchor bolts*; the third graph in each group shows the moment vs. rotation curves for the connection in which the anchor bolts are pretensioned, identified as (2) *anchor bolts*. In each graph, the moment vs. rotation behavior at four locations on the structure is shown. These locations are the top flange, the bottom flange, at the web centerline, and at the north connection cross plate. The applied moment is the product of the end load and its distance to the cap girder web, 96 inches. The rotation is given in milliradians.

The plots show that the rotation is a linear function of the applied moment. The stiffness of each connection is given by the slope of the moment vs. rotation curve. For the connection in which the anchor bolts are not engaged, only the out-of-plane flexural stiffness of the bearing resists the rotation. The slope of the curve is relatively flat. When the anchor bolts are snug tight, the anchor bolts on the west side of the web resist the rotation while those on the east side do nothing. The stiffness of the connection increases, as shown by the increase in the slope. For the connection in which the anchor bolts are pretensioned, all of the anchor bolts resist the rotation and this connection is the stiffest. The connection stiffness increases considerably when the center of rotation is moved up into the web. An extra restraining element is introduced into the connection, the out-of-plane shearing stiffness of the bearing (see Section 4.2.2, Bearing and Pier Column). The rotations at the top flange and at the web centerline are essentially the same, as are the rotations at the cross plate and the bottom flange. The top flange is connected only to the web, since the bearing stiffeners are half-depth; it follows the rotation of the top half of the web. If the web was perfectly rigid, the rotation at every point along the depth of the girder would be the same as that point about which the web rotates, the center of rotation. The rotational stiffnesses of the different configurations are shown in Table 8.4. The negative sign for

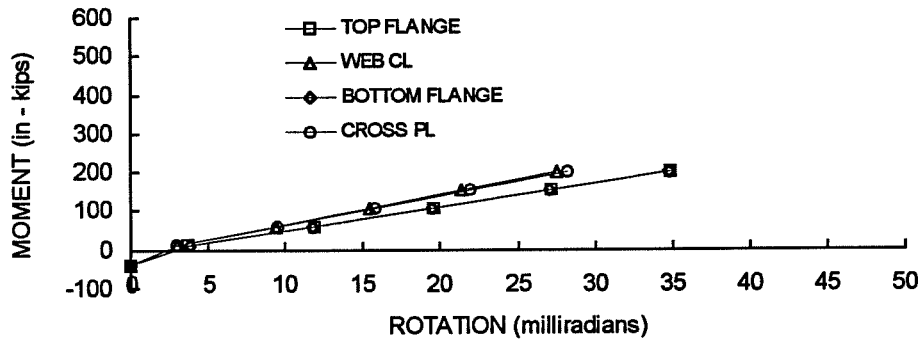


Figure 8.13 Moment vs. Rotation, (0) Anchor Bolts, CoR = 0"

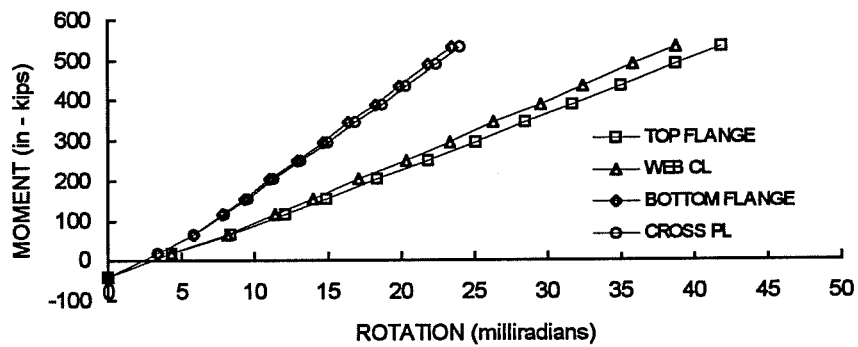


Figure 8.14 Moment vs. Rotation, (1) Anchor Bolts, CoR = 0"

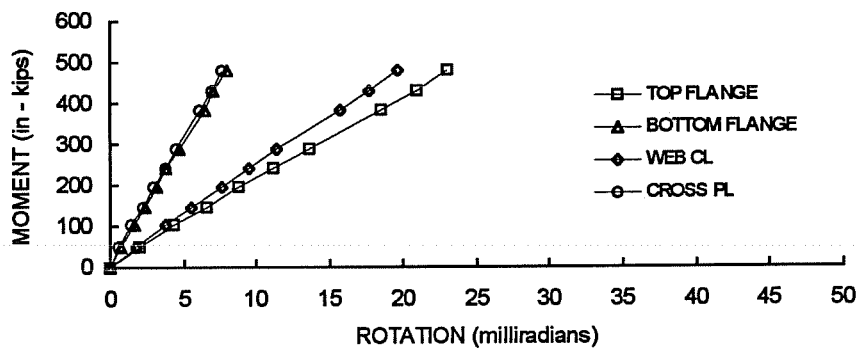


Figure 8.15 Moment vs. Rotation, (2) Anchor Bolts, CoR = 0"

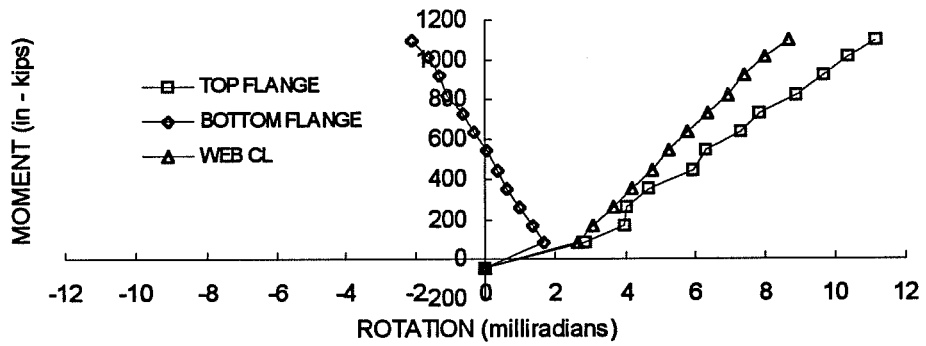


Figure 8.16 Moment vs. Rotation, (0) Anchor Bolts, CoR = 34"

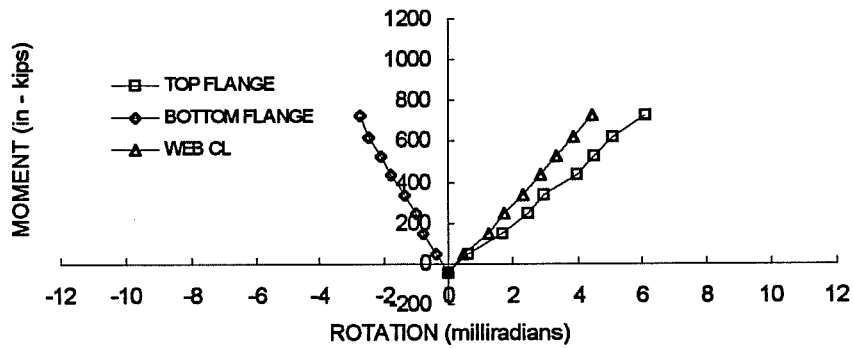


Figure 8.17 Moment vs. Rotation, (1) Anchor Bolts, CoR = 34"

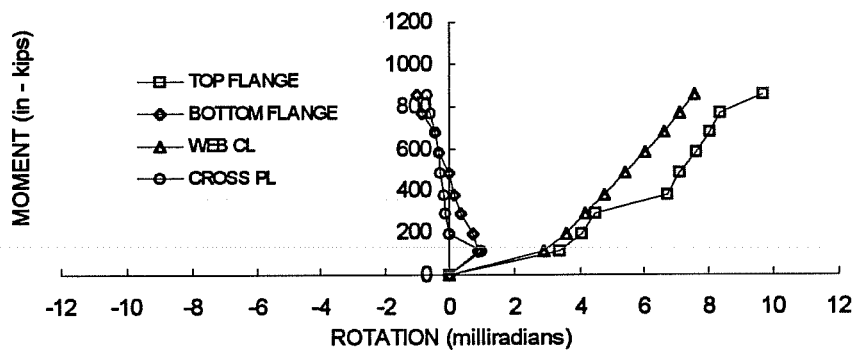


Figure 8.18 Moment vs. Rotation, (2) Anchor Bolts, CoR = 34"

the stiffnesses of the CoR = 34" tests indicates that the rotation was in the direction opposite to the direction of the applied moment. This "negative rotation" occurred because the stiffeners did not extend the full depth of the web. Shown in Figure 8.19 is an illustration of the behavior. If no stiffeners were present the bottom flange would rotate in a counterclockwise, or negative direction. If the stiffeners extend through the full depth of the web, all of the points along the depth must rotate in the same direction, which is clockwise or positive because that is the sense of the applied moment. Because the stiffeners of the detail only extend through half the depth of the web, the behavior will fall between the two extremes and in this case the bottom flange rotates counterclockwise, or in the negative direction.

Table 8.4 : Longitudinal Rotational Stiffness of Connection

ROTATIONAL STIFFNESS (inch-kips per radian)				
TEST	BOTTOM FLANGE	CROSS PLATE	WEB CL	TOP FLANGE
R1520000	6,500	8,040	8,290	6,520
R1520001	25,000	24,600	15,000	13,800
R1520080	59,500	60,900	23,900	20,300
R1523400	-275,000	-	172,000	123,000
R1523401	-275,000	-	174,000	128,000
R1523480	-385,000	-953,000	160,000	223,000
R0870040	50,300	-	-	18,040

Distribution of Moments

The applied moment is resisted by both the anchor bolts and the bearings. The amount of moment that is resisted by each is proportional to its stiffness. Figures 8.20 and 8.21 are graphs of the moment resisted by the anchor bolts and bearings vs. the applied moment at CoR = 0". If the anchor bolts, or the bearings, were the only resisting element, the slope of the curve would be one, i.e. the resisting moment would equal the applied moment. This relationship is represented in the graphs by the heavy solid line. A curve below this line indicates that the

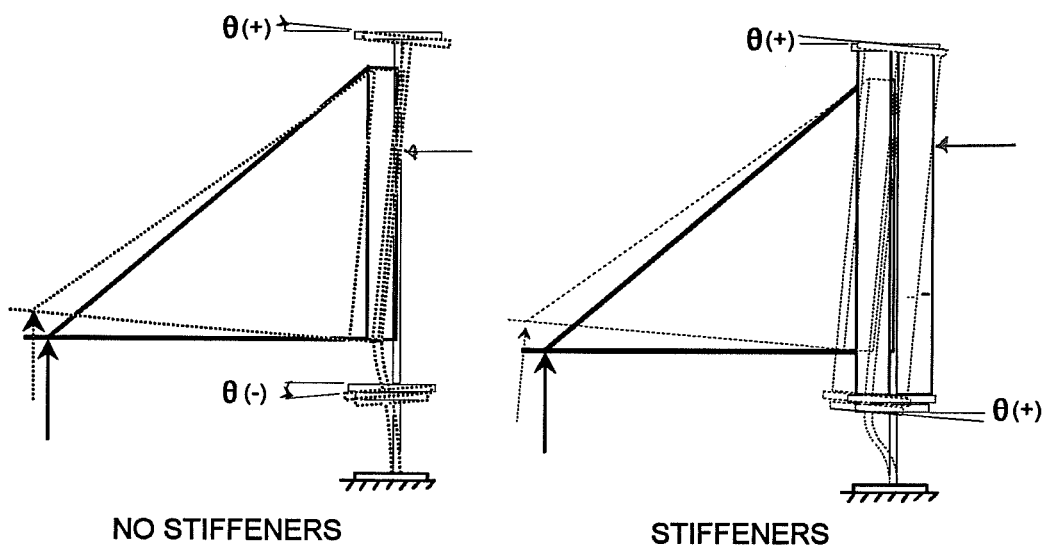


Figure 8.19 Illustration of Negative Rotation

resisting moment is some fraction of the applied moment. Except for the case in which no anchor bolts are engaged, the bearings resist a smaller percentage of the applied moment than do the anchor bolts. The fraction of the applied moment that the bearings and anchor bolts resist is shown in Figure 8.22. For the connection in which the anchor bolts are not engaged, the applied moment is resisted solely by the bearings. The moment in the bearings, which was constant along the depth of the bearing web, was calculated using the strain gages located at the top and bottom of the web. The moment calculated using the experimental stress readings was 7% greater than the applied moment. For the connection in which the anchor bolts were snugged, the anchor bolts on the west side of the web were engaged, and the bearings resisted 35% of the moment and the anchor bolts resisted 69% of the moment, with the sum 4% greater than the applied moment. When the anchor bolts are pretensioned, the anchor bolts on both sides of the web are engaged and the stiffness of the anchor bolt system increases. This is confirmed by the experimental data. The moment resisted by the anchor bolts increased to 89% and the moment resisted by the bearings fell to 12%, with the sum equal to the applied moment. Also shown are the results from the connection with the W12X87 bearings. The W12X87 bearing is much more flexible than the W12X152 bearing and as a result resists a much smaller fraction of the applied moment.

It is more difficult to calculate the resisting moments using experimentally determined values for the connection with the center of rotation at 34 inches. Though the anchor bolt moments may still be easily determined, the bearings resist the moment through rotation and horizontal shear. The horizontal shear mechanism of the bearings (Section 4.2.2) produced a rotational stiffness much greater than the rotational stiffness of the anchor bolts and as a result the anchor bolts resisted less than 5% of the anchor bolt moment for all connection configurations {R1523400 [(0) anchor bolts] - 0%, R1523401 [(1) anchor bolts] - 3%, R1523480 [(2) anchor bolts] - 4%}.

Cap Girder Web Stresses

The web stress vs. rotation curves are shown in Figures 8.23 and 8.24. Six curves are shown for each of tests (CoR = 0" and CoR = 34"). The label *PAR* indicates the stress parallel to the long axis of the cap girder; the label *PRP* indicates the stress perpendicular to the long axis of the cap girder. The stresses are the averages of the values from the north and south groups of gages. The curves show that for CoR=0" the web stresses at the stiffener termination point increase with an increase in the number of anchor bolts engaged. This is due to the increase in

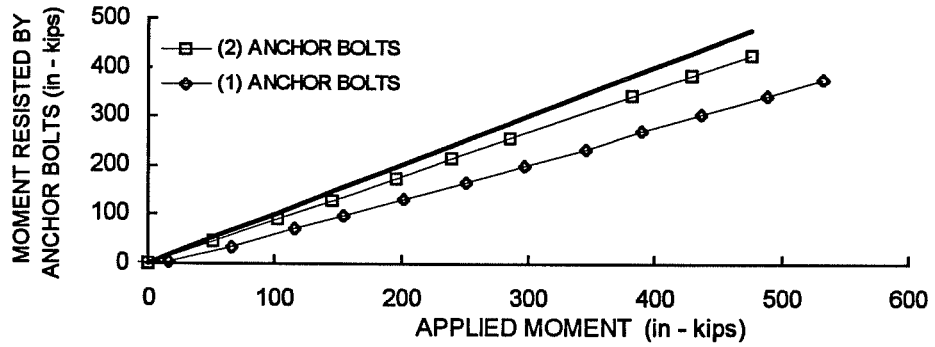


Figure 8.20 Moment in Anchor Bolts vs. Applied Moment, CoR = 0"

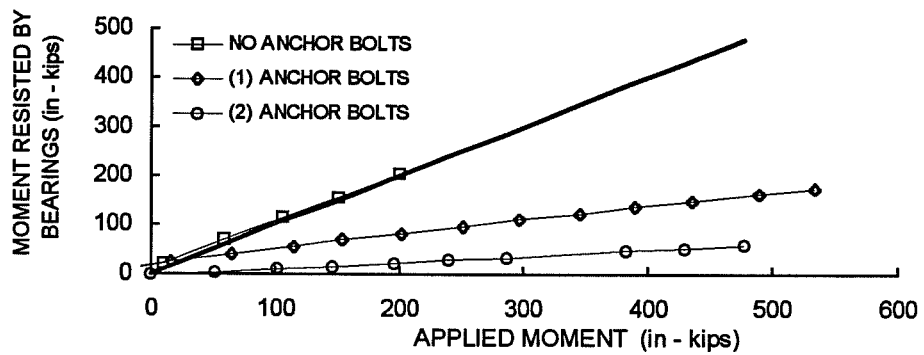


Figure 8.21 Moment in Bearings vs. Applied Moment, CoR = 0"

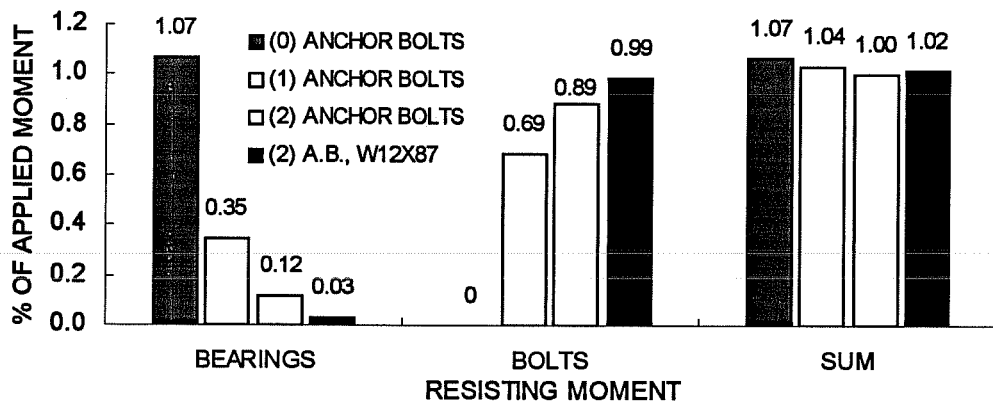


Figure 8.22 Resisting Moment as Percentage of Applied Moment, CoR = 0"

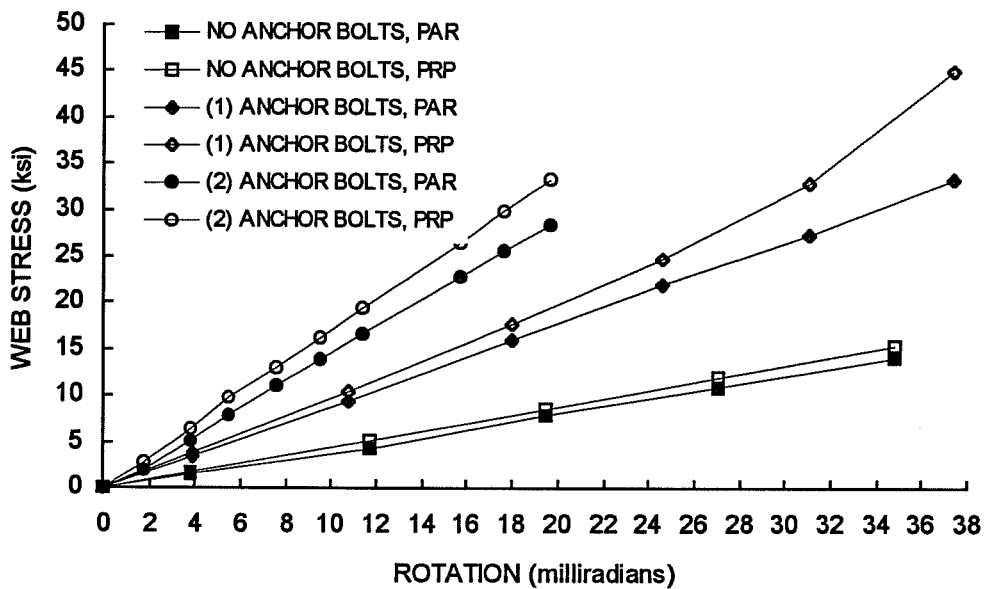


Figure 8.23 Web Stress vs. Web Centerline Rotation, CoR = 0"

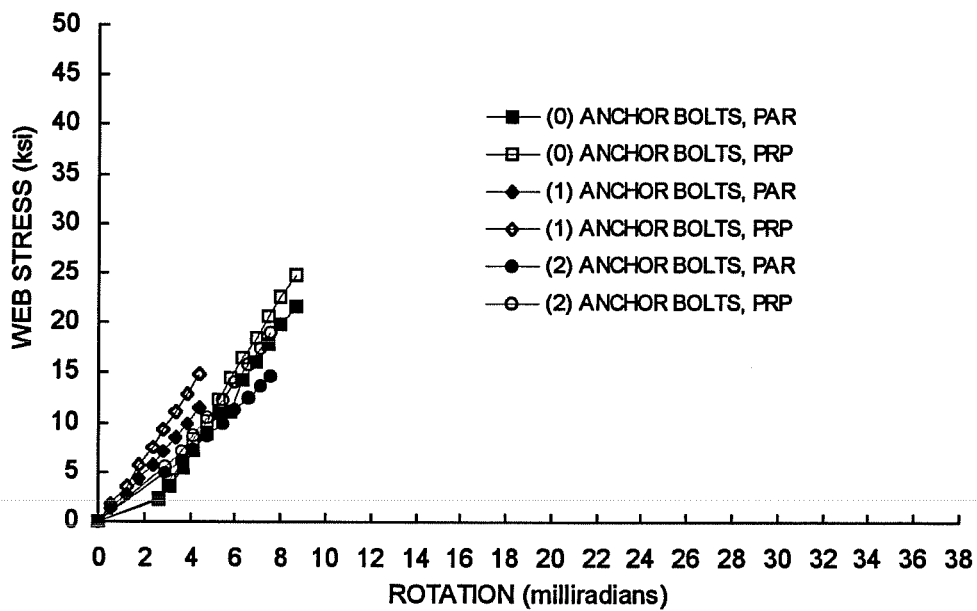


Figure 8.24 Web Stress vs. Web Centerline Rotation, CoR = 34"

the rotational stiffness of the anchor bolt system. The difference between the rotation at the web centerline and the cross-plate increases and so does the stress, as illustrated in Figure 8.23. The finite element analyses of the cap girder web subject to longitudinal rotation described in Section 4.2.4 showed the same trend. The web stress was not as sensitive to the rotational stiffness of the anchor bolt system for the CoR = 34" tests. The stress, however, was larger for the same rotation as compared with the CoR = 0" tests; at a rotation of 0.003 radians, the web stress at the stiffener termination point approached 10 ksi, which suggests that this area of the web may be susceptible to distortion-induced cracking (see Section 4.2.4). This problem could be eliminated by extending the stiffener the full depth of the web.

8.2.3 Transverse Direction Tests

Moment vs. Rotation Behavior

The moment vs. rotation curves for the transverse direction tests are shown in Figures 8.25 and 8.26. The curves show that the rotational stiffness of the connection system increases when the anchor bolts are pretensioned. For the connections with no anchor bolts (Txxx00), the bearings provide the resistance to rotation. When the anchor bolts are engaged but not pretensioned (Txxx01) the stiffness increases slightly. The connection system is at its stiffest when the anchor bolts are pretensioned. The rotation is primarily a function of the vertical deflection at each bearing. As was shown in Section 1.2.1 no significant deflection will occur in a pretensioned connection until the pretension force is overcome. The flattening of the moment vs. rotation curves for the pretensioned connections (Txxx80) is due primarily to the decrease in depth of the compression bearing as it failed. The rotational stiffness of the connection systems in the linear range are shown in Table 8.5.

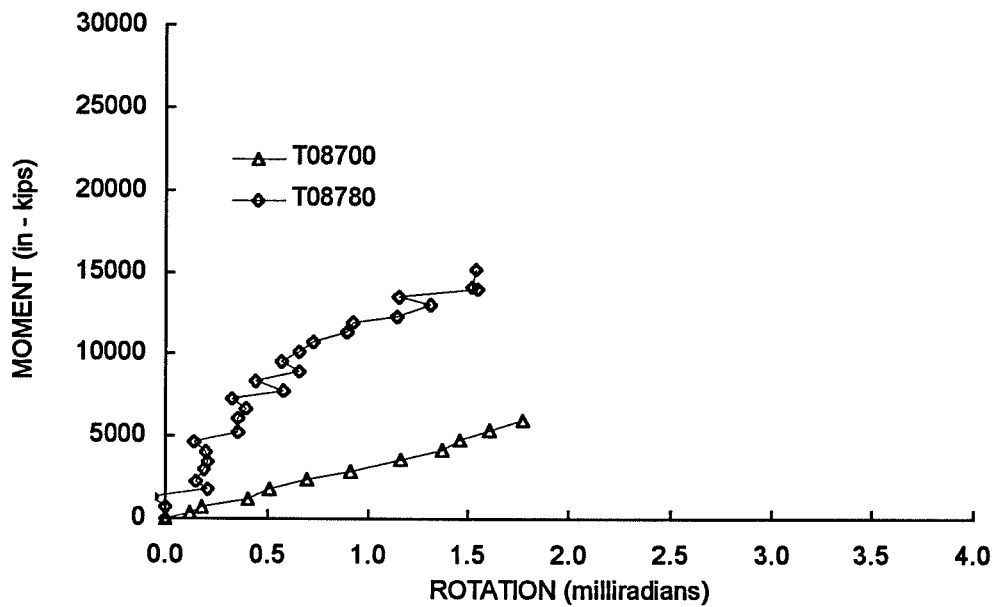


Figure 8.25 Moment vs. Rotation, Transverse Direction, W12X87

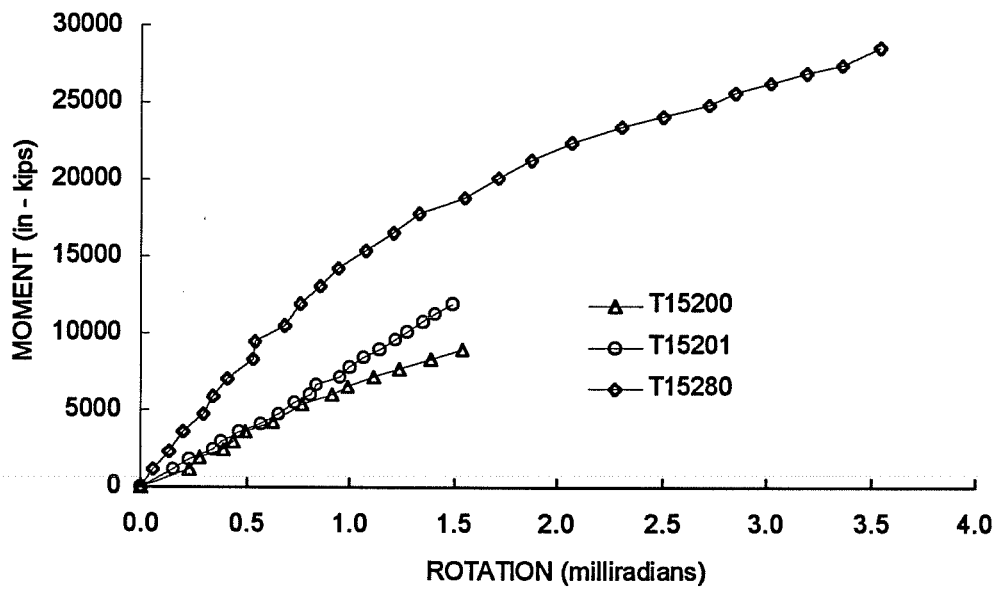


Figure 8.26 Moment vs. Rotation, Transverse Direction, W12X152

Table 8.5 : Transverse Rotational Stiffness

TEST	ROTATIONAL STIFFNESS (in-kips per radian)	
	W12X87 BEARINGS	W12X152 BEARINGS
Txxx00	3,250,000	6,000,000
Txxx01	-	8,000,000
Txxx80	8,850,000	15,200,000

Distribution of Bolt Forces and Pier Reactions

The statics of the cap girder are shown in Figure 8.27. The dead load of the cap girder was ignored. The reaction force for the uplift connection is tensile and is defined as $2P$ where P is load applied at the end of the cap girder. The reaction force for the compression connection is compressive and is defined as $3P$. The distribution of bolt forces and pier reactions for the tests with the pretensioned connections (T08780 and T15280) are shown in Figures 8.28 and 8.29. The individual data points are not shown. Each graph presents the data from both connections. The behavior of the uplift connection is represented by the curves to the right of zero, the heavy solid lines. The behavior of the compression connection is represented by the curves to the left of zero, the heavy dashed lines. The curves above zero show the response of the bolt group force as a function of the reaction force and the curves below zero show the response of the pier reaction as a function of the reaction force.

The pretension force in all of the bearings was approximately 310 kips. As load was applied to the end of the girder (P in Figure 8.27), the pier reaction decreased for the uplift bearing (heavy solid curve below zero) and increased for the compression bearing (heavy dashed curve below zero) in almost direct proportion to the reaction force (A in Figure 8.27). Note, however, that the change in the bolt group force (heavy curves above zero) was very small for the same increase in the reaction force. This behavior illustrates the great advantage of a pretensioned connection over a connection that is not pretensioned. For example, if the uplift bearing was not pretensioned, the bolt group would resist the entire reaction force (shown by the light dashed curve); a change in the reaction force of 250 kips would mean a change in the bolt

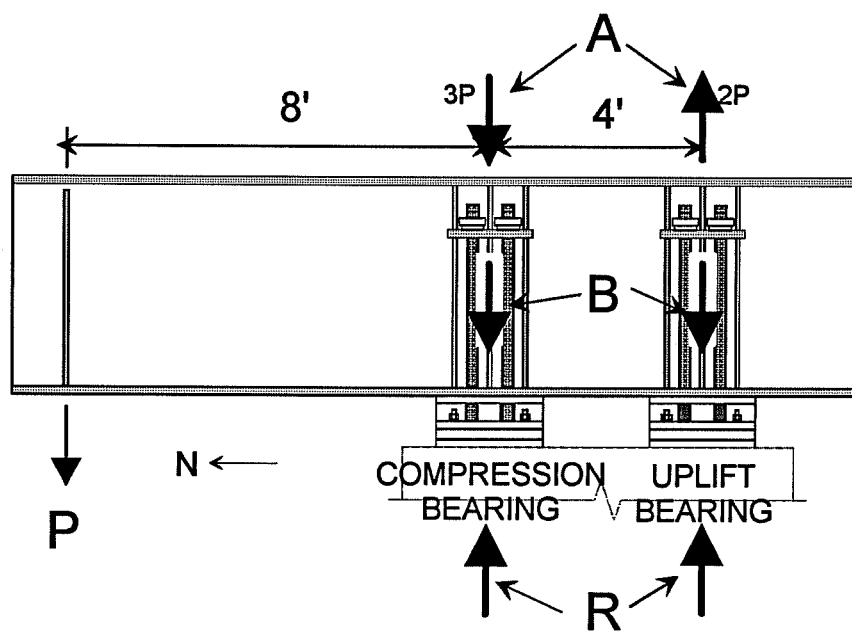


Figure 8.27 Connection Statics

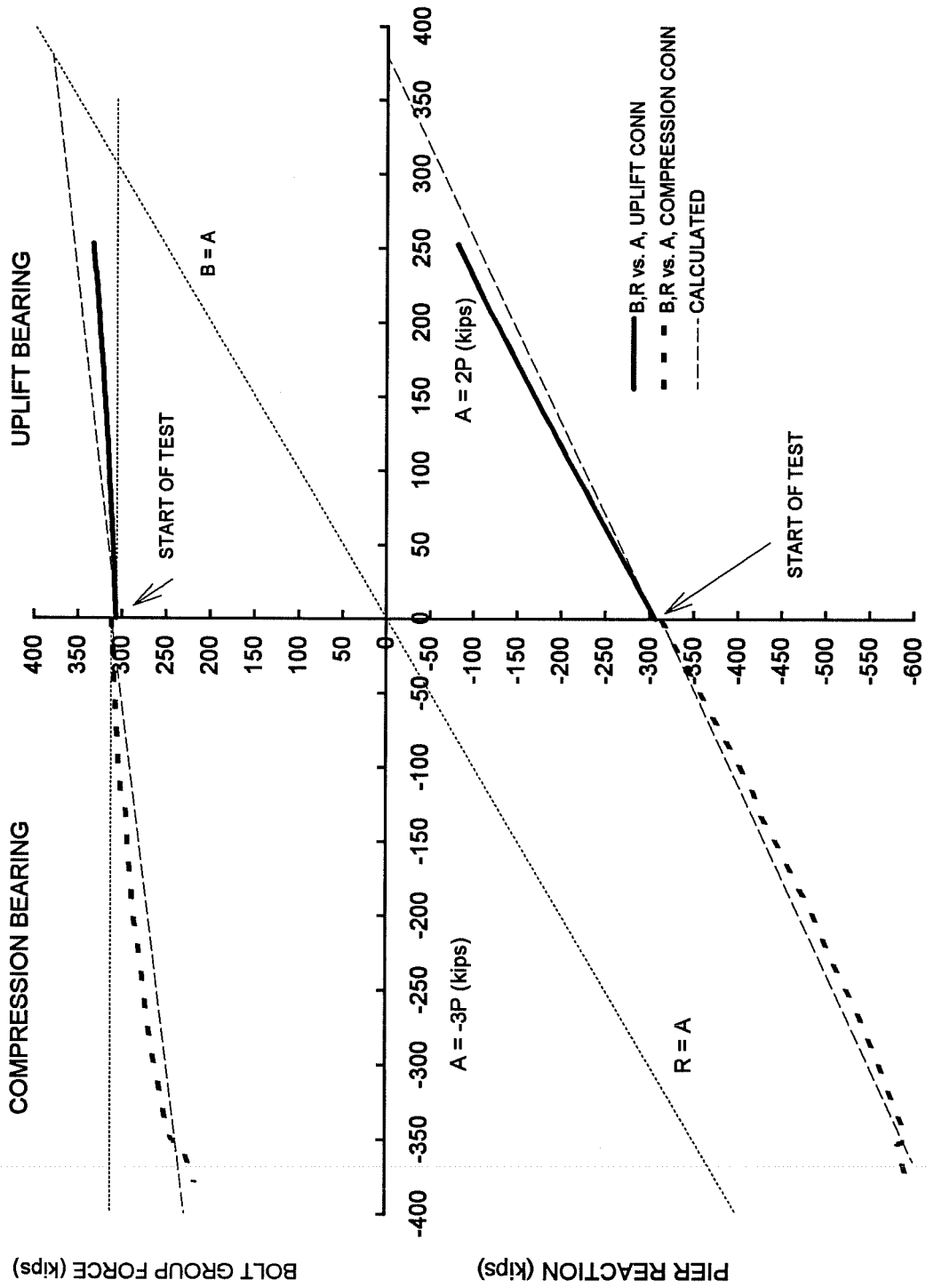


Figure 8.28 Bolt Group Force, Pier Reaction vs. Applied Load, W12X87 Large Scale Test

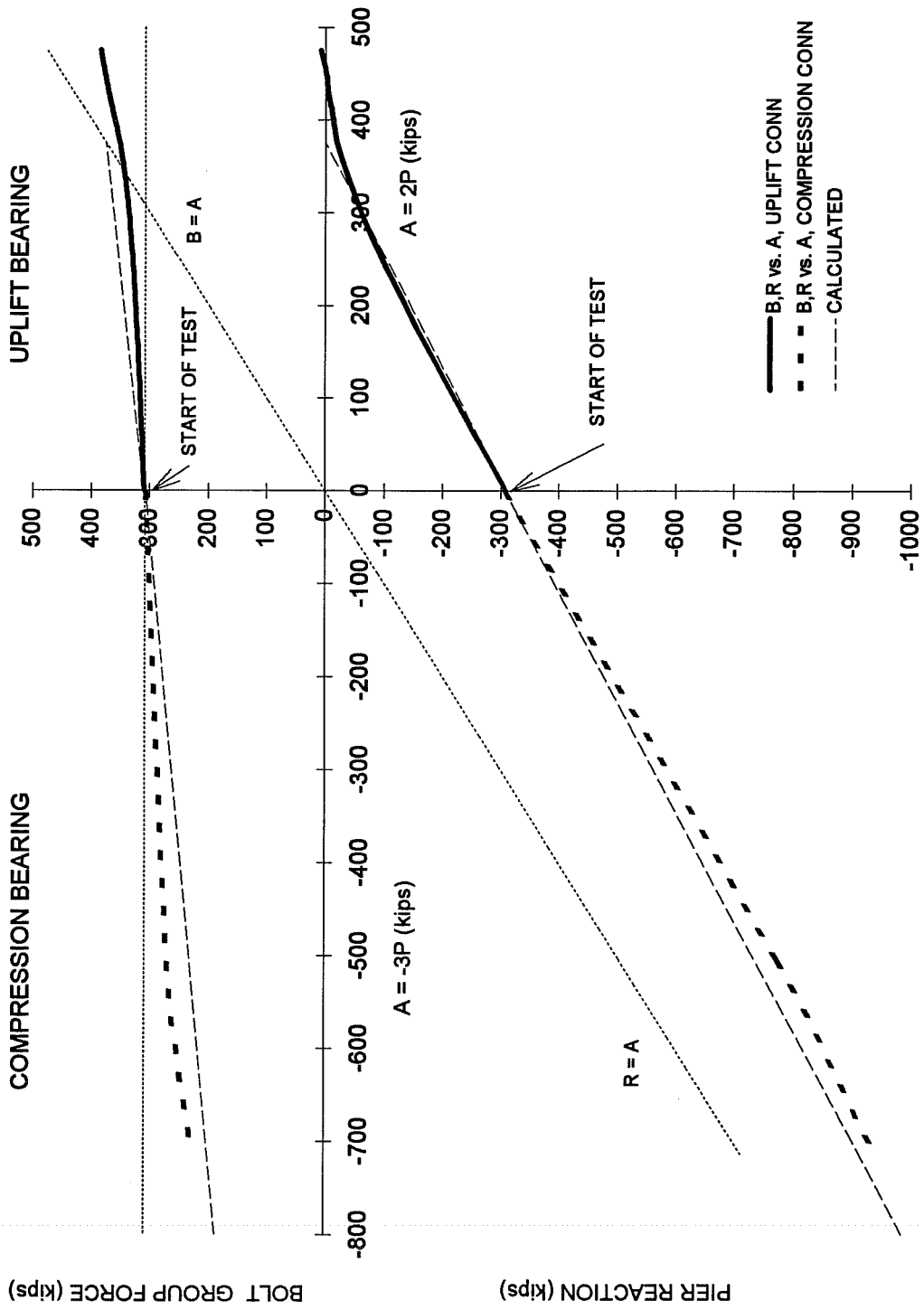


Figure 8.29 : Bolt Group Force, Pier Reaction, vs. Applied Load, W12X152 Large Scale Test

group force of 250 kips. The actual change in the bolt group force for the same change in reaction force, however, was only 35 kips for the pretensioned connection (heavy solid curve), a decrease of over 600%.

Uplift did not occur in the test with the W12X87 bearings, illustrated by the termination of the curve before it reached the B (bolt group force) = A (reaction force) line; the compression bearing failed first. Uplift did occur in the test with the W12X152 bearings. The bolt group, however, did not resist all of the applied load once uplift occurred; unlike the Phase I bearing, which can only resist compression, the Phase II bearing provides a positive connection to the pier cap and resists tension. The uplift is resisted by both the anchor bolts and the wide-flange bearing.

The bolt group force for the compression bearing decreased slightly with an increase in the applied compressive load. The decrease was linear for the test with the W12X152 bearings; the failure of the bearing was not accompanied by a large decrease in height. The failure of the W12X87 bearing was accompanied by a significant decrease in height and as a result the bolt group force decreased more rapidly toward the end the test.

The equations for the linear portions of the curves, obtained by a least squares fit of the data, are shown below, where B is the bolt group force, R is the pier reaction, and A is the reaction force ($A = 2P$ for the uplift bearing and $A = -3P$ for the compression bearing). These equations will be analyzed in the following section.

Uplift Bearing, Bolt Group Force vs. Applied Tensile Load

$$\begin{aligned} B &= 306 + 0.10 A && \text{W12X152 Bearings} \\ B &= 305 + 0.10 A && \text{W12X87 Bearings} \end{aligned} \tag{8.1}$$

Uplift Bearing, Pier Reaction vs. Applied Tensile Load

$$\begin{aligned} R &= -301 + 0.80 A && \text{W12X152 Bearings} \\ R &= -305 + 0.90 A && \text{W12X87 Bearings} \end{aligned} \tag{8.2}$$

Compression Bearing, Bolt Group Force vs. Applied Compressive Load

$$\begin{aligned} B &= 309 + 0.077 A && \text{W12X152 Bearings} \\ B &= 321 + 0.20 A && \text{W12X87 Bearings} \end{aligned} \quad (8.3)$$

Compression Bearing, Pier Reaction vs. Applied Compressive Load

$$\begin{aligned} R &= -309 + 0.92 A && \text{W12X152 Bearing} \\ R &= -321 + 0.80 A && \text{W12X87 Bearing} \end{aligned} \quad (8.4)$$

Stress Distribution in Bearings

The load in the compression bearing is the sum of the pretension load in the bolts and the reaction from the load applied at the end of the cap girder. The resultant of the compressive load produced by the anchor bolts coincides with the centerline of the bearing and should therefore produce a fairly uniform distribution of stress along the length of the bearing. The reaction produced by the applied load will not be concentric with the bearing, producing a non-uniform distribution of stress. Strain gages were placed at three locations along the top and bottom of the bearing to measure the stress distribution. Typical stress distributions are shown in Figure 8.30 to 8.33. The vertical axis shows the strain gage stress f_{sg} (at the center the average of the two gages was used) normalized to the nominal stress f_n , which is the compressive load in the bearing divided by the web area. The horizontal axis shows the nominal stress f_n normalized to the transverse direction yield stress F_y . The stresses are shown for three locations: at the north end of the bearing, 0 inches; at the center of the bearing, 12 inches; and at the south end of the bearing, 24 inches. The plots apply to the tests using the W12X87 bearings. The results show that the normalized stress at each location remains fairly constant at the 12 inch and 24 inch locations. The only deviation from this generalization occurred for the stress at the north top of the bearing. At the 0 inch location at the top the stress increased with an increase in load until a certain point and then began dropping. This indicated a redistribution of the stress from the north end of the bearing towards the south end.

Comparison of the average normalized strain gage stresses at the three locations for the tests in which the anchor bolts were pretensioned and the tests in which the anchor bolts were not

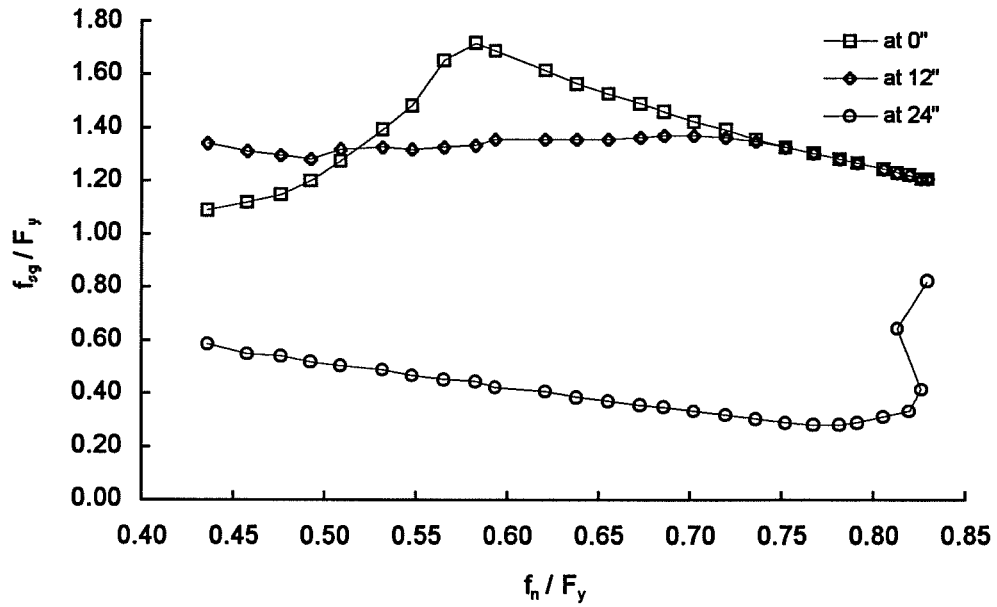


Figure 8.30 Normalized Stress at Top Gages vs. Nominal Web Stress, W12X87 Compression Bearing with Pretensioned Anchor Bolts

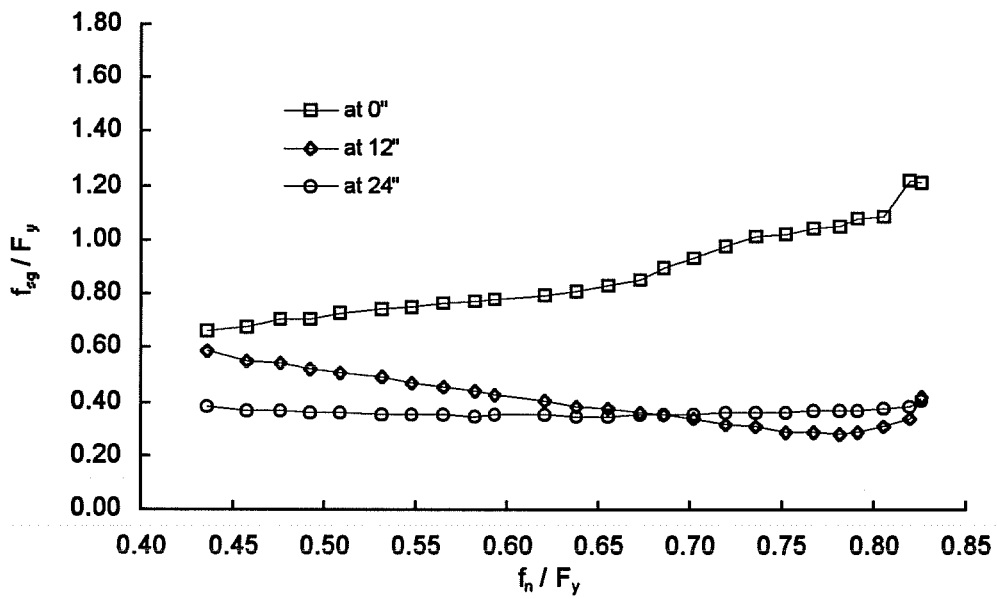


Figure 8.31 Normalized Stress at Bottom Gages vs. Nominal Web Stress, W12X87 Compression Bearing with Pretensioned Anchor Bolts

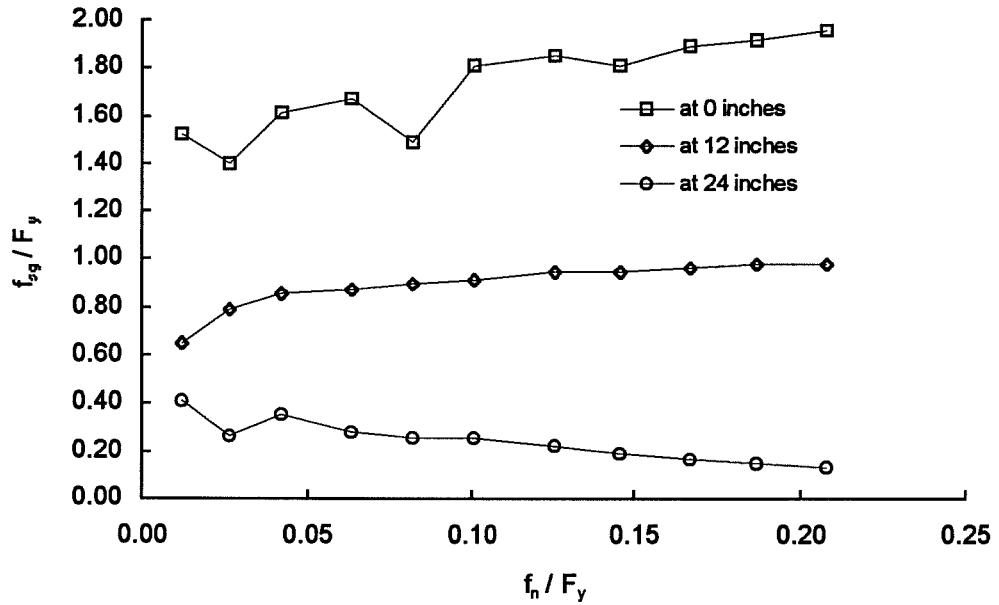


Figure 8.32 Normalized Stress at Top Gages vs. Nominal Web Stress, W12X87 Compression Bearing, no Pretension

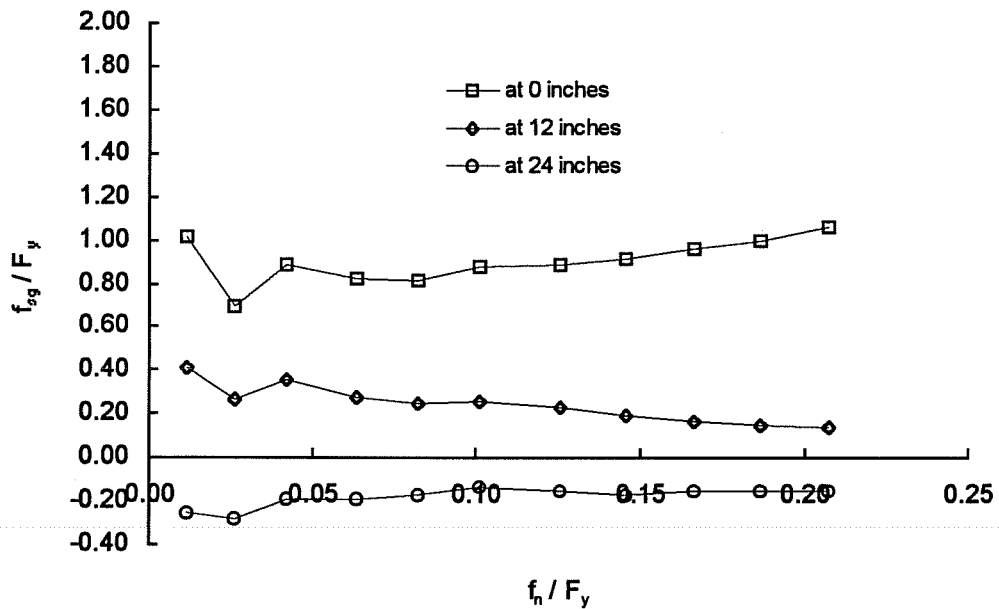


Figure 8.33 Normalized Stress at Bottom Gage vs. Nominal Web Stress, W12X87 Compression Bearing, no Pretension

pretensioned are shown in Figures 8.34 and 8.35. The stress was maximum at the north end of the bearing in all cases and decreased toward the south end. The difference between the north end stress and the south end stress was most extreme for the tests in which the anchor bolts were not pretensioned. Pretensioning the anchor bolts tended to lessen the difference in the stresses at the three locations; the concentric compression force produced by the anchor bolts mitigated the non-uniform stresses of the eccentric compression reaction produced by the applied load. In the actual bearing the dead load reactions should perform a function similar to the pretensioned anchor bolts and produce a relatively uniform stress distribution.

The non-uniform stress distribution did not have any affect on the ultimate strength of the bearings. The load vs. top flange lateral displacement curves of the two compression bearings are shown in Figure 8.36. The ultimate loads from the full scale tests are compared with the ultimate loads from the bearing compression tests in Table 8.6.

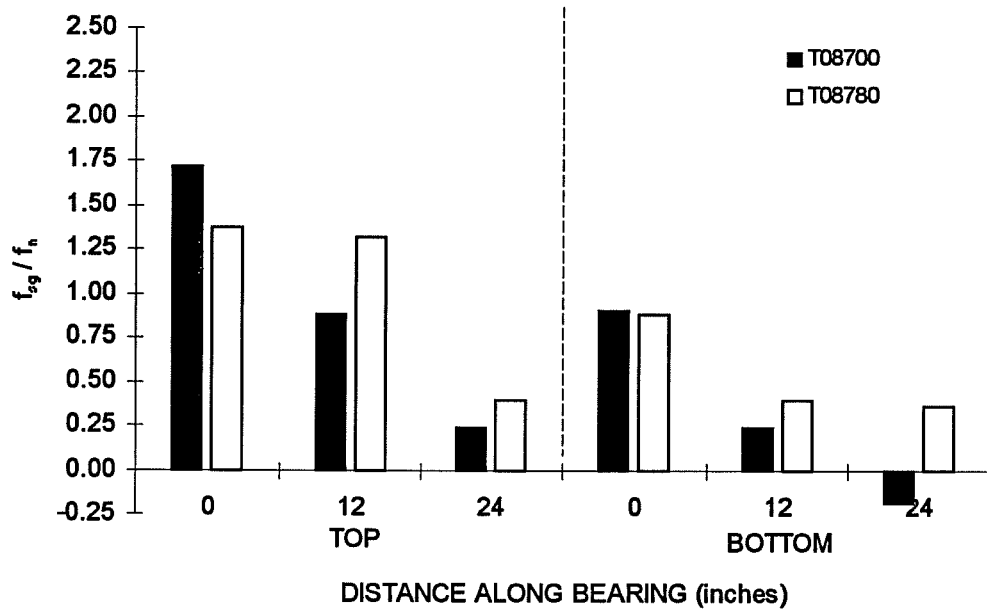


Figure 8.34 Average Ratio of Gage Stress to Nominal Stress (f_{sg}/f_n), W12X87

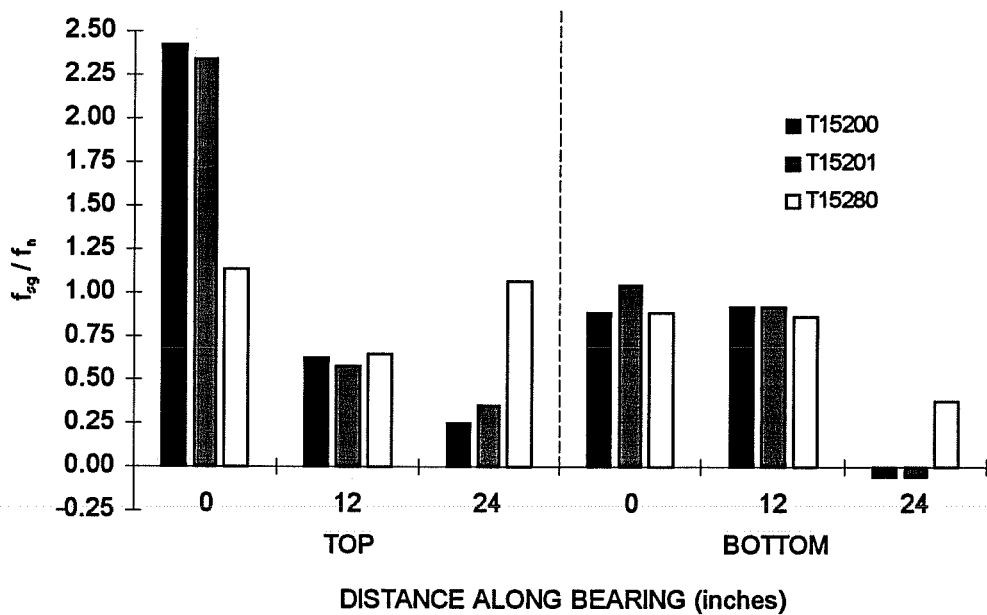


Figure 8.35 Average Gage Stress vs. Nominal Stress (f_{sg}/f_n), W12X152

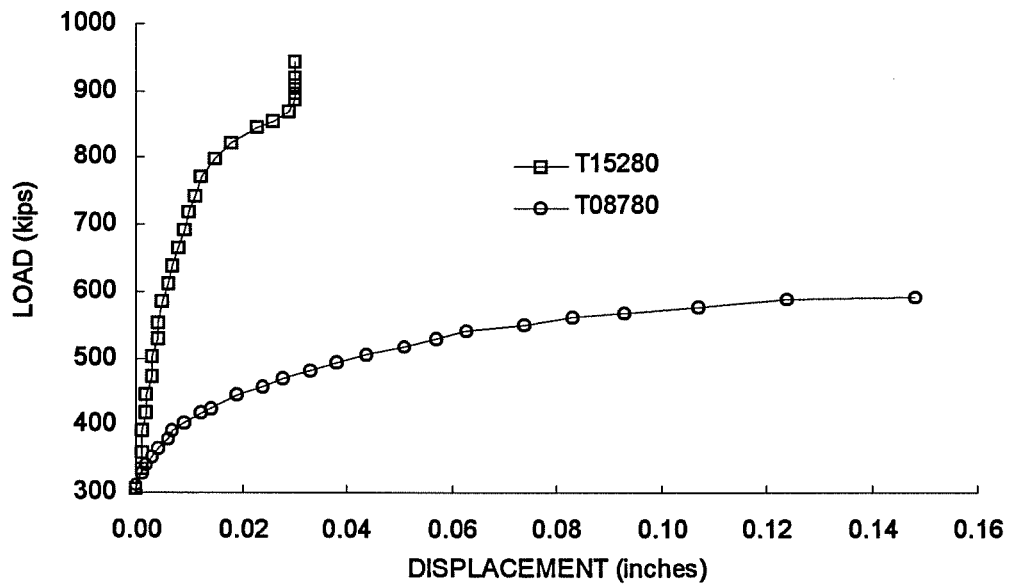


Figure 8.36 Load vs. Top Flange Lateral Displacement, Compression Bearing

Table 8.6 : Comparison of Bearing Capacities from Full Scale Tests and Component Tests

BEARING SIZE	TEST	F_{cr} / F_y
W12X87	Component	0.81
	Full Scale	0.83
W12X152	Component	1.13
	Full Scale	1.06

8.3 DISCUSSION OF RESULTS

8.3.1 Longitudinal Direction Behavior

The primary difference between the connection using the rocker and the connection using the wide-flange bearing is that the wide-flange bearing provides fixity from the cap girder to the pier cap. This fixity changes the horizontal and rotational stiffnesses of the pier column that were calculated in Section 4.2. The calculation of the anchor bolt system stiffness remains the same.

Pier Column

A model of the pier column and bearing with a horizontal force and a moment is shown in Figure 8.37. The force and the moment will produce a deflection and a rotation and by relating the displacements to the forces the stiffnesses of the pier column can be determined. The derivation of the stiffnesses is shown in Appendix D.

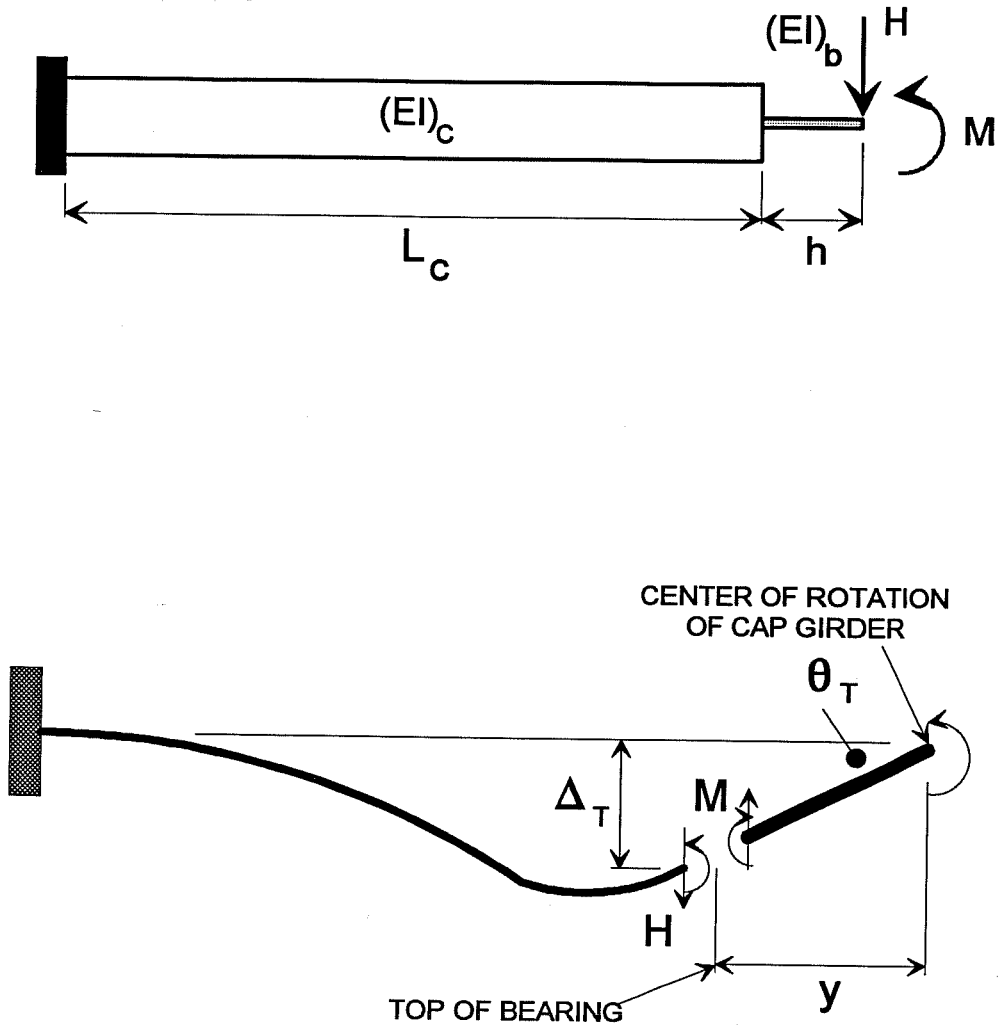


Figure 8.37 Pier Column and Bearing

The horizontal force was derived as

$$H = k_H \Delta_H \left[\frac{\left(1 + \frac{\beta_M}{k_M y} \right)}{\left(1 - \frac{k_H \beta_M}{k_M^2} \right)} \right] \quad (8.5)$$

and the moment at the top of the bearing was derived as

$$M = \beta_M \theta \left[\frac{\left(1 + \frac{y k_H}{k_M} \right)}{\left(1 - \frac{k_H \beta_M}{k_M^2} \right)} \right] \quad (8.6)$$

where

$$k_H = \frac{(EI)_c}{\frac{L_c^2 h}{2} + h^2 L_c + \frac{L_c^3}{3} + \frac{h^3}{3} \left(\frac{(EI)_c}{(EI)_b} \right)} \quad (8.7)$$

$$k_M = \frac{(EI)_c}{L_c h + \frac{L_c^2}{2} + \frac{h^2}{2} \left(\frac{(EI)_c}{(EI)_b} \right)} \quad (8.8)$$

$$\beta_M = \frac{(EI)_c}{L_c + h \left(\frac{(EI)_c}{(EI)_b} \right)} \quad (8.9)$$

and y is the distance from the top of the bearing to the center of rotation, L_c is the length of the column, h is the height of the bearing, and $(EI)_i$ is the stiffness of the column $(EI)_c$ or the bearing $(EI)_b$. In calculating the force and moment due to temperature change, the rotation may be approximated as zero since a uniform temperature change through the depth of the superstructure would produce equal horizontal displacements.

The horizontal force would be calculated as

$$H = \frac{k_H \Delta_t}{\left(1 - \frac{k_H \beta_M}{k_M^2}\right)} \quad (8.10)$$

and the moment would be

$$M = \frac{\beta_M \Delta_t \left(\frac{k_H}{k_M}\right)}{\left(1 - \frac{k_H \beta_M}{k_M^2}\right)} \quad (8.11)$$

where Δ_t is the horizontal displacement due to the temperature change. The largest horizontal force and moment will be produced by the temperature change and therefore Equations 8.10 and 8.11 should be used. The fatigue loads will be produced by the longitudinal rotation of the cap girder and Equations 8.5 and 8.6 should be used.

Comparison of Calculated and Experimental Stiffnesses

The rotational stiffnesses of the anchor bolt systems for the Phase II connection were determined using Equation 4.16. There are no bearing pads and the axial stiffness of the anchor bolts is shown below:

- W12X87 Bearings : $L_b = 75$ in : $A_b E = 25944$ kips : $k_b = 25944 / 75 = 346$ k / in
- W12X152 Bearings : $L_b = 77$ in : $A_b E = 25944$ kips : $k_b = 25944 / 74 = 337$ k / in

The rotational stiffness of the bearings with CoR = 0" is $(EI)_b / h$ and the rotational stiffness of the bearings with CoR = 34" can be determined by using Equation 8.6, dividing both sides by θ . The properties of the bearings are shown below:

- W12X87 Bearings : $h = 10.94$ in : $(EI)_b = 29000 (1/12) (2*24) (0.509)^3 = 15298$ in-k / rad
- W12X152 Bearings : $h = 11.01$ in : $(EI)_b = 29000 (1/12) (2*24) (0.891)^3 = 82052$ in-k / rad

A comparison of the experimental rotational stiffnesses, as measured at the bottom flange, and the calculated rotational stiffnesses are shown in Table 8.7. The calculated stiffnesses are the sum of the rotational stiffnesses of the anchor bolt system and the bearing.

Table 8.7 : Comparison of Experimental and Calculated Longitudinal Rotational Stiffnesses

TEST	EXPERIMENTAL (in-k / rad)	CALCULATED (in-k / rad)
R1520000	6,500	7,450
R1520001	25,000	34,800
R1520080	59,500	62,000
R1523400	-275,000 to 172,000	180,000
R1523401	-275,000 to 174,000	207,000
R1523480	-385,000 to 223,000	235,000
R0870040	50,300	57,500

The calculated stiffnesses for the connections with $CoR = 0$ correlate well for no anchor bolts engaged and both set of anchor bolts engaged, but diverge somewhat for one set of anchor bolts engaged. For the connections with $CoR = 34$ the calculated stiffnesses are of the same order of magnitude as the experimentally determined stiffnesses. The negative sign indicates the negative rotation that occurred at the bottom flange of the cap girder. This behavior was explained in Section 8.2.1 and had to do with stiffeners only extending up half the depth of the web. If the stiffeners extended up the entire length of the web the rotation at all points along the depth of the cap girder would be the same (ignoring any difference due to the bending of the stiffeners) and the values would correlate better with the calculated values. It will be recommended in a following section that the stiffeners should indeed extend the full depth of the web.

8.3.2 Transverse Direction Behavior

Equations Describing Connection Behavior

The behavior of a connection that has been pretensioned was described in section 1.2.1. If the bolt is pretensioned to a force B_0 , all of the material between the bolt heads are compressed by a force C_0 that is equal to B_0 . An applied tensile load produces an increase in the bolt force, but this increase is a small percentage of the applied load, as shown in Figure 1.7, region c). This behavior is quantified as

$$B = B_0 + A \left[1 - \frac{B_0}{B_s} \right] \quad (8.12)$$

where B is the bolt force, A is the applied load, and B_s is the force in the bolt at the initiation of uplift, or the point at which the plates separate, and is given as

$$B_s = B_0 \left[1 + \frac{k_b}{k_c} \right] \quad (8.13)$$

where k_b is the axial stiffness of the bolt and k_c is the axial stiffness of the elements in the grip. The applied tensile force necessary to produce uplift is termed T_s and is equal to B_s . For $A \geq T_s$, shown in Figure 1.7, region e.), the bolt force is equal to the applied load, or $B = T$. The magnitude of the reaction between the plates in response to applied tensile load, shown in Figure 1.8, region c.), is given as

$$R = B_0 \left[1 - \frac{A}{B_s} \right] \quad (8.14)$$

where R is the reaction between the plates. For $A \geq T_s$ the plates have separated and $R = 0$, shown in Figure 1.8, region e.).

Equation 8.12 and 8.14 also apply for an applied compressive load and similar to the tensile force that produces plate separation there is a compressive force that produces separation of the plates from the bolt, given as

$$C_s = - \frac{B_0 B_s}{B_s - B_0} \quad (8.15)$$

For $A \leq C_s$ the plates and the bolt have separated and $B = 0$ and $R = A$, shown in Figures 1.7 and 1.8, region f.).

To apply the above equations to the Phase II connection the following modifications will be made:

- The bolt will be defined as the bolt group.
- The elements in the grip will be defined as the stiffeners, cap girder web, and cross-plate; the bearing hardware, consisting of the wide-flange bearing and the grouting material; and the concrete above the bolt anchorage. These elements were described in Section 4.1.
- The applied load will be defined as $-3P$ for the compression connection and as $2P$ for the uplift connection.

Determine Element Stiffnesses

The axial stiffness of the bolt groups were determined using Equation 4.3.

- W12X87 Bearings: $L_b = 75$ in : $A_b E = 25944$ kips : $k_b = 4(346) = 1384$ k / in
- W12X152 Bearings : $L_b = 77$ in : $A_b E = 25944$ kips : $k_b = 4(337) = 1348$ k / in

The stiffness of the cap girder web, cross-plate, and stiffeners were determined using Equation 4.5.

- $h_{st} = 22"$: $A_{st} = 6(7)(0.75)(2/3) = 21 \text{ in}^2$: $A_{web} = 0.5625(24+16)/2 = 11.25 \text{ in}^2$
- $n = 4$: $S_{st} = 8 \text{ in}$: $I_{cp} = (1/12)(7)(2)^3 + (1/12)(4)(1.25)^3 = 5.32 \text{ in}^4$ {includes anchor plate}

$$k_{grd} = \left[\frac{1}{42511} + \frac{1}{231420} \right]^{-1} = 35914 \text{ k / inch}$$

The bearing hardware stiffnesses for the connections were determined from measured loads and deflections, as described in section 4.1.4. Equations 8.12 and 8.14 using the calculated values are superimposed on Figures 8.28 and 8.29.

- W12X152 : Compression Bearing : 11350 k / inch
: Uplift Bearing : 9300 k / inch
- W12X87 : Compression Bearing : 6780 k / inch
: Uplift Bearing : 7880 k / inch

The total stiffness of the compression elements is

$$k_c = \left[\frac{1}{k_{grd}} + \frac{1}{k_{conc}} + \frac{1}{k_h} \right]^{-1} \quad (8.17)$$

and the values of k_b , k_c , and Equations 8.12, 8.13, and 8.14 are listed in Table 8.8. Also shown are the experimentally determined equations presented in Section 8.2.3

Table 8.8 : Equations for Bolt Group Force and Pier Reaction

BEARING		k_b	k_c	Eq. 8.13	Eq. 8.12	Eq. 8.14
COMP		(k / in)	(k / in)	B_s (kips)	B (kips)	R (kips)
W12X87	Calc	1384	5131	$396\{1.27B_0\}$	$312 + 0.21A$	$-312 + 0.79A$
	Exp			$390\{1.21B_0\}$	$321 + 0.20A$	$-321 + 0.80A$
W12X152	Calc	1348	7375	$362\{1.18B_0\}$	$307 + 0.15A$	$-307 + 0.85A$
	Exp			$334\{1.08B_0\}$	$309 + 0.08A$	$-309 + 0.92A$
UPLIFT						
W12X87	Calc	1384	5735	$379\{1.24B_0\}$	$306 + 0.19A$	$-306 + 0.81A$
	Exp			$340\{1.11B_0\}$	$305 + 0.10A$	$-305 + 0.90A$
W12X152	Calc	1348	6451	$375\{1.21B_0\}$	$310 + 0.17A$	$-310 + 0.83A$
	Exp			$344\{1.12B_0\}$	$306 + 0.10A$	$-306 + 0.80A$

The results show that the tensile force needed to produce uplift varied from $1.B_0$ to $1.21B_0$ for the experimentally determined values and from $1.21B_0$ to $1.27B_0$ for the calculated values. The calculated and the experimentally determined values compare reasonably well. The behavior of the connection is dependent primarily on the ratio of the bolt group stiffness to the stiffness of the compressed elements. The stiffer the compressive elements with respect to the bolt group, the smaller the tensile force that is necessary to produce uplift and the smaller the tensile stress range that the bolt group can cycle through (see Figure 1.7, regions c and d). Conversely, the stiffer the bolt group is with respect to the compressed elements, the larger the tensile force that is necessary to produce uplift and the larger the tensile stress range that the bolt group can cycle through. The bearing hardware, which consisted of the wide-flange section and the grouting material, had the smallest stiffness of the compressed elements. The stiffness of the wide-flange bearing can be determined as $k_{bear} = AE / h$ where A is the area of the web. The grouting material, which was placed between two steel elements in the tests, will be placed between the bottom bearing plate and the concrete pier cap in the field. Localized deformations

of grout and the concrete under the bearing plate will reduce the stiffness of the concrete element. The deformation can be estimated as the centerline deflection of the bearing plate, as calculated assuming the bearing plate acts as a beam on an elastic foundation. This gives a stiffness of

$$k_{cpl} = 0.3E_c \sqrt{l_{eff}} b \quad (8.18)$$

where l_{eff} is the effective length of the bearing plate and b is the width of the bearing plate. The term $(1/k_{cpl})$ should be added to Equation 8.17.

8.3.3 Constructibility

One of the advantages of testing a full scale specimen is that the constructibility aspects of the connection could be realistically examined. The two primary constructibility issues are the erection and positioning of the cap girder and the tensioning of the anchor bolts.

Erection and Positioning of the Cap Girder

For the standard TxDOT connection the bearing hardware is placed on the pier cap, the cap girder is lifted and set on the hardware, and the anchor bolts are snug tightened to hold the cap girder in place. The longitudinal stringers are then attached to the cap girder. The rocker allows the cap girder to be rotated (by selectively tightening the anchor bolts on each side of the web) in the vertical plane to accommodate the attachment of the stringers, as shown in Figure 8.38, but it cannot be rotated in the horizontal plane due to the interlock between the sole plate and the rocker.

The primary difference between the standard TxDOT detail and the new detail is that bearing is a monolithic unit, that is the cap girder cannot be as easily rotated in the vertical plane as with the standard connection that has the rocker mechanism. To address this problem, two solutions are proposed. It is recommended that the bearings be attached to the cap girder before it is erected, rather than placing the bearings on the pier cap first. Given that, the first solution is to set the cap girder on two metal strips (one under each bearing), as shown in Figure 8.39a. The strip will roughly approximate the rocker of the standard connection, allowing the cap girder to be rotated in the vertical plane (using either the hold-down bolts or the anchor bolts if it is an uplift connection). It will also allow some movement of the cap girder in the horizontal plane since there is no interlock between the bearing and the strip (only sliding friction). The holes for

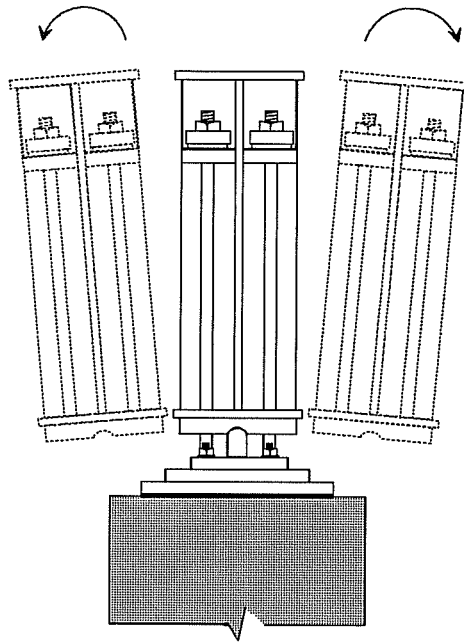


Figure 8.38 Vertical Adjustment of Cap Girder

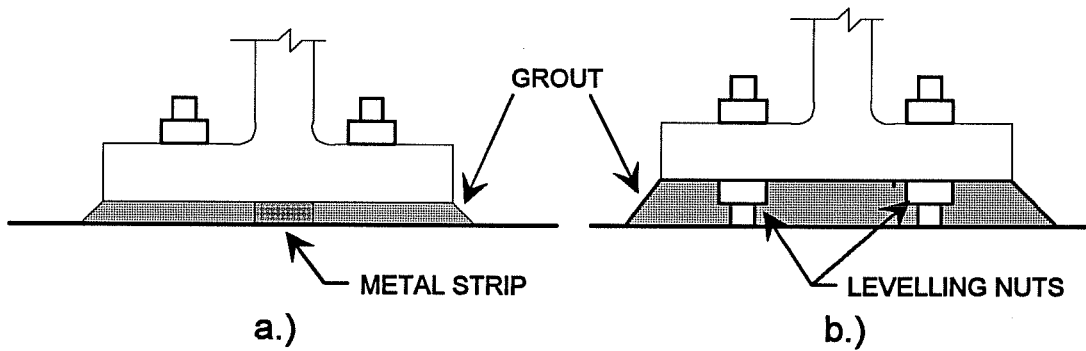


Figure 8.39 Setting of Bearing on Pier Cap

the hold-down bolts and anchor bolts, however, must be oversized to accommodate the movement. Once all of the stringers are in place the bearing can be grouted in. The second solution, shown in Figure 8.39b, is to use levelling nuts under the bearing. This will allow a more precise adjustment of the cap girder, but it may be difficult to turn the nuts if the dead load is too great. In addition, the flange will bend due to the axial load in the web and it must be checked.

The standard TxDOT connection simply sits on a 1/4 inch thick preformed fabric pad; no grout is used under the bearing plate. As a result, the groove in the sole plate and the rocker pin may be misaligned. This is why it seems unjustified that the surfaces are machined with such precision (see Section 5.2.2). A type of misalignment was noted in erecting the test specimen cap girder; it had a distorted bottom flange and as a result it could not be levelled properly without using a grouting material. The wide-flange bearings will not have perfectly parallel flanges, so it would be very difficult to get a level cap girder by setting the bearings only on preformed fabric pad. The grouting process mentioned above would take care of any misalignment problems.

Tensioning the Anchor Bolts

The initial intent for tensioning the anchor bolts necessary for an uplift resistant compression was to use a partial height stiffener and do the tensioning within the depth of the cap girder. The height necessary for the standard tensioning jacks, however, will preclude this option. It was necessary for the full scale tests to drill a hole through the top flange of the cap girder and do the tensioning from there (see Figure 8.9). A spacer had to be fabricated to prevent bending of the top flange. It is recommended that the anchor bolts extend through holes in the top flange and that the tensioning be done from that point. The stiffeners will have to extend the entire depth of the cap girder web and must bear against the top flange to prevent its bending. It was explained, however, in Chapter 5 that requiring a tight fit to both the top and bottom flange by one stiffener is time-consuming and expensive. Therefore, a stiffener detail is proposed that would provide a tight fit to both flanges but would use two sets of stiffeners instead of one. The detail is shown in Figure 8.40a. The long lower stiffeners would have a tight fit to the lower cap girder flange and resist the compressive bearing loads. The short upper stiffeners would bear against the top flange of the cap girder and resist the anchor bolt bearing forces. The upper stiffeners should be long enough to allow the stiffener forces to be transferred to the cap girder web through the weld. The upper stiffeners should also extend below the uppermost position of the clip angle connections to the longitudinal stringers; this will reduce the out-of-plane web flexure

of the cap girder above the termination of the lower stiffeners. For the connection in which no uplift is present and the anchor bolts are not needed, only the lower set of stiffeners is required and they should extend the full depth of the web, cut just short of the top flange. This is shown in Figure 8.40b. By extending the lower stiffener up to near the top of the top flange, the potential fatigue problem at the stiffener termination point is eliminated.

8.3.4 Design of Bolt Anchorage

The determination of the capacity of the bolt anchorage was not part of this research. It may be noted that there was no evidence of any problems in either the Phase I tests or the Phase II tests. There are two failure modes related to the anchorage capacity of the concrete; one is the pull-out capacity of the anchor bolt, in which the anchor bolt pulls out of the concrete. This leaves the concrete mostly intact. The second failure mode is the break-out capacity of the concrete, in which a bolt or group of bolts breaks out a cone of concrete. The first failure mode can be prevented by providing sufficient bearing area for the anchoring nut or plate. The second failure mode cannot occur if the anchor bolts are pretensioned and the pretension load is not exceeded. A procedure for determining the capacity of a bolt anchorage has been developed by Shipp and Haninger [17].

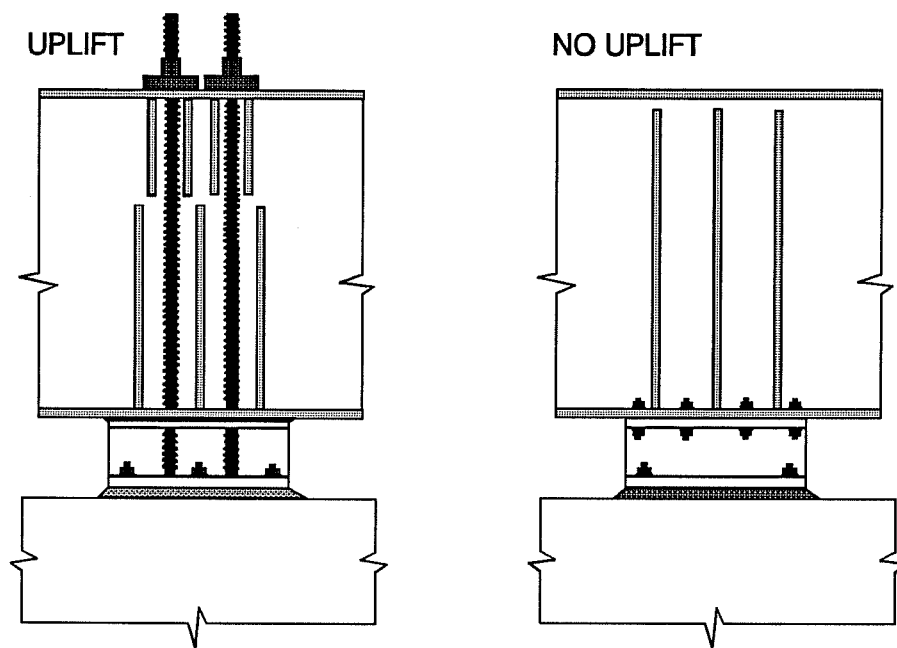


Figure 8.40 Recommended Connection Details

CHAPTER 9 : PHASE II CONCLUSIONS

9.1 CONCLUSIONS

Bearing Compression Tests

The bearing compression tests showed that the wide-flange web buckles essentially as column and that the buckling loads were in the inelastic and strain hardening range. A procedure was developed such that the web could be designed using a simple interaction equation by limiting the slenderness ratio of the web. The tests also showed that the design procedure that TxDOT uses for design of the bearing plates is conservative and not based on the actual bearing stress distribution. A method was developed for a more realistic design of the bearing plates, treating them more properly as a beam on an elastic foundation, that takes into account the actual bearing stress distribution.

Fatigue Tests

The fatigue tests showed that a wide-flange bearing subject to an out-of-plane shearing load could be treated as a category A detail. Because a category A detail rarely controls the design of a bridge member, fatigue should not be a concern in the new detail. The tests also showed that for calculating the out-of-plane shear stiffness of the bearing, the height of the bearing can be taken as $h = d - 2t_f$.

Longitudinal Direction Tests

The longitudinal direction tests showed that the rotation is a linear function of the moment and that the elements in the connection resist moment in proportion to their rotational stiffness. The tests showed that stresses developed at the stiffener termination point that may be high enough to cause a concern regarding distortion-induced fatigue cracking. Relatively simple equations were developed for calculating the forces and moments in the wide-flange sections that result both from longitudinal rotation and temperature.

Transverse Direction Tests

The transverse direction tests showed that the new connection may be used in situations in which uplift occurs. The connections behaved qualitatively as expected and that pretensioning of

the connections will reduce substantially the tensile stress range in the anchor bolts. There was no reduction in the compressive load carrying capacity of the bearing webs as a result of the eccentric loading produced by the cap girder. Pretensioning of the connections increased the rotational stiffness of the connection system.

9.2 COMPARISON WITH STANDARD TxDOT CONNECTION

Economic Comparison

A primary objective of the research was to develop a more cost-effective connection design. The cost of test specimens used in the Phase I and Phase II full scale tests were used as the basis of comparison. The following items were used in the cost estimates:

- Plate Girder
- Stiffeners
- Bearing Hardware
- Anchor Bolts and Anchorage
- Hold-Down Bolts

The cost of the Phase I connection, representing the standard TxDOT connection, given in both absolute and relative terms, is shown in Table 9.1. The connection details account for 64% of the connection cost. Of this, the stiffeners comprise 41% and the bearing hardware comprises 50% of the cost. It should be noted, however, that the cap girder used in the tests is about half the length that it would be in the field, so the percentages listed in table will be reduced for the connection details. If the costs are computed on the basis of dollars per pound, the following breakdown occurs:

● Bare Plate Girder :	\$ 0.78 / lb
● Anchor Bolts, Anchorage :	\$ 1.00 / lb
● Bearing Hardware :	\$ 1.40 / lb
● Stiffeners :	\$ 1.89 / lb
● Total :	\$ 1.13 / lb

The bearing hardware and the stiffeners are the most expensive components of the system on a per unit weight basis; this reflects the welding and machining necessary for the fabrication of the elements.

Table 9.1 : Cost Analysis of Phase I Connection

ITEM	ABSOLUTE COST	RELATIVE COST
Bare Plate Girder	2,978.29	36 %
Stiffeners Welded to Plate Girder	2,125.08	26 %
Bearing Hardware	2,615.91	32 %
2 Rocker Plates	669.97	26 %
2 Top Bearing Plates	968.47	37 %
2 Middle Bearing Plates	432.13	16 %
2 Bottom Bearing Plates	545.34	21 %
Anchor Bolts and Anchorage	529.41	6 %
8-1.5" ϕ A36 Galvanized rod	342.72	65 %
8 Nut-Washer Plates	137.25	26 %
24-1.5" ϕ Galvanized HH Nuts	49.44	9 %
Hold-Down Bolts	29.28	0 %
TOTAL \$8278		

The major difference between the Phase I connection and the Phase II connection is that a wide-flange bearing section replaces the bearing hardware and that the anchor bolts are not necessary for the new detail if uplift does not occur. The no uplift connection will be used as the basis of comparison. The upper stiffeners were also not present in the Phase II connection. The

cost of the Phase I stiffener detail was reduced by 30% to arrive at an estimate of the cost for the Phase II stiffener detail. The bearing used for the Phase II connection estimate was a two foot long W14X550, which has a web thickness (2.38") close to the thickness of the rocker pin used in the Phase I connection. It was assumed that the section cost \$0.50 per pound. The cost analysis of the Phase II connection is shown in Table 9.2. The analysis shows that new detail is less expensive than a comparable standard TxDOT detail. The major savings comes from the replacement of the bearing hardware with the wide-flange section, a reduction in cost from \$2616 to \$1100, from \$1.40 per pound to \$0.50 per pound.

Table 9.2 : Cost Analysis of Phase II Connection

ITEM	ABSOLUTE COST	RELATIVE COST
Bare Plate Girder	2,978.29	53 %
Stiffeners Welded to Plate Girder 70% of Phase I	1478.50	27 %
Bearing Hardware	1100.00	20 %
2- W14X550 @ \$0.50 / lb	1100.00	20 %
Hold-Down Bolts	29.28	0 %
TOTAL \$5586		

General Comparison

The advantages of the new detail are that it

- can be used to resist uplift
- is redundant with respect to uplift (both the wide-flange bearing and the anchor bolts (threadbars) can resist uplift)
- needs no anchor bolts if uplift does not occur
- is less expensive and less labor intensive to fabricate

The major disadvantage of the new detail is that it cannot simply be set on the pier cap as is the standard TxDOT detail. It must be grouted in place.

With regard to the performance aspects of the two connections, the new detail is obviously superior to the standard TxDOT connection in that it can resist uplift. It may, however, become impractical to use the new detail for connections that must resist extremely high compression loads. The maximum web thickness of a rolled section is limited, but the rocker pin can be made in almost any thickness.

CHAPTER 10 : DESIGN PROCEDURES

10.1 DESIGN PROCEDURE, NO UPLIFT

There are three basic steps in designing the new detail; determine the loads on the bearings, design the wide-flange bearing section, and design the stiffener detail. Design examples for the new connection detail are given in Appendix E. **All length units are inches and all force units are kips (unless otherwise specified).**

10.1.1 Determine the Loads on the Bearings

There are six load groups that are relevant to the design of a cap girder and its bearings; they are listed in Table 10.1. A description of the loads is given in Section 1.4.2. The first column shows the load group number as given in Table 3.22.1A of the AASHTO code; the second column shows the direction of the load; the third column shows the load combinations for each load direction; the fourth column shows the percent of the basic unit stress that may be used in the design. Load Group I is the basic gravity load case; Load Group II checks overturning due to wind load; Load Group III is Group I plus the wind loads, with an increase in the allowable stresses of 25%; Load Groups IV, V, and VI include the load combinations of the first three groups plus temperature, with a larger increase in the allowable stress.

The determination of the various loads was illustrated in Appendix A and will not be repeated here. The only load that cannot be determined prior to the beginning of the design process is the horizontal force due to temperature change. The horizontal shear stiffness of the bearing is needed before the temperature-induced force can be determined.

Table 10.1 : Load Combinations

LOAD GROUP	LOAD DIRECTION	LOAD COMBINATIONS	% BASIC UNIT STRESS
I	VERTICAL	$D + (L+I)_n + CF$	100
	LONGITUDINAL	-	
	TRANSVERSE	CF	
II	VERTICAL	$D + W$	125
	LONGITUDINAL	-	
	TRANSVERSE	-	
III	VERTICAL	$D + (L+I)_n + CF + 0.3W + WL$	125
	LONGITUDINAL	$0.3W + WL + LF$	
	TRANSVERSE	$CF + 0.3W + WL$	
IV	VERTICAL	$D + (L+I)_n + CF$	125
	LONGITUDINAL	T	
	TRANSVERSE	CF	
V	VERTICAL	$D + W$	140
	LONGITUDINAL	T	
	TRANSVERSE	-	
VI	VERTICAL	$D + (L+I)_n + CF + 0.3W + WL$	140
	LONGITUDINAL	$0.3W + WL + LF + T$	
	TRANSVERSE	$CF + 0.3W + WL$	

10.1.2 Design of Wide-Flange Bearing

Size the Bearing Web

- 1.) Determine the section size (W12 or W14) and establish the minimum web thickness.

$$t_w \geq \frac{h \sqrt{F_y}}{60} \quad (10.1)$$

- 2.) Size for Group I loading, axial load only.

$$f_a = \frac{P}{b t_w} \leq 0.472 F_y \quad (10.2)$$

Check Design for Remaining Load Cases

- 1.) Calculate longitudinal direction stiffness constants

$$k_H = \frac{(EI)_c}{\frac{L_c^2 h}{2} + h^2 L_c + \frac{L_c^3}{3} + \frac{h^3}{3} \left(\frac{(EI)_c}{(EI)_b} \right)} \quad (10.3)$$

$$k_M = \frac{(EI)_c}{L_c h + \frac{L_c^2}{2} + \frac{h^2}{2} \left(\frac{(EI)_c}{(EI)_b} \right)} \quad (10.4)$$

$$\beta_M = \frac{(EI)_c}{L_c + h \left(\frac{(EI)_c}{(EI)_b} \right)} \quad (10.5)$$

- 2.) Determine horizontal displacement due to temperature change.

$$\Delta_T = \frac{L_{cs}}{2} (.0003) \quad (10.6)$$

•The term 0.003 represents the temperature change, 50°F, times the coefficient of thermal expansion, 6.5E-6 / °F.

3.) Calculate maximum bending stress in bearing due to temperature change.

$$M = \frac{\beta_M \Delta_T \left(\frac{k_H}{k_M} \right)}{\left(1 - \frac{k_H \beta_M}{k_M^2} \right)} \quad (10.7)$$

$$f_b = \frac{M}{S_{bearing}} \quad (10.8)$$

4.) Calculate maximum bending stress in bearing due to LF (longitudinal force).

$$M = LF \frac{\beta_M}{k_M} \quad (10.9)$$

$$f_b = \frac{M}{S_{bearing}} \quad (10.10)$$

5.) Check interaction equations and resize bearing if necessary.

$$\text{Group III} : \frac{f_a}{0.472F_y} + \frac{f_b}{0.55F_y} \leq 1.25 \quad (10.11)$$

$$\text{Group IV} : \frac{f_a}{0.472F_y} + \frac{f_b}{0.55F_y} \leq 1.25 \quad (10.12)$$

$$\text{Group V} : \frac{f_a}{0.472F_y} + \frac{f_b}{0.55F_y} \leq 1.40 \quad (10.13)$$

$$\text{Group VI} : \frac{f_a}{0.472F_y} + \frac{f_b}{0.55F_y} \leq 1.40 \quad (10.14)$$

Check Bottom Flange for Bearing

- 1.) Determine length of bearing flange effective

$$l_{eff} = 3.25 \sqrt{\frac{E_s}{E_c} \left(\frac{P}{0.55F_y b} \right)^{0.75}} \quad (10.15)$$

- 2.) Check flange thickness

$$t_f \geq \sqrt{\frac{Pl_{eff}}{F_y b}} \quad (10.16)$$

- 3.) Check bearing stress on concrete

$$f_{bc} = \frac{P}{l_{eff} b} \leq f'_c \quad (10.17)$$

Check Fatigue of Bearing

- 1.) Determine Stress Range

$$M = \beta_M \theta_r \left[\frac{\left(1 + \frac{y k_H}{k_M} \right)}{\left(1 - \frac{k_H \beta_M}{k_M^2} \right)} \right] \quad (10.18)$$

$$S_r = \frac{M}{S_{bearing}} \quad (10.19)$$

- 2.) Check Stress Range Against Allowable

$$2.2S_t < S_c \quad , \quad 1.1S_r < S_{rp} \quad (10.20)$$

- θ_r is the rotation range caused by the live load (or fatigue truck, see below), S_t is the tensile stress range (check against θ_t , rotation range that produces tensile stress range), S_c is the compressive dead load stress, S_r is the total stress range caused

by θ_r and S_{rp} is the permissible stress range (category A detail).

- Equation 10.20 from "Guide Specifications for Fatigue Design of Steel Bridges," AASHTO, 1989. [1]

10.1.3 Design Bearing Stiffeners

This procedure is well established and has not been modified for the new detail. See Appendix A for sample calculations. The stiffeners should extend the full depth of the cap girder web, cut just short of the top flange.

10.1.4 Connection of Bearing to Pier Cap and Cap Girder

Since there is no uplift on the bearing the connection details used for the standard TxDOT connection should suffice. The erection and grouting procedures were explained in Section 8.3.3.

10.2 DESIGN PROCEDURE FOR UPLIFT

Prior to the design of the uplift connection, the bearing size should be determined using the procedure described in Section 10.1.

10.2.1 Determine the Uplift Load

Uplift Force (T_{up}) Calculated as Largest of:

- 1.) 100% of calculated uplift, with live load plus impact increased by 100%
- 2.) 150% of calculated uplift at working load levels
 - the uplift resisting elements are designed at 150% of allowable basic stress
- 3.) For fatigue loads, use 100% of live load (or fatigue truck [1])

10.2.2 Design Uplift Resisting Elements

Hold-Down Bolts

$$T_a = 1.5 n (0.55 F_y A_t) \geq T_{up} \quad (10.21)$$

Wide-Flange Bearing

$$T_a = 1.5 (0.55 F_y b t_w) \geq T_{up} \quad (10.22)$$

Anchor Bolts

$$T_a = 1.5 n (0.66 f_{pu} A_{ps}) \geq T_{up} \quad (10.23)$$

Connection Between Bearing Top Flange and Cap Girder

- Recommend using fillet weld all-around, as is done with the standard TxDOT detail. Size weld to resist uplift load T_{up} .

Design Anchorage

- 1.) Recommend using "Design of Headed Anchor Bolts" by J.G. Shipp and E.R. Haninger, Engineering Journal, AISC, Second Quarter 1983. [17]
- 2.) Check capacity and development length of vertical steel.

10.2.3 Determine Pretension Force

$$B_0 = T_D + T_L$$

- T_D is the working dead load uplift force, T_L is the working live load uplift force

10.2.4 Check Fatigue of Anchor Bolts

Calculate Connection Stiffnesses

1.) Anchor Bolt

$$k_b = \frac{A_{ps} E}{L_b} \quad (10.24)$$

- L_b is the free length of the anchor bolt plus one-half of the embedded length

2.) Cap Girder and Stiffeners

$$k_{gird} = \frac{E (A_{st} + A_{web})}{h_w} \quad (10.25)$$

3.) Wide-Flange Bearing

$$k_{bear} = \frac{b t_w E_s}{h} \quad (10.26)$$

4.) Concrete

$$k_{conc} = \frac{(A_{bear} + A_{anch}) E_c}{2 L_d} \quad (10.27)$$

5.) Concrete Under Bearing Plate

$$k_{cpl} = 0.3 E_c \sqrt{l_{eff}} b \quad (10.28)$$

6.) Total Compression Element Stiffness

$$k_c = \left[\frac{1}{k_{gird}} + \frac{1}{k_{bear}} + \frac{1}{k_{conc}} + \frac{1}{k_{cpl}} \right]^{-1} \quad (10.29)$$

Transverse Direction

$$T_r = n A_{ps} S_{rp} \left[1 + \frac{k_c}{k_b} \right] > T_L \quad (10.30)$$

- T_L is the working live load uplift; the live load uplift from the fatigue truck may also be used (see Guide Specs cited above)

Longitudinal Direction

$$S_r = \frac{d k_b \theta_r}{A_{ps}} < S_{rp} \quad (10.31)$$

- Check Against Category E detail at 2 million cycles (for truck loading)
- d is distance from centerline of bolt to centerline of cap girder web

Since holes must be drilled in the top flange of the cap girder to accommodate the anchor bolts, it should be checked for fatigue as well, as a category B detail.

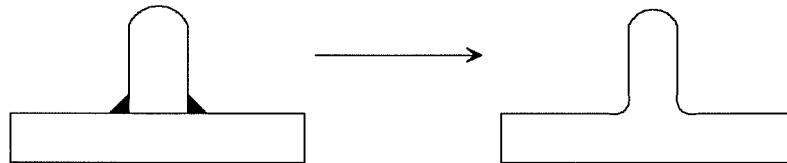
10.2.5 Design Bearing Stiffeners

- 1.) Use split stiffeners that bear against the top and bottom flanges of cap girder. Length of top stiffener should be sufficient to transfer load through weld to cap girder. Upper stiffeners should also extend below the top of the clip angle connection.
- 2.) Lower stiffeners should extend to bottom of upper stiffeners.

10.3 DESIGN PROCEDURE FOR STANDARD TxDOT CONNECTION

10.3.1 Design of Bearing Plates

The design procedure for the standard TxDOT connection is modified from the current design procedure by designing the bearing plates using Equations 10.15, 10.16, and 10.17. The top bearing and rocker pin may also be replaced with the half-section of a wide-flange, as shown below.



10.3.2 Design of Neoprene Bearing Pad

The neoprene bearing pad used in series with the anchor bolt can be sized based on the pretension force (using turn-of-the-nut method) in the bolt. It is recommended that a 1 inch thick, 70 durometer pad be used.

Determine Stiffness of Bearing Pad-Anchor Bolt System

1.) Determine Compressive Stiffness of Bearing Pad

$$E_p = 3G (1 + 2CS^2)$$

- G is the shear modulus, use 150 psi for 70 durometer pad
- C is a constant that is 0.55 for a 70 durometer pad
- S is the shape factor, compressed area divided by area free to bulge
- E_p is given in psi; estimate size of pad to determine shape factor

$$k_p = \frac{A_p E_p}{t_p}$$

- t_p is the thickness of pad, use 1 inch

2.) Determine Axial Stiffness of Anchor Bolt

$$k_b = \frac{A_b E_s}{L_b}$$

- L_b is free length of anchor bolt plus one-half of embedded length

Determine Pretension Force in Anchor Bolt

$$B_0 = \frac{\text{number of turns past snug / pitch}}{\left[\frac{1}{k_p} + \frac{1}{k_b} \right]}$$

- B_0 is the pretension force in one anchor bolt

Check Stress in Bearing Pad

$$f_a = \frac{B_0}{A_p} \leq 880 \text{ lbs per square inch}$$

- allowable stress in non-reinforced pad with no shear is 880 psi (AASHTO 14.4.1.1)

CHAPTER 11 : SUMMARY AND CONCLUSIONS

11.1 PROBLEM STATEMENT

At congested highway interchanges, horizontal and vertical clearance requirements may dictate the use of narrow piers and shallow depth cap girders to accommodate the various roadways and overpasses. In situations such as this the state of Texas uses horizontally curved steel plate girders as the bridge structural system, supported on integral steel cap girders at single column piers. Two bearings are used to connect the steel cap girder to the concrete pier. Due to the narrow pier, unbalanced loading may produce a transverse overturning moment at the pier; the bearings resist this moment by developing a couple, with one bearing loaded in compression and one bearing loaded in tension. When the unbalanced loading is caused by truck traffic, which is cyclic, the bearing resisting the uplift is subject to fatigue loading.

The standard connection used by the state of Texas is an in-house design that comprises a line rocker bearing, which accommodates the longitudinal direction rotation, and embedded anchor bolts that are used to both resist potential uplift and to provide a positive connection from the cap girder to the pier. The anchor bolt stress that develops due to the longitudinal direction rotation is reduced by placing a bearing pad in series with each anchor bolt. The behavior of this connection is not well understood and the detailing is complex. The objectives of the research were to examine and categorize the behavior of the existing connection and to develop a new detail that is simpler and cost-effective.

This research produced a new detail for the connection between an integral steel cap girder and a concrete pier and provided a better understanding of the behavior of the current standard detail. The primary research method was the experimental testing of large scale connection models, which was supplemented with mathematical modelling and finite element analyses.

11.2 PHASE I SUMMARY AND CONCLUSIONS

Summary

The first phase of the research centered on the experimental testing of the standard connection detail. Seven tests were conducted in both the longitudinal (direction parallel to traffic

flow) and transverse (direction perpendicular to traffic flow) directions. The variable in the tests was the type of bearing pad used in series with the anchor bolt; the purpose of the bearing pad is to increase the longitudinal direction flexibility of the connection. Mathematical models were developed to describe the behavior of the connection. Methods were developed for easily determining the longitudinal rotation at the pier supports, for calculating the rotational stiffness of the connection and the forces produced in the elements of the connection as a result of the rotation, and for estimating the pretensioning force.

Conclusions

The major conclusions of the first phase of testing are listed below.

- 1.) The standard connection detail behaves as expected in the longitudinal direction. The stresses produced in the anchor bolts due to the longitudinal rotation are typically much less than the allowable tensile stress range of 8 ksi if a neoprene bearing pad is used in series with each anchor bolts.
- 2.) The standard connection detail should not be used to resist uplift. The connection flexibility necessary for the longitudinal direction adversely affects the performance of the connection in the transverse direction. Since some portion of the uplift force comes from the cyclic truck loading, the uplift bearing is subject to fatigue. The fatigue problem may be solved by pretensioning the anchor bolts, but this option is precluded because of the connection flexibility cited above.
- 3.) A cost-breakdown from the fabricator showed that the rocker bearing is responsible for a significant portion of the cost of the cap girder and the connection.

11.3 PHASE II SUMMARY AND CONCLUSIONS

Summary

Based on the results of the first phase of the research, a new connection detail was developed that addressed the deficiencies of the standard connection. The first deficiency was that the standard connection may not be used in situations in which uplift occurs. This problem was solved by replacing the threaded anchor bolts with high-strength threadbar. Since threadbar is specifically designed for pretensioning, the damaging tensile stress ranges that the uplift connection would be subject to can be reduced. The second deficiency was the cost and labor intensive fabrication requirements of the rocker element. This problem was solved by replacing the rocker with a wide-flange bearing section. A significant decrease in the cost of the connection was realized by this change.

The second phase of the research was focused on three areas; determination of the compressive strength of the wide-flange bearing web, establishment of the fatigue characteristics of the wide-flange bearing when subject to an out-of-plane shearing force, and the performance of the connection system in the transverse and longitudinal directions. Nine compression tests were conducted on three different wide-flange sections, W12X87, W12X152, and W12X230. The specimens were either two feet or three feet wide and the failure loads ranged from 450 kips for a two foot wide W12X87 to 1850 kips for a two foot wide W12X230. The same section sizes were used for the fatigue tests. Ten tests were conducted on specimens that were twelve inches wide; in six of the tests, two specimens were tested simultaneously and in four of the tests one specimen was loaded. Two sets of full scale tests were conducted on connection systems using the W12X87 section and the W12X152 section as bearings. Longitudinal direction and transverse direction tests were conducted in a manner similar to the large scale tests conducted in the first phase of the research. The behavior of the system with and without pretensioned anchor bolts was examined. The transverse direction tests were taken to failure.

Conclusions

The major conclusions of the second phase of the research are listed below.

- 1.) The bearing compression tests showed that the wide-flange web buckles essentially as column and that the buckling loads were in the inelastic

and strain hardening range. A procedure was developed such that the web could be designed using a simple interaction equation by limiting the slenderness ratio of the web. The tests also showed that the design procedure that TxDOT uses for design of the bearing plates is conservative and not based on the actual bearing stress distribution. A method was developed for a more realistic design of the bearing plates, treating them more properly as a beam on an elastic foundation, that takes into account the actual bearing stress distribution.

- 2.) The fatigue tests showed that a wide-flange bearing subject to an out-of-plane shearing load could be treated as a category A detail. Because a category A detail rarely controls the design of a bridge member, fatigue should not be a concern in the new detail. The tests also showed that for calculating the out-of-plane shear stiffness of the bearing, the height of the bearing can be taken as $h = d - 2t_f$.
- 3.) The longitudinal direction tests showed that the rotation is a linear function of the moment and that the elements in the connection resist moment in proportion to their rotational stiffness. The tests showed that stresses developed at the stiffener termination point that may be high enough to cause a concern regarding distortion-induced fatigue cracking. It was recommended that the stiffeners extend the full depth of the cap girder web, cut just short of the top flange. Relatively simple equations were developed for calculating the forces and moments in the wide-flange sections that result both from longitudinal rotation and temperature.
- 4.) The transverse direction tests showed that the new connection may be used in situations in which uplift occurs. The connections behaved qualitatively as expected and that pretensioning of the connections will reduce substantially the tensile stress range in the anchor bolts. There was no significant reduction in the compressive load carrying capacity

of the bearing webs as a result of the eccentric loading produced by the cap girder.

11.4 FUTURE RESEARCH

The following are areas in which research is needed.

Fatigue strength of high-strength threadbar : No comprehensive research has been done in this area and the data would be useful for connections that are subject to uplift and require pretensioned high-strength threadbar.

Bearing plate design and concrete bearing strength : Simple methods were developed to design the bearing plates based on a few experimental results. More research is needed to verify the design methods proposed and to determine more accurately the bearing stress distribution in the concrete.

Flexural stiffness of concrete pier column : The horizontal forces developed in the bearings are very sensitive to the length of the supporting pier column. In design the length of the pier column is increased to account for the flexibility of the foundation. The determination of this added length is at the discretion of the engineer. A more accurate method for determining this foundation flexibility is needed.

Behavior of cap girder web : The interaction of the bearing stiffeners and the web when it is subject to out-of-plane rotation needs further study. Some preliminary finite element analysis studies were conducted to identify fatigue sensitive areas at the termination points of partial height stiffeners. A more comprehensive study should be done to understand more fully the behavior of the cap girder web and its susceptibility to distortion-induced cracking caused by the rotation of the longitudinal stringers.

APPENDIX A : DESIGN EXAMPLE FOR STANDARD TxDOT CONNECTION

The following example has been adapted from a design by an engineer at the Texas Department of Transportation. The parameters of the bridge are listed below:

- Three span continuous plate girder bridge : $160' - 208' - 180' = 548'$ total length
- Radius of curvature is 1910'
- Seven longitudinal girders spaced at 8' - 8", composite with concrete deck
- Bearings at pier supports spaced at 12'
- Pier Columns are 18' high with a 12' X 4' cross - section

A plan view and cross - section view of the bridge system is shown in Figure 1.* This example focuses on the bearing connection design; the design of the cap girder itself is not covered, nor is the design of the concrete pier cap. A schematic showing the location of the forces acting on the cap girder and bearings are is shown in Figure A.1. The sections of the AASHTO bridge code relevant to each step are listed at the far right.

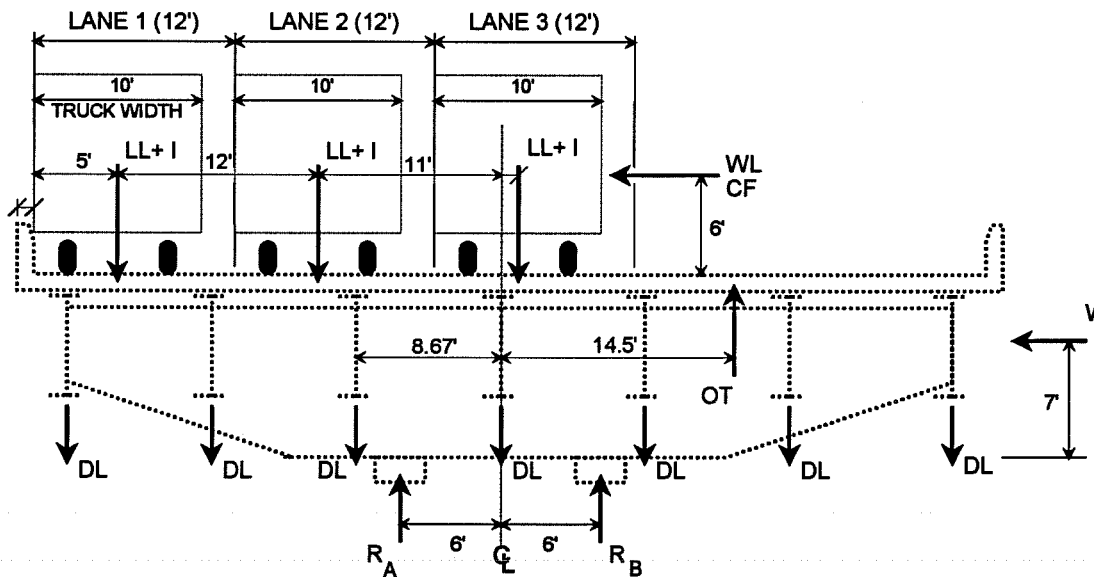


Figure A.1 Loads on Cap Girder

Determine the Loads on the Cap Girder and Bearings

DEAD LOAD

1.) Reactions from the longitudinal girders, including the V - loads

GIRDER	LOAD (kips)	DISTANCE TO CL (feet)	MOMENT (ft - kips)
1	299	26.00	7,774.00
2	297	17.33	5,147.01
3	294	8.67	2,548.98
4	284	0.00	0.00
5	282	-8.67	-2,444.94
6	278	-17.33	-4,817.74
7	276	-26.00	-7,176.00
Σ	2010		1,031.31

2.) Assumed self - weight of cap girder

- 72" X 1" web; 24" X 2" flanges; 52' long

$$D_{\text{cap}} = \{ 2 (72 \cdot 1 + 2 \cdot 24) / 144 \} \cdot 490 \cdot 52 = 30 \text{ kips}$$

LIVE LOAD

# OF LANES	LOAD (kips)	DISTANCE TO CL (feet)	MOMENT (ft - kips)
1	211	23	4,853
2	422	23,11	7,174
3	570	23,11,-1	6,963

TRANSVERSE LOADS

1.) Centrifugal Forces

AASHTO '92 : 3.10.1

$$C = \frac{6.68 S^2}{R}$$

where S = truck speed in miles per hour

where R = radius of curve in feet

- $F_{CF} = C * (\text{truck weight}) = (6.68 * 70^2 / 1910) * 72 = 12.2 \text{ kips / truck}$
- $M_{CF} = F_{CF} * (\text{distance to top of bearings})$

# OF LANES	LOAD (kips)	DISTANCE TO BEARINGS (feet)	MOMENT (ft - kips)
1	12.2	14.5	176.90
2	24.4	14.5	353.80
3	33.0	14.5	478.50

WIND LOAD

- 1.) On Superstructure in Transverse Direction (W_T) AASHTO '92 : 3.15.2.1.1

$$W_T = 50 \text{ psf}$$

which acts on the depth of the superstructure

- $F_{WT} = h * L * W_T = 10 * 194 * 0.05 = 97 \text{ kips}$
- $M_{WT} = F_{WT} * h_b = 97 * 7 = 679 \text{ ft - kips}$

- 2.) On Superstructure in Longitudinal Direction (W_L) AASHTO '92 : 3.15.2.1.3

$$W_L = 12 \text{ psf}$$

which acts on the depth of the superstructure

- $F_{WL} = h * L * W_L = 10 * 194 * 0.012 = 23.3 \text{ kips}$

- 3.) On Live Load in Transverse Direction (WL_T) AASHTO '92 : 3.15.2.1.2

$$WL_T = 100 \text{ plf}$$

which acts on an assumed line of trucks

- $F_{WLT} = L * WL_T = 194 * 100 = 19.4 \text{ kips}$
- $M_{WLT} = F_{WLT} * (\text{distance to top of bearings}) = 19.4 * 14.5 = 281 \text{ ft - kips}$

4.) On Live Load in Longitudinal Direction (WL_L) AASHTO '92 : 3.15.2.1.3

$$WL_L = 40 \text{ plf}$$

which acts on an assumed line of trucks

- $F_{WLL} = L * WL_L = 194 * 40 = 7.76 \text{ kips}$

5.) On Underside of Bridge (OT) AASHTO '92 : 3.15.3

$$OT = 20 \text{ psf}$$

which acts \uparrow on windward side at 1/4 point of deck

- $F_{OT} = D_W * L * OT = 58 * 194 * 20 = 225 \text{ kips}$

- $M_{OT} = F_{OT} * D_W * 0.25 = 225 * 58 * 0.25 = 3263 \text{ ft - kips}$

LONGITUDINAL FORCES

1.) Truck Braking Forces (LF) AASHTO '92 : 3.9

$$LF = 5 \%$$

which acts on the weight of a lane of trucks

- $F_{LF} = LL * LF * n = 182 * 0.05 * n = 9.1n \text{ kips}$
 $= 9.1 \text{ kips (1 lane)}, 18.2 \text{ kips (2 lanes)}, 24.6 \text{ kips (3 lanes)}$

2.) Temperature (T) AASHTO '92 : 3.16

$$\delta_l = \alpha \Delta T = 0.0000065 / ^\circ F (50^\circ) = 3.25 E^{-4} \text{ ft / ft}$$

which acts on the length of the center span

The horizontal force on the bearings is determined by multiplying the flexural stiffness of the column by the displacement of the column. The displacement of the column is:

$$\Delta_{pc} = \delta \frac{L_{cs}}{2} = 3.25E-4 \frac{208}{2} = .0338 \text{ ft}$$

The flexural stiffness of the column is shown below. The length of the pier column includes the depth of the pier cap and is increased by ten feet to account for the base flexibility and the modulus of elasticity is decreased by 60% to account for creep.

$$k_{flex} = \frac{1.2 (EI)_{pc}}{(L_{pc} + L_{base})^3} = \frac{1.2 (3420 * 1327104)}{(261 + 120)^3} = 98.5 \text{ k / inch}$$

- $F_T = k_{flex} \Delta = 98.5 * .0338 * 12 = 40 \text{ kips}$

LOADS ON BEARINGS

AASHTO '92 : Table 3.22.1A

LC	DL	LL+I	CF	W	OT	WL	T	LF	Σ	RA	RB
β	1.0	1.0	1.0	0.3	0.3	1.0	1.0	1.0	-	-	-
VERTICAL LOADS											
1P	2040	211	-	-	-68	-	-	-	2183	1718	465
1M	1031	4853	177	204	979	275	-	-	7519	-	-
2P	2040	422	-	-	-68	-	-	-	2394	2032	362
2M	1031	7174	354	204	979	275	-	-	10017	-	-
3P	2040	570	-	-	-68	-	-	-	2542	2099	443
3M	1031	6963	479	204	979	275	-	-	9931	-	-
HORIZONTAL TRANSVERSE LOADS											
3V	-	-	33	29	-	19	-	-	81	41	41
HORIZONTAL LONGITUDINAL LOADS											
3V	-	-	-	7	-	8	40	25	80	40	40

The β factor is a reduction factor to account for the improbability of the maximum wind occurring simultaneously with the other loads. The loads and moments are listed as iP for vertical load, iM for overturning moment, and iV for horizontal load where i is the number of lanes of live load.

Design of the Bearing Plates and Pin

Lower Bearing Plate

- 1.) Calculate Allowable Bearing Stress on Concrete **AASHTO '92 : 8.15.2.1.3**

$$f_b = 0.3 f'_c \sqrt{A_2 / A_1} \leq 0.3 f'_c \sqrt{2}$$

- let $f'_c = 3600$ psi; $A_2 = 48$ in²; $A_1 = 36^2$
- $f_b = 0.3 (3.6) \sqrt{(48^2 / 36^2)} = 1.44$ ksi

- 2.) Determine Size of Plate

- $L = \sqrt{(P_{\max} / f_b)} = \sqrt{(2099 / 1.44)} = 38.2$ inches
- $f_b = 0.3 (3.6) \sqrt{(48^2 / 38^2)} = 1.36$ ksi
- $L = \sqrt{(2099 / 1.36)} = 39.3$ inches → use 39" X 39" plate

Middle and Upper Bearing Plates

- 1.) Assume Three Stepped Plates (shown in Figure A.2)

- $P_{\max} = 2099$ kips; $L = 39$ "
- $w = 2099 / 39^2 = 1.38$ kips / inch
- $M = 1.38 (39^2) / 4 = 262$ inch - kips

The bearing plates are assumed to resist a uniformly distributed load.

- 2.) Determine Allowable Bending Stress

$$F_B = 27 \text{ ksi for 50 ksi steel}$$

AASHTO '92 : Table 10.32.1A

3.) Determine Thickness

- $S = M / F_b = 262 / 27 = 9.70 \text{ in}^3$

- $t_{\text{req}} = \sqrt{(6S)} = \sqrt{(6*9.7)} = 7.63 \text{ inches}$

→ use 3 plates @ 2.5"

→ step plates 2.25" on each side

The plates are treated as a monolithic unit.

4.) Check Cantilever Bending at Point A

- $w = 2099 / (39*34.5) = 1.56 \text{ k / in}$

- $M = 1.56 (2.25)^2 / 2 = 3.95 \text{ inch - kips}$

- $S = 3.95 / 27 = 0.15 \text{ in}^3$

- $t_{\text{req}} = \sqrt{(0.15*6)} = 0.94 \text{ in} < 2.5 \text{ in, O.K.}$

Design of Rocker Pin (see Figure A.3)

1.) Determine Allowable Bearing Stress on Pin AASHTO '92 : Table 10.32.1A

$$F_b = 20 \text{ ksi for 50 ksi steel}$$

2.) Determine Thickness

- $t_{\text{req}} = 2099 / (39*20) = 2.7 \text{ inches}$

→ use 3.5"

3.) Check Bending due to Horizontal Force

- $M = 53*6 = 318 \text{ inch - kips}$

- $S = 39*3.5^2 / 6 = 11.78 \text{ in}^3$

- $f_b = 318 / 11.78 = 4 \text{ ksi} < 27 \text{ ksi, O.K.}$

4.) Check Weld Assuming 0.5" Groove Weld on Both Sides

- $R_w = 0.3*70*1.0 = 21 \text{ k / in}$

- $f_v = 53 / 39 = 1.36 \text{ k / in} < 21 \text{ k / in, O.K.}$

Sizing of Cap Girder Web

1.) Determine Maximum Shear on Web

- $V_{\max} = \{299+297+294\} \text{DL} + \{30*20/52\} \text{CAP} + \{211*1.5\} \text{LL} = 1218 \text{ kips}$

2.) Determine Required Thickness

- Assume depth = 84" and no transverse stiffeners will be used

$$F_v = \frac{7.33E7}{(D/t_w)^2} \leq \frac{F_y}{3}$$

AASHTO '92 : 10.34.4.1

- $f_v = 1218000 / (84*t_w) = 7.33E7 / (84/t_w)^2 : t_w = 1.12 \text{ inches use } 1.25"$
- $f_v = 1218 / (84*1.25) = 11.6 \text{ ksi} < F_y / 3 = 50 / 3 = 17 \text{ ksi, O.K.}$

Design Bearing Stiffeners (see Figure A.4)

1.) Determine Minimum Thickness to Preclude Buckling

AASHTO '92 : 10.34.6.1

$$t_{\min} = \frac{b'}{12} \sqrt{\frac{F_y}{33,000}}$$

- $t_{\min} = (13.5/12)\sqrt{(50000/33000)} = 1.38 \text{ inches} \rightarrow \text{use } 1.5"$

2.) Calculate Properties

- Area = $(13.5*1.5*6) \text{ST} + (2*14+2*9*1.25)*1.25 \text{WEB} = 184.625 \text{ in}^2$
- $I_{XX} = (1/12)*4.5*28.25^3 = 8454 \text{ in}^4$
- $r = \sqrt{(8454/184.625)} = 6.77 \text{ in}$
- $Kl/r = 0.75*84.5/6.77 = 9.37 \text{ in}$

3.) Determine Compression Capacity

AASHTO 10.32.1.A

$$F_a = 23580 - 1.03 \left(\frac{Kl}{r} \right)^2$$

- $P_{cr} = F_a A = (23580 - 1.03 * 9.37^2) * 184.625 = 23.5 * 184.625 = 4339 \text{ kips} > 2099 \text{ kips}$, O.K.

4.) Check Stiffener Bearing Stress

AASHTO '92 : 10.32.1.A

$$F_b = 40 \text{ ksi for } 50 \text{ ksi steel}$$

- $f_b = 2099 / (6 * 12 * 1.5) = 19.5 \text{ ksi} < 40 \text{ ksi}$, O.K.

The 1.5" cope reduced stiffener width from 13.5" to 12".

5.) Check Stiffener to Web Weld

AASHTO '92 : 10.3.2

$$F_v = 19 \text{ ksi for } 70 \text{ ksi weld material}$$

- assume minimum fillet weld; size = 5/16"

- $f_v = 2099 / (12 * 76 * .707 * .3125) = 10.42 \text{ ksi} < 19 \text{ ksi}$, O.K.

Size Anchor Bolts and Related Plates

1.) Minimum Anchor Bolt Diameter for Spans over 100 feet

AASHTO '92 :10.29.6.2

$$d_b = 1.5 \text{ inches}$$

- Anchor bolts embedded $20d_b = 20(1.5) = 30 \text{ inches}$

- Use A193 B7 steel rod, $F_y = 105 \text{ ksi}$

- Use 4 anchor bolts and 4 hold - down bolts per connection

The above specifications are typical. Since uplift will not occur, the anchorage need not be checked.

2.) Design Cross Plate

The cross plate is designed as a fixed - fixed beam with a point load at the center.

The magnitude of the load is equal to the allowable tensile stress of the anchor bolt.

AASHTO '92 : 10.32.1.A

$$f_a = 0.55 F_y$$

- $B_a = A_t (0.55F_y) = 1.41 \cdot 0.55 \cdot 105 = 81$ kips
- $M = PL/8 = 81 \cdot 14 / 8 = 141$ in-kips
- $S_{req} = 142 / 27 = 5.26 \text{ in}^3$: $t_{req} = \sqrt{(6 \cdot 5.26 / 14)} = 1.50$ inches → use 1.5" plate

3.) Size Bearing Pad

Design bearing pad to resist allowable tensile load of anchor bolt.

- Use 1 inch thick preformed fabric pad (*typical*)
- Allowable stress = 2000 psi
- $A_{req} = (81/2) = 40.5 \text{ in}^2$: $t = \sqrt{40.5} = 6.36$ in → use 6.5" X 6.5" pad

4.) Size Nut - Washer Plate

*Determine thickness in manner similar to bearing plate design. See Figure 1.**

- $w = 81 / 6.25^2 = 1.91 \text{ k / in / in}$: $M = 1.91 \cdot 3.25^2 / 2 = 10.09$ in - kips
- $S_{req} = 10.09 / 27 = 0.37 \text{ in}^3$: $t_{req} = \sqrt{(6 \cdot 0.37)} = 1.49$ inches → use 1.5" plate

5.) Design Anchorage

The anchorage comprises a template to hold the anchor bolts and hold - down bolts in place and two heavy hex nuts to hold the bolts in place.

Design Schematics of the cap girder and the connection detail are given in Figure A.5, A.6, and A.7.

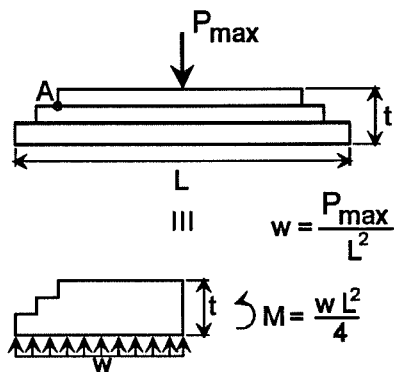


Figure A.2 Model of Stepped Bearing Plates

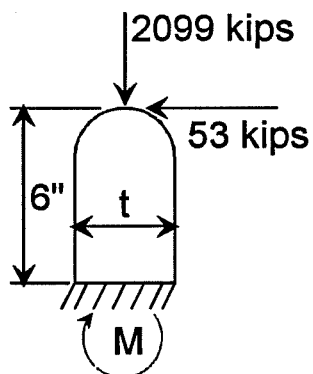


Figure A.3 Forces on Rocker Pin

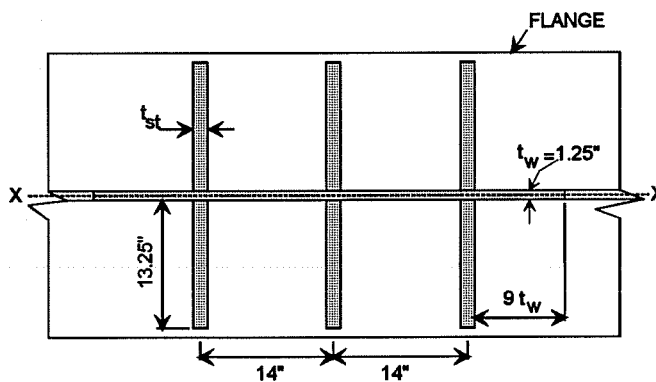


Figure A.4 Bearing Stiffener Properties

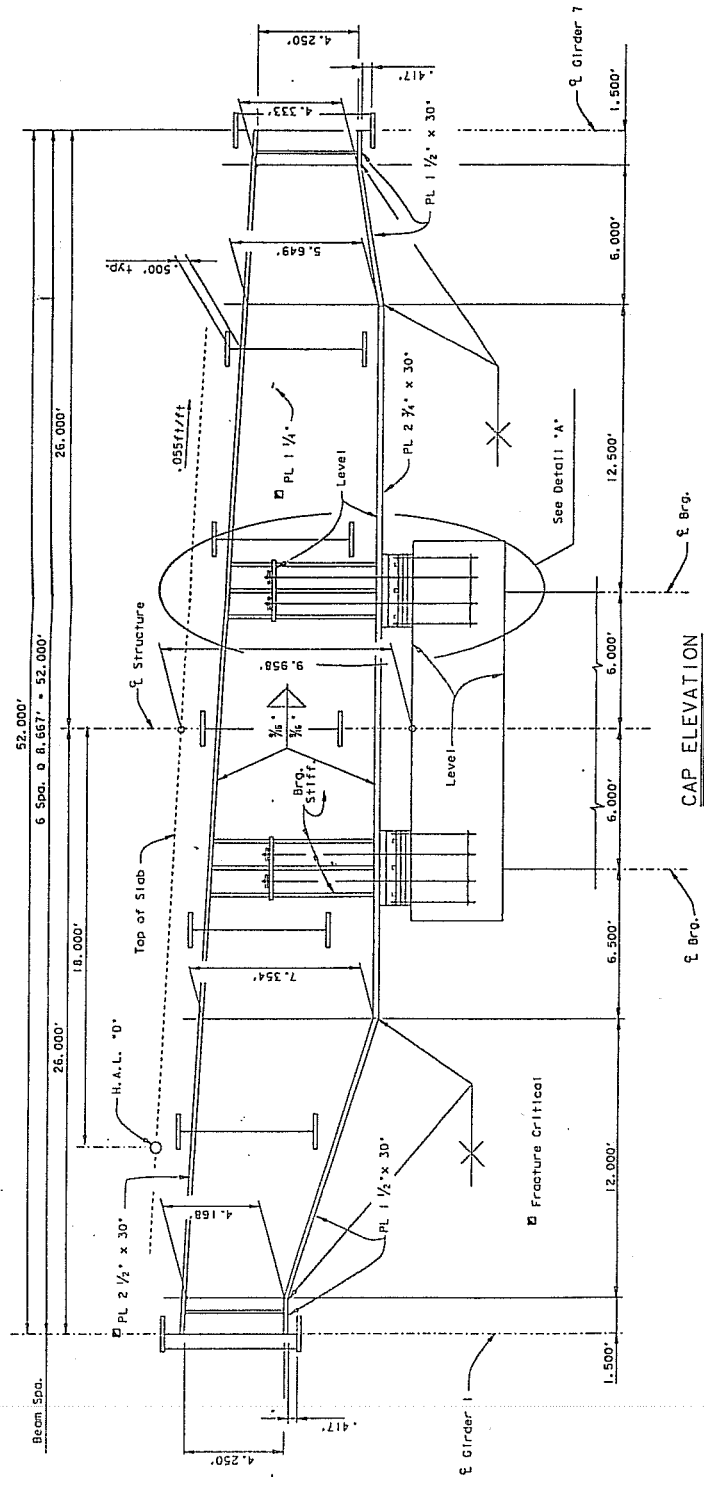


Figure A.5 Schematic of Cap Girder
(from Texas Department of Transportation)

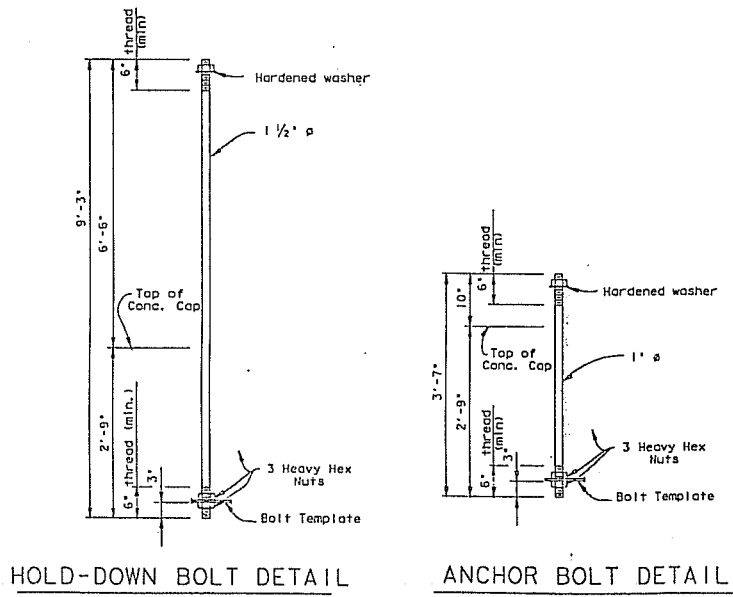
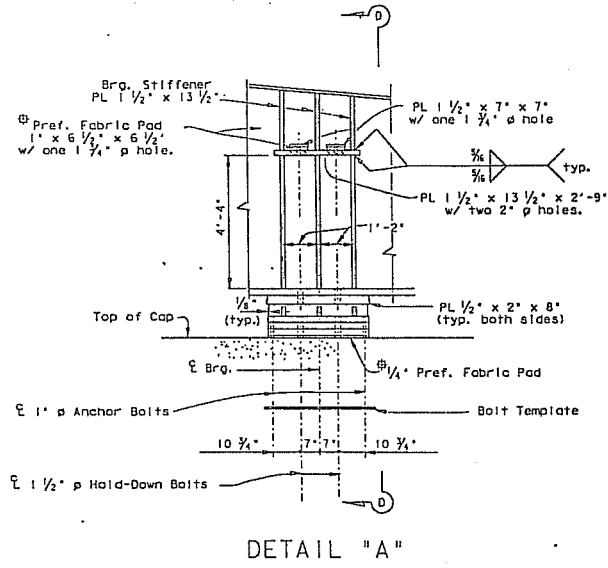
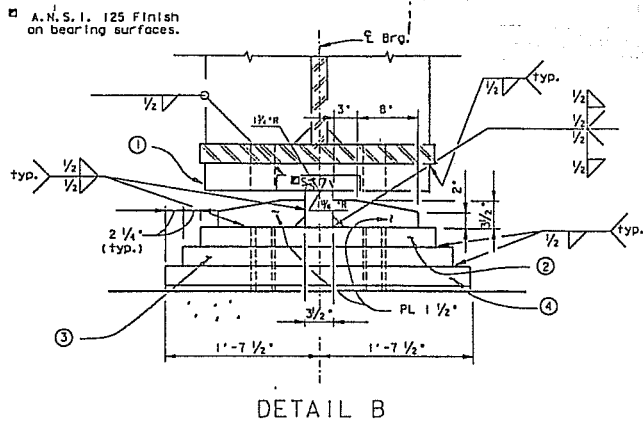
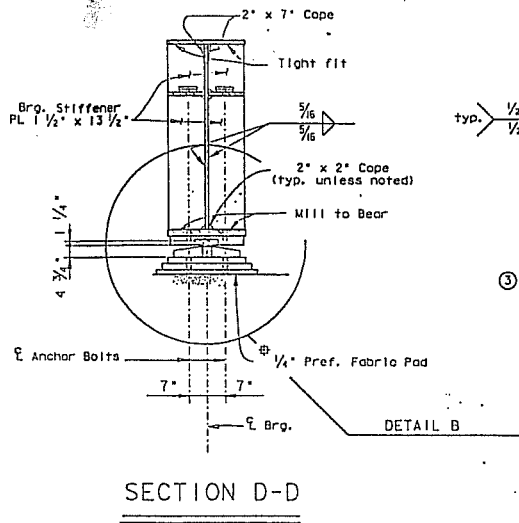


Figure A.6 Schematic of Connection Details
 (from Texas Department of Transportation)



- ① PL 3 1/2" x 28 1/4" x 3'-3"
(w/ 4 - 2" ϕ holes)
- ② PL 2 1/2" x 30" x 3'-3"
(w/ 4 - 2" ϕ holes &
4 - 1 1/8" ϕ holes)
- ③ PL 2 1/2" x 34 1/2" x 3'-3"
(w/ 4 - 2" ϕ holes &
4 - 1 1/8" ϕ holes)
- ④ PL 2 1/2" x 39" x 3'-3"
(w/ 4 - 2" ϕ holes &
4 - 1 1/8" ϕ holes)

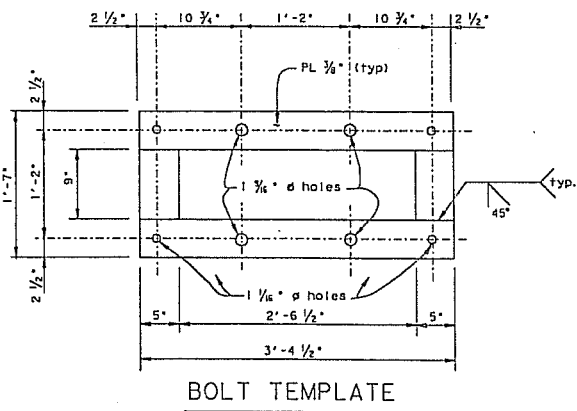
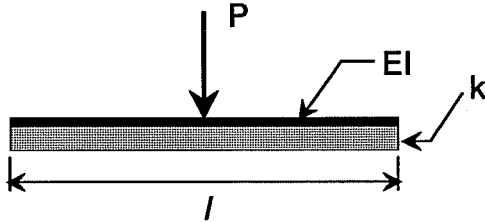


Figure A.7 Schematic of Connection Details (cont.)
 (from Texas Department of Transportation)

APPENDIX B : DERIVATION OF BEARING PLATE EQUATIONS



$$\beta = \sqrt[4]{\frac{k}{4EI}} \quad (\text{B.1})$$

where k is the foundation modulus per unit width, EI is the beam stiffness per unit width [5].

Determination of k

For uniform stress over surface of square on elastic medium (concrete or grout),

$$\Delta_{avg} = \frac{ql(1 - \nu^2)}{E_c \sqrt{l}} \quad (\text{B.2})$$

where Δ_{avg} is the average deflection and q is the uniform load per unit width [19,20]. If $k = q / \Delta_{avg}$,

$$k = \frac{E_c}{(1 - \nu^2)} \frac{1}{\sqrt{l}} \quad (\text{B.3})$$

If $\nu = 0.17$ for concrete, $1 - \nu^2 = 0.97 \approx 1$, which gives

$$k = \frac{E_c}{\sqrt{l}} \quad (\text{B.4})$$

Substituting Equation B.4 into Equation B.1 gives

$$\beta = \sqrt[4]{\frac{3}{\sqrt{l}} \frac{E_c}{E_s} \frac{1}{t^3}} \quad (\text{B.5})$$

Determine Effective Length of Bearing

At $\beta l > 3$, portions of plate ineffective. Letting $\beta l = 3$ and solving Equation B.5 for l gives

$$l = 2.56 \sqrt[3.5]{\frac{E_s}{E_c} t^3} \quad (\text{B.6})$$

The maximum moment at centerline of plate is

$$M_{cent} = \frac{P}{4\beta} \frac{\cosh \beta l - \cos \beta l}{\sinh \beta l + \sin \beta l} = 0.09Pl \text{ at } \beta l = 3 \quad (\text{B.7})$$

which gives the maximum stress as

$$f_{max} = \frac{M_{cent}}{S} = 0.54 \frac{Pl}{b t^2} \quad (\text{B.8})$$

Limiting the maximum stress to the allowable bending stress F_b and solving for t gives

$$t_{min} = \sqrt{\frac{0.54Pl}{F_b b}} \quad (\text{B.9})$$

where b is the width of the beam. Substituting Equation B.9 into Equation B.6 gives

$$l_{eff} = 3.25 \sqrt{\frac{E_s}{E_c} \left[\frac{P}{F_b b} \right]^{0.75}} \quad (\text{B.10})$$

At $F_b = 0.55F_y$ where $F_y = 50$ ksi

$$l_{eff} = 0.27 \sqrt{\frac{E_s}{E_c} \left[\frac{P}{b} \right]^{0.75}} \quad (\text{B.11})$$

Deflection Under Load P

For determination of Equation 8.18. The deflection under load P is

$$\delta = \frac{P}{b} \frac{\beta}{k} \frac{2 + \cos \beta l + \cosh \beta l}{\sin \beta l + \sinh \beta l} \quad (\text{B.12})$$

At $\beta l = 3$ and $k = E_c / \sqrt{l}$, localized stiffness of concrete under bearing plate is

$$k_{cpl} = \frac{P}{\delta} = 0.3 E_c \sqrt{l_{eff}} b \quad (\text{B.13})$$

APPENDIX C : WIDE-FLANGE BEARING PROPERTIES

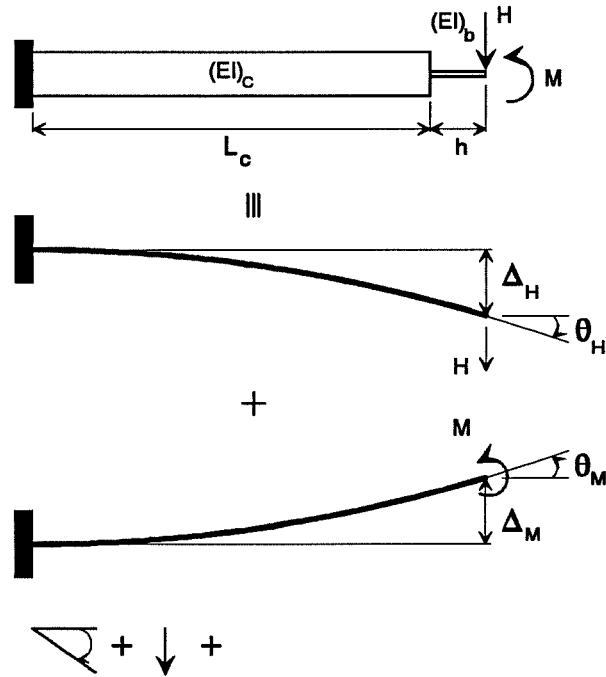
Properties of Fatigue Specimens

ID	d	d-2tf	T	t _{w,avg}	t _{w,sg}	k
	(in)	(in)	(in)	(in)	(in)	(in)
F087LT1D	12.53	10.88	9.62	0.508	0.525	1.455
F087LB1D	12.53	10.88	9.62	0.508	0.525	1.455
F087LT1D	12.53	10.88	9.62	0.508	0.525	1.455
F087LT1D	12.53	10.88	9.62	0.508	0.525	1.455
F087LB1D	12.53	10.88	9.62	0.508	0.525	1.455
F087UT2D	12.51	10.89	9.59	0.509	0.537	1.460
F087UB2D	12.51	10.89	9.59	0.509	0.544	1.460
F087LT2D	12.52	10.89	9.46	0.509	0.539	1.530
F087LB2D	12.52	10.89	9.46	0.509	0.542	1.530
F087UT2D	12.51	10.89	9.59	0.509	0.539	1.460
F087UB2D	12.51	10.89	9.59	0.509	0.543	1.460
F087LT2D	12.52	10.89	9.46	0.509	0.540	1.530
F087LB2D	12.52	10.89	9.46	0.509	0.543	1.530
F152UT1D	13.89	11.00	9.43	0.884	0.869	2.230
F152UB1D	13.89	11.00	9.43	0.884	0.864	2.230
F152LT1D	13.90	11.00	9.21	0.885	0.865	2.345
F152LB1D	13.90	11.00	9.21	0.885	0.869	2.345
F152UT2D	13.90	11.00	9.40	0.878	0.878	2.250
F152UB2D	13.90	11.00	9.40	0.878	0.866	2.250
F152LT2D	13.88	10.96	9.13	0.877	0.869	2.375
F152LB2D	13.88	10.96	9.13	0.877	0.870	2.375
F152UT3D	13.89	10.99	9.39	0.873	0.864	2.250
F152UB3D	13.89	10.99	9.39	0.873	0.869	2.250
F152LT3D	13.88	10.98	9.42	0.876	0.865	2.230
F152LB3D	13.88	10.98	9.42	0.876	0.869	2.230
F152UB1S	13.90	11.00	9.21	0.885	0.869	2.345
F230UT1D	15.00	10.82	9.63	1.30	1.32	2.685
F230UB1D	15.00	10.82	9.63	1.30	1.31	2.685
F230LT1D	15.00	10.84	9.60	1.30	1.32	2.700
F230LB1D	15.00	10.84	9.60	1.30	1.31	2.700
F230LB1S	15.00	10.84	9.60	1.30	1.31	2.700
F230UB1S	15.00	10.82	9.63	1.30	1.31	2.685
F230LB2S	15.00	10.82	9.63	1.30	1.31	2.685
NOMINAL VALUES FROM AISC MANUAL						
W12X87						
T=9.5"	d = 12.53"	t _w = 0.515"	k = 1.5"			
W12X152						
T = 9.5"	d = 13.71"	t _w = 0.870"	k = 2.125"			
W12X230						
T = 9.5"	d = 15.05"	t _w = 1.285"	k = 2.75"			

Properties of Compression Specimens

TEST	SIZE	b	t _w	h (d - 2t _f)	h/r	A	I _{avg}	S _{avg}	t _{eg, top}	t _{eg, bot}
		in	in	in		in ²	in ⁴	in ³	in	in
C087361	W12X87	36	0.511	10.91	71.17	18.40	0.400	1.567	0.524	0.529
C087362	W12X87	36	0.511	10.91	71.17	18.40	0.400	1.567	0.524	0.525
C087241	W12X87	24	0.516	10.89	70.35	12.38	0.275	1.065	0.531	0.532
C087242	W12X87	24	0.506	10.89	71.74	12.14	0.259	1.024	0.527	0.528
C087243	W12X87	24	0.512	10.89	70.90	12.29	0.268	1.049	0.540	0.535
C152361	W12X152	36	0.893	11.01	41.10	32.15	2.136	4.785	0.906	0.886
C152362	W12X152	36	0.879	11.01	41.75	31.64	2.037	4.636	0.865	0.869
C230241	W12X152	24	1.300	10.85	27.82	31.20	4.394	6.760	1.310	1.320
C230242	W12X230	24	1.300	10.85	27.82	31.20	4.394	6.760	1.320	1.300
T08724N	W12X87	24	0.507	10.94	71.93	12.17	0.261	1.028	0.525	0.528
T08724S	W12X87	24	0.510	10.94	71.50	12.24	0.265	1.040	0.531	0.529
T15224N	W12X152	24	0.889	11.01	41.28	21.34	1.405	3.161	0.873	0.870
T15224S	W12X152	24	0.892	11.02	41.18	21.41	1.419	3.183	0.882	0.880

**APPENDIX D : DERIVATION OF MOMENT AND FORCE EQUATIONS
FOR WIDE-FLANGE BEARINGS**



Displacement due to horizontal force

$$\Delta_H = \frac{H}{(EI)_c} \left[\frac{L_c^2 h}{2} + L_c h^2 + \frac{L_c^3}{3} + \frac{h^3}{3} \left(\frac{(EI)_c}{(EI)_b} \right) \right] \quad (D.1)$$

Rotation due to horizontal force

$$\theta_H = \frac{H}{(EI)_c} \left[L_c h + \frac{L_c^2}{2} + \frac{h^2}{2} \left(\frac{(EI)_c}{(EI)_b} \right) \right] \quad (D.2)$$

Displacement due to moment

$$\Delta_M = \frac{M}{(EI)_c} \left[L_c h + \frac{L_c^2}{2} + \frac{h^2}{2} \left(\frac{(EI)_c}{(EI)_b} \right) \right] \quad (D.3)$$

Rotation due to moment

$$\theta_M = \frac{M}{(EI)_c} \left[L_c + h \left(\frac{(EI)_c}{(EI)_b} \right) \right] \quad (D.4)$$

Stiffness Constant Definitions

$$k_H = \frac{H}{\Delta_H} : k_M = \frac{H}{\Delta_M} : \beta_M = \frac{M}{\theta_M}$$

$$k_H = \frac{(EI)_c}{\frac{L_c^2 h}{2} + h^2 L_c + \frac{L_c^3}{3} + \frac{h^3}{3} \left(\frac{(EI)_c}{(EI)_b} \right)} \quad (D.5)$$

$$k_M = \frac{(EI)_c}{L_c h + \frac{L_c^2}{2} + \frac{h^2}{2} \left(\frac{(EI)_c}{(EI)_b} \right)} \quad (D.6)$$

$$\beta_M = \frac{(EI)_c}{L_c + h \left(\frac{(EI)_c}{(EI)_b} \right)} \quad (D.7)$$

Compatibility

$$\Delta_T = \Delta_H - \Delta_M : \Delta_T = \frac{H}{k_H} - \frac{M}{k_M} : H = k_H \left[\Delta_T + \frac{M}{k_M} \right] \quad (D.8)$$

$$\theta_T = \theta_H - \theta_M : \theta_T = \frac{H}{k_M} - \frac{M}{\beta_M} : M = \beta_M \left[-\theta_T + \frac{H}{k_M} \right] \quad (D.9)$$

Solve Equations D.8 and D.9 for H and M

$$H = \frac{k_H \left(\Delta_T - \frac{\beta_M \theta_T}{k_M} \right)}{\left(1 - \frac{k_H \beta_M}{k_M^2} \right)} \quad (\text{D.10})$$

$$M = \frac{\beta_M \left(-\theta_T + \frac{k_H \Delta_T}{k_M} \right)}{\left(1 - \frac{k_H \beta_M}{k_M^2} \right)} \quad (\text{D.11})$$

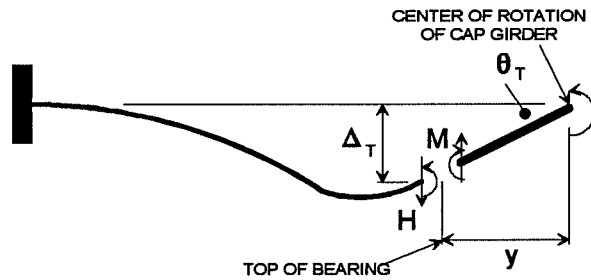
Rearranging Equations D.10 and D.11

$$H = k_H \Delta_T \left[\frac{1 - \left(\frac{\theta_T}{\Delta_T} \right) \left(\frac{\beta_M}{k_M} \right)}{\left(1 - \frac{k_H \beta_M}{k_M^2} \right)} \right] \quad (\text{D.12})$$

$$M = \beta_M \theta_T \left[\frac{\left(\frac{\Delta_T}{\theta_T} \right) \left(\frac{k_H}{k_M} \right) - 1}{\left(1 - \frac{k_H \beta_M}{k_M^2} \right)} \right] \quad (\text{D.13})$$

Assume a rotation is applied as shown below

- let $\Delta_T = -y \theta_T$ [rotation (-)]
- $\Delta_T / \theta_T = -y$
- $\theta_T / \Delta_T = -(1/y)$



Rearranging Equations D.12 and D.13

$$H = k_H \Delta_T \left[\frac{1 + \left(\frac{1}{y} \right) \left(\frac{\beta_M}{k_M} \right)}{\left(1 - \frac{k_H \beta_M}{k_M^2} \right)} \right] \quad (D.14)$$

$$M = \beta_M \theta_T \left[\frac{y \left(\frac{k_H}{k_M} \right) + 1}{\left(1 - \frac{k_H \beta_M}{k_M^2} \right)} \right] \quad (D.15)$$

For force and moment due to temperature change, assume $\theta_T = 0$ (from Eqs. D.10, D.11)

$$H = \frac{k_H \Delta_T}{\left(1 - \frac{k_H \beta_M}{k_M^2} \right)} \quad (D.16)$$

$$M = \frac{\beta_M \Delta_T \left(\frac{k_H}{k_M} \right)}{\left(1 - \frac{k_H \beta_M}{k_M^2} \right)} \quad (D.17)$$

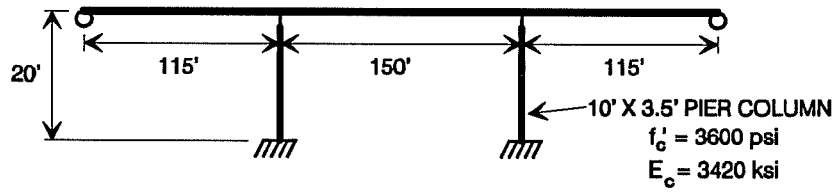
For the moment at the top of the bearing due to longitudinal force (from Eq. D.9, $\theta_T = 0$)

$$M = H \left(\frac{\beta_M}{k_M} \right) \quad (\text{D.18})$$

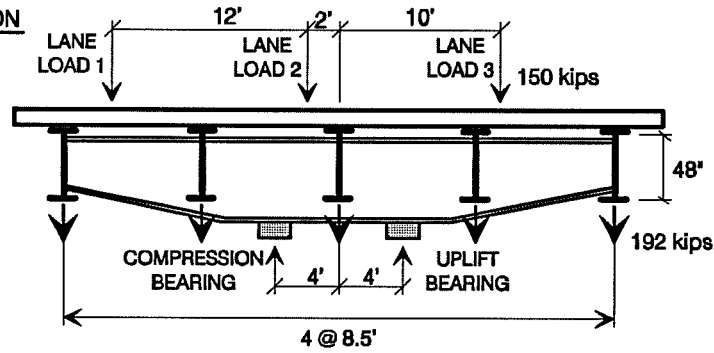
APPENDIX E : DESIGN EXAMPLES FOR PROPOSED CONNECTION

NO UPLIFT

PLAN VIEW



CROSS-SECTION



LOADS

COMPRESSION BEARING (2 LANES CONTROLS)

	GROUP I	GROUP II	GROUP III	GROUP IV	GROUP V	GROUP VI
	↓ 938 k	↓ 645 k	↓ 979 k ← 3 k	↓ 938 k ← T	↓ 645 k ← T	↓ 979 k ← 3 k+T
% UNIT STRESS	100%	125%	125%	125%	140%	140%

● BY INSPECTION, GROUP I, III, IV, OR VI WILL CONTROL

UPLIFT BEARING (2 LANES CONTROLS)

	GROUP I	GROUP II	GROUP III	GROUP IV	GROUP V	GROUP VI
	↓ 338 k	↓ 331 k	↓ 297 k ← 3 k	↓ 338 k ← T	↓ 331 k ← T	↓ 297 k ← 3 k+T
% UNIT STRESS	100%	125%	125%	125%	140%	140%

● UPLIFT DOES NOT OCCUR

Design of Wide-Flange Bearing

Size the Bearing Web

1.) Try W14 using 50 ksi steel.

$$t_w \geq \frac{h \sqrt{F_y}}{60} = \frac{12.6 \sqrt{50}}{60} = 1.48 \text{ inches} \quad (10.1)$$

2.) Size for Group I loading, axial load only.

- $F_a = 0.472F_y = 0.472(50) = 23.6 \text{ ksi} : A_{\text{req}} = 938 / 23.6 = 39.8 \text{ in}^2$
- @ $t_w = 1.48 \text{ inches}$, $b = 26.9 \text{ inches} : \underline{\text{Try W14X370 @ 28 inches}}$

$$f_a = \frac{P}{b t_w} = \frac{938}{28 (1.655)} = 20.3 \text{ ksi} < 23.6 \text{ ksi, O.K.} \quad (10.2)$$

Check Design for Remaining Load Cases

1.) Calculate longitudinal direction stiffness constants

- Bearing : $I_b = (1/12)(28*2)*1.655^3 = 21.15 \text{ in}^4 : (EI)_b = 613,477 \text{ k-in}^2$
- Pier Column : $I_c = (1/12)(10*12)(3.5*12)^3 = 740,880 \text{ in}^4 : (EI)_c = 2.53E9 \text{ k-in}^2$
- $h = 12.6 \text{ in} : L_c = (20+10)(12) = 360 \text{ inches} : (EI)_c / (EI)_b = 4130$

$$k_H = \frac{2.53E9}{\frac{360^2 (12.6)}{2} + 12.6^2 (360) + \frac{360^3}{3} + \frac{12.6^3}{3} (4130)} = 132 \text{ kips / inch} \quad (10.3)$$

$$k_M = \frac{2.53E9}{360 (12.6) + \frac{360^2}{2} + \frac{12.6^2}{2} (4130)} = 6370 \text{ kips / rad} \quad (10.4)$$

$$\beta_M = \frac{2.53E9}{360 + 12.6 (4130)} = 48284 \text{ inch-kips / rad} \quad (10.5)$$

2.) Determine horizontal displacement due to temperature change.

$$\Delta_T = \frac{150 \cdot 12}{2} (6.5E-6 \cdot 50) = 0.29 \text{ inches} \quad (10.6)$$

3.) Calculate maximum bending stress in bearing due to temperature change.

$$M = \frac{\beta_M \Delta_T \left(\frac{k_H}{k_M} \right)}{\left(1 - \frac{k_H \beta_M}{k_M^2} \right)} = \frac{48284 (0.29) \left(\frac{132}{6370} \right)}{\left(1 - \frac{132 (48284)}{6370^2} \right)} \frac{1}{2} = 172 \text{ inch-kips} \quad (10.7)$$

$$f_b = \frac{M}{S_{bearing}} = \frac{172}{(10.58 / 0.83)} = 13.5 \text{ ksi} \quad (10.8)$$

4.) Calculate maximum bending stress in bearing due to LF (longitudinal force).

$$M = LF \frac{\beta_M}{k_M} = (3) \frac{48284}{6370} = 22.7 \text{ inch-kips} \quad (10.9)$$

$$f_b = \frac{M}{S_{bearing}} = \frac{22.7}{(10.58 / 0.83)} = 1.78 \text{ ksi} \quad (10.10)$$

5.) Check interaction equations and resize bearing if necessary.

$$\bullet 0.472F_y = 23.6 \text{ ksi} : 0.55F_y = 27.5 \text{ ksi}$$

$$\underline{\text{GROUP III}} : f_a = 979 / 46.34 = 21.1 \text{ ksi} : f_b = 1.78$$

$$\frac{f_a}{0.472F_y} + \frac{f_b}{0.55F_y} = \frac{21.1}{23.6} + \frac{1.78}{27.5} = 0.96 \leq 1.25, \text{ O.K.} \quad (10.11)$$

$$\underline{\text{GROUP IV}} : f_a = 20.2 \text{ ksi} : f_b = 13.5 \text{ ksi}$$

$$\frac{f_a}{0.472F_y} + \frac{f_b}{0.55F_y} = \frac{20.2}{23.6} + \frac{13.5}{27.5} = 1.35 > 1.25, \text{ N.G.} \quad (10.12)$$

- Increase bearing width by 15%, $b = 32$ inches, $A = 1.655(32) = 53.0 \text{ in}^2$
- $f_a = 938 / 53 = 17.7 \text{ ksi}$: $f_b = 13.5 (28/32) = 11.8 \text{ ksi}$
- No need to recalculate stiffness constants

$$\frac{f_a}{0.472F_y} + \frac{f_b}{0.55F_y} = \frac{17.7}{23.6} + \frac{11.8}{27.5} = 1.18 < 1.25, \text{ O.K.}$$

GROUP VI : $f_a = 21.1(28/32) = 18.5 \text{ ksi}$: $f_b = 11.8 + 1.78(28/32) = 13.4 \text{ ksi}$

$$\frac{f_a}{0.472F_y} + \frac{f_b}{0.55F_y} = \frac{18.5}{23.6} + \frac{13.4}{27.5} = 1.27 < 1.40, \text{ O.K.} \quad (10.14)$$

Check Bottom Flange for Bearing

- 1.) Determine length of bearing flange effective

$$l_{eff} = 3.25 \sqrt{\frac{E_s}{E_c} \left(\frac{P}{0.55F_y b} \right)^{0.75}} = 3.25 \sqrt{\frac{29000}{3420} \left(\frac{979}{27.5 (32)} \right)^{0.75}} = 10.3 \text{ inches} \quad (10.15)$$

- 2.) Check flange thickness

$$t_f \geq \sqrt{\frac{Pl_{eff}}{F_y b}} = \sqrt{\frac{979(10.3)}{50 (32)}} = 2.51 \text{ inches} < t_f = 2.66 \text{ inches}, \text{ O.K.} \quad (10.16)$$

- 3.) Check bearing stress on concrete

$$f_{bc} = \frac{P}{l_{eff} b} = \frac{979}{10.3 (32)} = 2.97 \text{ ksi} < f'_c = 3.6 \text{ ksi}, \text{ O.K.} \quad (10.17)$$

Check Fatigue of Bearing

1.) Determine Stress Range

- $\theta_{\text{end}} = .000654$ radians : $\theta_{\text{center}} = 0.000775$ radians : $\theta_r = 0.00143$ radians
- Above rotations are maximum produced by fatigue truck on one stringer
- Position to produce max rotations at left interior support is shown in Fig. E.1
- Assume center of rotation at mid-depth of longitudinal stringer, $y = 24 + 18 = 42$ "

$$M = \beta_M \theta_r \left[\frac{\left(1 + \frac{y k_H}{k_M} \right)}{\left(1 - \frac{k_H \beta_M}{k_M^2} \right)} \right] = 48284 (.00143) \left[\frac{\left(1 + \frac{(42) 132}{6370} \right)}{\left(1 - \frac{(132) 48284}{6370^2} \right)} \right] \frac{1}{2} = 76.5 \text{ inch-k}$$

(10.18)

$$S_r = \frac{M}{S_{\text{bearing}}} = \frac{76.5}{(10.58 / 0.83)} = 6 \text{ ksi}$$

(10.19)

$$S_t = (6) \frac{.000775}{.00143} = 3.25 \text{ ksi}$$

2.) Check Stress Range Against Allowable

- $S_c = 488 / (53) = 9.21 \text{ ksi}$: $S_{rp} = 8.8 \text{ ksi}$ for Category A

$$2.2S_t = 2.2 (3.25) = 7.15 \text{ ksi} < S_c = 9.21, \text{ O.K.}$$

(10.20)

- Since tensile stress range does not exceed dead load compressive stress
no need to check full stress range.

Design Bearing Stiffeners

- use three 9" wide stiffeners spaced at 11"
- let web thickness = 7/8" and web height = 56"

1.) Determine Minimum Thickness to Preclude Buckling AASHTO '92 : 10.34.6.1

$$t_{\min} = \frac{b'}{12} \sqrt{\frac{F_y}{33,000}}$$

- $t_{\min} = (9/12) \sqrt{(50000/33000)} = 0.92$ inches → use 1"

2.) Calculate Properties

- Area = $(9 \cdot 1 \cdot 6)_{\text{ST}} + (2 \cdot 11 + 2 \cdot 9 \cdot 1) \cdot 1_{\text{WEB}} = 94 \text{ in}^2$
- $I_{\text{XX}} = (1/12) \cdot 3 \cdot 18.875^3 = 1681 \text{ in}^4$
- $r = \sqrt{(1681/94)} = 4.23 \text{ in}$
- $Kl/r = 0.75 \cdot 56 / 4.23 = 9.93 \text{ in}$

3.) Determine Compression Capacity AASHTO 10.32.1.A

$$F_a = 23580 - 1.03 \left(\frac{Kl}{r} \right)^2$$

- $P_{\text{cr}} = F_a A = (23580 - 1.03 \cdot 9.93^2) \cdot 94 = 23.5 \cdot 94 = 2207$ kips > 938 kips, O.K.

4.) Check Stiffener Bearing Stress AASHTO '92 : 10.32.1.A

$$F_b = 40 \text{ ksi for } 50 \text{ ksi steel}$$

- $f_b = 938 / (6 \cdot 7.5 \cdot 1) = 20.8 \text{ ksi} < 40 \text{ ksi}$, O.K.

The 1.5" cope reduced stiffener width from 9" to 7.5".

5.) Check Stiffener to Web Weld AASHTO '92 : 10.3.2

$$F_v = 19 \text{ ksi for } 70 \text{ ksi weld material}$$

- assume minimum fillet weld; size = 5/16"
- $f_v = 938 / (12 \cdot 56 \cdot 0.707 \cdot 0.3125) = 6.32 \text{ ksi} < 19 \text{ ksi}$, O.K.

The connection is illustrated in Figure E.2.

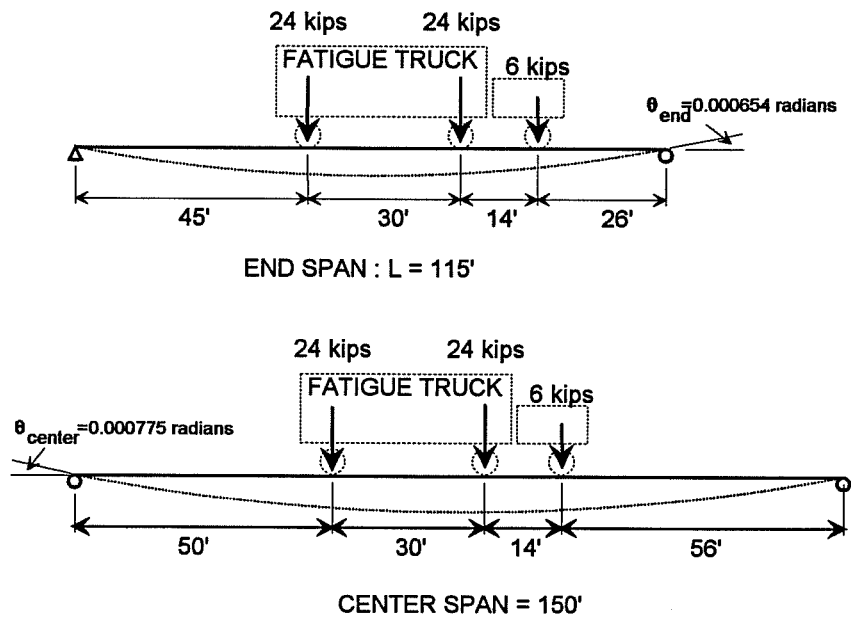


Figure E.1 Fatigue Truck Position for Maximum End Rotations

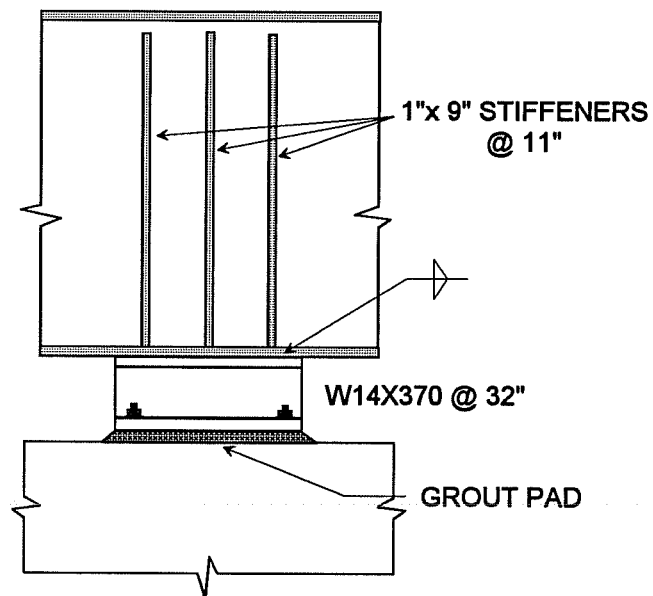
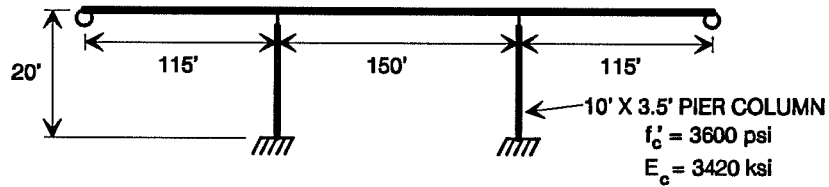


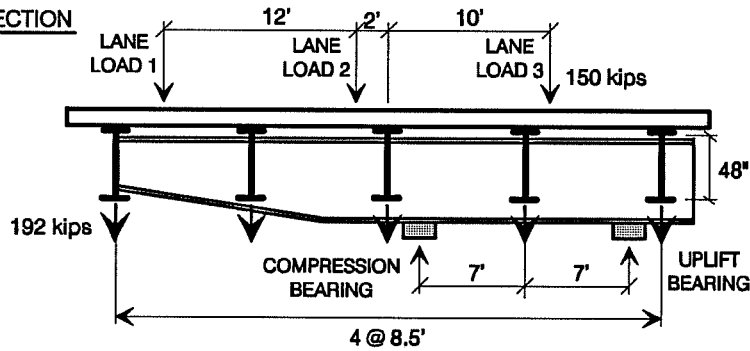
Figure E.2 Connection Detail, No Uplift

UPLIFT

PLAN VIEW



CROSS-SECTION



LOADS

COMPRESSION BEARING (2 LANES CONTROLS)

	GROUP I	GROUP II	GROUP III	GROUP IV	GROUP V	GROUP VI
	↓ 1574 k	↓ 1096 k	↓ 1601 k ← 3 k	↓ 1574 k ← T	↓ 1096 k ← T	↓ 1601 k ← 3 k+T
% UNIT STRESS	100%	125%	125%	125%	140%	140%

• BY INSPECTION, GROUP I, III, IV, OR VI WILL CONTROL

UPLIFT BEARING (2 LANES CONTROLS)

	GROUP I	GROUP II	GROUP III	GROUP IV	GROUP V	GROUP VI
	↑ 298 k 203 L	↑ 120 k	← 3 k ↑ 325 k 203 L	← T ↑ 298 k 203 L	← T ↑ 331 k	← 3 k+T ↑ 325 k 203 L
% UNIT STRESS	150%	150%	150%	150%	150%	150%

• GROUP III CONTROLS

From design of compression bearing (calculations not shown)

- W14X455 @ 36" : $t_w = 2.015$ " : $F_y = 50$ ksi

Determine the Uplift Load

- $T_D = 122$ k : $T_L = 203$ k (Group III)
- 1.) $T_{up} = 122 + 2(203) = 528$ kips → controls
- 2.) $T_{up} = 1.5(203 + 122) = 488$ kips

Design Uplift Resisting Elements

Hold-Down Bolts

- use A193 GR B7 @ $F_y = 105$ ksi, 1.5 " diameter
- $n_{req} = 528 / (0.55 * 105 * 1.41 * 1.5) = 4.32$ bolts → use 6 bolts

Wide-Flange Bearing

$$T_a = 1.5 (0.55 (50) (36) 2.015) = 2993 \text{ k} \geq T_{up} = 528 \text{ k, O.K.} \quad (10.22)$$

Anchor Bolts

- use 150 ksi threadbar (ASTM A722), 4 anchor bolts per connection
- anchor bolts on 11" X 11" pattern
- $A_{req} = 528 / (4 * 0.66 * 150 * 1.5) = 0.88 \text{ in}^2$ → use 1.25" ϕ , $A_{ps} = 1.25 \text{ in}^2$

Connection Between Bearing Top Flange and Cap Girder

- perimeter length of top flange = $2 * 36 + 2 * 16.84 = 106$ inches
- use fillet weld, $F_v = 0.27 (65) = 17.6$ ksi
- $t_{req} = 528 / (0.707 * 17.6 * 1.5 * 106) = 0.27$ in → use 5/16" (minimum size)

Design Anchorage

- See "Design of Headed Anchor Bolts" by J.G. Shipp and E.R. Haninger, Engineering Journal, AISC, Second Quarter 1983.

- Embedment depth necessary to produce anchor bolt failure before concrete breakout is 84 inches. A typical anchorage is shown in Figure 8.5.

Determine Pretension Force

- $B_0 = T_D + T_L = 122 + 203 = 325$ kips

Check Fatigue of Anchor Bolts

Calculate Connection Stiffnesses

1.) Anchor Bolt

- $L_b =$ free length (69") + one-half of embedment depth (42") = 109"
- use $E = 30,000$ ksi for threadbar

$$k_b = \frac{4 (1.25) (30000)}{109} = 1376 \text{ kips per inch}$$

2.) Cap Girder and Stiffeners

- same properties as in No Uplift Example
- $h_w = 56"$, $A_{st} = 54 \text{ in}^2$, $A_{web} = (22 + 36) \cdot 5 \cdot (7/8) = 25.4 \text{ in}^2$

$$k_{gird} = \frac{29000 (54 + 25.4)}{56} = 41120 \text{ kips per inch} \quad (10.25)$$

3.) Wide-Flange Bearing

$$k_{bear} = \frac{36 (2.015) 29000}{12.6} = 167000 \text{ kips per inch} \quad (10.26)$$

4.) Concrete

$$\bullet A_{\text{bear}} = 36(16.48) = 593 \text{ in}^2, A_{\text{anch}} = 11(11) = 121 \text{ in}^2$$

$$k_{\text{conc}} = \frac{(594 + 121) 3420}{2 \cdot 84} = 14560 \text{ kips per inch} \quad (10.27)$$

5.) Concrete Under Bearing Plate

$$k_{\text{cpl}} = 0.3 (3420) \sqrt{13.5} \cdot 36 = 135700 \text{ kips per inch} \quad (10.28)$$

6.) Total Compression Element Stiffness

$$k_c = \left[\frac{1}{41120} + \frac{1}{167000} + \frac{1}{14560} + \frac{1}{135700} \right]^{-1} = 9400 \text{ kips per inch} \quad (10.29)$$

Transverse Direction

- for lane loading, $S_{rp} = 13$ ksi for category E; should also check for multiple truck loading and single truck loading (AASHTO '92 10.3.1 - 10.3.2) or use Guide Spec [1].

$$T_r = 4 (1.25) 13 \left[1 + \frac{9400}{1376} \right] = 509 \text{ k} > T_L = 203, \text{ O.K.} \quad (10.30)$$

Longitudinal Direction

- let $\theta_r = 0.006$ from simplified method described in Section 4.2.3

$$S_r = \frac{5.5 (1376/4) 0.006}{1.25} = 9.1 \text{ ksi} < S_{rp} = 13 \text{ ksi}, \text{ O.K.} \quad (10.31)$$

Design Bearing Stiffeners

Determine Length of Top Stiffeners

- use minimum weld size 5/16"
- $L_{req} = 528 / (12 * 0.707 * .3125 * 17.6 * 1.5) = 7.54 \text{ in}$: add 1.5" for cope
- use 10" long stiffener, leave 2" gap for lower stiffener
- use 44" lower stiffener, all stiffeners 1" X 9"

Uplift detail shown below.

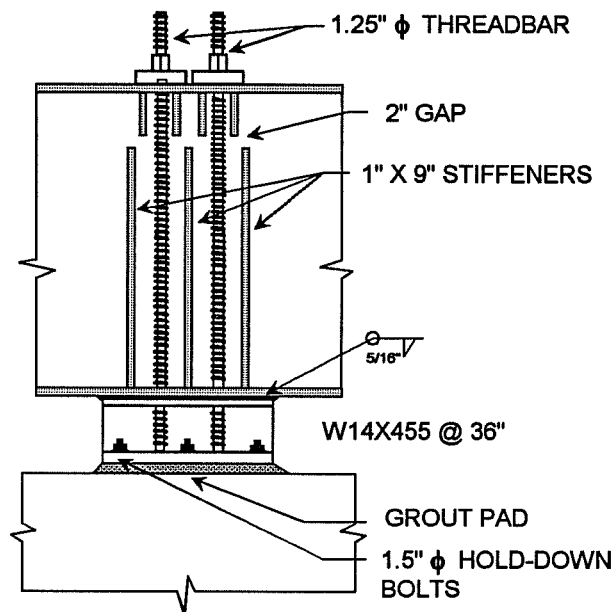


Figure E.3 Connection Detail, Uplift

APPENDIX F : THERMOELASTIC STRESS ANALYSES

SPATE Theory

Thermoelastic stress analysis is a method of determining the stresses in a material by correlating the change in temperature of the material to the change in stress. If an element of a material is compressed the temperature of that element increases, much in the same way that a gas heats up when compressed. If the element is pulled, or subject to tension, the temperature decreases. The theory is based on the relationship shown in equation F.1, which states that the change in the sum of the principal stresses at a point in a material is proportional to the change in temperature at that point [14],

$$\Delta \sigma = - \Delta T \left(\frac{\alpha T}{\rho C_p} \right) \quad (\text{F.1})$$

where $\Delta \sigma$ is the change in the sum of the principal stress, ΔT is the change in temperature, α is the coefficient of thermal expansion, T is the ambient temperature, ρ is the density of material, and C_p is the coefficient of specific heat at constant pressure. This relationship is valid for homogeneous materials loaded in the linear elastic range and under adiabatic conditions. To ensure adiabatic conditions, the specimen must be loaded cyclically. The procedure of determining the stresses in a material by thermoelastic stress analysis is commonly referred to as the SPATE technique, or Stress Pattern Analysis by Thermal Emission. The SPATE technique uses infrared detection to measure heat radiating from a surface. The instrument used in this research was the SPATE 9000, manufactured by Ometron [14].

3.7.2 Application

The components of the SPATE 9000 system are listed in Figure F.1. They comprise the scan head, which uses an infrared light for detection; a correlator, which processes the temperature signal and load signal; a computer, which controls the system and manipulates the data; and a video monitor, which provides a real-time plot of the stress distribution. The SPATE 9000 scans an area one point at a time. The maximum grid density is 255 by 255 points. The

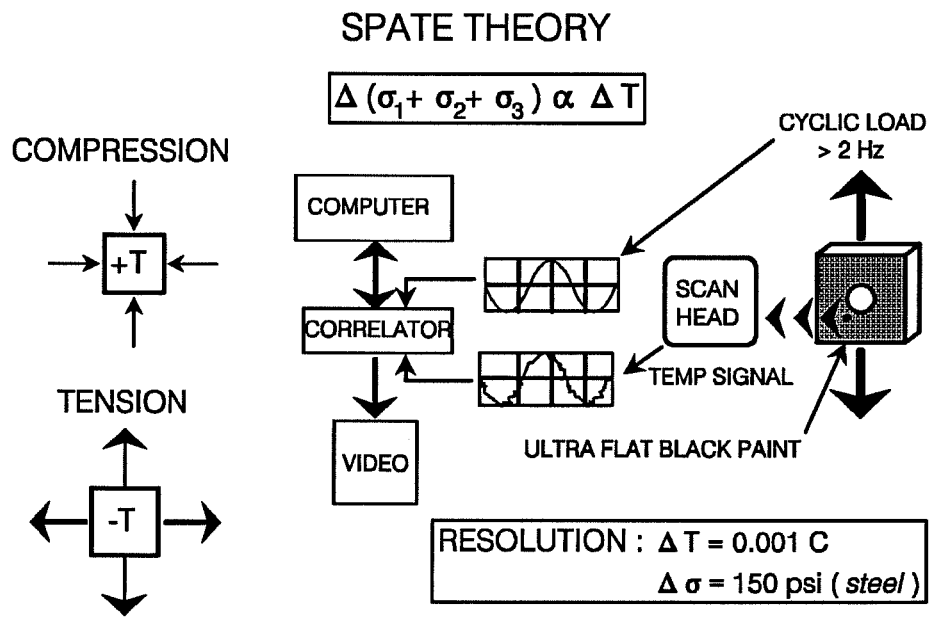


Figure F.1 SPATE Theory and Components of System

SPATE 9000 has a temperature resolution of 0.001° C, which translates to a stress resolution of 150 psi for steel. The procedure for conducting a SPATE scan is listed below:

- 1.) Define the scan area : The detection beam from the scan head is accompanied by a light dot and crosshairs. The crosshairs are correlated to a cursor on the video screen; by using a trackball and arrow keys, the cursor and crosshairs are simultaneously moved around the screen and scan area. In this way, the boundaries of the scan area on the test specimen are defined.
- 2.) Set the parameters of the scan : These include the sensitivity of the detection beam, the number of points in the scan area, and the length of time spent at each point. The larger the number of points in the scan and the longer the time spent at each point, the more accurate the scan.
- 3.) Scan the area : The surface of the scan area should be painted with an ultra flat black paint or other type of paint with high emissivity. This increases the heat radiating efficiency of the surface. The specimen must be loaded cyclicly and the load signal must have a symmetric waveform.

Thermoelastic Analyses of Cross Plate and Stiffeners, Phase I Tests

Cross Plate

A SPATE scan of the north cross-plate of the east connection and 3 inches of the stiffeners above and below the cross-plate (see Figure 2.6) is shown in Figure F.2. The purpose of this scan was to qualitatively verify the accuracy of the SPATE 9000. The cross plate is essentially a two span continuous beam with point loads (the anchor bolts) at the midspan of each beam and elastic restraints at the ends (the stiffeners). The contour bar shows the relative stress values of the different colors. A calibration procedure is necessary to obtain the actual stresses. The dark green, blue, and black colors indicate compression stress and the light green, yellow, and red colors indicate tension. The locations of the tension and compression stress contours correlate qualitatively with the predicted locations. The maximum tensile stresses occur at the bottom

center of each span of the cross plate and the maximum compressive stresses occur at the top center of each span of the cross plate. The magnitudes of the stresses are larger in the left span; this is because the first line of the anchor bolts picks up more load than the second line of anchor bolts.

Stiffeners

SPATE scans of an upper and lower stiffener are shown in Figures F.3 and F.4. The purpose of these scans was to determine the distribution of the axial stress in the stiffeners. The stiffeners scanned were on the north side of the east connection, on the west end. The compressive stress region at the bottom of the upper stiffener results from the bending of the stiffener, as described in the previous section. The tensile stress region near the web-stiffener weld is produced by the tension load in the anchor bolts. The lower stiffener, which is below the cross plate, resists the anchor bolt tensile load through compression. The upper stiffener, which is welded to the cross plate, is pulled down by the cross plate and resists through tension. The stresses in the upper stiffener are very low, on the order of 1 to 2 ksi for the anchor bolt load range of about 50 kips (sum of all four bolts). The stresses were obtained from strain gages placed on the upper stiffener for the ultimate strength test. The 1 to 2 ksi range is the range that occurred when the load range in the ultimate strength test matched the load range used in the SPATE analysis. The upper stiffener serves no structural purpose and may in fact be detrimental because it develops tensile stresses. Fatigue becomes a possibility.

The section of the lower stiffener near the cross plate is in tension, due to stiffener bending. The thermoelastic analysis shows that the entire length of the stiffener is not necessary to resist the compressive load caused by the reacting anchor bolts. Most of the axial stress has been transferred to the web in the top third of the stiffener. There is no necessity for the mill-to-bear specification of the stiffener to the bottom flange.

Scan of Wide-Flange Fillet Area, Phase II Fatigue Tests

Thermoelastic stress analyses of the fillet area of the wide-flange bearing sections were conducted to determine if any stress concentrations were present as a result of the horizontal shear produced by the longitudinal rotation (see Chapter 7). A SPATE scan of the lower half of the bottom specimen (see Figure 7.4) is shown in Figure F.5. An enlarged scan of a fillet (similar to that shown in Figure 7.1, fillet ID UT in Figure 7.4) is shown in Figure F.6. The stress distribution is symmetrical and there are no obvious stress concentrations in the fillet area. The scan also shows that the stress dissipates rapidly in the transition from the web to the fillet. A SPATE scan of a fillet that developed a crack is shown in Figure F.7 (fillet ID LT). The flange is not shown. The scan picked up the high stresses at the crack tip and showed how the stress redistributes in response to a crack.

Stresses in Cap Girder Web Due to Longitudinal Rotation, Phase II Tests

The stress distribution in the cap girder web due to longitudinal rotation for the Phase II large scale tests was examined using a thermoelastic stress analysis. A SPATE scan of the area is shown in Figure F.8. The rotation range was 0.0023 radians and the stress at the stiffener termination point as measured by the T-gages (as shown in Figure 8.6) was 5.5 ksi for the parallel gage and 5.4 ksi for the perpendicular. The results correlate very well qualitatively with the results of the finite element analyses presented in Section 4.2.4. Stress concentrations center around the stiffener termination point and in the portion of the cap girder web between the toe of the clip angle and the bottom flange. At a rotation range of 0.003, the maximum range recommended in section 4.2.1, the corresponding stresses at the stiffener termination point would be 8.2 ksi (parallel) and 8.0 ksi, approaching the allowable recommended 10 ksi stress range (Section 4.2.4).

REFERENCES

- 1.) AASHTO, *Guide Specifications for Fatigue Design of Steel Bridges*, American Association of State Highway and Transportation Officials: Washington, D. C., 1989.
- 2.) AASHTO, *Standard Specifications for Highway Bridges*, 15th Edition, American Association of State Highway and Transportation Officials: Washington, D. C., 1992.
- 3.) AISC, *Manual of Steel Construction, Allowable Stress Design*, 9th Edition, American Institute of Steel Construction: Chicago, IL., 1989.
- 4.) Chen, W. F. and Lui, E. M., *Structural Stability: Theory and Implementation*, Elsevier Science Publishing Co., Inc.: 1987.
- 5.) Den Hartog, J. P., *Advanced Strength of Materials*, Dover Publications, Inc.: New York, 1952 (pp. 141-162).
- 6.) Denio, R. J., "Behavior of Reinforced Concrete Pier Caps Under Concentrated Bearing Loads," Master's Thesis, The University of Texas at Austin: August 1993.
- 7.) DSI, *DYWIDAG Threadbar Posttensioning System*, Brochure, DYWIDAG Systems International: April 1988.
- 8.) DUPONT, *Design of Neoprene Bridge Bearing Pads*, Second Printing, Elastomer Chemical Department, E. I. du Pont de Nemours and Co., Inc.: Delaware, November 1959.
- 9.) Fischer, F. L., and Frank, K. H., "Axial Tension Fatigue Strength of Anchor Bolts," *Research Report 172-1*, Center for Highway Research, The University of Texas at Austin: March 1977.

- 10.) Haaijer, G. and Thurlimann, B., "Inelastic Buckling in Steel," *ASCE Transactions*, Vol. 125: 1960 (pp. 308-338).
- 11.) Joehnk, J. M., "Fatigue Behavior of Welded Joints Subjected to Variable Amplitude Stresses," Master's Thesis, The University of Texas at Austin: August 1982.
- 12.) Kulak, G. L., Fisher, J. W., and Struik, J. H. A., *Guide to Design Criteria for Bolted and Riveted Joints*, 2nd Edition, John Wiley and Sons: New York, 1987.
- 13.) Olsen, J. E., " Parametric Study of Curved Steel Girders Supported on Narrow Concrete Piers," Master's Thesis, The University of Texas at Austin: May 1994.
- 14.) Ometron, *SPATE 9000 User's Manual*, John Deere and Co.: 1988
- 15.) Salmon, C. G., and Johnson, J. E., *Steel Structures: Design and Behavior*, 3rd Edition, Harper and Row, Publishers, Inc.: New York, 1990.
- 16.) SASI, *ANSYS User's Manual*, Revision 5.0, Swanson Analysis Systems, Inc.: Houston, PA, 1992.
- 17.) Shipp, J. G., and Haninger, E. R., "Design of Headed Anchor Bolts," *Engineering Journal*, AISC, Second Quarter: 1983 (pp. xx-xx).
- 18.) Timoshenko, S. P. and Gere, J. M., *Theory of Elastic Stability*, 2nd Edition, McGraw - Hill Book Co.: 1961.
- 19.) Timoshenko, S. P. and Goodier, J. N., *Theory of Elasticity*, 3rd Edition, McGraw-Hill Book Co.: New York, 1961.

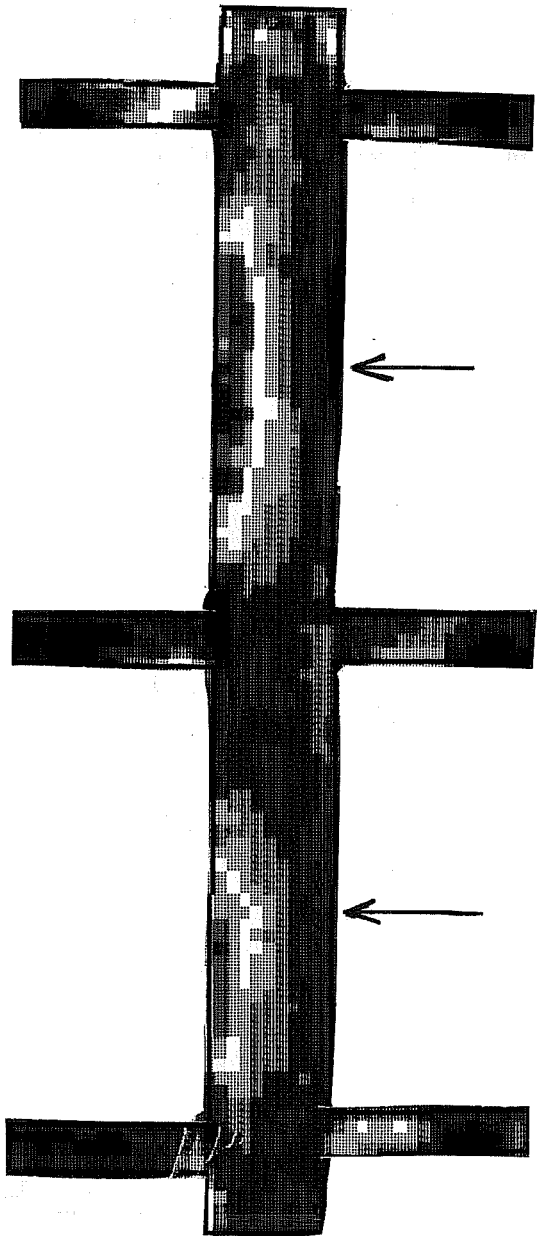
- 20.) Salmon, C. G., Schenker, L., and Johnston, B. G., "Moment-Rotation Characteristics of Column Anchorages," *ASCE Transactions*, Vol. 122: 1957 (pp. 132-154).
- 21.) TSDHPT, *Bridge Design Guide*, 1st Edition, Texas State Department of Highways and Public Transportation: 1990.
- 22.) USS, "V-Load Analysis: An Approximate Procedure,..." *United States Steel Corp. Highway Structures Design Handbook*, Chapter 12, Volume 1: 1984.
- 23.) Witecki, A. A. and Raina, V. K., "Distribution of Longitudinal Horizontal Forces Among Bridge Supports," *Concrete Bridge Design*, ACI Paper SP 23 - 45: Detroit, MI, 1969 (pp. 803 - 815).

VITA

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CROSSPLATE

1998.
1730.
1454.
1198.
932.
665.
399.
133.
-133.
-399.
-665.
-932.
-1198.
-1454.

Figure F.2 SPATE Scan of Cross-plate and Stiffeners

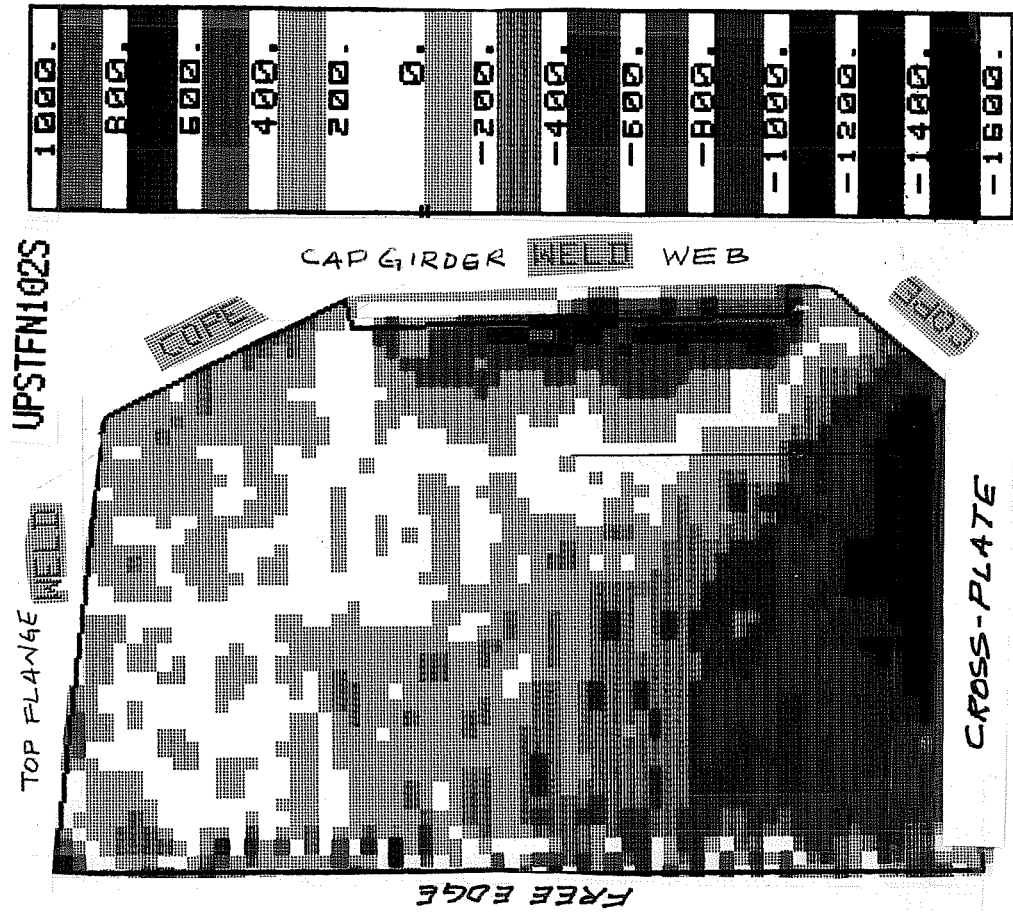


Figure F.3 SPATE Scan of Upper Stiffener

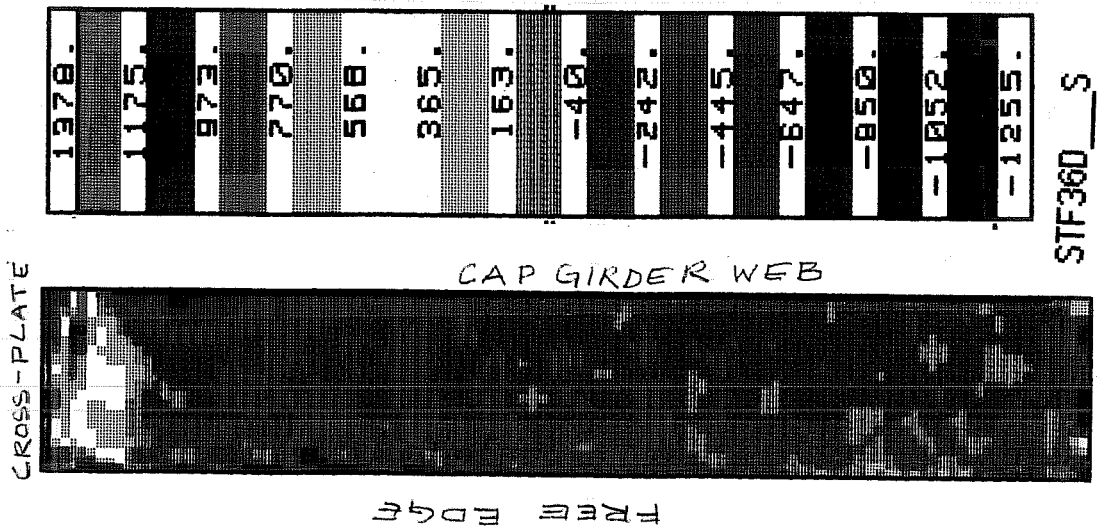


Figure F.4 SPATE Scan of Lower Stiffener

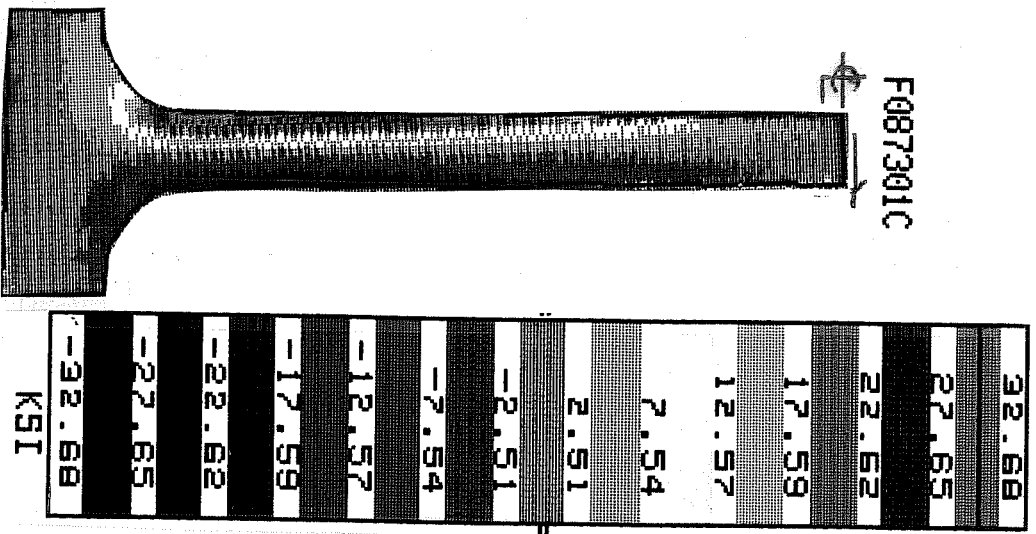


Figure F.5 SPATE Scan of Web, Fatigue Specimen

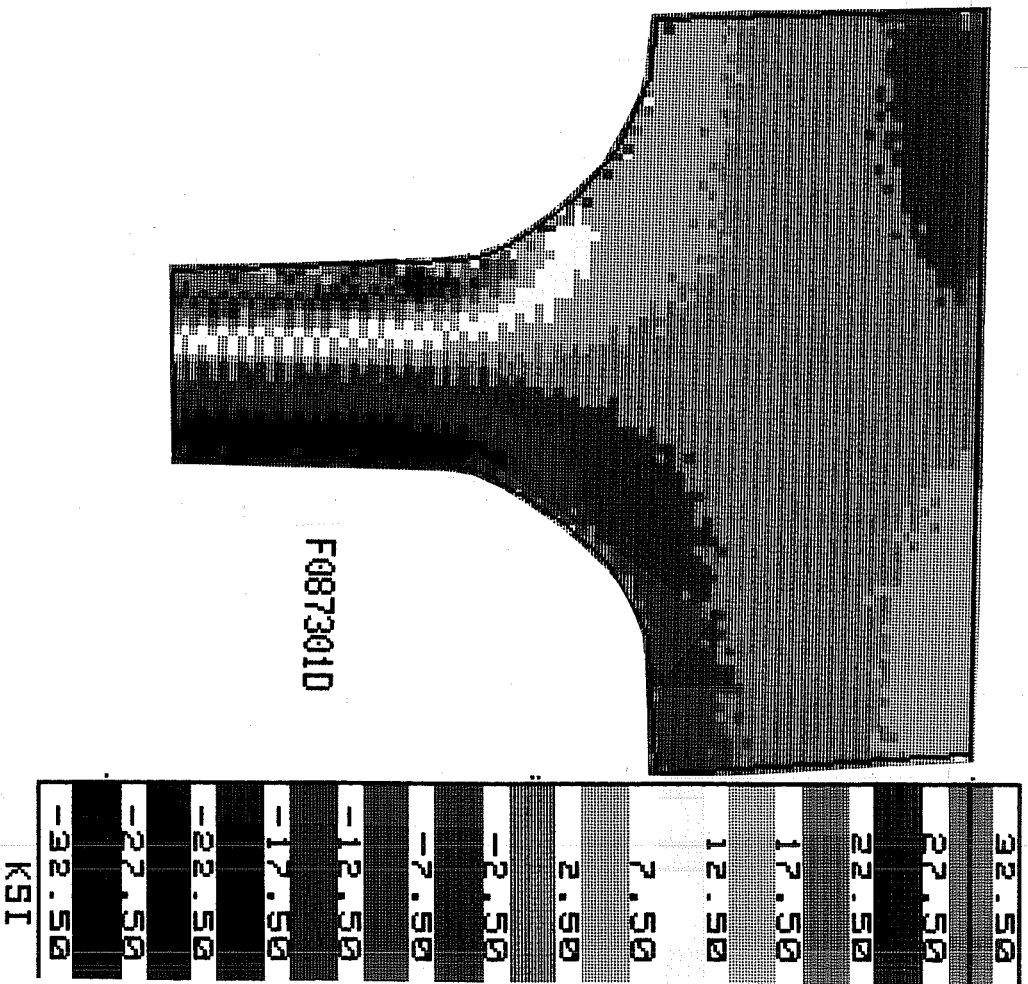
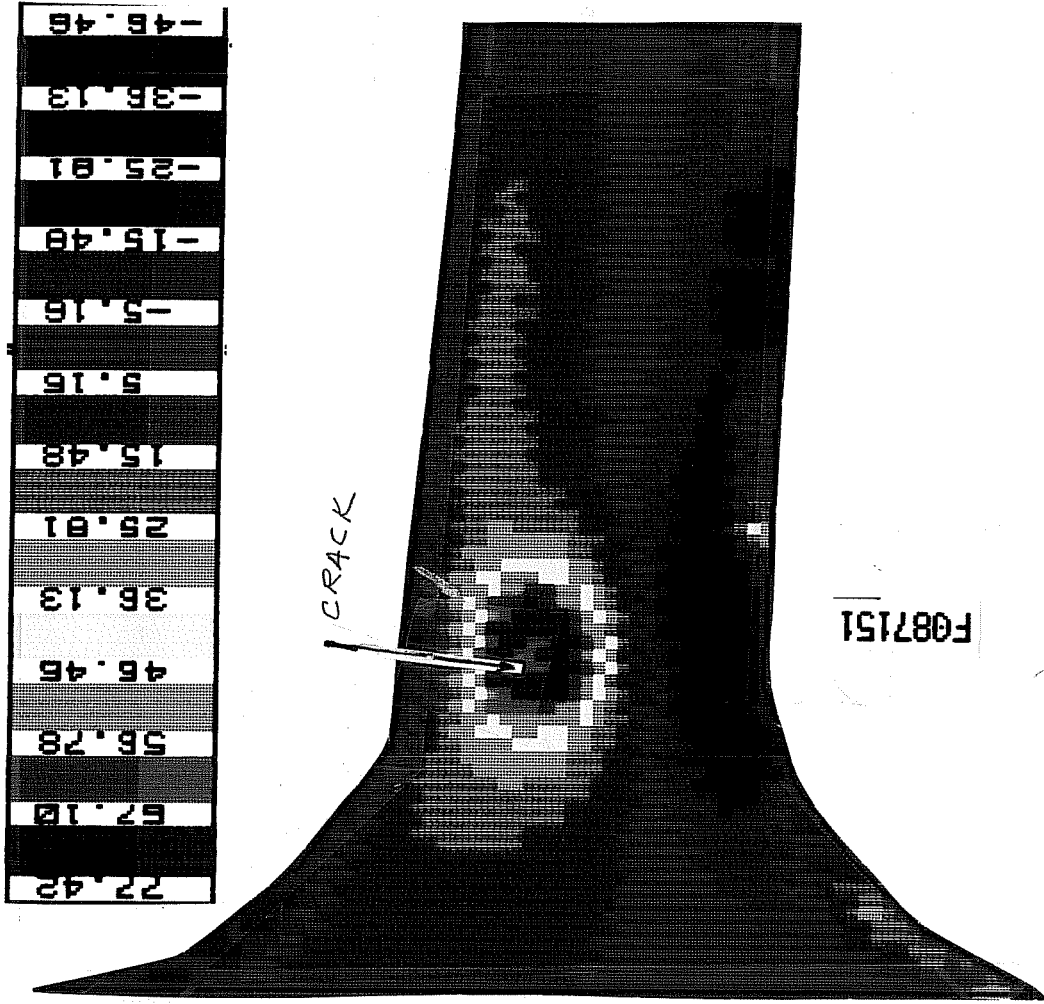
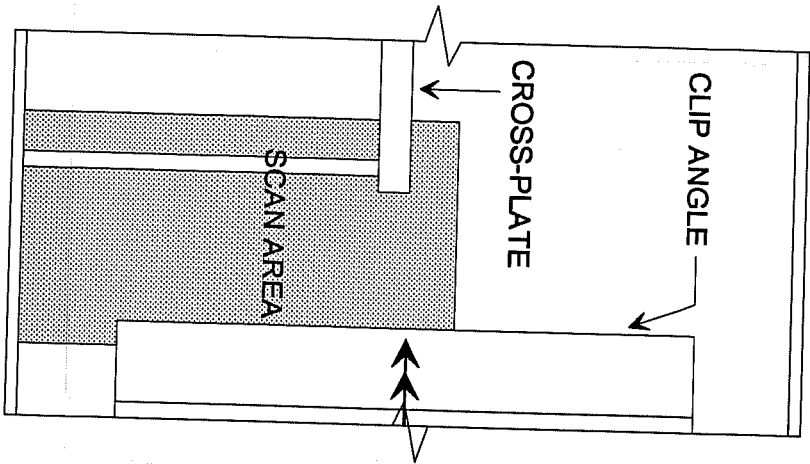


Figure F.6 SPATE Scan of Fillet Area

Figure F7 SPATE Scan of Fillet with Crack

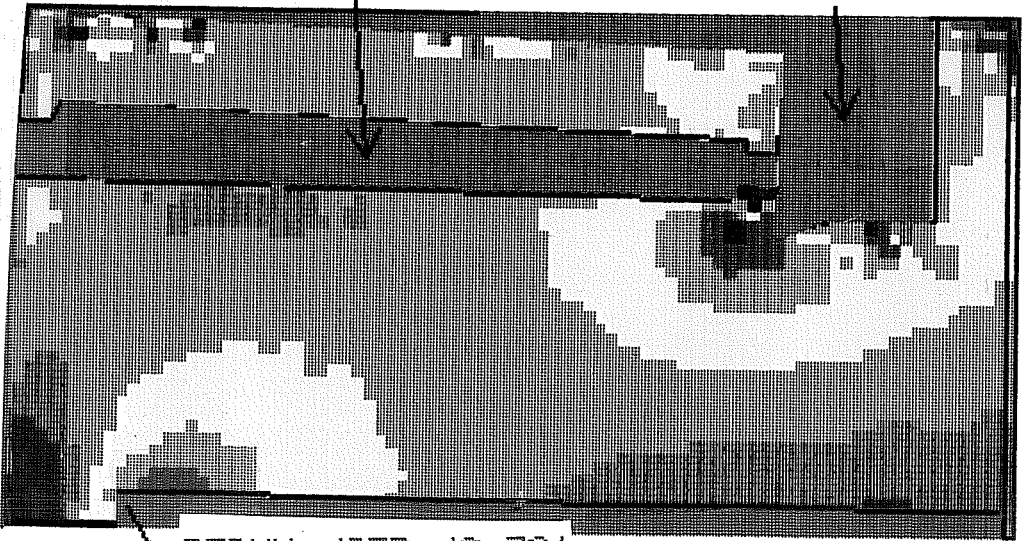




ROT8 S

CROSS PLATE

STIFFENER



10.13
7.88
5.63
3.38
1.13
-1.13
-3.38
-5.63
-7.88
-10.13

Figure F.8 SPATE Scan of Cap Girder Web

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FIND THE INCLINATION ANGLE OF THE BOTTLE STRUT, θ

The location of node 2 is the intersection of struts C1, C4, and C5, and is defined by x_{cg} and θ , the angle of inclination, as shown in Figure 4.25. The inclination angle, θ , can be found knowing the centroid of strut C5, x_{cg} , and the depth of reinforcing, d .

$$d = h - \text{cover} - d_b/2$$

Assuming a #11 bar in the top layer, $d_b = 1.41''$

$$\text{so } d = 48 - 2.25 - 1.41/2 = 45.0''$$

$$\theta = 0.5 \arcsin [2x_{cg}/d] = 72.8^\circ$$

$$\theta = 90^\circ - 0.5 \arcsin [2x_{cg}/d] = 72.8$$

90°
should need like this

add this

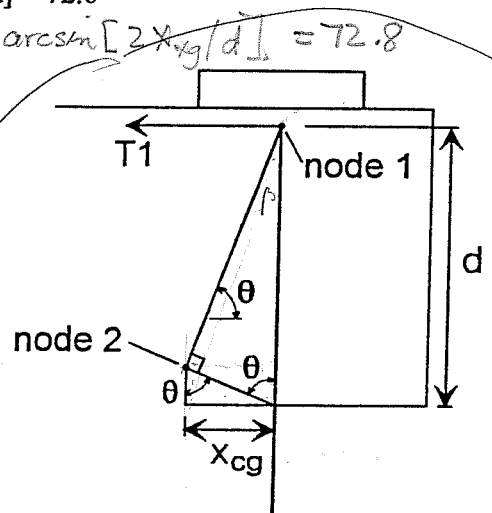


Figure 4.25 Location of Node 2 in the Strut and Tie Model

KNOWING θ , CHECK T1 AND ASSUMED d

$$T1 = P_n / \tan \theta = 2,130 / \tan 72.8 = 659 \text{ kips}$$

$$AT1 = \text{area of steel for } T1 = T1 / f_y = 11.0 \text{ in}^2$$

use 8 #11 bars, with $A_s = 12.48 \text{ in}^2$

⇒ this implies that 2 layers of steel will be needed

RECALCULATE θ USING TWO LAYERS OF STEEL FOR T1

for 2 layers of steel, use clear spacing of $2d_b = 2.82''$ for a #11 bar

$$d = h - \text{cover} - d_b - \text{clear spacing}/2 = 42.9''$$

$$\text{so new } \theta = 0.5 \arcsin [2x_{cg}/d] = 71.8$$

INSERT 90° -

Check T1 using the new θ

$$T1 = P_n / \tan \theta = 2,130 / \tan 71.8 = 700 \text{ kips}$$

$$AT1 = T1 / f_y = 11.7 \text{ in}^2 < 12.48 \text{ in}^2 \text{ provided, OK}$$

FIND ALL MEMBER FORCES

Compute the diffusion angle:

$$\Phi = 12 + 3/[(w/h)^{0.5}] = 16.2^\circ$$

$$P_n = 2,130 \text{ kips}$$

$$T1 = 700 \text{ kips}$$

$$\theta = 71.8^\circ$$

$$C1 = P_n/\sin\theta = 2,242 \text{ kips}$$

$$C2 = 0.5 C1/\cos \Phi = 1,167 \text{ kips}$$

$$C3 = C1/2 = 1,121 \text{ kips}$$

$$C4 = T1 = 700 \text{ kips}$$

$$C5 = P_n = 2,130 \text{ kips}$$

$$T2 = 0.5 C1 \tan \Phi = 326 \text{ kips}$$

$$T2_{\text{horiz}} = T2 \sin \theta = 310 \text{ kips} = \text{horizontal component of } T2$$

$$T2_{\text{vert}} = T2 \cos \theta = 101 \text{ kips} = \text{vertical component of } T2$$

CHECK STRESS AT THE CCC NODE, NODE 2

11,32

The layout of the CCC node is shown in Figure 4.26. AC1 is the area of compression strut C1 at this node. Since AC5 is a combination of a circular and a rectangular section, AC1 is found by projecting from AC5.

$$AC1 = AC5/\sin\theta = 740/\sin 71.8 = 779 \text{ in}^2$$

wC4 = the width of strut C4

$$0.5 wC4 = c/\tan \theta = 3.09''$$

$$wC4 = 6.18''$$

The column width is used to find AC4 as the column width limits the strut width.

$$AC4 = B2 wC4 = 260 \text{ in}^2$$

σ_{C5} = the stress on strut C5 = $C5/AC5$

$$\sigma_{C5} = 2.88 \text{ ksi} = f_{cd} \text{ OK}$$

$$\sigma_{C1} = C1/AC1 = 2.88 \text{ ksi} = f_{cd} \text{ OK}$$

$$\sigma_{C4} = C4/AC4 = 2.67 \text{ ksi} < f_{cd} \text{ OK}$$

All strut stresses at the node are less than or equal to the limiting concrete design stress, but a hydrostatic state of stress does not exist as all three stresses are not equal. Since the stress at the node is not hydrostatic, the checks suggested by Schlaich (1987) are used:

Ce
[26]

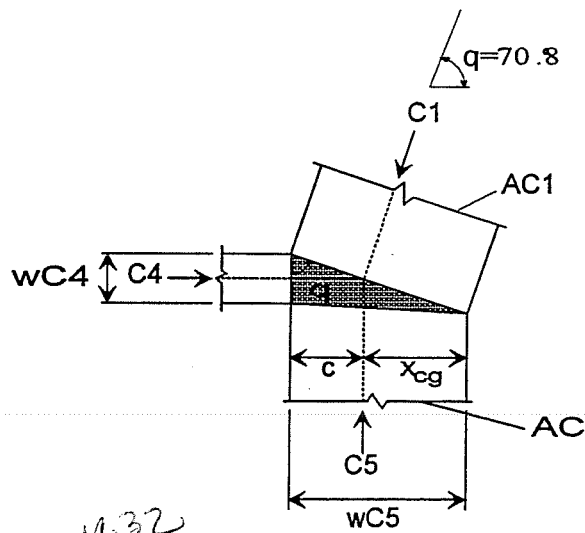


Figure 4.26 Geometry of the CCC Node (Node 2 in the Strut and Tie Model)

1. All strut stresses at the node are within design limits.
2. The smallest stress ratio between faces of the node is greater than 0.5.
Check: $2.67/2.88 = 0.93 > 0.5$ OK.

Since both checks are met, this node is satisfactory.

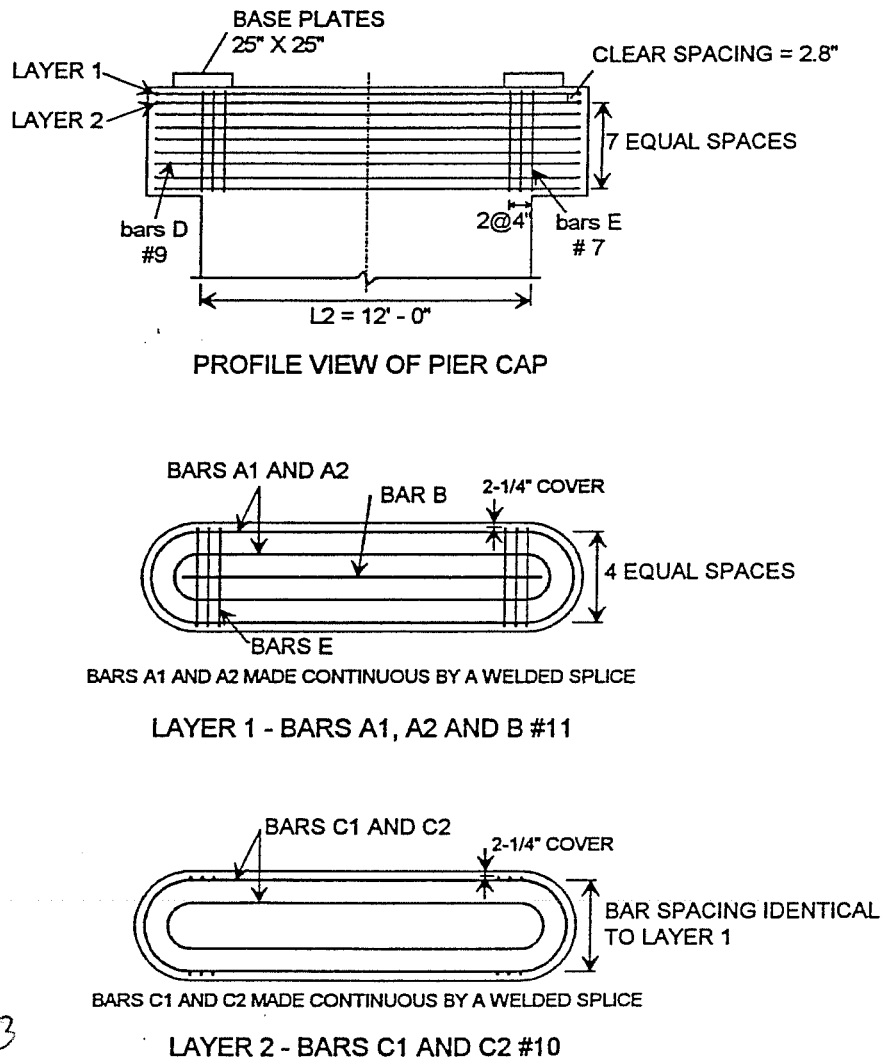
DESIGN STEEL FOR T1 AND THE COMPONENTS OF T2

$T1 = 700$ kips

$AT1 = T1/f_y = 11.67 \text{ in}^2 =$ area of steel required for T1

For T1, use 5 #11 bars and 4 #10 bars, $A_s = 12.88 \text{ in}^2$

The layout of T1 steel at the top of the cap is shown in Figure 4.27.



6.33
Figure 4.27

Steel Reinforcing Pattern from the Example Problem

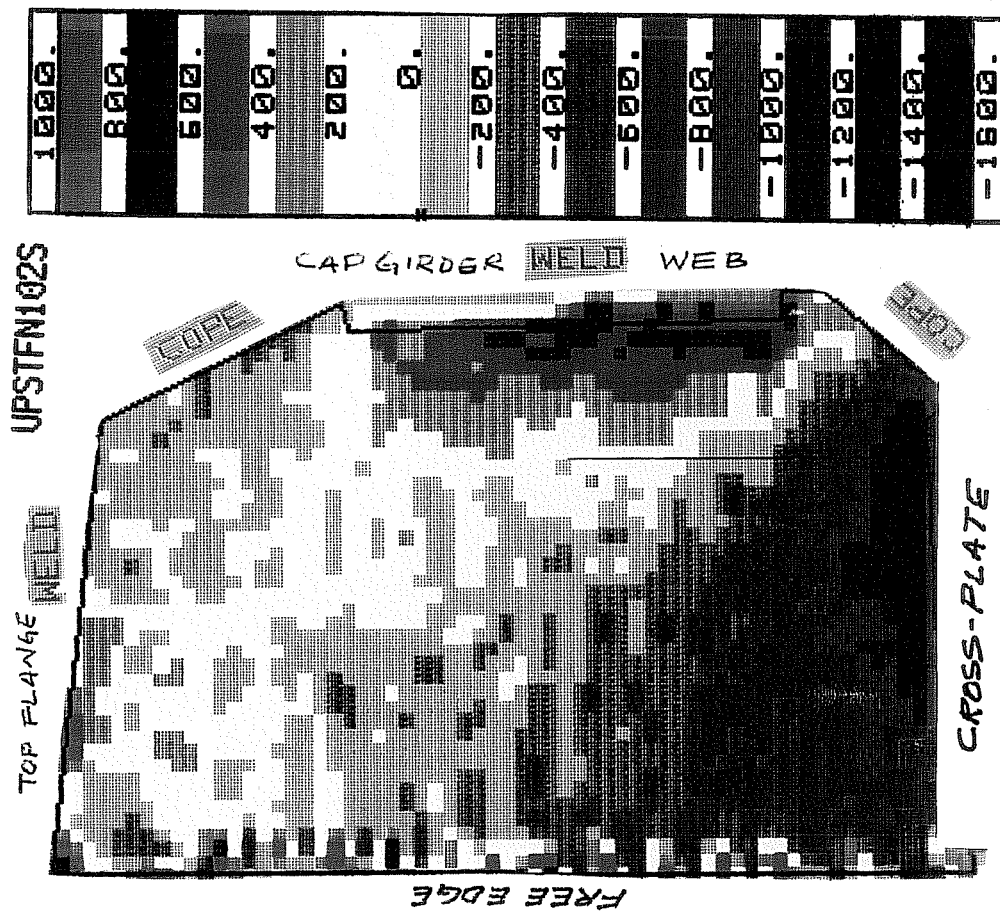


Figure F.3 SPATE Scan of Upper Stiffener

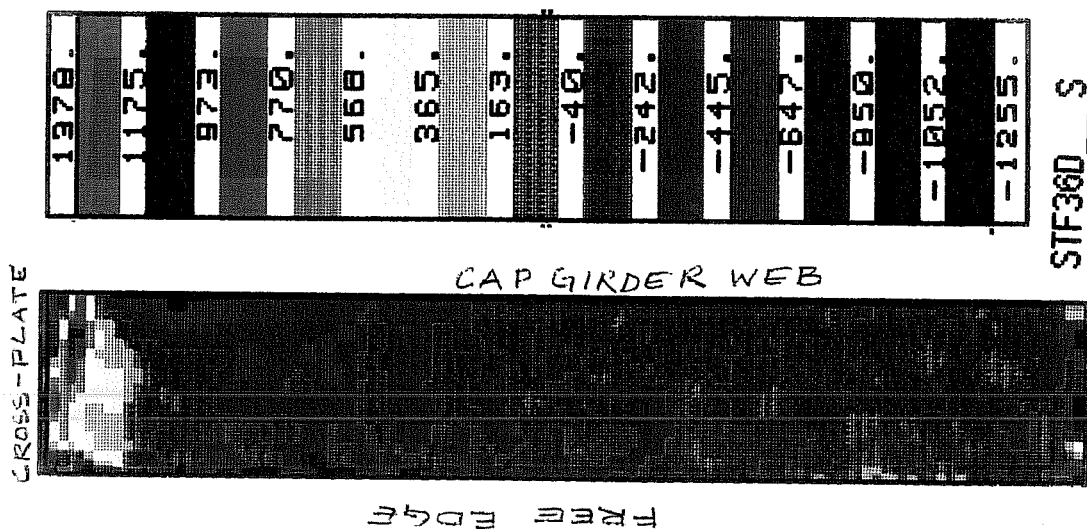


Figure F.4 SPATE Scan of Lower Stiffener

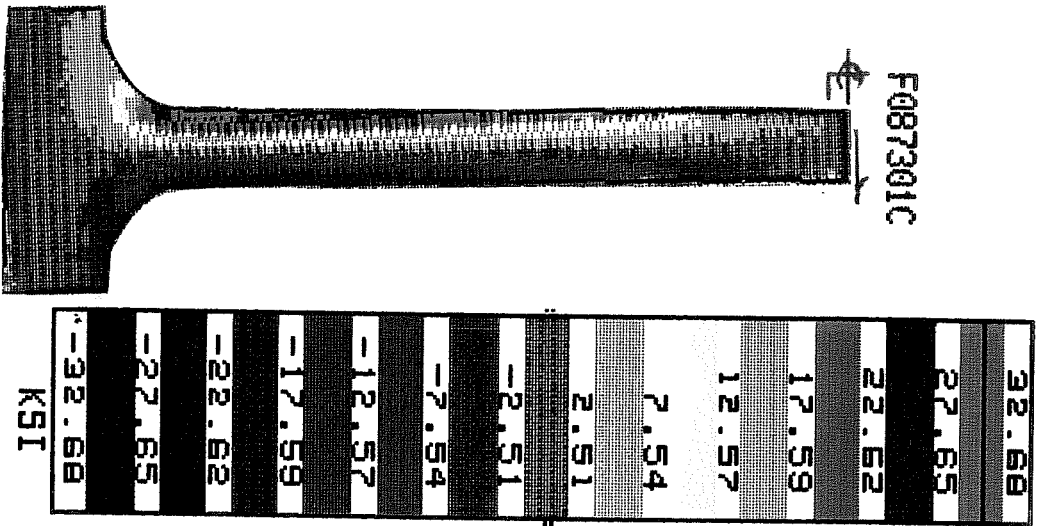


Figure F.5 SPATE Scan of Web, Fatigue Specimen

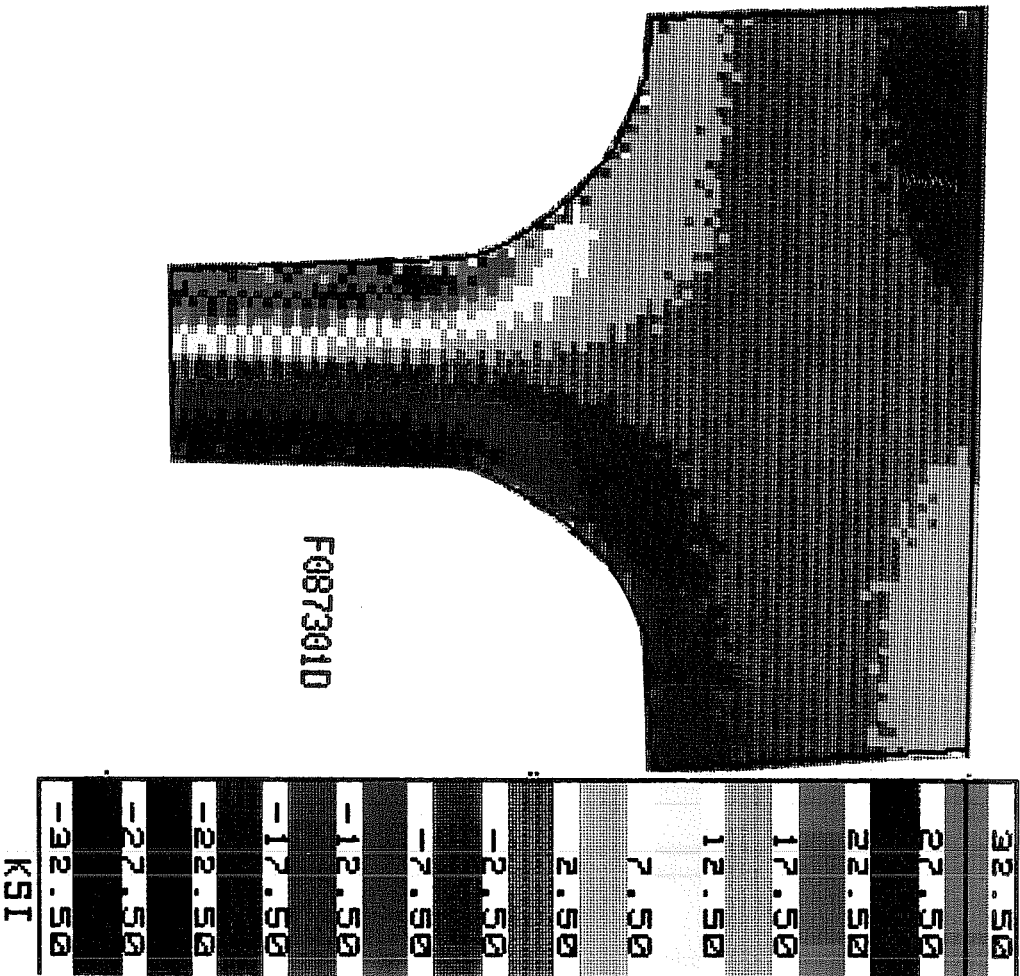


Figure F.6 SPATE Scan of Fillet Area

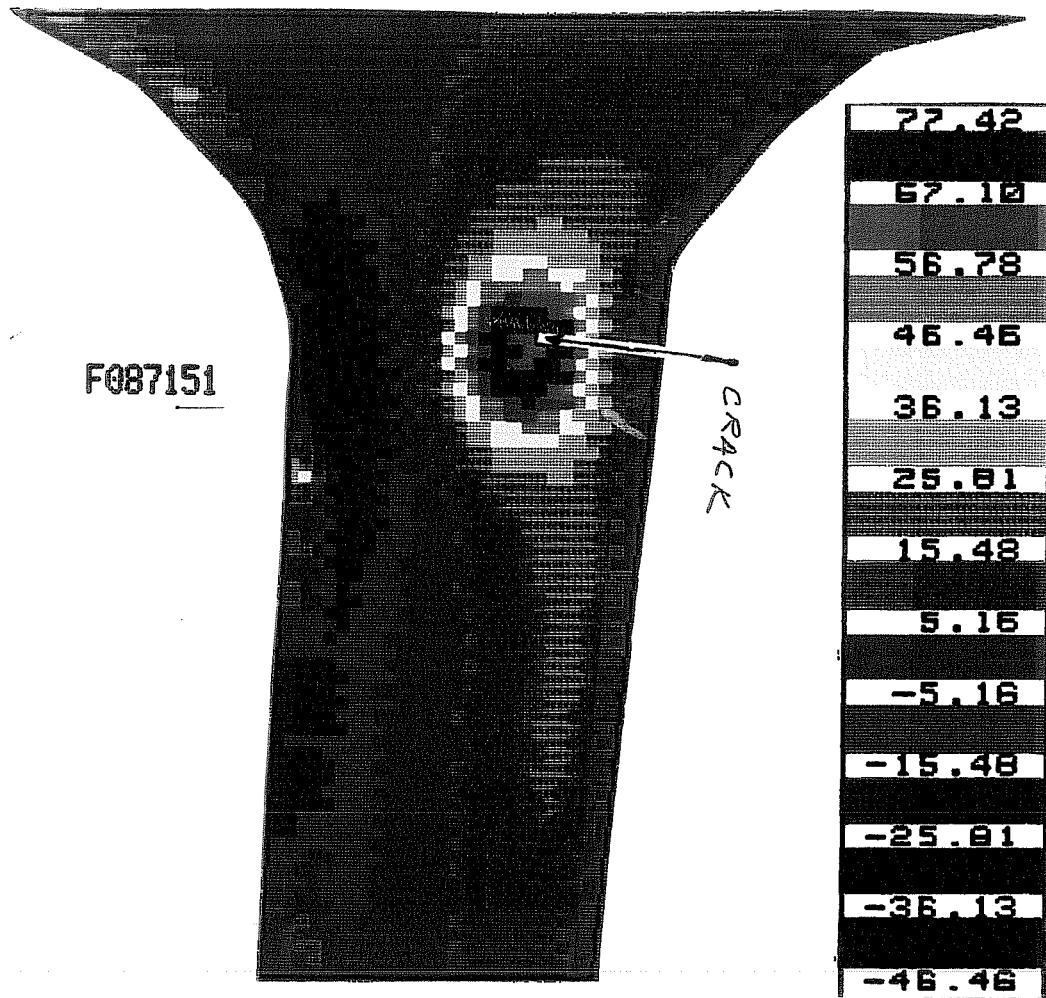


Figure F.7 SPATE Scan of Fillet with Crack

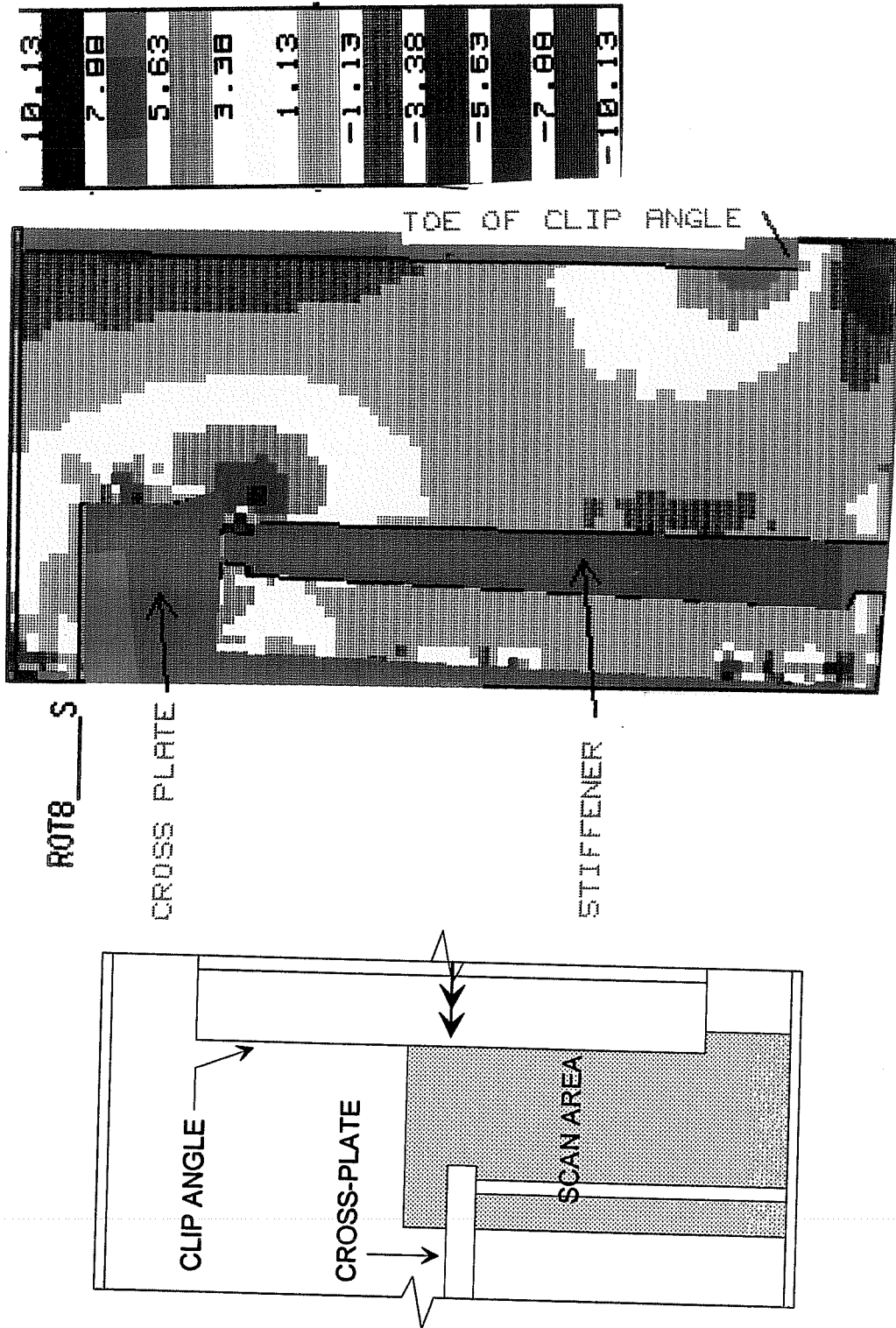


Figure F.8 SPATE Scan of Cap Girder Web

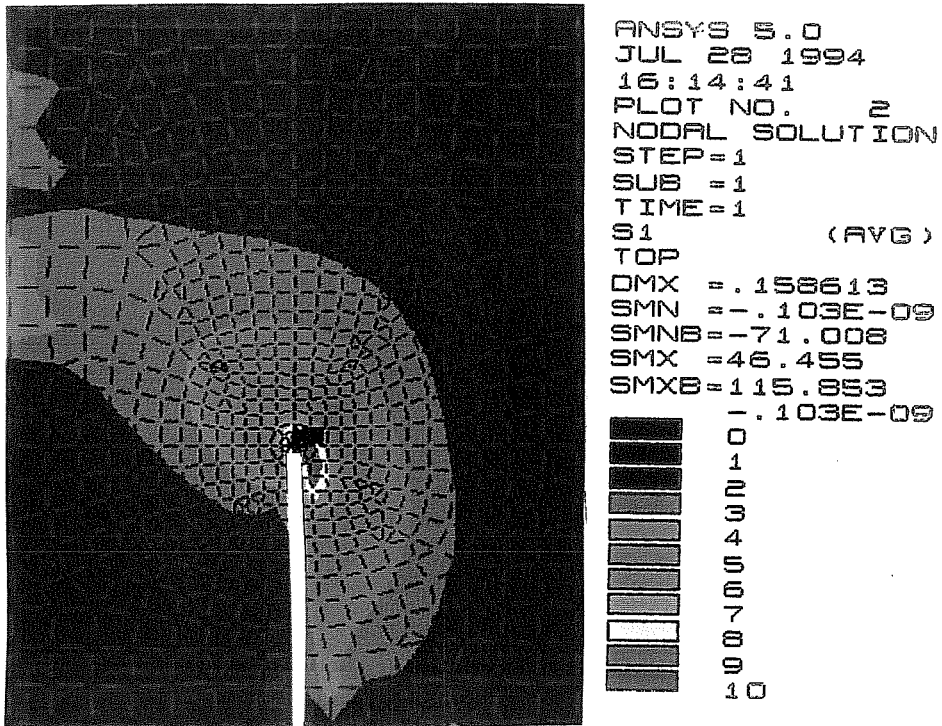


Figure 4.15 Typical Plot of Principal Tensile Stress for Refined Mesh

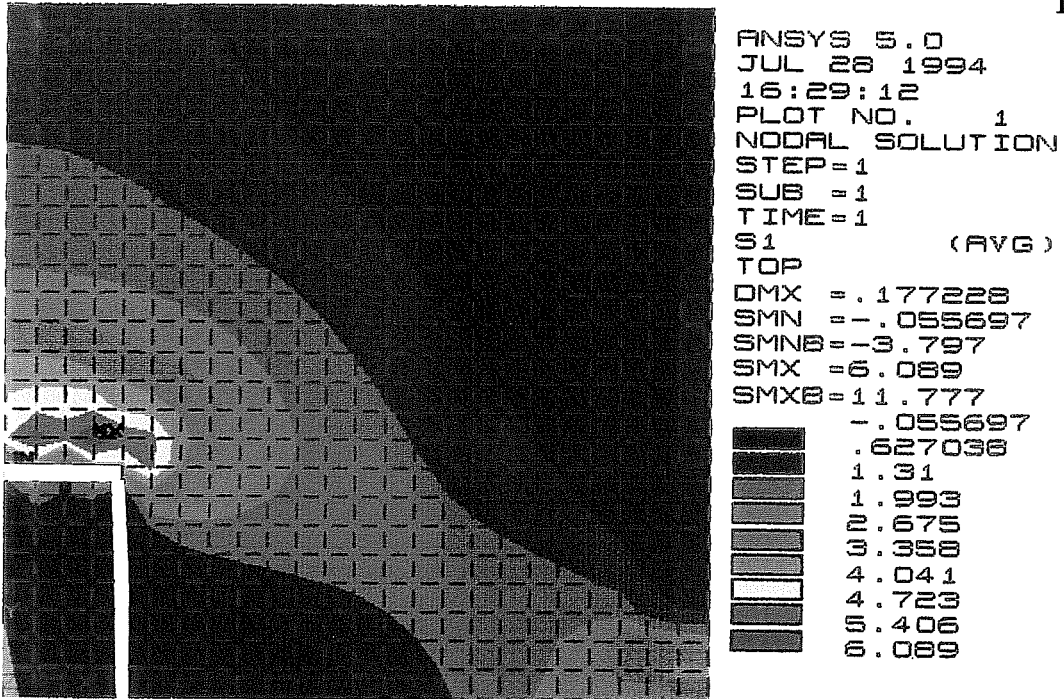


Figure 4.9 Typical Plot of Principal Tensile Stress

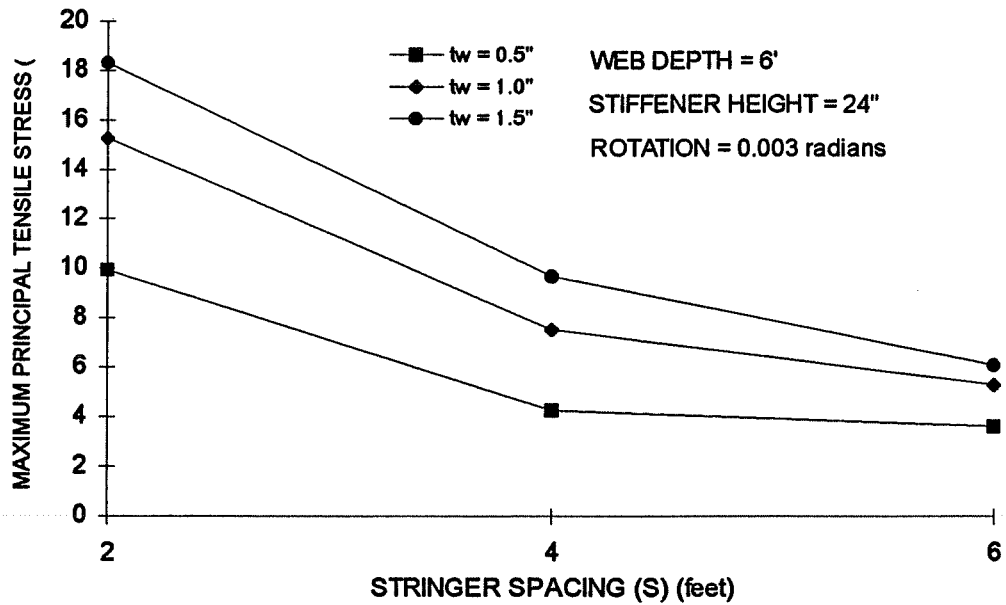


Figure 4.10 Principle Tensile Stress vs. Stringer Spacing